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Virtual full-scale testing for investigating strength characteristics of a composite wind turbine blade

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Abstract. Full-scale structural tests enable us to monitor mechanical response of the blades under various loading scenarios. Yet these tests must be accompanied with numerical simulations, so that the physical basis of the progressive damage development can be captured and interpreted correctly. Within the scope of this paper the previous work of the authors concerning the strength analysis of an existing 5-m GFRP wind turbine blade using Puck failure criteria is revisited. An important outcome of the previous study was that nonlinear Puck material model was found to be necessary for a more realistic simulation of failure mechanisms. In the current work, under extreme load cases internal flange at the leading edge and trailing edge of the blade are identified as the mainly damaged regions. Moreover, dominant failure mechanism is expected to be the de-bonding at the trailing and leading edges. When extreme load case is applied as a combination of edge-wise and flap-wise loading cases, less damage is observed compared to the pure flap-wise loading case. This damage evolution is attributed to the stiffer structural behavior of the blade under combined loading condition.

1 Introduction

As fundamental eco-friendly renewable energy resources, wind turbines are designed to operate over a lifespan of 20 years. According to Holmes et al. (2007) long-term structural reliability of wind turbine components is vital when high cost of manufacturing, inspection and repair, especially for turbines located in remote regions are considered. Composite blades are among the most critical components of a wind turbine, which are subjected to complex loading conditions. A rotor blade failure can have a significant impact on turbine downtime and safety. In order to assure sufficient mechanical resistance structural testing and/or analysis must be conducted. However structural testing methods such as full-scale testing of the blade are expensive and troublesome due to the construction of a test set-up. Chen et al. (2017) states that, in order to capture and understand the physical basis of the progressive damage development correctly, tests need to be accompanied with numerical analysis methods. Moreover, structural analyses are utilized to calibrate structural blade test set-ups for different loading conditions.

In the literature, there are many studies on the structural behavior of composite turbine blades. A novel methodology for the structural design and analysis of 8-m tidal current turbine blade is presented based on the Puck phenomenological failure

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criteria for fibre and inter-fibre failure by Fagan et al. (2016). The methodology that is developed in the study predicted damage values for different load cases. This methodology is an iterative design process with respect to failure criteria to check structural strength of the blade. Their turbine blade Finite Element Model did not include geometric nonlinearity. Passipoularidis et al. (2011) developed a fatigue damage simulator (FADAS) utilizing Puck failure criteria for the life prediction of GFRP laminates, which are commonly used for the construction of wind turbine blades under variable amplitude loading. In this study failure analysis is done on ply level based on Classical Lamination Theory (CLT). Puck failure criteria are implemented to predict failure initiation and sudden stiffness degradation. The predictions of FADAS agree well with fatigue data and shows that the algorithm is able to take into account load sequence effects, as well. Another study conducted by Jensen et al. (2006) is about the full-scale test and nonlinear FEM simulation of a 34-meter composite wind turbine blade under flapwise loading. Load-displacement curves are used to predict the location of failure initiation which lead to ultimate failure. After comparing the test data and simulation results delamination of the skin and the buckling followed by delamination was found out to be the main mechanism which lead to ultimate collapse. Chen et al. (2017) revisited his previous study concerning the structural collapse of a 52.3 m composite rotor blade in a full-scale bending test and explained the sequence of failure events from the video record of the test. In addition to this, they introduced a new methodology for predicting delamination and skin/core failure modes, which must be analyzed according to the DNV GL standard (2015).

In their paper Haselbach and Branner (2016) discuss the initiation and development of trailing edge failure in the full-scale test of a 34-m wind turbine blade. In their study the influence of buckling on the damage onset in the trailing edge and sandwich panel failure is highlighted. As a further outcome of the study, they show that modeling technique utilizing fracture mechanics approach for the failure in trailing edge delivers good agreement with experiments. Haselbach (2017) investigated different trailing edge modeling methods in his work. He analyzed the trailing edge failure under edge-wise and flap-wise loading conditions. As a conclusion he pointed out that modeling the adhesive bonding in the trailing edge with solid brick elements and connecting them to the shell elements of the skin with Multi Point Constraints (MPC) shows the best agreement with experiments. Recently, Castelos and Balzani (2016) studied the effect of geometric nonlinearities on the fatigue analysis of the trailing edge bonding in wind turbine blades. From their results they point out that superposition of stresses for the fatigue may be misleading especially for modern flexible rotor blades in which geometric nonlinearities in the stress calculation must be considered. In addition to this they propose a novel methodology for calculating stresses with a new load application method that reduces geometric nonlinear behavior of the blade.

Within the scope of this work the authors of this study refer back to their previous paper concerning the strength analysis of an existing 5-meter GFRP turbine blade using Puck failure criteria (Ozyildiz et al., 2018). In the previous work the FE Model of the blade was built in ANSYS ACP environment. ANSYS APDL Code was developed to carry out progressive damage analysis and degradation rules. As part of the previous study, linear Puck material model was compared with the nonlinear progressive Puck material model. They have seen from that the load patterns change as the elements fail when

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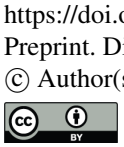

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progressive Puck criteria are used. Therefore, they came to the conclusion that progressive failure analysis is necessary to capture a more realistic simulation of failure mechanisms prior to testing.

The goal of this work is to identify the main failure mechanisms and/or modes and locations which may lead to the collapse of the wind turbine blade during the flap-wise, edge-wise and combined flap-wise and edge-wise bending virtual full-scale test with the help of progressive damage analysis. For this purpose, structural behavior of the blade under these extreme loading conditions are investigated. After an in-depth understanding of the damage development, the failure initiation regions are determined to be the trailing edge and internal flange and the main failure mechanism is expected to be de-bonding/delamination in this region. Based on the main failure mechanism it is seen that under flap-wise loading spar design needs improvement, but under edge-wise loading the current design is overconservative. On top of this, in combined edge-wise and flap-wise loading less damage is observed compared to the pure flap-wise loading case. This damage evolution is explained by the stiffer structural behavior of the blade under combined loading condition.

The existing blade was designed as part of a project between METUWIND – METU Center for Wind Energy and Core Team of the University of Patras. The blade was designed for a wind turbine that has 30 kW nominal power capacity at 10 m/s wind speed. According to the wind turbine characteristics, the blade optimized aerodynamic and geometric design was finalized by the blade manufacturer. The existing blade consists of five main parts: suction side, pressure side, internal flange, “hat shaped” chassis/spar, and flange as seen in Figure 1.

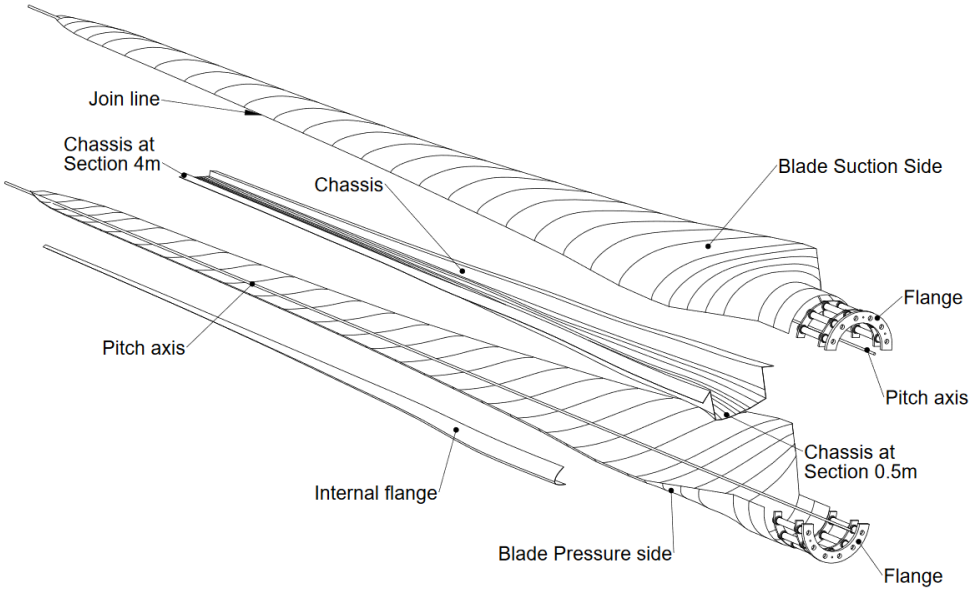

Figure 1. Blade assembly for 5-meter METUWIND turbine blade (Philippidis and Roukis, 2013).

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2 Methodology

For the progressive failure analysis of the blade, Puck criteria, which are explained briefly in the following paragraphs are used. Puck failure criteria (Puck and Schuermann, 2002) are one the most commonly used and well-established criteria for the assessment of composite laminate strength. In this study, Puck's failure criteria are implemented for the evaluation of stress results of unidirectional and tri-axial laminate composite materials.

For fiber failure, Puck's criteria are as follows:

$$f_{E(FF)}^{T} = \frac{\sigma_1}{X_T} = 1 \text{ if } \sigma_1 > 0 \quad (1)$$

$$f_{E(FF)}^{C} = \frac{\sigma_1}{X_C} = 1 \text{ if } \sigma_1 < 0 \quad (2)$$

where $f_{E(FF)}^{T}$ and $f_{E(FF)}^{c}$ are stress exposures for fiber failure under tension and compression loading cases. σ_1 is the stress value in fiber direction, X_T and X_c are tensile and compressive strengths in fiber direction, respectively. Puck's inter fiber failure use different equations depending on the failure mode which is detected. Under two dimensional (2-D) biaxial loading the failure modes, which can be detected are summarized in Figure 2 below. In Figure 2 the transition point from failure mode B to failure mode C is denoted by the point $(\tau_{21c}, R_{\perp\perp}^{A})$ and is calculated by the ratio $R_{\perp\perp}^{A}/\tau_{21c}$. Their values are calculated by the expressions below:

$$R_{\perp\perp}^{A} = \frac{S}{2p_{\perp\parallel}^{(-)}}\left[\sqrt{1 + 2p_{\perp\parallel}^{(-)}\frac{Y_c}{S}} - 1\right] \quad (3)$$

$$\tau_{21_c} = R_{\perp\parallel}\sqrt{1 + 2p_{\perp\perp}^{(-)}} \quad \text{and} \quad p_{\perp\perp}^{(-)} = p_{\perp\parallel}^{(-)}\frac{R_{\perp\perp}^{A}}{S} \quad (4)$$

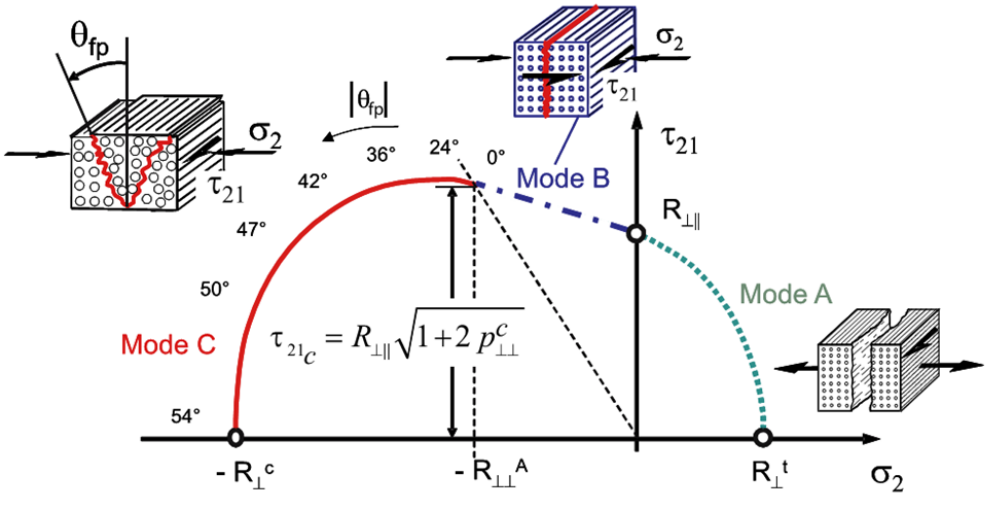

Figure 2. Failure envelope under biaxial loading (Knops, 2008).

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Depending on the region of the failure envelope, the following inter fiber failure expressions are written:

$$f^{A}_{E(IFF)} = \left[\left(\frac{\sigma_6}{S}\right)^2 + \left(1 - p^{(+)}_{\perp II}\frac{Y_T}{S}\right)^2\left(\frac{\sigma_2}{Y_T}\right)^2\right]^{1/2} + p^{(+)}_{\perp II}\frac{\sigma_2}{S} = 1 \text{ if } \sigma_2 \geq 0 \tag{5}$$

$$f^{B}_{E(IFF)} = \frac{1}{S}\left\{\left[(\sigma_6)^2 + \left(p^{(-)}_{\perp II}\sigma_2\right)^2\right]^{1/2} + p^{(-)}_{\perp II}\sigma_2\right\} = 1 \text{ if } \begin{cases} \sigma_2 < 0 \\ \left|\frac{\sigma_2}{\sigma_6}\right| \leq \frac{R^A_{\perp\perp}}{\tau_{21C}} \end{cases} \tag{6}$$

$$f^{C}_{E(IFF)} = \left[\left(\frac{\sigma_6}{2(1+p^{(-)}_{\perp\perp})S}\right)^2 + \left(\frac{\sigma_2}{Y_C}\right)^2\right]\frac{Y_C}{(-\sigma_2)} = 1 \text{ if } \begin{cases} \sigma_2 < 0 \\ \left|\frac{\sigma_2}{\sigma_6}\right| \geq \frac{R^A_{\perp\perp}}{\tau_{21C}} \end{cases} \tag{7}$$

In the equations above $p^{(+)}_{\perp II}, p^{(-)}_{\perp II}$ and $p^{(-)}_{\perp\perp}$ represent inclination parameters that control the shape of the failure envelope. According to Puck and Schürmann (2002), $p^{(+)}_{\perp II} = 0.3$ and $p^{(-)}_{\perp II} = 0.25$. σ_2 is the stress value in the transverse fiber direction, Y_T and Y_c are tensile and compressive strengths in the transverse fiber direction. Shear stress and shear strength are represented by σ_6 and S, respectively. If the value of failure exposure (f_E) exceeds 1, failure initiation occurs. Mode A is caused by tensile and shear stresses. Modes B occurs under compressive and shear stresses. Mode C is a dangerous failure mode in compressive shearing which may lead to ultimate failure.

Degradation rules are applied to the elements which fail according to the specific Puck's failure criteria that are inter fiber failure (IFF) mode A, B or C (Eq. (5), (6) and Eq. (7), respectively). As presented by Passipoularidis et al. (2011), based on degradation rules in Table 1, transverse elasticity and shear moduli of the damaged elements are reduced accordingly. Recommended parameters c, η_r and ξ for the degradation function in Eq. (8) are taken from Knops and Bögle (2006). $f^{A}_{E(IFF)}$, $f^{B}_{E(IFF)}$ and $f^{C}_{E(IFF)}$ are the failure exposure values that are considered for determining the failure mode during the analysis.

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Table 1: Degradation rules according to failure mode.

Failure Mode	Degradation Rule
FF (Tension / Compression) or IFF(C) in 3 plies or more	Failure of the laminate
IFF (A)	$E_2 = \eta E_2$ $G_{12} = \eta G_{12}$ $\nu_{12} = \eta \nu_{12}$
IFF (B)	$E_2 = \eta E_2$ $G_{12} = \eta G_{12}$ $\nu_{12} = \eta \nu_{12}$
IFF (C) in one or two plies	$E_2 = 0.1 E_2$ $G_{12} = 0.1 G_{12}$ $\nu_{12} = 0.1 \nu_{12}$

In Eq. (8) η is known as the degradation factor and can be expressed according to the equation below:

$$\eta = \frac{1 - \eta_r}{1 + c(f_{E(IFF)} - 1)^{\xi}} + \eta_r \tag{8}$$

The summary of the algorithm of the FE analysis based progressive failure analysis of a composite laminate using Puck failure criteria is shown in Figure 3. The complete algorithm is implemented using ANSYS Parametric Design Language (APDL). First using ANSYS APDL script different material numbers are given to each lamina, which constitute elements of the composite laminates. This step is necessary, because during the execution of the progressive damage propagation each lamina are subjected to different degradation rules. Then, extreme load case is applied incrementally to the model and static analysis is run. Afterwards, in the post processing module stresses are read. From Puck failure criteria for FF (Eq. (1) and (2)) and IFF (Eq. (5), (6) and (7)) failure exposures are calculated. Depending on the rules presented in Table 1whether ply failure happens is checked. If ply failure occurs due to First-Fiber-Failure (FFF) or if IFF(C) is observed in 3 plies or more, element failure is assumed to take place. If IFF A or B or IFF C in less than 3 plies is seen gradual degradation rules are applied and after the assembly of the new constitutive material model, load is incremented and the analysis is re-run. This calculation procedure is run until unconverged solution is observed. If solution does not converge, the analysis aborted. As seen from the flow chart, as long as no FF or IFF failure occurs, without updating the constitutive material model load is incremented and the analysis is executed.

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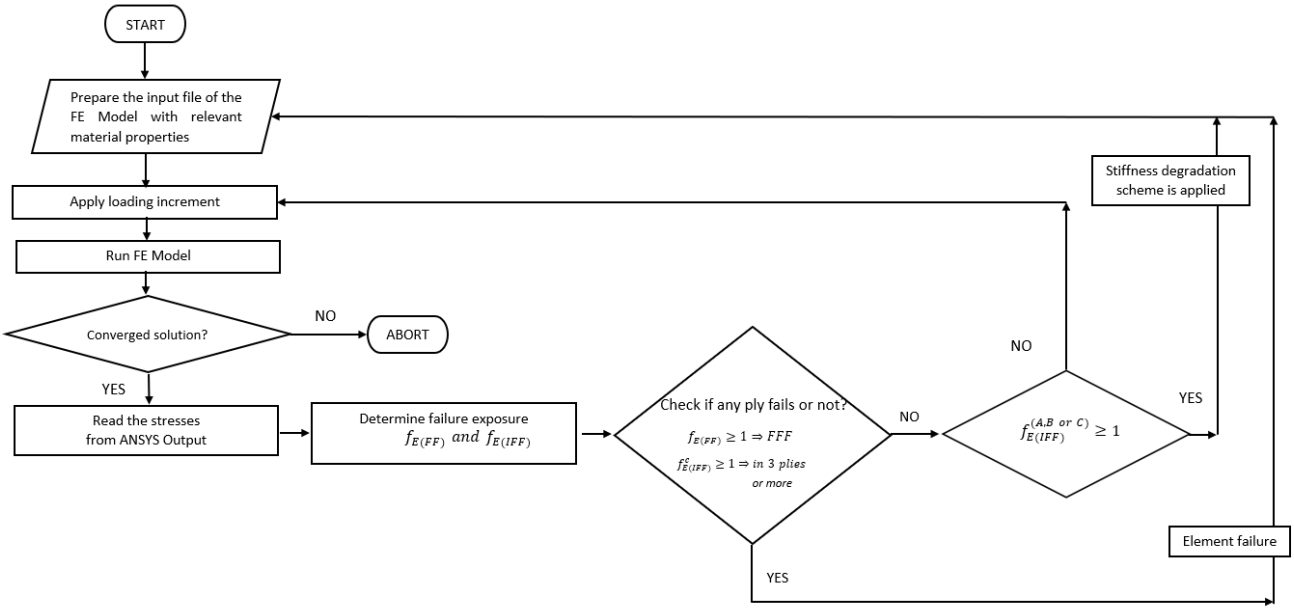
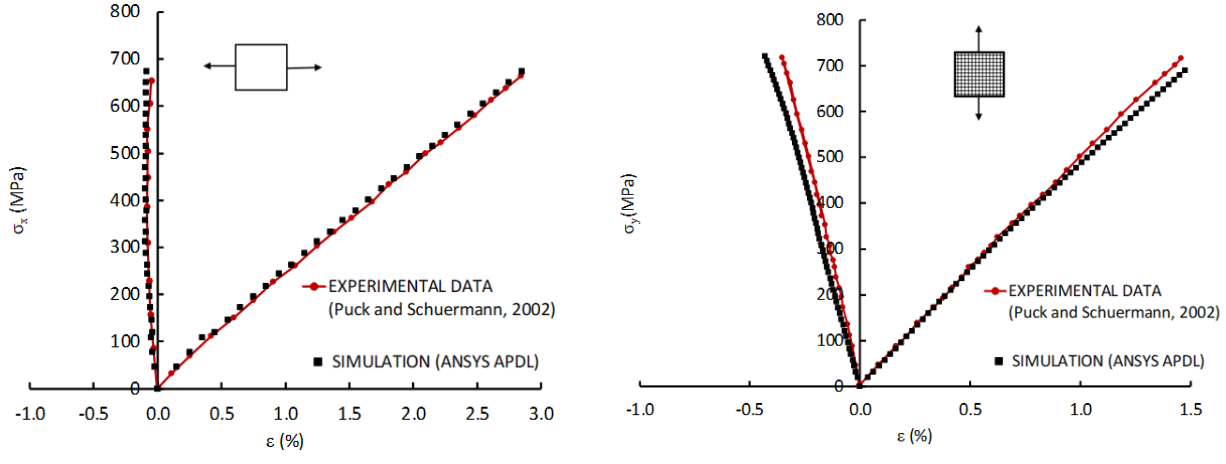


Figure 3. Flowchart of the FE Analysis Based Strength Analysis of a Composite Laminate Using Puck Failure Criteria.

Validation of the APDL Code against experimental data provided by Puck and Schuermann (2002) is done for the $[0/90]_s$ GFRP laminate/MY750 and $[0/\pm45/90]_s$ CFRP laminate/AS4 3501-6 under uniaxial tension loading. As seen from Fig. 4 there is a good agreement between APDL Code and experimental data for the progressive failure analysis of $[0/90]_s$ GFRP/MY750 and $[0/\pm45/90]_s$ CFRP/AS4 3501-6 laminates.



(a) (b)

Figure 4. Validation of the APDL Code for the progressive failure analysis of (a) $[0/90]_s$ GFRP/MY750 laminate under σ_x uniaxial tension (b) $[0/\pm45/90]_s$ CFRP/AS4 3501-6 laminate under σ_y uniaxial tension.

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2.1 Finite Element Model of the METUWIND Blade

The two-dimensional blade technical drawings which include the blade aerodynamic design details such as cord length and twist angle along the blade were provided by the blade manufacturer Compblades. By using these given two-dimensional blade drawings, the three-dimensional CAD model of the blade is prepared in NX 12.0 environment. In Table 2 the material properties and design allowables of the blade materials for static analysis is listed as given in the technical report written by Philippidis and Roukis (2013). Referring to Germanischer Lyod (2010), design allowables are obtained from the knockdown of the experimental strength values by the material safety factor 2.406.

Table 2. Material properties and design allowables of METUWIND blade for static analysis

Material Property		Unidirectional Laminate	Steel	Gel Coat	CSM 300	Divinycell H45
Density, ρ	[kg/mm^3]	1896	7850	1200	1896	200
Thickness, h	[mm]	0.716	5.3	0.9	0.358	5 or 10
$\mathbf{E_1}$	[GPa]	24.84	210	3.98	9.14	$55x10^{-3}$
$\mathbf{E_2}$	[GPa]	9.14				$55x10^{-3}$
$\mathbf{\nu_{12}}$		0.29	0.3	0.34	0.29	0.4
$\mathbf{G_{12}}$	[GPa]	2.38				$15\ x10^{-3}$
$\mathbf{X_T}$	[MPa]	191.73	581.8	35.29	16.86	1.4
$\mathbf{X_C}$	[MPa]	101.16				0.6
$\mathbf{Y_T}$	[MPa]	16.86				1.4
$\mathbf{Y_C}$	[MPa]	50.41				0.6
S	[MPa]	11.29				0.56

The skin of the blade is composed of unidirectional and tri-axial laminates, whereas only tri-axial laminates are used for the spar. The lay-up sequence for the pressure and suction side differs only in the area from 1.25m to 2.0m where an extra unidirectional glass fabric was placed in the suction side of the blade. The root part of the blade is composed of unidirectional laminate, tri-axial laminates and steel. The outer surface of the blade is covered with transparent Gel Coat and a layer of chopped strand mat, 300 g/m^2 CSM 300. In addition to this, the Divinycell H45 foam used in the trailing edge is of 10 mm thickness in the area from 0.7m to 2.0m and 5mm thickness from 2.0m to 3.0m. Since gel coat, CSM 300 and foam do not have a significant contribution to the strength of the blade, these materials are not included in the Finite Element Model.

After geometric modeling of the blade, the material model of the blade structure is prepared in Ansys ACP/Pre module. Plane stress SHELL 181 quadrilateral elements are used to mesh the blade entirely in ANSYS Workbench as seen in Figure 5. The total number of elements in FE Model is 48,326. Depending on the stress concentration regions, mesh density is increased in specific regions of the blade. The fine mesh density is chosen on the blade leading and trailing edges for detailed investigation of the stress distribution in these areas as depicted in Figure 5. Decision for the mesh refinement is based on the compromise between accuracy and computing time. The analysis takes about four hours with an Intel Core i7 Desktop PC with 16 GB RAM.

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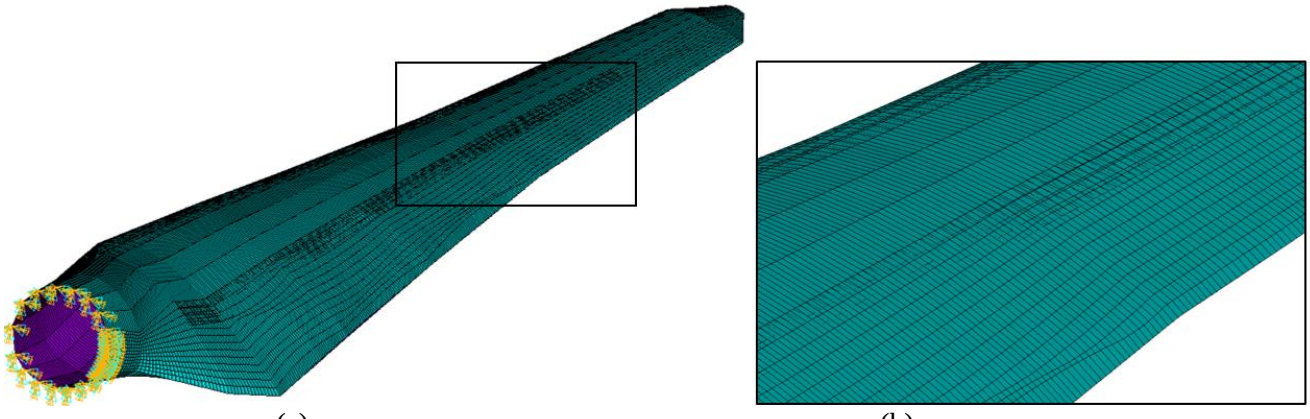
For the simulation of the virtual full-scale test, all rotational and translational degrees of freedom are fixed at the blade root as seen in Figure 5. We note that, through-the-thickness stresses cannot be calculated with plane stress element type SHELL 181 used in the model. Adhesive materials are used for connections: spar - suction side - pressure side, internal flange - suction side - pressure side along leading edge and finally between suction side - pressure side along trailing edge. These connections are simulated using bonded contact in the FE Model. Geometric nonlinearity is not included in the simulations.



(a) (b)

Figure 5. (a) Boundary conditions and the mesh density of the FE Model, (b) blade detail.

As stated in the technical report prepared by Weinzierl and Pechlivanoglu (2013), the turbine specifications are obtained from the meteorological data in Ankara, Turkey. These were analysed so that average wind speed, occurrence of gusts and wind speeds are determined. Based on this information the turbine specifications were selected according to British Standard IEC 61400-2 standard (2006). Worst case load scenario is chosen among the complete set of IEC extreme loads provided from aero-elastic simulations. The extreme loads are computed using the wind turbine aero-hydro-servo-elastic software tool FAST version v7.01.00a-bjj. During the simulations the turbine is simulated as a stall regulated constant speed turbine at 83 rpm with a gearbox and simple induction generator. Using this data, the blade is analysed under extreme loads in the flap-wise and edge-wise directions. Loads are calculated at 28 stations along the blade span direction.

According to the recommendations of IEC 61400-23:2002(2015) standard a partial safety factor of 1.35 is included in the loads for the virtual full-scale static tests. After considering partial safety factors Figure 6 shows the calculated values of the flap-wise and edge-wise bending moment in radial sections along blade span length, which are calculated from flap-wise and edge-wise loads. In a similar manner, extreme flap-wise and edge-wise external load distributions are plotted in Figure 7 below. As stated in IEC 61400-23 standard, for the virtual full-scale test, external flap-wise and edge-wise loads are incrementally increased, until collapse of the blade occurs.

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In order to avoid stress singularities in the FE Model, extreme flap-wise and edge-wise external loads are given over 28 stations of the blade suction side as displayed in Figure 8(a) and Figure 9(a). From Figure 8(b) and Figure 9(b) it is seen that the load at these stations are distributed among the nodes along the spar width on the suction and pressure sides.

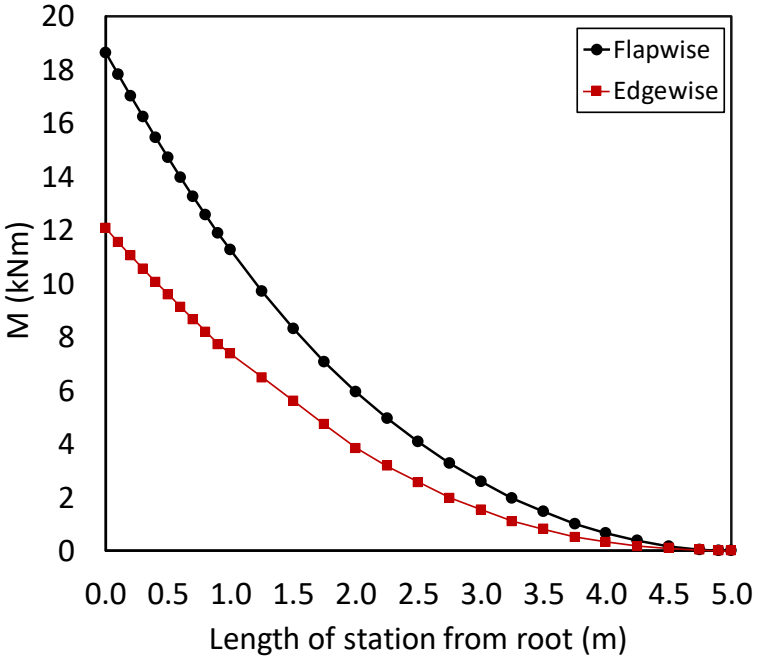

Figure 6. Extreme moment distribution along span length.

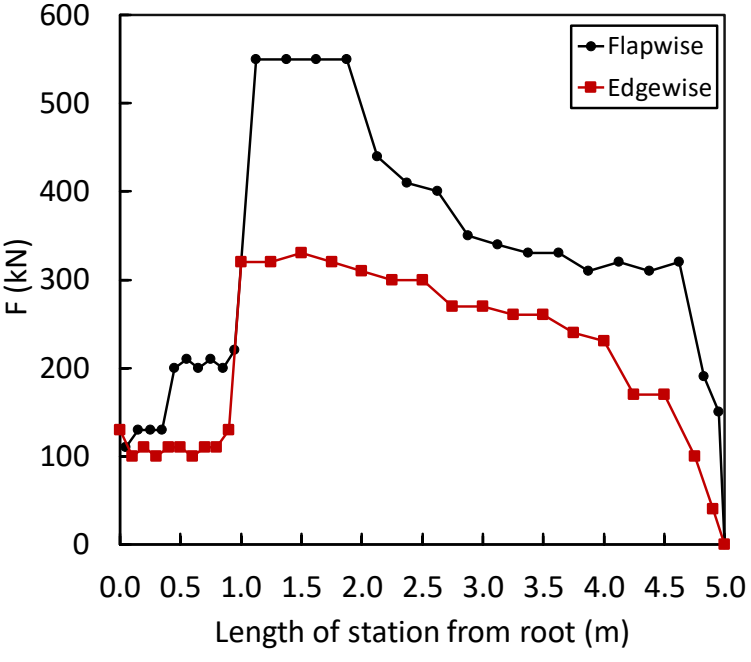

Figure 7. Extreme external load distribution along span length.

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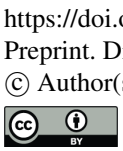

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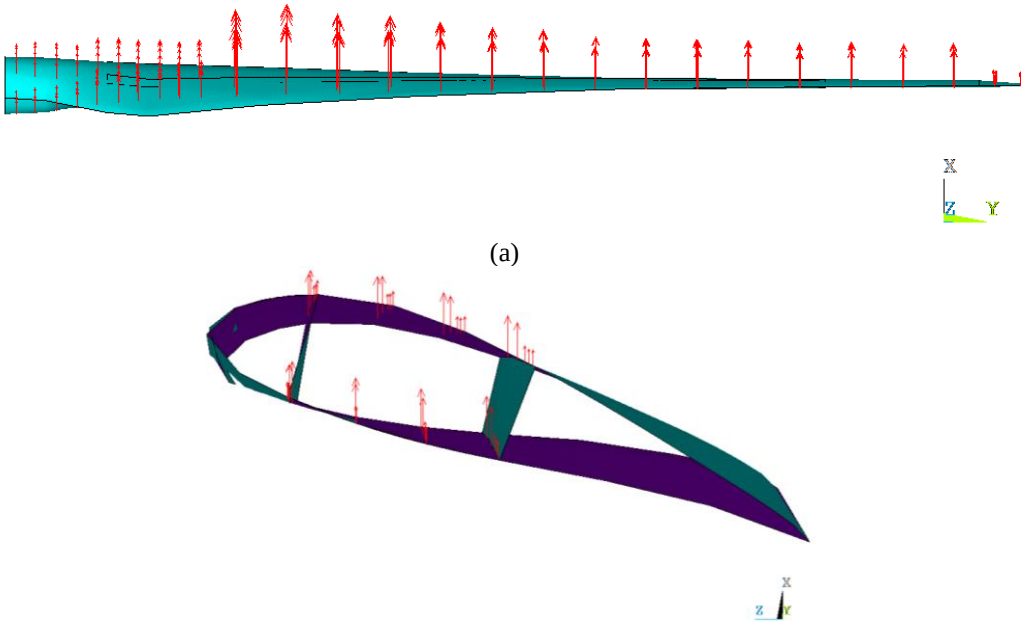
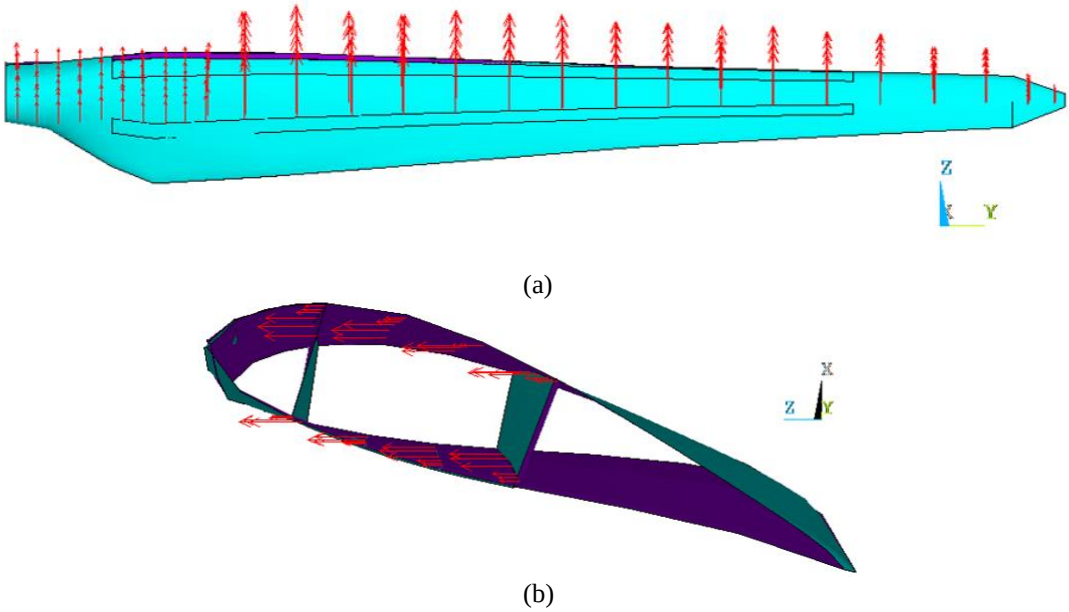


(a)

(b)

Figure 8. Extreme flap-wise load application to blade FE Model: (a) top and (b) side view.



(a)

(b)

Figure 9. Extreme edge-wise load application to blade FE Model: (a) top and (b) side view.

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3 Results and discussion

In this section, results from the virtual full-scale of the blade under flap-wise, edge-wise and combined edge-wise and flap-wise loading extreme loading conditions are presented. At the end of the section a comparative study of these three loading scenarios are given.

3.1 Virtual full-scale testing under flap-wise loading

Load displacement curves in the range between 10% - 150% of extreme flap-wise loading of the blade are displayed for the linear Puck model and nonlinear progressive Puck model in Figure 10. Loads are computed from the reaction forces at the blade root and the displacement is measured at a node near blade tip. We observe from the figure that, up to 90% of the extreme flap-wise loading the stiffness for both models remain almost the same. After 90% loading, stiffness reduction starts in the nonlinear progressive Puck model due to the first element failure at the internal flange tip. According to the simulation output data, elements begin to fail due to FF and/or IFF(C) starting from this loading increment as will be shown in the next section. Element failure failures in the internal flange are followed by the failures in the trailing edge. In addition to this, the output data show that degradations in the transverse elasticity, shear moduli and Poisson's ratio starting at 40% loading occur, but this does not play a major role in the deflection of the blade as can be seen from the figure. When loading is further increased up to 130% the deflection of the nonlinear model is more than twice the deflection of the linear model. At this point blade is considered to be close to ultimate failure. Finally, the blade collapses after 150% of extreme flap-wise loading.

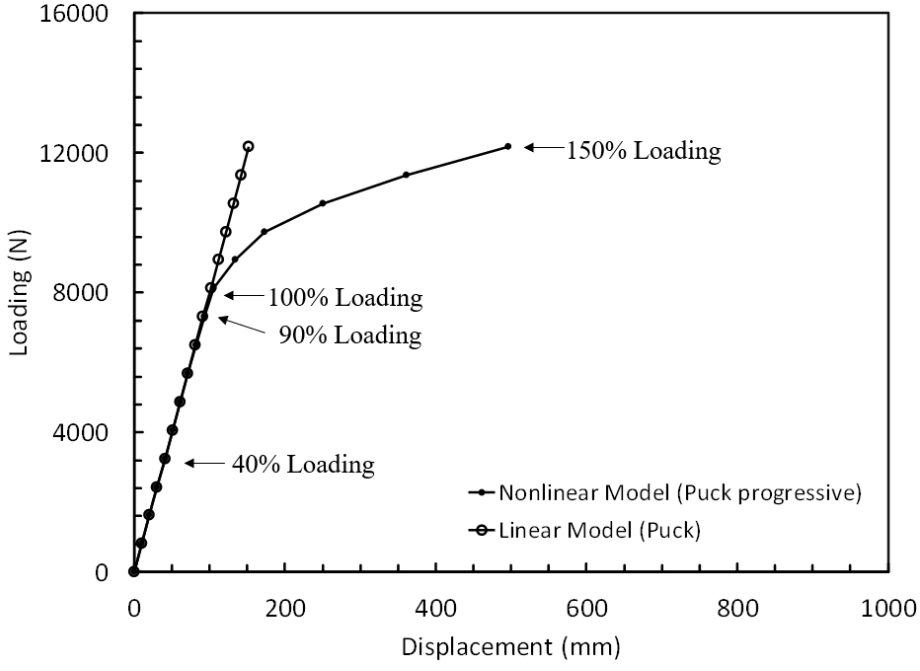

Figure 10. Load displacement curves of the blade using linear Puck model and nonlinear Puck progressive model under flap-wise loading.

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The total deformation of the nonlinear blade model versus undeformed model under 100% extreme flap-wise loading is displayed in Figure 11. Maximum blade deflection at the blade tip is 111 mm.

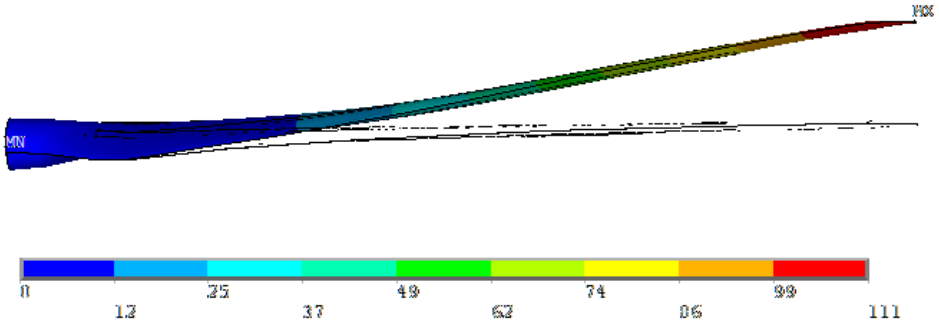

Figure 11. Total deformation of the nonlinear METUWIND blade model vs. undeformed model under 100% extreme flap-wise loading condition (scale factor: x5).

During full-scale test, internal flange and suction side of the blade are the two parts, which are damaged primarily. Since internal flange is rather thin, for visualization reasons damage propagation in the suction side of the blade is presented. Figure 12 shows the evolution of the failed elements in the suction side of the blade at 90%, 100%, 120% and 150% of extreme flap-wise loading. According to the implemented methodology, an element fails if FF or IFF(C) in three or more plies of an element is detected. Element failure begins at 90% loading in the trailing edge and the failed elements are shown in black. As seen from the figure number of failed elements increase along the trailing edge towards the blade tip and root as the load is further increased to 120% and the suction side heavily damaged at 150%.

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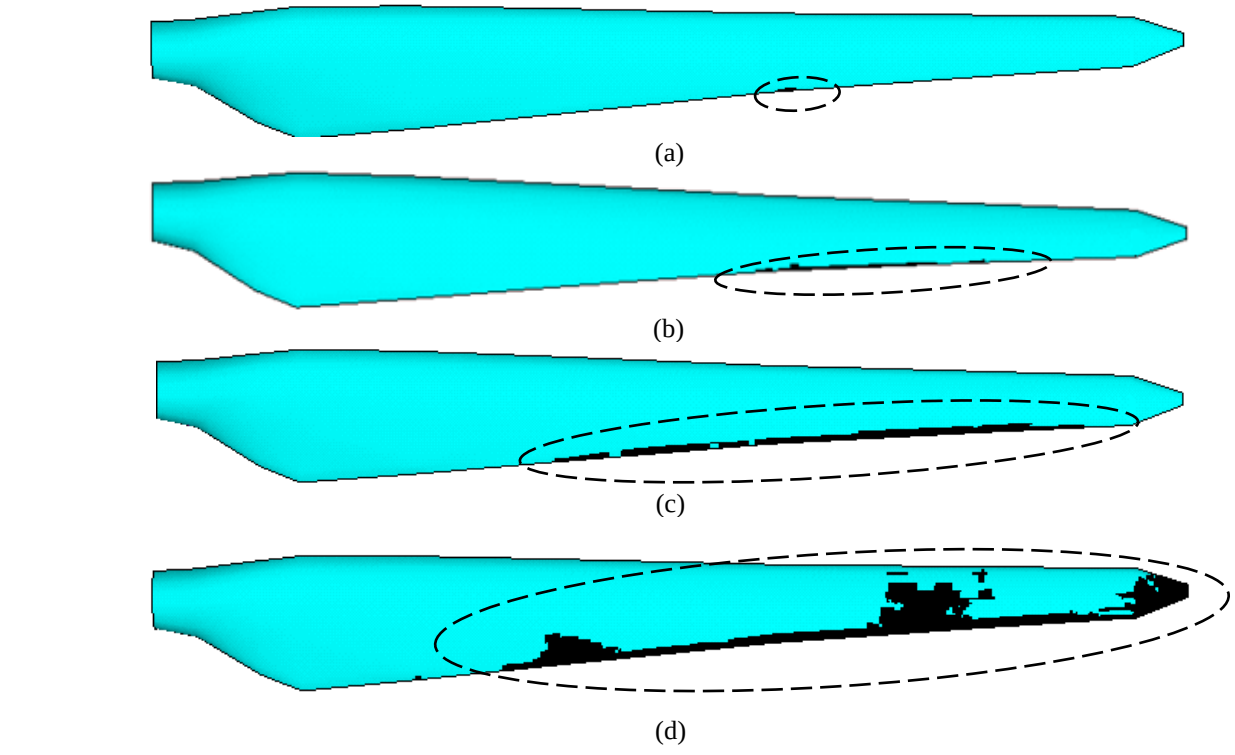

Figure 12. Element failure progression on the suction side of the blade at (a) 90%, (b) 100%, (c) 120% and (d) %150 of extreme flap-wise loading.

In Figure 13. inter-fiber failure mode A (IFF(A)) and inter-fiber failure mode B (IFF(B)) distribution in the suction side are shown on the same plot. If both failure exposures are present in an element the higher failure exposure IFF(A) or IFF(B) is shown. Based on the output data from the virtual full-scale test of the blade, the damage initiation begins at 40% of extreme flap-wise loading due to IFF(A) and/or IFF(B). Figure shows that, inter-fiber failure starts at the trailing edge of the suction side towards spar end. As discussed in the methodology chapter, failure exposures greater than or equal to one indicate damage and damaged regions are shown in red. Under 40% to 100% loading damage propagates by increasing load increments as shown in Figure 14. From the regions, where inter-fiber failure values are high detailed section D is created on the blade suction side. This detailed section is used to study the damage evolution in Figure 14. After the beginning of failure at 40%, the damaged regions grow along the trailing edge towards blade tip and root. It is worth noting that, IFF(A) and IFF(B) do not lead to the element failure. When IFF(A) or IFF(B) occur, only the transverse, shear moduli and Poisson's ratio are reduced according the degradation rules. As seen from Figure 10, degradation in the stiffness and Poisson's ratio does not lead to the change in the slope of the force-displacement curve. This means there are not considerable changes in the blade stiffness. However, at 100% load, there are failed elements due to FF and/or IFF(C) as will be discussed in the preceding paragraphs and at this point the slope of the force-displacement curve, i.e. blade stiffness decreases. Since the IFF(A) and/or IFF(B) of the failed elements are zero, they appear as dark blue regions inside red areas. When the elements

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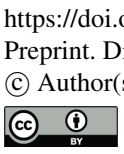

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fail, they do not contribute to strength anymore and the load will be carried by the neighbouring elements. As a result, IFF(A) and/or IFF(B) evolve around the failed elements.

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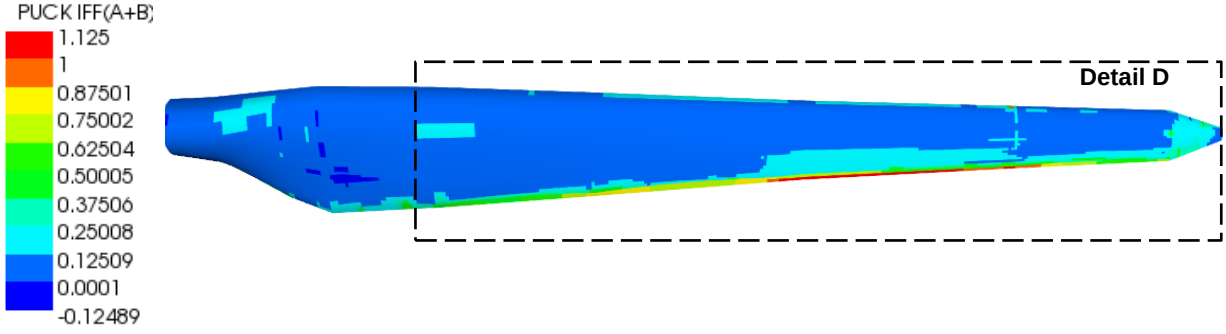

Figure 13. Detail section D from the suction side of the blade at 40% of extreme flap-wise loading.

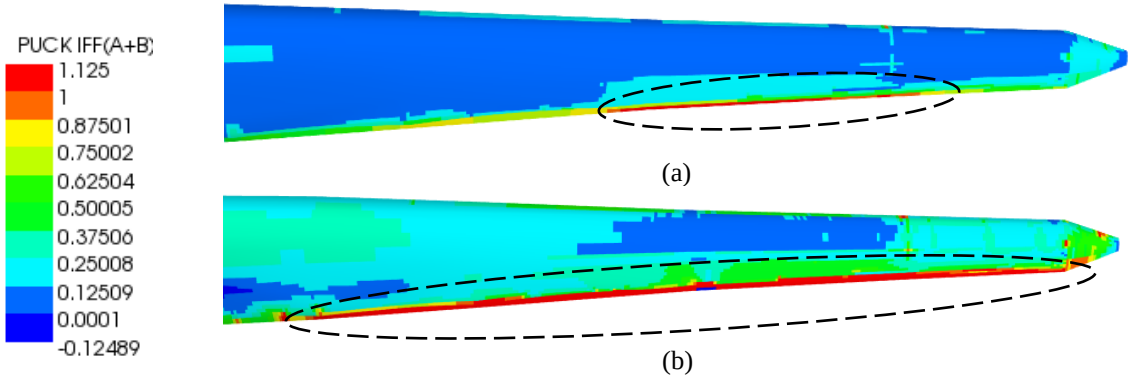

Figure 14. Failure progression on the suction side of the blade at (a) 40% and (b) 100% of extreme flap-wise loading in detail D.

270 Damage progression regarding IFF(C) and FF in detail section D of the blade is further investigated in Figure 15. In Figure 15 fiber-failure (FF) and inter-fiber failure mode C (IFF(C)) distribution in the suction side are shown on the same plot. If both failure exposures are present in an element the higher failure exposure FF or IFF(C) is shown. In the figure, failure exposures are plotted under 90%, 100%. 120% and 150% loading. At 90% loading the regions depicted in red are the regions where the value of the failure index exceeds 1 and the stiffnesses of these elements are set to zero. As a consequence, in the 275 following load increment of 100% loading the regions, which appear in red at 90% loading become dark blue. This is due to the fact that, after setting the stiffness to zero their FF is zero and these regions do not contribute to the strength of the blade anymore. We note that these dark blue regions correspond to the failed elements in Figure 12. At 110% loading the failed

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region evolves along the trailing edge as depicted in red. The new failed regions together with the failed elements from previous runs are represented by the failed elements in black in Figure 12. We note FF and IFF(C) initiate in the same location as IFF(A) and/or IFF(B).

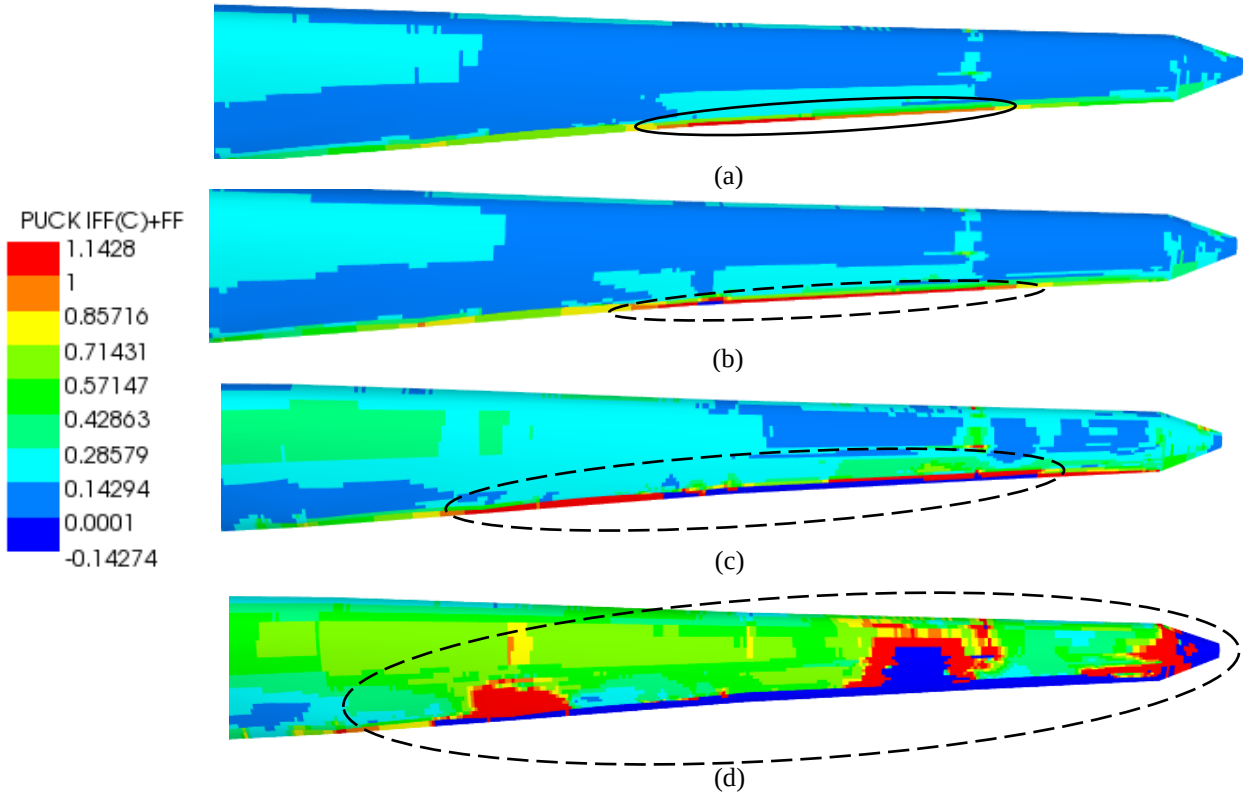

Figure 15. Failure progression on the blade suction side at (a) 90%, (b) 100%, (c) 120% and (d) %150 of extreme flap-wise loading in detail D.

3.2 Virtual full-scale testing under edge-wise loading

Load displacement curves in the range between 10% - 350% of extreme flap-wise loading of the blade are displayed for the linear Puck model and nonlinear progressive Puck model in Figure 16. Loads are computed from the reaction forces at the blade root and the displacement is measured at a node near blade tip. We observe from the figure that, up to 320% of the extreme flap-wise loading the stiffness for both models remain almost the same. After 320% loading, stiffness reduction starts in the nonlinear progressive Puck model due to the first element failure at the internal flange tip. Finally, at %350 loading blade is close to collapse.

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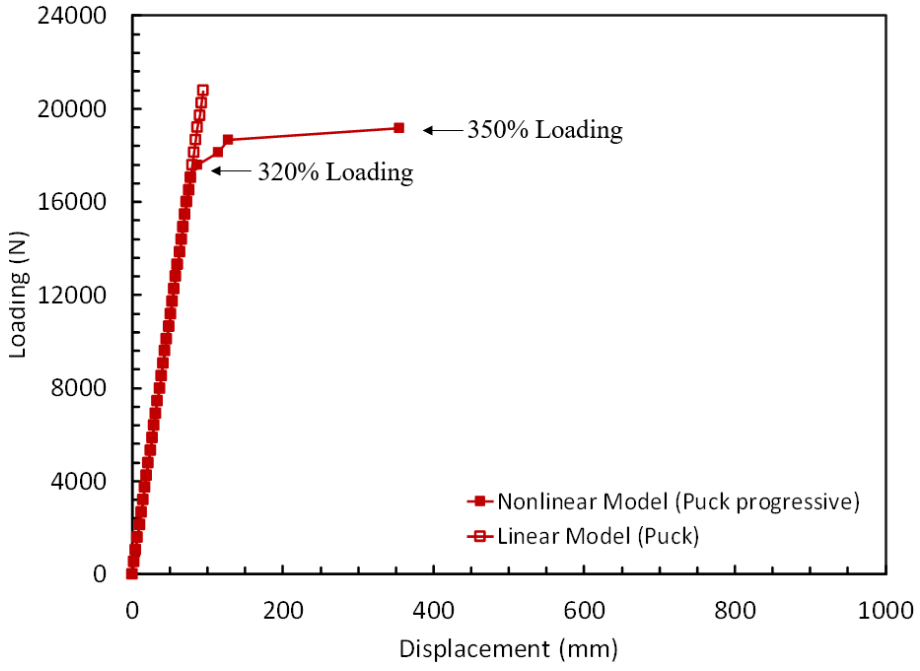

Figure 16. Load displacement curves of the blade using linear Puck model and nonlinear Puck progressive model under edge-wise loading.

The total deformation of the nonlinear blade model versus undeformed model under 100% extreme flap-wise loading is displayed in Figure 17. Maximum blade deflection at the blade tip is 27 mm and much less than the deformation compared to pure flap-wise loading. Figure 18 shows the evolution of the failed elements due to FF and IFF(C) in the suction side of the blade at 320%, 330%, 340% and 350% of extreme edge-wise loading. Element failure in the trailing edge near root begins at 330% loading and the failed elements are shown in black. We observe from Figure 18, that since the spar has more structural strength than the suction side in edge-wise direction, damage grows inside the suction side further and ends at the boundary where the spar is located until %340 loading. As seen from the figure number of failed elements increase in the trailing edge towards the spar and root as the load is further increased to 350%. Starting from 350% the part of the suction side connected to the spar caps is heavily damaged.

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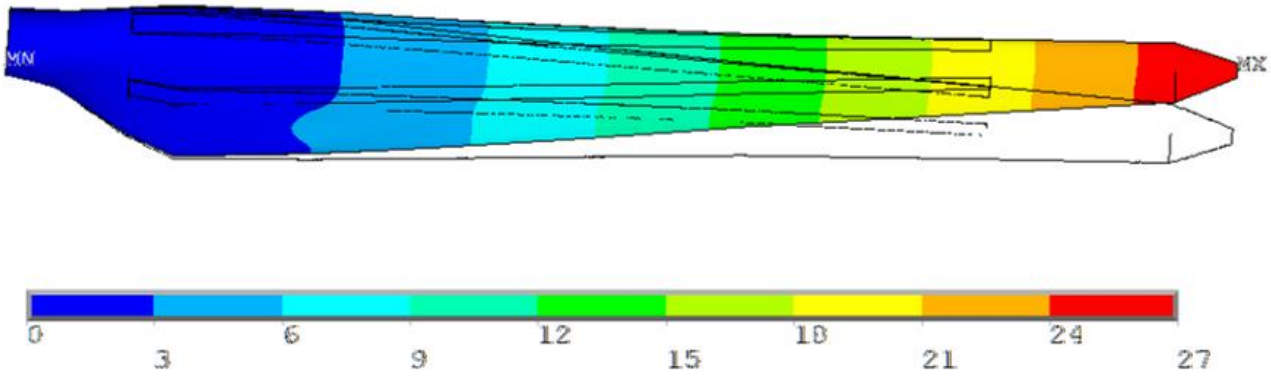

Figure 17. Total deformation of the nonlinear METUWIND blade model vs. undeformed model under 100% extreme edge-wise loading condition (scale factor: x10).

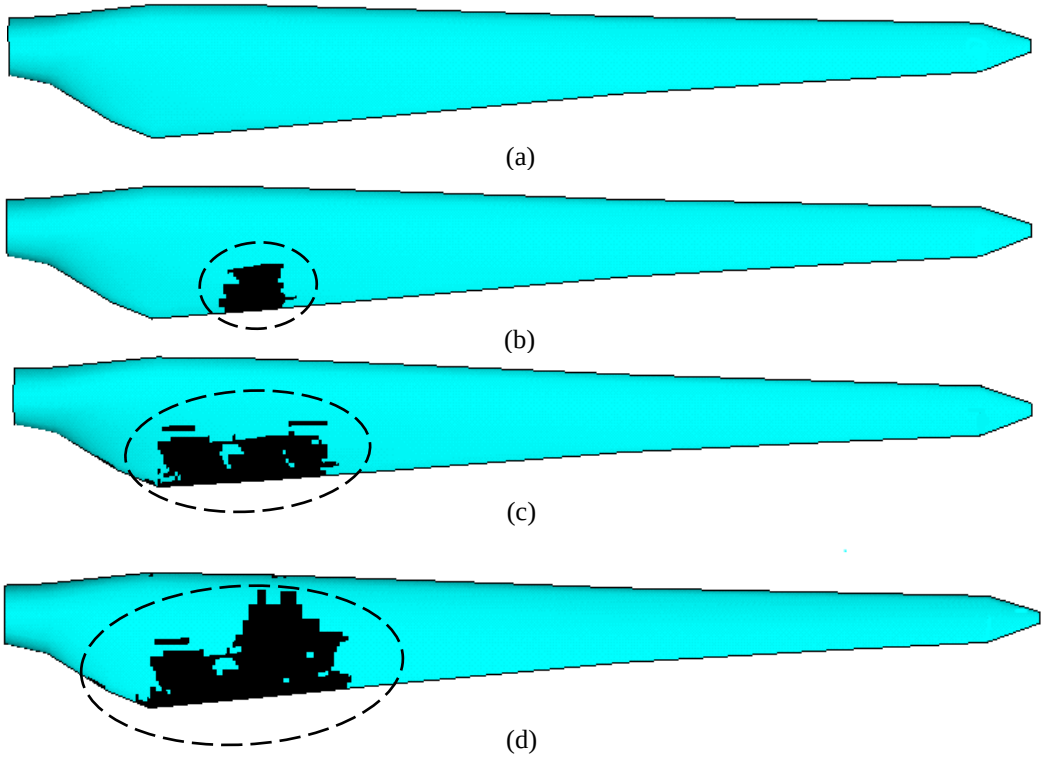

Figure 18. Element failure progression on the suction side of the blade at (a) 320% (no element failure) (b) 330%, (c) 340% and (d) 350% of extreme edge-wise loading.

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In Figure 19 fiber-failure (FF) and inter-fiber failure mode C (IFF(C)) distribution in the suction side are shown on the same plot. In the figure, failure exposures are plotted under 320%, 330%, 340% and 350% loading. In the regions where the value of the failure index exceeds 1 and the stiffnesses of these elements are set to zero. The regions appear as dark blue regions and correspond to the failed elements in Figure 18. At 340% and %350 loading cases the failed region evolves along the trailing edge as depicted in red. The new failed regions together with the failed elements from previous runs are represented by the failed elements in black in Figure 18. Based on the results, blade design exhibits excessive safety in edge-wise direction and is considered to be overconservative for this type of loading.

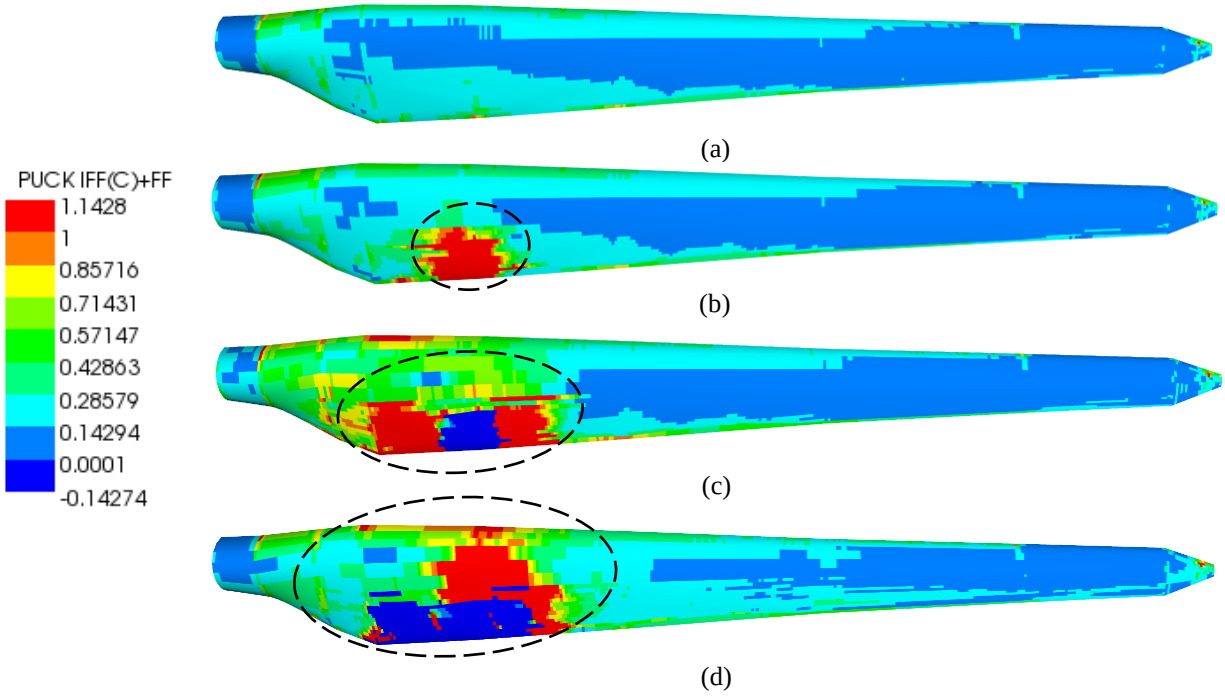

Figure 19. Failure progression on the blade suction side at (a) 320%, (b) 330%, (c) 340% and (d) 350% of extreme edge-wise loading.

3.3 Virtual full-scale testing under combined edge-wise and flap-wise loading

Load displacement curves in the range between 10% - 160% of combined extreme flap-wise and edge-wise loading of the blade are displayed for the linear Puck model and nonlinear progressive Puck model in Figure 20. Loads are computed from the reaction forces at the blade root and the displacement is measured at a node near blade tip. We observe from the figure that, up to 110% of the extreme flap-wise loading the stiffness for both models remain almost the same. After 110% loading, stiffness reduction starts in the nonlinear progressive Puck model due to the first element failure at the internal flange tip. Finally, at %160 loading blade is close to collapse.

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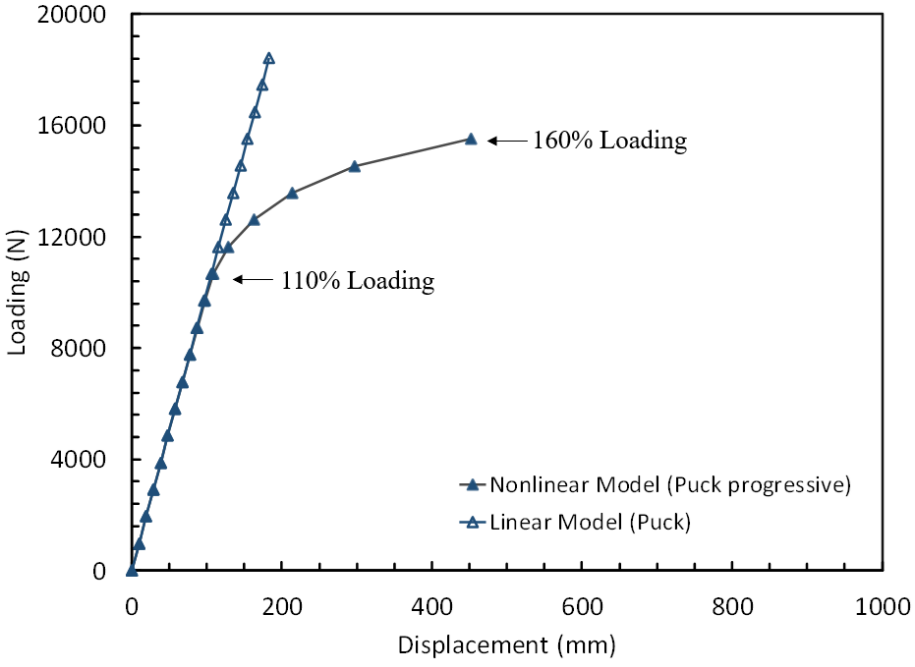

335 **Figure 20.** Load displacement curves of the blade using linear Puck model and nonlinear Puck progressive model under combined edge-wise and flap-wise loading.

The total deformation of the nonlinear blade model versus undeformed model under 100% combined extreme flap-wise and edge-wise loading is displayed in Figure 21. Maximum blade deflection at the blade tip is 103 mm and less than the deformation compared to pure flap-wise loading. We further note that, the deflection in edge-wise direction is less than the
340 deflection in span-wise direction. Figure 22 shows the evolution of the failed elements due to FF and IFF(C) in the suction side of the blade at 100%, 110%, 120% and 160% of extreme edge-wise loading. As in pure flap-wise loading case element failure distribution represented in black. Yet under the same load increment the number failed elements are less compared to pure flap-wise loading case. This observation is due to resistance, i.e. stiffer behavior of the blade to bending under combined loading case. Element failure in the trailing edge begins at 110% loading. Similar to the pure flap-wise loading,
345 number of failed elements increase in the trailing edge towards the spar and root as the load is further increased up to 160% loading. We observe from Figure 22, that since stiffness of the spar in flap-wise direction is less compared to pure edge-wise loading at 160% combined loading damage grows inside the suction side further where the spar is located.

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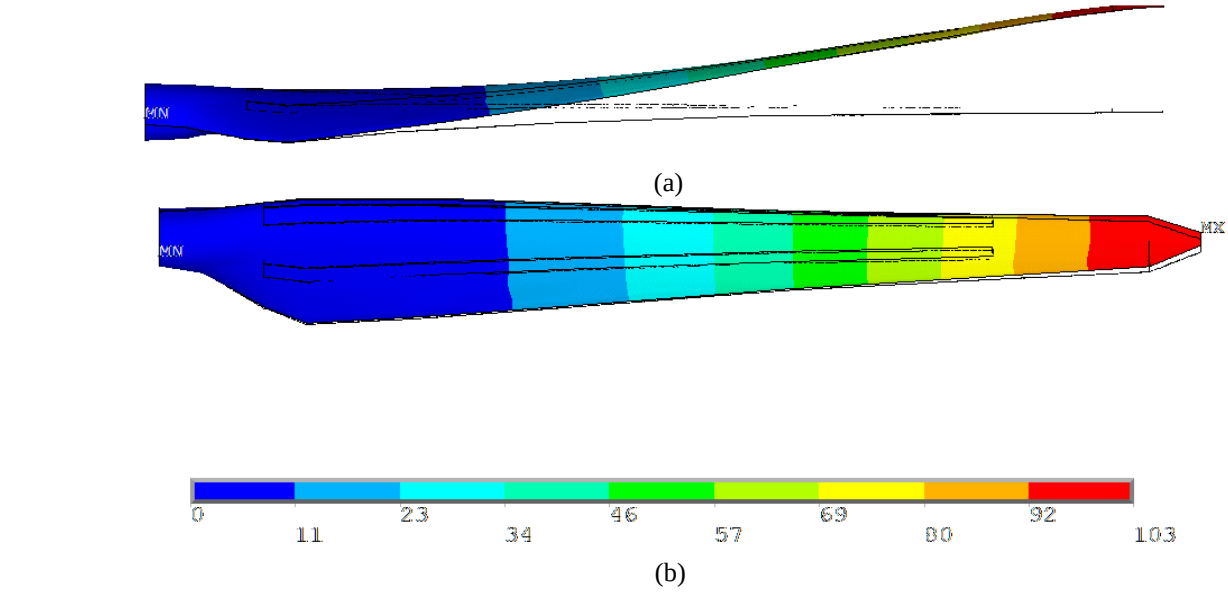

Figure 21. Total deformation of the nonlinear METUWIND blade model vs. undeformed model under 100% combined extreme edge-wise and flap-wise loading condition (a) side view (b) top view (scale factor: x5).

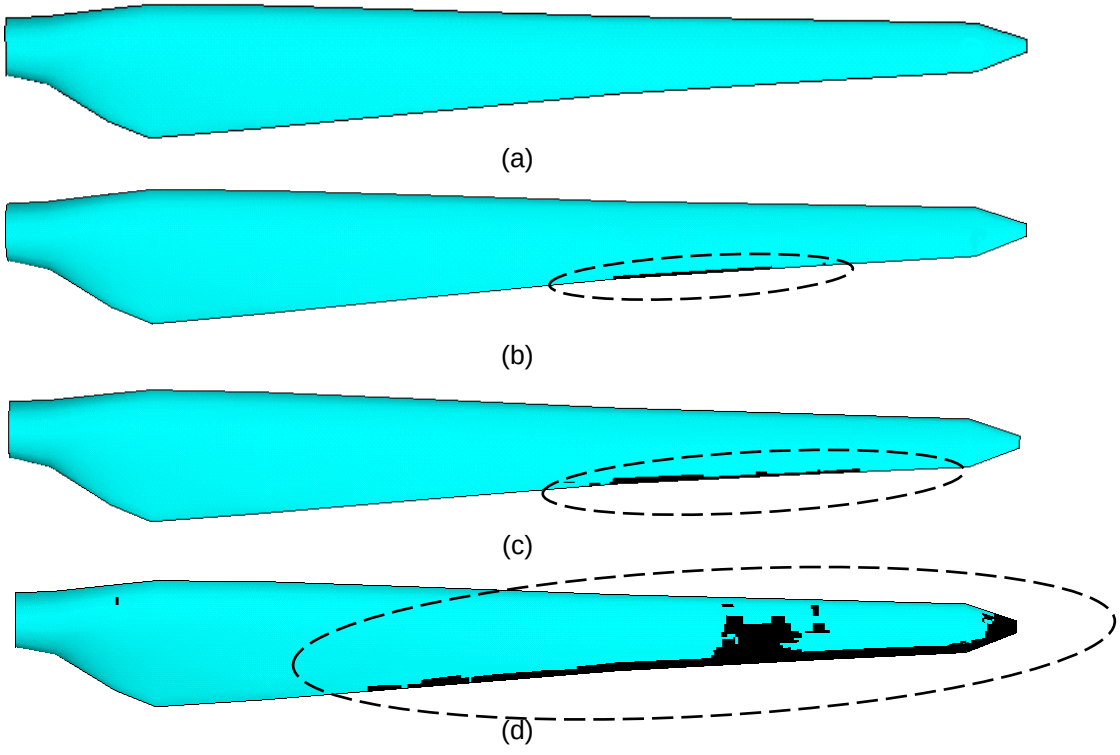

Figure 22. Element failure progression on the suction side of the blade at (a) 100%, (b) 110%, (c)120% and (d) 160% of combined extreme flap-wise and edge-wise loading.

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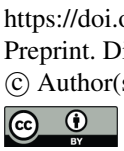

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In Figure 23 fiber-failure (FF) and inter-fiber failure mode C (IFF(C)) distribution in the suction side caused by combined flap-wise and edge-wise loading are shown. In the figure, failure exposures are plotted under 100%, 110%, 120% and 160% loading. The regions where the value of the failure index exceeds 1 are depicted in red and stiffnesses of these elements are set to zero. The regions appear as dark blue regions and correspond to the failed elements in Figure 22. At 120% and %160 loading cases the failed region evolves along the trailing edge as depicted in red. The new failed regions together with the failed elements from previous runs are represented by the failed elements in black in Figure 22. When the results in Figure 23 are compared with Figure 15, one can conclude that the damaged area under combined loading is less than pure flap-wise loading case. In combined loading case and pure flap-wise loading the damage growth begins and continue along the trailing and leading edges. Since adhesive is the weakest interface at these locations, main failure mechanism is expected to be debonding and/or delamination in this region before FF or IFF(C) occurs. Furthermore, bearing in mind that adhesive is modelled as bonded contact, current modeling technique is not capable of simulating the progression of debonding. At this point it is worth noting that, plane stress elements, which are used to model the blade are not able to show through-the-thickness stresses, which may help to trigger debonding/delamination failure mechanisms. Based on the results, blade design is considered to be safe for the combined load case.

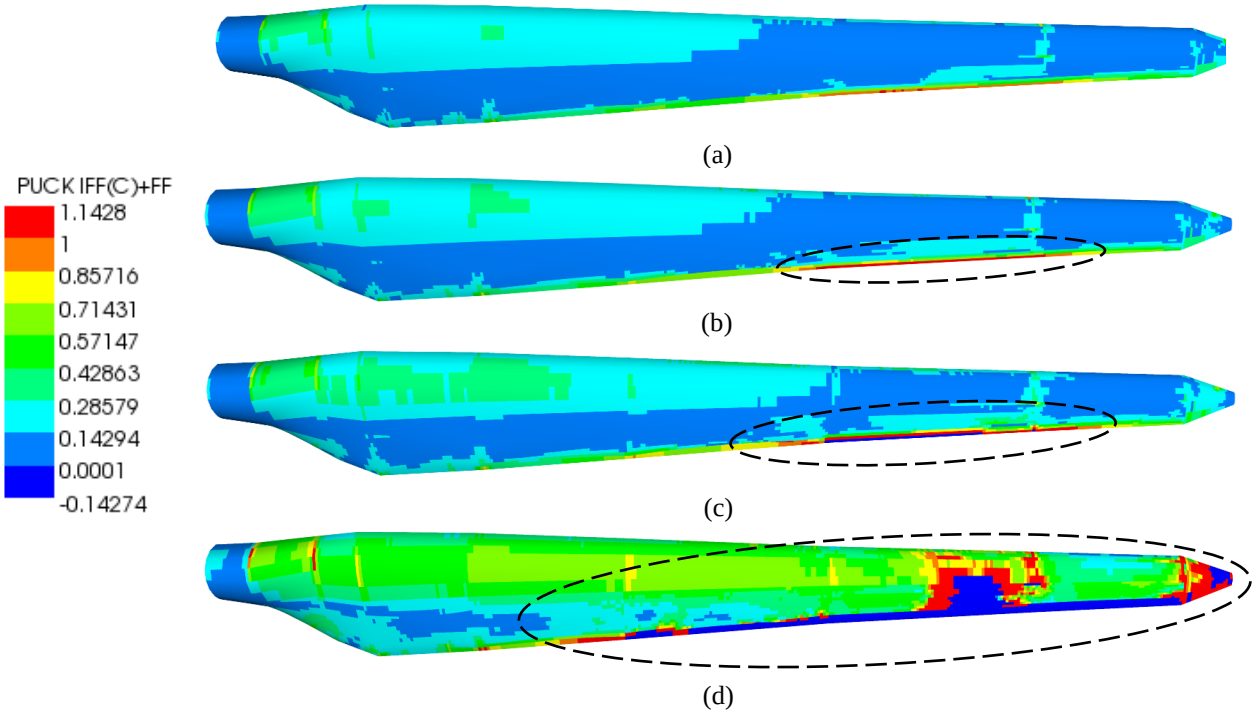

Figure 23. Failure progression on the blade suction side at (a) 100%, (b) 110%, (c)120% and (d) 160% of combined extreme flap-wise and edge-wise loading.

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3.4 Comparative study of virtual full-scale testing under flap-wise, edge-wise and flap-wise and edgewise loading

In Figure 24 load-displacement curves of the blade under edge-wise, flap-wise and combined edge-wise plus flap-wise loading conditions are plotted on the same graph for comparison reasons. In the figure resultant loads at the blade root versus. total displacements at a location near blade tip are plotted. We see that the slope of the load-displacement curve is highest for edge-wise loading followed by edge-wise plus flap-wise loading. For flap-wise loading the slope of the load-displacement curve is the lowest. Since slope is a measure of stiffness, the blade exhibits stiffest behavior in the pure edge-wise loading direction, followed by the edge-wise plus flap-wise loading and the pure flap-wise loading.

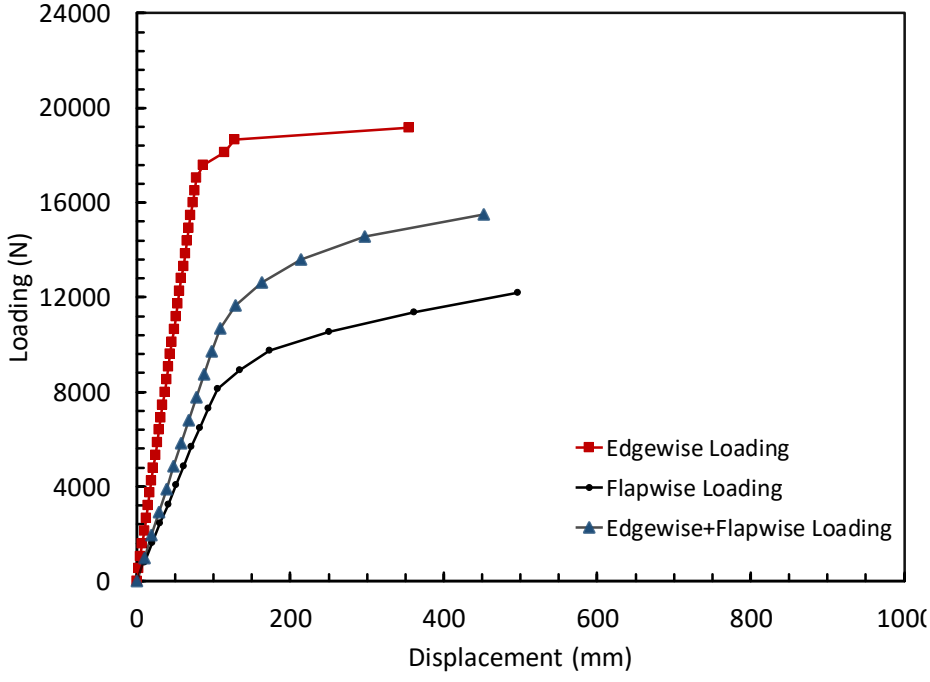

Figure 24. Load displacement curves of the blade under edgewise, flap-wise and edgewise plus flap-wise loading.

As a further investigation, the structural responses of the blade under pure flap-wise versus combined edgewise loading and pure edgewise versus combined edgewise and flap-wise loading are depicted in Figure 25 and Figure 26, respectively. In Figure 25 instead of the resultant component of the combined edgewise and flap-wise loading its flap-wise component, i.e. load and displacement in flap-wise direction is plotted. In a similar manner in Figure 26 for the combined loading case edgewise component, i.e. load and displacement in edgewise direction is plotted. It is seen that in both cases, the blade exhibits a stiffer behavior under combined loading condition compared to pure loading cases. In Figure 27 variation the METUWIND blade deflection along spanwise direction measured from the chord center is shown. It is seen that, in all load cases the slope of the curve is small at the blade root and increases towards the tip and tends to decrease at the blade tip.

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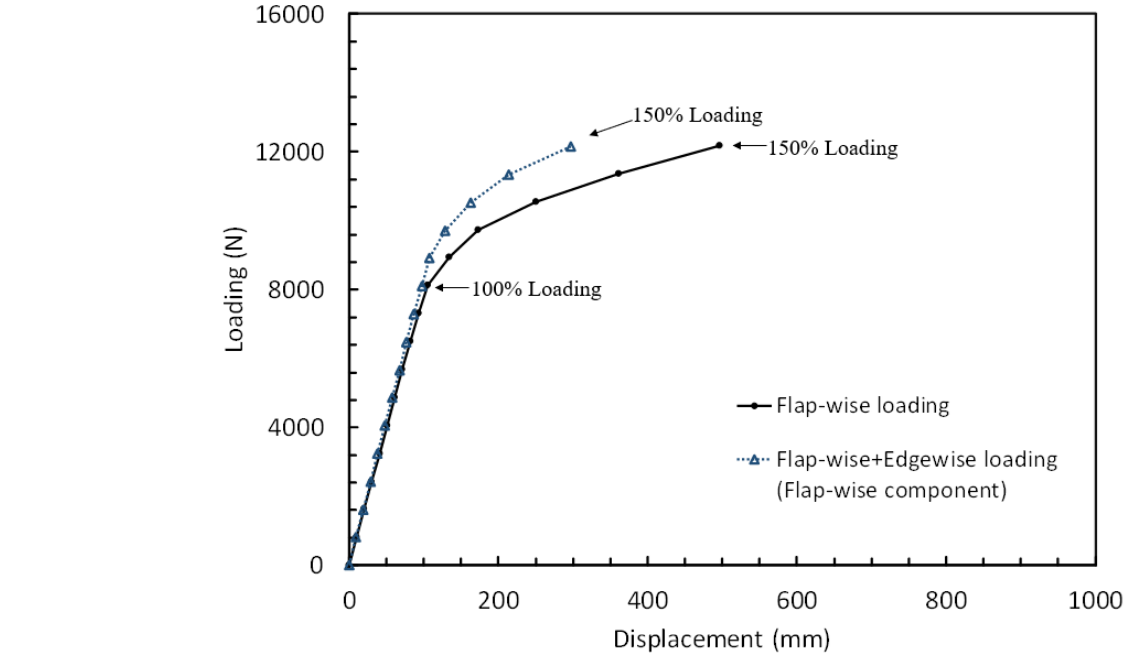

Figure 25. Load-displacement curves of the blade under pure flap-wise and combined edgewise plus flap-wise loading with flap-wise component plotted.

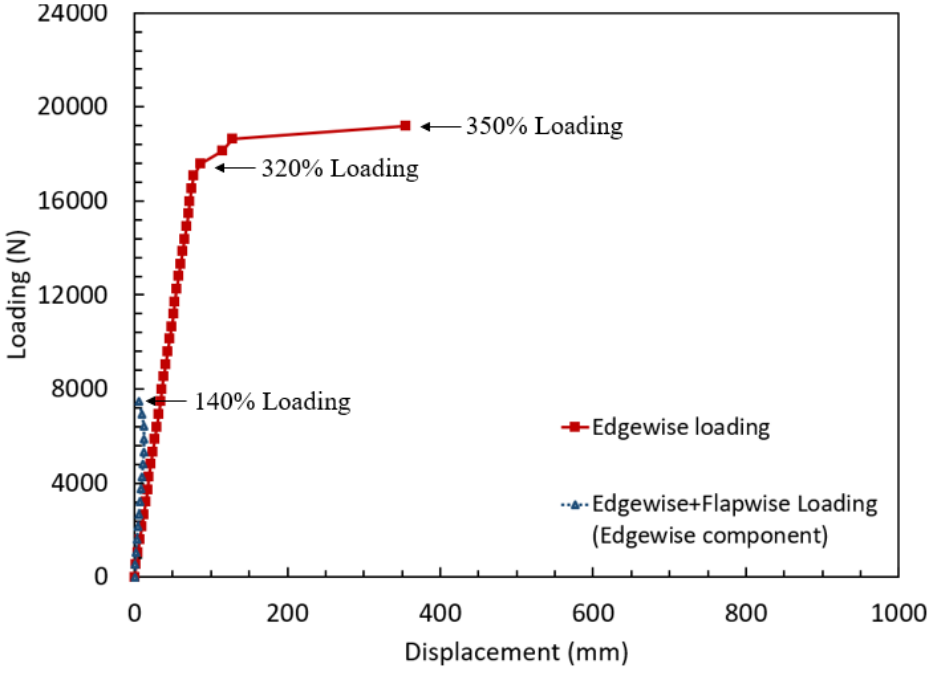

Figure 26. Load-displacement curves of the blade under pure edgewise and combined edgewise plus flap-wise loading with edgewise component plotted.

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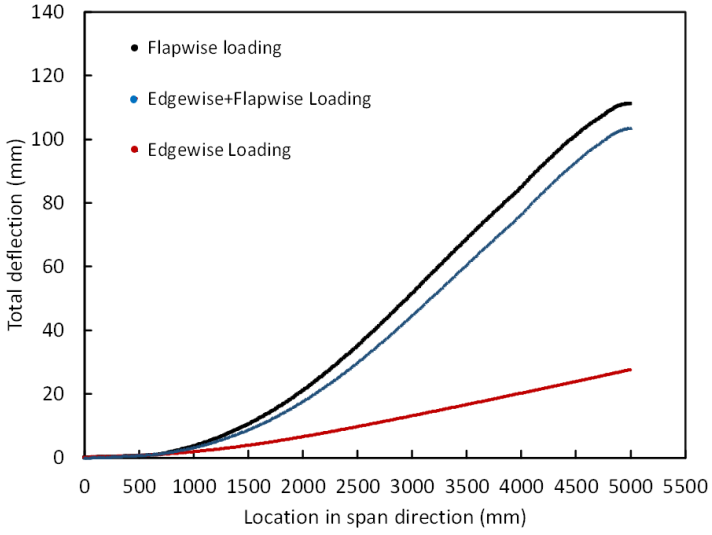

Figure 27. Variation of the deflection of the METUWIND blade along spanwise direction under 100% loading condition measured at cord center.

In Figure 28 element failure progression is compared for edge-wise, flap-wise and combined loading scenarios under %120 resultant loading. We observe that the degree of failed region is highest for flap-wise loading case, followed by combined loading and there is no failed region in the edge-wise loading case. This failure development can be explained by the blade stiffness as the blade shows the same trend for stiffness variation under the related loading conditions as discussed in Figure 24 and Figure 27.

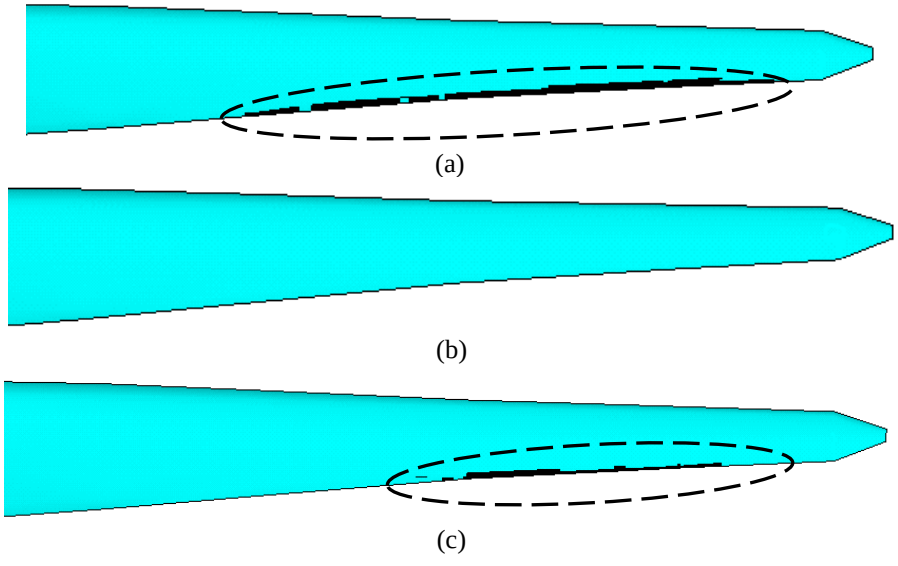

Figure 28. Element failure progression on the suction side of the blade at (a) flap-wise, (b) edge-wise (no element failure) and (c) combined flap-wise and edge-wise 120% extreme loading case.

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4 Conclusions

In this work, virtual full-scale test of an existing 5-meter composite wind turbine blade under extreme flap-wise, edge-wise and combined flap-wise and edge-wise loading conditions is conducted using Puck material model in order to predict dominant failure modes and failure exposures before testing. Failure on laminate level is investigated using Puck failure criteria (2D). The main conclusions are as follows:

1. Failure of elements are observed and the blade is found to deflect extensively after 90% of the pure extreme flap-wise loading, ultimate failure is expected to occur. In contrast, element failures are not observed under pure edge-wise and combined edge-wise and flap-wise extreme load cases until 100% extreme loading.
2. Less damage evolution is observed under flap-wise loading compared to combined edge-wise and flap-wise loading. This damage progression can be explained by the fact that the blade exhibits more resistance to bending, i.e. stiffer when it is subjected to combined loading.
3. Plots of the load displacement curves of the blade under extreme loading conditions show that slope reduction in the curves, i.e. stiffness reduction becomes notable only after failure of elements due to IFF(C) and/or FF. Shear and transverse stiffness reductions caused by IFF(A) and IFF(B) do not lead to significant changes in the overall structural stiffness of the blade.
4. Failure initiation begins with inter-fiber failure (IFF(A) and/or IFF(B)) and as the load is further increased element failure due to fiber failure or explosive failure mode IFF(C) occurs in the same failure initiation region.
5. Based on the identified failure mechanism and simulation results, spar design seems to be overconservative under edgewise-loading, but strength improvement is needed for pure flap-wise loading. This can be achieved by increasing area moment of inertia of the spar for pure flap-wise loading case.
6. Since adhesive is the weakest interface, the main failure mechanism is expected to be the de-bonding at the trailing and leading edges. Plane strength elements are not capable of capturing the main failure mechanism, which is triggered by the through-the-thickness stresses.

Author contributions. CM developed the method, conducted numerical simulations and wrote the paper. DC is the supervisor and guided CM for the conception of the ideas and participated in structuring and review of the paper.

Data availability. The data that supports the results of this research are not publicly available due to confidentiality reasons.

Competing interests. The authors declare that they have no conflict of interest.

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