# Reply to the reviewers' comments for paper "On the power and control of a misaligned rotor – Beyond the cosine law"

The authors would like to thank the two reviewers for their time and for the useful feedback. All inputs that they provided have contributed to the improvement of the paper.

A list of point-by-point replies to the reviewers' comments is reported in the following. The reviewer's comments are in black, and our replies in blue.

In addition to the extensive rewriting of entire sections, which also contain new results and additional bibliographical references, we have taken the opportunity of this deep revision to make several small editorial changes to the text in order to improve readability.

A revised version of the manuscript is attached to the present reply, with additions highlighted in blue and deletions marked in red.

The authors

# **Reviewer 1**

The paper presents a new analytical relationship to predict the variation of thrust and power coefficients with the yaw angle. This is an important and timely topic, as an accurate estimation of these two dimensionless quantities is of great importance in implementing wake steering. The power coefficient directly determines the amount of power loss on the turbine that is yawed, and the thrust coefficient is an important parameter in engineering wake models used to estimate the amount of wake deflection in yawed conditions. Two assumptions are made to develop the model: (i) the relative flow angle is small, which is valid for high-tip speed ratios, and (ii) the induction factor is uniformly distributed across the rotor disk. The latter assumption is supported by a discussion on the significance of the non-uniformity for misaligned rotors. Model predictions are compared with both LES and wind tunnel data. Overall, the paper is well-written. The work is timely and of interest to the wind energy community. An interesting coordinate transformation was performed to group both yaw and tilt effects into one combined angle (called the misalignment angle) in a bespoke coordinate system. It is great to see that all coordinate systems and parameters are very carefully and clearly defined. Detailed discussions were made on the impact of inflow shear and its effect on the asymmetrical distribution of thrust and power. I believe this work will have a high impact, especially in the field of wake steering. However, before publication, I would like to ask the authors to address the following comments (sorted by the line number) to improve the clarity and completeness of their work. Thank you for the useful feedback.

- 1. Line 44: The paragraph is too short. It can be merged with the following one. We have merged the paragraphs accordingly.
- Line 84: Verb missing in "... and Sect. 3 for its validation..." We have edited the text accordingly.
- 3. Line 112 and 117: Line 112 says that Mu is always positive, but in line 117, it has a negative value.

True, this is a mistake, we have edited the text accordingly.

- 4. Line 116 and Figure 2: I agree with the argument that different distributions of yaw and tilt angles produce a similar wake in the II plane if their misalignment angle is the same. However, this is only valid if we neglect ground effect. Cases with a large tilt angle may have a strong interaction with the ground. This can be clarified here. Thank you, we agree and we have edited the text accordingly.
- 5. Line 128: The radial and azimuthal coordinates can be shown in Figure 1d. We have modified the figure and the text as requested.
- 6. Line 125: A brief discussion on the relevance of linear shear in comparison to widely used power law or logarithmic profiles can be mentioned here. In other words, you can better justify why linear shear was used instead of those profiles. The choice was simply based on a desire to reduce the complexity of the derivations, and this has now been noted in the text.
- 7. Line 150: "r" is first defined in 128 and later in 150 with apparently two different definitions. In both cases *r* represents the radial coordinate of the rotor plane. We have now removed the second definition of *r*, which was unnecessary and confusing.
- 8. Line 199: I am not sure if I understand what 0P means here. Some clarification in the text would be useful.

As written, 0P (or zero P) stands for "constant-over-the-rotor". This term is also defined in the "Nomenclature" section.

- 9. Figure 4: I think the angle shown in the figure should be mu instead of gamma. Thank you, we have corrected the figure.
- 10. Line 207: Authors can better justify the validity of the small inflow angle approximation. For instance, the value of phi for the blade tip of a turbine with a TSR of 8 can be reported here. Thank you for the suggestion, we have added a quantification of the angle for a typical operating condition.
- 11. Line 210: Typo: "we" and "list" should be replaced with "be" and "lift". Thank you, we have corrected the typos.
- 12. Line 210: I believe the assumption of C\_L=C\_L,alpha\*alpha is only valid for a symmetrical aerofoil. This can be clarified in the text. We actually measure the angle of attack with respect to the zero-lift direction. In this case, the expression C\_L=C\_L,alpha\*alpha is valid also for a non-symmetric airfoils. This choice was made to eliminate one extra term from the (already quite complex) expressions. We understand however that we initially did not clearly explain this fact, so we have now expanded the text accordingly.
- 13. Effect of tilt angle: It is good to consider the uptilt angle effect, but it would be useful to show how big the effect is on C\_T and C\_P. Arguably, the effect on turbine performance and its wake should not be very significant. With that in mind, I suggest mentioning that the developed model will also be very useful for tilt angle control, which could become popular in the future generation of wind turbines, especially in floating turbines.

Thank you for the suggestion, we have expanded the text as per your recommendation.

14. Line 315: It is useful to report the difference between the values of C\_D and C\_L,alpha obtained from the optimization study with the typical values reported for the blade aerofoil in 2D studies.

We have now greatly expanded this part, as part of a complete writing of the previous Sect. 2.9, now expanded into the new Sect. 3. The new sections 3.4 and 3.5 explain both a data-driven calibration of the model parameters and a simplified choice of the same parameters, the latter designed to ease the use of the model when tuning data is not available. In the new Sect. 4.2 we now provide a comparison between calibrated and actual 2D values, and we demonstrate the performance of the simplified choice of parameters by comparison with LES simulations. Thank you for this suggestion, because we believe that the explanation of the choice of model parameters is of great importance for the practical use of the proposed model.

- 15. Section 4.1: The discussion in this section is interesting. One thing that I however struggled to understand (perhaps I'm missing something here) is that Figure 15d shows that the power loss for the first turbine is more significant when the optimal model is used. However, this contrasts with what is mentioned in lines 439 and 488. Thank you for spotting this mistake, the text has now been corrected.
- 16. Section 4.1 C\_T effect: The manuscript points out that the thrust coefficient for the optimal model used in section 4.1 is higher for yawed cases compared with the standard approach. It is mentioned that C\_T has an effect on the wake felt downstream. I agree with the authors, but I suggest elaborating on this more. As we know, C\_T has two effects on the wake: (i) it increases the streamwise velocity deficit and (ii) also increases the amount of wake deflection. These two have counteracting effects on the downwind turbine. It is interesting to understand this effect in greater detail, at least discussing it in the paper.

Thank you for the suggestion, we have expended the text as suggested.

17. Missing references: I appreciate that the authors did a thorough literature review, but some relevant references are missing. For instance, the streamtube model in section 2.5 and finding the wake spanwise velocity in the farm wake based on lifting theory are highly relevant to Shapiro et al. (2018). For instance, eq 14 in this manuscript could be compared with equation 2.13 in that paper. Heck et al. (2022): It is true that your developed model improving the state-of-the-art by including tilt and shear effects in greater details. However, I believe it is still informative for the reader to show whether your model, compared with the one proposed in Heck et al., provides similar predictions if they are used for similar operating conditions.

Thank you for these relevant comments. We have now deeply rewritten the previous Sect. 3. The new Sect. 3 provides a much more in depth description of the implementation of the model, and includes an extensive discussion on the integration with arbitrary control strategies (now in Sect. 3.2). This has also the purpose of clarifying one of the main differences between the proposed model and the one of Heck et al. 2023, which was not explained clearly enough in the previous version of the manuscript. Specifically, the method of Heck et al. 2023 is limited to operation in region II, when the modified trust coefficient that is at the basis of their formulation remains constant as a function of the misalignment angle.

On the other hand, our formulation includes a complete thrust model, and takes as input the tip speed ratio and pitch angle. This crucial difference means that our method can be used with any desired control strategy, in region II, III, II1/2 (for noise mitigation, thrust clipping, etc.), and also in derated conditions. These important differences between the two methods are now briefly mentioned in the introduction and conclusions, and explained in great detail in Sect. 3.2. The new Sect. 4 presents detailed comparisons between our model and the one of Heck et al. 2023, whenever the latter is applicable. We believe that these extensive modifications have greatly improved the paper.

## **Reviewer 2**

The submitted paper studies the power of wind turbines in yaw misalignment. An induction model is coupled with a simple blade element model, and the resulting model outputs are compared to large eddy simulations (LES) of actuator line modeled (ALM) turbines and wind tunnel experiments. Overall, the paper could be useful for the community. Modelling the power of yawed turbines is important. The contribution of this paper is coupling an induction model with a blade element model. The LES and experimental validation campaign is thorough.

However, it appears that several components of the proposed model have been already developed in the following two papers:

1. Shapiro, Gayme & Meneveau Journal of Fluid Mechanics (2018) [1]

This paper develops a lifting line model for the transverse velocity (downwash) associated with a yawed turbine. The present paper appears to follow the same analysis, resulting in the same final answer (compare: Eq. 2.7 in Shapiro to Eq. (13) here), but reference to Shapiro et al. (2018) [1] is missing.

2. Heck, Johlas & Howland Journal of Fluid Mechanics (2023) [2] This paper develops a model for the induction, thrust, power, and wake velocities for a yawed actuator disk, using the lifting line model of Shapiro, Gayme & Meneveau (2018) [1], but also accounting for how the induction affects the transverse velocity. The induction and wake velocity model developed in Heck, Johlas & Howland (2023) [2] is the same as the induction and wake velocity model in this present paper (compare: combining Eqs. (2.15) and E1 in Heck to Eq.(14) here), but this is not stated in the current manuscript.

To summarize, Section 2.5 in the present paper can be replaced with references to [1] and [2]. Explaining the progress of yaw modelling based on existing literature and how the present paper has contributed will be helpful for the readers. Overall, the main contribution of this submitted paper is to build on Shapiro, Gayme & Meneveau (2018) [1] and Heck, Johlas & Howland (2023) [2], by coupling their induction model with a simple blade element model, and the detailed comparisons to LES and experimental data. These are useful contributions to the literature, but the framing, comparisons to baseline methods, and other comments below should be reconsidered in the authors' revision.

Thank you for your comments. As explained when answering to reviewer 1, we have now deeply rewritten extensive parts of the paper. We believe that our deep revision clarifies the differences between previous publications and our proposed method. The extensive comparisons with the work of Heck et al. 2023 also provides ample evidence of the much broader applicability and superior accuracy of our approach.

In passing, we would also like to mention that the score of "poor" given by this reviewer seems not to be objective nor fair. Although the previous version of the manuscript certainly had deficiencies, this reviewer should have recognized that this paper improves on the state of the art. We hope that this extensively revised version makes this point even clearer.

# **General comments**

1. The authors have not compared their modified model (coupling with a simple blade element model) version to baseline approaches, including the Glauert induction model (model for rotor averaged induction in yaw) and the actuator disk induction model from Heck, Johlas & Howland (2023) [2] (same as present model without the blade element coupling).

Thank you for the comment. Unfortunately, in the previous version of the manuscript we gave for granted that a reader would be able to appreciate the main difference between our approach and the one of Heck et al. 2023. This was a mistake, for which we apologize. We have now tried to correct this error by expanding the text in various parts (see especially the new Sect. 3.2), and by adding extensive comparisons with Heck et al. 2023 in the new Sect. 4. In reality, also the reviewer seems to have missed the main point. In fact, the statement "same as present model without the blade element coupling" (which seems to imply a minor difference between what we proposed and what was written by Heck et al.) fails to consider the following:

- a. Not having a blade element model (which our approach does have), means that Heck et al. 2023 is forced to assume that their modified thrust coefficient CT' remains constant as the turbine yaws out of the wind. Unfortunately, this is hardly true in general, and only applies (under some conditions) when the turbine operates in region II.
- b. Crucially, our proposed approach does include a blade element model, which brings into the formulation the dependency on the pitch setting and tip speed ratio. This means that our method, contrary to Heck et al 2023, is applicable to any operational region, and with arbitrary control strategies in those regions.

We believe that the new writing finally clarifies the fundamental points listed above. These theoretical differences between the two methods are also supported by new extensive comparisons among LES simulations, wind tunnel experiments, our model and the one of Heck et al 2023, in the cases where the latter is applicable.

In addition, we also note that, as explained in the new Sect. 4.3, our method is also capable of excellent performance in the extreme case of a large dependency of the aerodynamic characteristics on the Reynolds number. Although this is a rather specific case, which is limited to small-scale wind tunnel models, it is again only due to the presence of a blade element model in our approach. On the other hand, the use of the global CT' parameter used by Heck et al. 2023, is blind to such effects, because it does not include the blade aerodynamic characteristics. Therefore, even from this point of view, "same as present model without the blade element coupling" is not a fair assessment of the crucial differences between the two approaches.

Regarding the comparison with Glauert, we believe that the method of Heck et al. 2023 is much more sophisticated than Glauert, and adding other lines to the plots would decrease their readability, without adding any useful information.

2. It appears that tunable parameters in the model are calibrated based on the same data that they are tested against, which is not ideal practice. Can this be considered model validation? How should this be done in general?

Thank you for this comment. We have added Sect. 3.3 and 3.4, which expanded on the problem of model calibration. In addition, the new Sect. 4 includes new examples where the model is compared to data that was not used for calibration.

3. Many figures are quite small, making it challenging to discern the accuracy of the model. Thank you for the comment. We have regenerated most of the plots and provided new ones, also enlarging them when necessary. We hope the new version is now more readable. 4. The paper states that a major contribution of the modelling is to capture asymmetry from wind speed shear, but the quantitative effect of wind speed shear presented in the results seems to be (visually) very small. I suggest quantifying its impact to help see its effect more clearly.

Thank you for this comment. As replied in response to point 1 above, in reality the main contribution of the method is probably not the one regarding shear, but the very broad applicability to any operating conditions and control strategy. In any case, we have now added the new Figs. 10 and 18, where we more clearly show the effects of shear.

# Line comments

1. Line 28: The formatting of the power equation looks as though all the variables are in the denominator.

We have modified the text accordingly.

2. Line 73: "Additionally, the new model clarifies the behavior of power capture with respect to some rotor design parameters and – even more importantly – with respect to the way a rotor is governed when it is misaligned. This is an effect that has been neglected in all analyses conducted so far, and that most probably explains the large scatter observed by various author" This statement is not correct. Howland et al. (2020) [3] developed a blade element model for the power-yaw relationship that incorporates rotor design parameters and the way a rotor is governed in misalignment. However, Howland et al. (2020) [3] did not have a model for induction in yaw.

Thank you for the comment. The difference with respect to Heck et al. 2023 has now been explained in detail (see also our reply to point 1 above).

3. Section 2.2: Why have the authors assumed inflow with linear vertical shear? This seems to be a limiting decision in the context of a paper which focuses on building a model for yawed turbines in general. This should be justified in more detail. For example, Liew et al. (2020) [4] identified that waked inflow modifies the power-yaw relationship, but this inherently cannot be captured in the present model that only considers linear vertical shear. Also, wind shear in the stratified boundary layer is very rarely linear. In the modelling and field experiment study of Howland et al. (2020) [3], the joint effects of low-level jets and wind veer were found to be important. They can be modeled using blade element modelling [3].

Thank you for the comment. We have opted for a linear shear distribution in order to simplify the equations of the model. In fact, the coordinate transformation to move from the ground-fixed reference frame to the wake-deflection intrinsic frame introduces cumbersome equations. We have also used a power-law in Eq. (2), observing similar results to the linear approximation, but with much heavier and complex final equations. For these reasons, we believe the present approximation to be appropriate, as it favors the understanding and usability of the model. The inclusion of low-level jets and veer is also clearly possible, once again at the price of much more involved equations.

We thank the reviewer for the suggestion of expanding the work in this direction, which is certainly relevant and might extend the applicability of our approach. However, we consider these aspects -that would not impact the main assumptions and theory of the method- as out of scope here, and material for future work.

4. Section 2.2: Why have the authors elected to neglect wind veer, which has been shown to be important in wake steering [5] and in power-yaw modelling [3] in published papers? Please see the reply to the previous point.

5. Equations 5a and 5b: Tangential induction has been neglected. This should be mentioned and justified.

A discussion on the neglection of tangential induction, together with various other effects, has been added at the end of Sect. 2.4. A similar discussion is repeated in the new Sect. 3.2, when explaining the use of loss functions. In a nutshell, it is clear that tangential induction, together with a myriad other effects, could be added to the present formulation. However, this way the model would be become a fully-blown BEM implementation, together with its complexity and computational cost. Notwithstanding these limitations, the results indicate that the present simplified analytical model is able to capture the trends of thrust and power changes remarkably well, considering yaw misalignment, arbitrary control setpoints, and shear.

6. Section 2.4: The structure of presentation in this section is a little odd. It starts by claiming that the non-uniform induction does not affect the results, then shows the equations, then neglects it for the remainder of the study. I suggest moving this section to the Appendix, and also including the quantitative evidence (referenced but not shown) that it is negligible in your cases.

We respectfully disagree on the comment on the structure of the presentation. We start with a non-uniform formulation of the induction, because this is what one would expect based on the theory of helicopters (for which, since they typically fly in wind-misaligned conditions, these similar analyses are text-book material). However, we have verified that the addition of the 1P harmonics is not strictly necessary in the present context, and we have explained this finding in Sect. 2.4, thereby dropping these extra terms from the subsequent derivations. Therefore we believe that the present explanation is logical, and provides the necessary context to the reader.

However we agree with the other suggestion by the reviewer, and we have now added a new appendix (B), where we show the difference between analyses conducted with and without the 1P terms.

7. Page 9 footnote: "This interpretation also reveals that the so-called curled shape of laterally deflected wakes (see e.g. Martínez-Tossas et al. (2021) and references therein) is nothing else than the effect of the horseshoe vortex structure generated behind a lifting wing, albeit with the addition of the swirl caused by the rotor rotation."

This explanation is exactly the one provided by Shapiro et al. (2018) [1] which the authors have not referenced in their study. This statement should be removed and references to Shapiro et al. (2018) [1] must be added.

We have added the requested reference. However, the footnote in reality was simply expanding on a citation of a classical book on helicopter theory (Johnson, 1995). In fact, the idea of seeing a misaligned rotor as a finite span wing is decades old, and classically taught in helicopter courses to scores of students. As this concept predates Shapiro et al. 2018 by many years, a classic as Johnson 1995 seems to be a much better fit in this case.

Equation (13): This is the same lateral velocity equation derived by Shapiro et al. (2018) [1] derived using Prandtl's lifting line theory, except that Shapiro et al. (2018) [1] assumed that thrust varies with cos(yaw)<sup>2</sup>. Heck et al. (2023) [2] extended the lifting line model of Shapiro et al. (2018) [1] to no longer assume thrust follows cos(yaw)<sup>2</sup>, and the final answer in Heck et al. (2023) [2] (Eq. (2.15)) is the same as Eq. (13) here.

We have added a reference to Heck et al. (2023), and we stress here once again the dramatic difference between that work and the present approach: the former lacks a generic thrust model.

9. Equation (14): Similarly, this is the same induction model derived by Heck et al. (2023) [2], although it is presented in a slightly different form in the previously published paper. Appendix E from Heck et al. (2023) [2] is pertinent (i.e. combine Eq. (2.15) with Eq. (E1) to arrive at the induction model form below that can be compared to Eq. (14) in the present paper). From Heck et al. (2023) [2], the induction model equation is:

$$a_n = \frac{2C_T - 4 + \sqrt{16 - 16C_T - C_T^2 \sin^2(\gamma)}}{-4 + \sqrt{16 - 16C_T - C_T^2 \sin^2(\gamma)}}$$

which appears to yield identical predictions to Equation (14) in this study. In summary, Section 2.5 is a repeat of existing literature and can be removed, with appropriate references added.

We believe that a complete derivation of the relevant equations is necessary here, in order for this article to be understandable and self-contained. The references to the relevant literature are provided in the manuscript, including references and extensive comparisons with Heck et al. 2023.

10. Line 206: Please justify the assumption of the small inflow angle, especially in the context of yaw and tilt misalignment and shear.

The choice of a small inflow angle is used for simplifying the equations. Clearly, this approximation somehow limits the applicability of the model, as stated in the accompanying text. However, we have now added an example that shows this angle to be small for a typical operating condition. Additionally, the extensive results and comparisons with LES and experimental measurements -in a wide variety of operating conditions- further support this choice.

- 11. Equations (18a) and (18b): Have the authors assumed that the lift and drag coefficients are constant along the wind turbine blade? Please explain. Not exactly, these model parameters represent "equivalent" values that render the effects of the actual spanwise-variable quantities on the global power and thrust produced by the rotor. This approach is similar to classical ones widely used for helicopter rotors (Jonhson, 1995). We have expanded on this concept in the next Sect. 3.3 and 3.4, and provided extra results in Sect. 4.2.
- 12. Line 239: "The power model reveals that vertical shear is the culprit for the observed lack of symmetry with respect to yaw misalignment." I don't quite understand this sentence. When the authors state "observed lack of symmetry," are they referring to existing published literature or to their own data (which to this point has not been presented). Previous studies have already explained and modeled that wind speed shear and wind direction veer cause the asymmetric power with respect to yaw misalignment. This current study neglects veer, which also seems limiting.

This sentence seems rather clear to us. We are not aware of other previous studies, including the most recent Heck et al 2023, that can explain why the power of a misaligned rotor is non-symmetric with respect to negative and positive yaw misalignments. On the other hand, our model presents a clear analytical expression of the effect of shear, and we discuss this at length in Sect. 2.8 and in the results Sect. 4.

Any effect of veer is a different topic, and not discussed here. As mentioned in an earlier reply, veer -together with non-standard shear, low level jets, horizontal shear etc.- could be readily

added to the proposed model. However, we consider these as interesting material for a follow-up study.

13. Figure 5: The effect of wind speed shear is very small. I expect the effect of veer is much larger, and especially when there is both shear and veer.

The reviewer's assertion that the effect of veer is much bigger is speculative at best, and would require specific investigations to the proven or rejected. As noted above, we consider this topic to be out of scope for this study, which seems to be ample enough.

14. Paragraph beginning on line 274: It's good the authors state assumptions and simplifications here, but they should also all be stated and explained within the derivation. Otherwise, it seems as if the authors are making ad hoc choices about what is important and what approximations are made.

We are not sure what is meant here: of course we make hypotheses, assumptions and choices in order to derive the model, as in any theory. We also try to explain as clearly as possible all our choices, and we repeat them when we believe it is necessary for clarity. In the lines mentioned here by the reviewer, we have simply repeated the limits of the model. However, we have revised the text, and -as previously noted when replying to other remarks-added various further comments on the assumption of small angles and given reasons for neglecting the tangential induction and other effects.

15. Equation (28) and associated discussion: I do not understand the motivation for simplifications to be applied to the model and then the tuning of more unknown parameters. How does this affect the result? How can this be done in general? Do the authors expect these parameters to be universal, and if not, how can this model be applied to a new wind turbine model? Do we need power data for turbines in yaw misalignment to tune this model? If so, that is not necessarily useful as a predictive model.

We have expanded the discussion on this important point, adding the new Sect. 3.3 and 3.4, where we propose a simplified choice of the model parameters. Furthermore, we have demonstrated the procedure in the new Sect. 4.2, testing the model on the NREL 5 MW and on a modified DTU 10 MW reference turbines. These examples replicate what one would do in the absence of dedicated tuning data, showing the general applicability of the proposed approach and its predictive ability.

16. The authors need to include an Appendix that describes the tuning process in much more detail. What do the authors mean when they say "a different random 50% subset of the available data"? Is this the training-testing split? Presumably the authors are not performing model tuning with the same data that are used to test the model accuracy, as this is improper practice and can bias the results. In the added Appendix I am requesting, model results without any tuning must be shown.

We expanded the text and clarified the tuning approach, which now has a dedicated new Sect. 3.3. Additionally, we have added a new experimental dataset that was purely used for validation purposes and not for tuning.

17. β: It strikes me as a bit strange to have a tunable parameter in the model represent a known geometrical feature such as blade twist.

It is not strange, as the model requires an "equivalent" value of twist, as for other blade characteristics. We have expended the text, providing an explanation in Sect. 3.4 of where twist should be measured (which, as for helicopter rotors, falls at about 2/3 of the blade span). We have also verified this value on four different wind turbines, as now explained in the new Sect. 4.2.

- 18. Figure 7: the authors show four results in this figure that are almost identical. It is very challenging to discern any notable differences among the subfigures, so it's reasonable to ask whether the authors have really tested the limits and applicability of their modelling framework. For example, why has the tip-speed ratio been kept within such a small region? Thank you for the comment. The numerical analysis was based on datasets that were available to us at the time. Now good part of the results based on LES comparisons has been completely rewritten, and it includes a new complete set of conditions in region II, III, and in transitions between the two. Additionally, a wider range of tip speed ratios is considered in the experimental part of this work.
- 19. Line 324: "However, as thrust is decreased (and pitch increased), power capture at positive γ values is larger than for negative misalignments." The asymmetry is almost not visible in Figure 7 to me. Perhaps quantify to make it more clear? Thank you. We have added the new Figures 10 and 18 to better show the effects of shear.
- 20. Line 327: "trust coefficient" -> "thrust coefficient" Thank you. We have corrected the sentence.
- 21. Figure 8: Why have the authors chosen to only consider low thrust coefficients with a maximum of C\_T=0.6? It is interesting to show higher thrust coefficients. For example, Heck et al. (2023) [2] found that the thrust should increase with yaw to reduce the power loss. However, it seems that their induction model (same as your Equation (14)) is less accurate at higher C\_T.

The C\_T that was reported in fig. 8 was the average one for plus/minus 30 degrees of yaw, hence the relatively low values. The ranges were selected to have a complete dataset. In fact, data for interpolation of higher C\_Ts was not available for TSR<9.5. This figure is no more present in the revised manuscript, and replaced by fig. 10.

22. Line 341: "[...] and is capable of describing even relatively minor effects of the complex behavior of a misaligned wind turbine rotor in a sheared inflow." I did not follow what the authors meant by this statement

We mean that -even though the physics of misaligned rotors is quite complex- a simple model like ours is able to correctly capture the trends and phenomena involved. The sentence seems correct and understandable to us.

23. Line 345: Since the rotor aerodynamic characteristics are necessary in your model, please include an Appendix which describes all relevant characteristics in this paper, so that the paper is self-contained.

We have now added Sect. 3.4, which explains how the model parameters can be computed from the aerodynamic data of the turbine. Since we use four different wind turbines, to avoid excessively increasing the length of the article, we decided not to report these data, which can be found in the relevant (cited) bibliographical references and associated online repositories.

# 24. What is the thrust coefficient of the experimental turbine?

The experimental turbine model operated at a wide range of thrust coefficients, depending on the control strategy and on the controller inputs. The text has been expanded for clarity.

- 25. Figure 11: It seems that a lot of tunable parameters are fit within this model. I am again unclear as to what is within sample of tuning and what should be considered as model validation (which requires out of sample data). We have added an additional experimental dataset that was purely used for validation purposes and not for tuning. The new version of the text is hopefully now clearer on these aspects of our work.
- 26. Figure 13: The results are summarized as having "very good" agreement with limited discussion, but there are several occasions where the model predictions are outside experimental uncertainty. It would be better to discuss these in detail. The critical points that do not lie within the uncertainty bands are obtained with very low tip-speed ratios. In these conditions the small angle approximation is not accurate, which impacts the accuracy of the results. The text has been expended to explain this fact.
- 27. Section 4.1: The results of this section align exactly with the published study of Heck et al. (2023) [2], who found that the thrust coefficient should be increased as the yaw is increased to reduce the power loss.Thenk you for this comment. We have now mentioned the agreement with Heck et al. (2022).

Thank you for this comment. We have now mentioned the agreement with Heck et al. (2023).

- 28. Section 4.2: How does the operation of the leading turbine affect the wake? The specific equations should be shown. It should affect the initial streamwise and spanwise wake deficits. This was done in Heck et al. (2023) [2] (Figure 9 and Appendix C). As noted by the reviewer, this was explained in Heck et al 2023. Therefore, we prefer not to repeat the analysis here, as the paper is already quite long.
- 29. Figure 15: It seems that there is almost no benefit from the modified model compared to baseline FLORIS, from the lines in Figure 15(f). Why is the additional benefit so negligible? The benefit is on the order of 1% power gain, which, in our opinion is not negligible. The reason for this is the trade-off generated by increasing the C\_T upstream. On one hand, more power can be produced upstream, but at the same time the wake expands and presents lower momentum, thereby impacting the production downstream.
- 30. Line 444: "The LES-ALM results confirm the findings based on the FLORIS engineering wake model: less power losses for the front turbine, and more gains for the downstream one." Looking at Figure 15, this statement appears to be incorrect. It seems that the "Opt. (Model)" approach actually increases the turbine 1 power loss and increases turbine 2 power gain. Rather than decreasing power loss for the front turbine with more gains for the downstream one.

Correct, thank you for pointing this out. We have corrected the text accordingly.

- 31. References: I am not sure why the authors have chosen to cite arXiv versions of papers that have been published before this present paper was submitted, but that must be corrected. Thank you for pointing out this. We have edited this accordingly.
- 32. I recommend a title change, since other published papers have previously gone 'beyond the cosine law.' It is better to be specific about what contributions this paper contains. We respectfully disagree, the title clearly indicates the content of the paper. The differences with other previous publications are now very extensively described in the revised manuscript, and could not be captured by a title, which clearly has to be relatively short.

# On the power and control of a misaligned rotor – Beyond the cosine law –

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#### Abstract.

We present a new model to estimate the performance of a wind turbine operating in misaligned conditions. The model is based on the classic momentum and lifting-line theories, considering a misaligned rotor as a lifting wing of finite span, and accounts for the combined effects of both yaw and uptilt angles.

- 5 Improving on the classical empirical cosine law in widespread use, the new model reveals the dependency of power not only on the misalignment angle, but also on some rotor design parameters and – crucially – on the way a rotor is governed when it is yawed out of the wind. We show how the model can be readily integrated with arbitrary control laws below, above and around rated wind speed. Additionally, the model also shows that a sheared inflow is responsible for the observed lack of symmetry for positive and negative misalignment angles. Notwithstanding its simplicity and insignificant computational cost, the new
- 10 proposed approach is in excellent agreement with large eddy simulations (LES) and wind tunnel experiments.

Building on the new model, we derive the optimal control strategy for maximizing power on a misaligned rotor. Additionally, we maximize the total power of a cluster of two turbines by wake steering, improving on the solution based on the cosine law.

#### 1 Introduction

Wind farm control by wake steering consists of deflecting the wake away from downstream rotors to boost the total power of

- 15 a plant (Meyers et al., 2022). The effectiveness of this control strategy has been proven numerically (Jiménez et al., 2010), experimentally in the wind tunnel (Campagnolo et al., 2016), as well as in field tests (Fleming et al., 2019; Doekemeijer et al., 2021). At the core of the power-boosting ability of wake steering is a trade-off: on the one hand, there is an enhanced momentum of the inflow at a downstream turbine when a wake is shifted laterally away from it; on the other hand, some power is lost at the upstream misaligned rotor, because it does not point into the wind anymore. In general the trade-off budget is
- 20 positive, in the sense that the power that is gained downstream is larger than the one lost upstream. The problem is however highly complex: downstream, power capture is determined by the interaction of the impinged rotor with the wake that, in turn, is influenced by the ambient conditions and those of the wake-shedding turbine; upstream, power losses depend on the inflow characteristics, but also on the rotor and on the way it is governed. Understanding and controlling this delicate balance between upstream and downstream behavior is clearly of paramount importance for improving the power capture of wind farms by
- 25 wake steering. Great progress has been made in recent years to understand, model and control wakes (see for example the

review in Meyers et al. (2022)), i.e. on the "downstream" aspect of the problem. However, the "upstream" aspect remains much less explored and understood. How much power does a yawed turbine really lose? And what inflow, rotor and rotor-control parameters influence its behavior? It is a major ambition of this paper to try and answer these questions.

- The aerodynamic power P of a wind turbine is customarily written as  $P = 1/2\rho Au_{\infty}^3 C_P P = \frac{1}{2}\rho Au_{\infty}^3 C_P$ , where  $\rho$  is the 30 air density, A the rotor swept area,  $u_{\infty}$  the ambient free-stream wind speed, and  $C_P$  the power coefficient. When a turbine is misaligned with respect to the wind vector by an angle  $\gamma$ , the rotor-orthogonal velocity component becomes  $u_{\infty} \cos \gamma$ . Accordingly, one would expect the yaw-induced power loss to be  $\eta_P = P/P_0 = \cos^3 \gamma$ , where  $P_0$  is the aerodynamic power produced for  $\gamma = 0$ . Unfortunately, this is only a naive interpretation of the true behavior of a misaligned rotor, and its predictions are not confirmed by experimental and numerical observations (Liew et al., 2020). To reflect this fact, a pragmatic solution has been
- 35 adopted by most of the literature, where power losses due to misalignment are assumed to obey the simple law  $\eta_P \approx \cos^{p_p} \gamma$ , where  $p_p$  is a tunable parameter.

Unsurprisingly, since such a model is not based on actual physics, a large spread of values for  $p_p$  has been reported in the literature. In wind tunnel experiments with scaled models, **?**-Campagnolo et al. (2020) measured  $p_p = 2.1$ , Krogstad and Adaramola (2012) and Bartl et al. (2018) reported  $p_p \approx 3$ , whereas Medici (2005) found a value  $p_p = 2$ . Numerically, Fleming

- 40 et al. (2015) measured  $p_p = 1.88$  on the NREL 5 MW wind turbine (Jonkman et al., 2009), whereas Draper et al. (2018) obtained values between 1.3 and 2.5 for scaled wind turbine models operating in waked inflow conditions. The power production in misaligned conditions has also been measured in multiple field tests. For example, Fleming et al. (2017) reported a value of 1.41 for an Envision 4 MW turbine; Dahlberg and Montgomerie (2005) published a range of values for  $p_p$  between 1.9 and 5.1 at an offshore plant. More recently, Hulsman et al. (2022) observed  $2 < p_p < 2.5$  at an onshore wind farm in the north of
- 45 Germany.

The high large scatter characterizing the  $p_p$  coefficients reported in the literature is a relevant source of uncertainty, creating a significant hindrance to the development of power-boosting wind farm control strategies.

The large scatter in the cosine exponent suggests, and suggesting that some relevant phenomena are not captured by the  $\cos^{p_p}$  law. In hindsight, this is to be expected, because this simple model fails to explicitly represent how the power coefficient  $C_P$  changes when a turbine is misaligned, and somehow absorbs this effect into the tunable exponent. Some indications that there is more to this problem than a simple power cosine law have already been reported by various authors. Based on experiments and numerical simulations, Campagnolo et al. (2023), Cossu (2021a, b) and Heck et al. (2023) suggested that the power of a misaligned rotor strongly depends on its loading, in the form of the thrust coefficient  $C_T$  (which, clearly, has also-; clearly, in turn this has a strong effect on the behavior of the wake (Cossu, 2021a, b). Other variables that have been shown to play a role on power losses are related to the inflow. Recently, Draper et al. (2018) and Liew et al. (2020) have observed that power losses in misaligned conditions differ depending on whether a rotor is waked or not. Howland et al.

(2020) observed a significant influence of shear and veer, while Simley et al. (2021) measured a strong dependency on inflow speed. The behavior of power losses has also been shown to depend on the direction of yaw misalignment, and not only on its magnitude as implied by the power cosine law. This asymmetric behavior of yaw misalignment has been observed by 60 Fleming et al. (2015); Schottler et al. (2017); Fleming et al. (2018); ?Fleming et al. (2015); Schottler et al. (2017); Fleming et al. (2018); 9

, among others. However, an agreement on which misalignment direction yields more or less power has not been reached yet. In this paper, we present a new analytical model for misaligned wind turbine rotors. The proposed approach combines the classic momentum and lifting-line theories, considering a misaligned rotor as a lifting wing of finite span, in close parallel to the analysis conducted for helicopter rotors in forward flight (Johnson, 1995). Very recently, Heck et al. (2023) published a

- 65 misaligned rotor model based on similar arguments, although their approach does While existing approaches do not explain the lack of symmetry with respect to yaw direction. The, the present model includes the effects of wind shear, which is shown to be the culprit for the observed break of symmetry with respect to the misalignment direction. For improved accuracy, the model also includes the effects of the uptilt angle, as it contributes to the overall misalignment of the rotor with respect to the wind vector. The resulting Thanks to this feature, the proposed methodology is also readily applicable to vertical wake
- 70 steering control, which could be implemented with floating wind turbines (Nanos et al., 2022) or downwind teetering rotors. The resulting model equations are integrated over the blade span and averaged over one rotor revolution, leading to a semianalytical formulation of negligible computational cost that can be readily coupled with engineering wake models such as FLORIS (NREL, 2023b) or PyWake (Pedersen et al., 2019). However, the model governing equations could also be integrated numerically and embedded into blade element momentum (BEM) codes (Hansen, 2015), such as the AeroDyn package (NREL,
- 75 2023a) implemented in OpenFAST (NREL, 2023c).

Very recently, Heck et al. (2023) published a misaligned rotor model based on similar arguments. However, their approach does not include the effects of shear, and therefore fails to capture the asymmetric behavior of yaw direction. More importantly, their formulation uses a modified thrust coefficient  $C'_T$ , which is assumed to remain constant between aligned and misaligned conditions. This hypothesis is indeed verified when the turbine operates in the partial load region. Departing from this approach,

- 80 the method proposed here is based on a completely general dependency of the thrust coefficient on the misalignment angle, and therefore can readily accommodate arbitrary regulation strategies in the partial, full and intermediate regulation regions – including thrust clipping and derating (Campagnolo et al., 2023). Moreover, the model of Heck et al. (2023) cannot predict power losses higher than  $\cos^3 \gamma$ , which have however been reported in the literature. A detailed comparison of the new proposed model and the one of Heck et al. (2023) is developed in the following pages.
- The proposed semi-analytical model shows that the behavior of a misaligned rotor does not follow the  $\cos^{p_p}$  law, contradicting this empirical formula in widespread use. Additionally, the new model clarifies the behavior of power capture with respect to some rotor design parameters and – even more importantly – with respect to the way a rotor is governed when it is misaligned. This is an effect that has been neglected in all analyses conducted so far, and that but that – as already noted by Howland et al. (2020) – most probably explains the large scatter observed by various authors. Building on these results the
- 90 unique ability of the proposed method to handle arbitrary control policies, we derive the optimal strategy for maximizing power capture when pointing a rotor away from the wind. Finally, we implement the semi-analytical model in FLORIS and we optimize the power of a cluster of two turbines. We obtain setpoints that differ from those that can be computed with the empirical  $\cos^{p_p}$  law, and that lead to a slight improvement of the cluster power.

The new models exhibits an excellent match with high-fidelity LES simulations obtained with a TUM-modified version

95 of NREL's large eddy simulator actuator line model (LES-ALM) SOWFA (Fleming et al., 2014; Wang et al., 2019, 2018). Additionally, the model is further validated with wind tunnel data from experiments conducted with the TUM G1 scaled wind turbines (Bottasso and Campagnolo, 2022a; Campagnolo et al., 2020).

The paper is organized as follows: Sect. 2 presents the new formulation, and Sect. 4-3 explains its implementation in an engineering wake model, including the integration with arbitrary control strategies. Next, Sect. 4 considers its validation with

100 respect to simulated and experimental data, while Sect. 5 analyzes the effects of the new model on wake steering. Finally, Sect. 6 draws conclusions and offers an outlook towards future work.

#### 2 Misalignment model

#### 2.1 Frames of reference

Three reference frames are necessary to completely characterize a misaligned rotor interacting with the wind, as shown in 105 Fig. 1: a *ground-fixed* reference frame and a *nacelle-fixed* reference frame, which together describe the relative orientation of the rotor with respect to the ground, and a *wake-deflection intrinsic frame*, which describes the relative orientation of the rotor with respect to the incoming wind vector.

The ground-fixed wind-aligned frame of reference is indicated with a subscript g and is defined by the right-handed triad of unit vectors  $\mathcal{F}_g = \{x_g, y_g, z_g\}$ .  $z_g$  points vertically down towards the ground,  $x_g$  is parallel to the terrain pointing downstream 110 and is contained in the plane formed by the wind vector  $u_\infty$  and  $z_g$ ; finally,  $y_g$  completes a right-handed triad. In the following,

for simplicity we consider the wind vector to be parallel to the terrain, i.e.  $u_{\infty} \parallel x_{g}$ , although this is not strictly necessary.

The nacelle-fixed frame of reference is indicated with a subscript n and is defined by the triad of unit vectors  $\mathcal{F}_n = \{x_n, y_n, z_n\}$ .  $\mathcal{F}_n$  is obtained from  $\mathcal{F}_g$  by two successive rotations: a first rotation by the *tilt* angle  $\delta$  about the horizontal axis  $y_g$ , followed by a second rotation by the *yaw* angle  $\gamma$  about the vertical axis  $z_g$ . Both rotations are positive about their respective axes according to the right hand rule (notice that, according to this definition, the typical uptilt of an upwind turbine

results in a negative value for  $\delta$ ).

However, the interaction of the rotor with the flow depends only on their mutual orientation, and not on how they are oriented with respect to the ground, which is a fundamental principle of fluid mechanics known as Galilean relativity. Therefore, a third frame is necessary, which is termed here *wake-deflection intrinsic frame* and is indicated with a subscript *d*. The frame is

120 formed by a right-handed triad of unit vectors  $\mathcal{F}_d = \{x_d, y_d, z_d\}$ . Vector  $x_d$  is parallel to the rotor axis, i.e.  $x_d = x_{n_z}$  while vectors  $y_d$  and  $z_d$  are contained in the rotor disk plane  $\Psi$ . Together, the rotor axis  $x_d$  and the wind velocity vector  $u_\infty$  define the  $\Pi$  plane. The angle in the  $\Pi$  plane between these two vectors is the *true misalignment angle*  $\mu$ :

$$\cos\mu = \frac{u_{\infty}}{u_{\infty}} \cdot \boldsymbol{x}_n,\tag{1}$$

where  $u_{\infty} = |\boldsymbol{u}_{\infty}|$  is the scalar <u>ambient</u> wind speed. The unit vector  $\boldsymbol{z}_d$  is orthogonal to the  $\Pi$  plane, i.e.  $\boldsymbol{z}_d = \boldsymbol{x}_d \times \boldsymbol{u}_{\infty}/(u_{\infty} \sin \mu)$ , while unit vector  $\boldsymbol{y}_d$  is finally chosen to form a right-handed triad. Using the coordinate transformations in Appendix A, it can

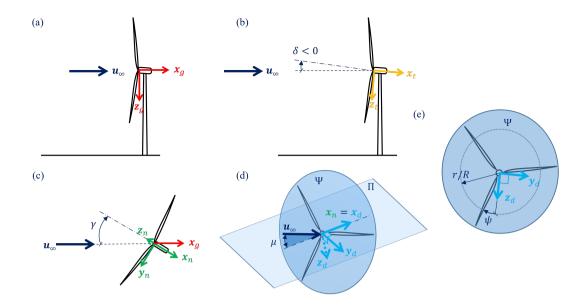


Figure 1. Reference frames used in the derivation of the model. Ground-fixed wind-aligned reference frame (in red, subscript g) (**a**); intermediate frame, obtained by the tilt rotation  $\delta$  about the horizontal axis  $y_g$  (in orange, subscript t) (**b**); nacelle-fixed reference frame, obtained from t by a yaw rotation  $\gamma$  about the vertical axis  $z_g$  (in green, subscript n) (**c**); plane  $\Pi$  formed by the wind vector  $u_{\infty}$  and the rotor axis  $x_n$ ; the plane contains the  $x_d$  and  $y_d$  unit vectors of the wake-deflection intrinsic frame (in light blue, subscript d) (**d**); rotor plane  $\Psi$ , formed by the  $y_d$  and  $z_d$  unit vectors, with rotor azimuthal angle  $\psi$  and radial position r/R (**e**). Rotations are positive according to the right hand rule; notice that the value of the uptilt angle of the rotor in panel (**b**) is therefore negative.

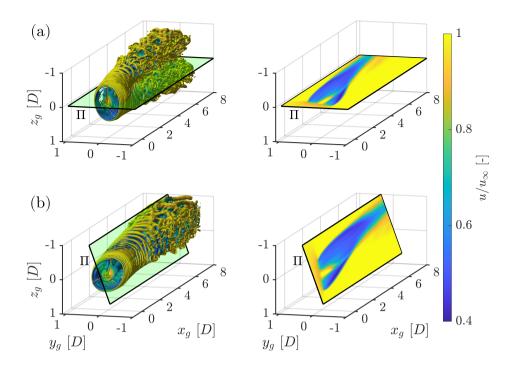
be readily shown that  $\cos \mu = \cos \delta \cos \gamma$ , i.e. the total misalignment is caused by both the tilt and yaw angles, the former typically being neglected in most wake models. Notice that, given its definition, the misalignment angle  $\mu$  is always positive, because  $z_d$  flips from one side of the  $\Pi$  plane to the other, depending on the relative orientation of the wind velocity and rotor axis vectors. When the wind comes from the right looking upstream in the  $\Pi$  plane,  $z_d$  points downwards (see Fig. 1d), whereas it points upwards when the wind comes from the left.

Figure 2 shows a visualization of the wakes developing behind a wind turbine rotor for two different pairs of tilt and yaw values:  $\delta = 0^{\circ}$ ,  $\gamma = -30^{\circ}$ ; and  $\delta = -28.43^{\circ}$ ,  $\gamma - 10^{\circ}$ . Both pairs correspond to a same true misalignment  $\mu = -30^{\circ} \mu = 30^{\circ}$ . The figure confirms that the wake is invariant for an observer on the  $\mathcal{F}_d$  frame. This is particularly evident in the images of the longitudinal speed on the  $\Pi$  plane (marked with a black solid border), which are clearly identical in the two cases. Clearly, for

# 135 large values of tilt the interaction of the wake with the ground or with a sheared inflow would break the $\Pi$ -frame invariance.

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Because of what noted above, in the following the wake analysis is developed in the  $\Pi$  plane, instead of the horizontal one as customarily done. Transformation matrices that map vector components from one frame to the other are reported in Appendix A.



**Figure 2.** Visualization of the wakes developing behind a wind turbine operating in steady inflow conditions for different pairs of tilt and yaw values, all corresponding to a same total (true) misalignment  $\mu = -30^{\circ} \mu = 30^{\circ}$ . The deflection of the wake occurs in the  $\Pi$  plane, marked with a black border. Top row (**a**):  $\delta = 0^{\circ}$ ,  $\gamma = -30^{\circ}$ ; bottom row (**b**):  $\delta = -28.43^{\circ}$ ,  $\gamma = -10^{\circ}$ . Left column: iso surfaces of *Q*-criterion; right column: image of the longitudinal flow speed  $u/u_{\infty}$  on the  $\Pi$  plane. Distances are expressed in rotor diameters *D*. Interactive 3D versions of the figures are available at the following links: https://tinyurl.com/btcl-fig-2-a (**a**); https://tinyurl.com/btcl-fig-2-b (**b**).

#### 2.2 Sheared inflow

140 Considering a linear vertical shear of the inflow, the ambient wind speed writes

$$u_{\infty}(z_g) = u_{\infty,\text{hub}}\left(1 - k\frac{z_g}{R}\right).$$
<sup>(2)</sup>

Here  $u_{\infty,\text{hub}}$  is the ambient wind speed at hub height, k is the vertical linear shear coefficient, and  $z_g$  is the vertical coordinate in the ground frame of reference, centered at the hub., and R is the rotor radius. The choice of a linear shear distribution was made just to simplify the derivations, and other choices are clearly possible, for example to model the more common power

<sup>145</sup> law or the presence of low-level jets. Additionally, it would be interesting to include also the effects of a horizontal shear, to account for waked conditions, and of veer. These further model improvements are however deferred to a continuation of this study.

By applying the coordinate transformation of Appendix A, the ambient wind speed of Eq. (2) can be written in terms of the radial r and azimuthal  $\psi$  coordinates on the rotor plane, yielding

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$$u_{\infty}(r,\psi,\delta,\gamma) = u_{\infty,\text{hub}} \left( 1 - k \frac{r}{R} \frac{\cos\delta}{\sin\mu} \left( \sin\gamma\cos\psi - \cos\gamma\sin\delta\sin\psi \right) \right).$$
(3)

Here  $\psi$  is positive about  $x_d$  according to the right hand rule (i.e. clockwise looking downstream), and it is measured starting from the  $z_d$  unit vector (which flips from one side of the  $\Pi$  plane to other depending on whether the wind blows from the right or left looking upstream, as explained in Sect. 2.1; see also Fig. 1).

#### 2.3 Force and velocity components at a blade section

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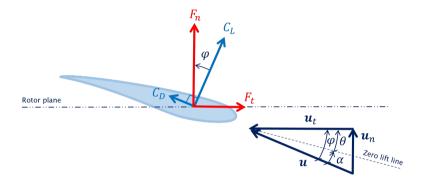


Figure 3. Blade cross section, with triangle of velocities (in blue), lift and drag (in light blue), and resulting aerodynamic force components (in red).

155 With reference to Fig. 3, the tangential  $F_t$  and normal  $F_n$  components of the aerodynamic force at a blade section are

$$F_t = \frac{1}{2}\rho u^2 c \left(C_L \sin\varphi - C_D \cos\varphi\right),\tag{4a}$$

$$F_n = \frac{1}{2}\rho u^2 c \left(C_L \cos\varphi + C_D \sin\varphi\right),\tag{4b}$$

where  $\varphi = \tan^{-1} u_n / u_t$  is the inflow angle,  $u = \sqrt{u_t^2 + u_n^2}$  is the total flow speed at the blade section, c is the sectional chord length, and finally  $C_L$  and  $C_D$  are the lift and drag coefficients, respectively. Using the coordinate transformations of Appendix A, the tangential  $u_t$  and normal  $u_n$  velocity components write

$$u_t = \Omega r + u_\infty \sin \mu \cos \psi, \tag{5a}$$

$$u_n = u_\infty \cos \mu (1-a),\tag{5b}$$

where  $\Omega$  is the angular speed of the rotor and a is the axial induction factor, which expresses how much the rotor-orthogonal component of the free-stream speed  $u_{\infty}$  is slowed down at the rotor disk.

#### 165 2.4 Induction model

It is well known that a non-uniform description of the induction is necessary in order to accurately capture the azimuthal variation of loads on a rotor operating in non-axial conditions (Johnson, 1995). However, it appears that this is not necessary when computing integral rotor quantities such as power, torque and thrust, as in the present case. To show this, the induction is modeled here with an expansion limited to one-per-revolution (1P) harmonics, i.e.

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$$a = a_0 \left( 1 + \kappa_{1s} \frac{r}{R} \sin \psi + \frac{k \kappa_{1c}}{R} \frac{r}{R} \cos \psi_g \right), \tag{6}$$

where  $a_0$  is the constant-over-the-rotor (0P) induction, r is the blade spanwise coordinate, and R is the rotor radius while  $a_0 \kappa_{1s}$ and  $a_0 \kappa_{1c}$  are the 1P sine and cosine harmonic amplitudes, respectively.

Following the classical approach used for helicopter rotors in forward flight (Johnson, 1995), the sine term accounts for the tilting of the induction plane caused by the misalignment  $\mu$  of the rotor with the incoming wind. As such, it is written in terms of  $\psi$ , which is measured starting from the  $\chi$ , written and therefore it expresses a rotation of the induction plane about the

175 of  $\psi$ , which is measured starting from the  $z_d$  unit vector, and therefore it expresses a rotation of the induction plane about the axis normal to the wake-deflection intrinsic frame  $\Pi$ . The coefficient  $\kappa_{1s}$  can be modeled according to Coleman et al. (1945) and Pitt and Peters (1981), resulting in the expression

$$\kappa_{1s} = -\frac{15\pi}{32} \tan\left(\frac{\chi}{2}\right),\tag{7}$$

where the initial wake skew angle is χ = μ + sin μC<sub>T</sub>/2 (Jiménez et al., 2010), and C<sub>T</sub> = 2T/(ρAu<sup>2</sup><sub>∞,hub</sub>) is the thrust coefficient. Notice that the definition of the skew angle differs from the one given by Eq. (20) of Jiménez et al. (2010), because of the different definition of the thrust coefficient used in that publication.

The cosine term is introduced to account for the effects on the induction caused by vertical shear. As such, it is written as a function of the azimuthal angle  $\psi_g$ , which is measured from the (vertical)  $z_g$  unit vector, and therefore it expresses a rotation of the induction plane about the (horizontal) unit vector  $y_g$ . Using Eq. (A4b), it is readily found that  $\psi_g = \psi \cos \mu$ . Following Meyer Forsting et al. (2018), the cosine term is proportional to both the shear k and the thrust  $C_T$  coefficients, i.e.

$$\kappa_{1c} = \kappa_{1c}^* k C_T. \tag{8}$$

The sine cosine term significantly complicates the analytical derivations of power, torque and thrust, which must now be expressed in terms of Bessel functions (Abramowitz et al., 1988) because of the term  $\cos(\psi \cos \mu)$ . Before attempting the modeling of the proportionality coefficient  $\kappa_{1c}^*$ , this term was numerically optimized to best fit the numerical simulations and experimental measurements, as explained later in Sect. 3.

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The inclusion of the sine and cosine induction terms has only an extremely modest effect on the quality of the results. In fact the match of  $C_P$  improves by 0.35% when the sine term is included, and by 0.60% when both terms are used, as more precisely shown in Appendix B. Because of their modest effects, these terms are dropped from the following discussion, to simplify the resulting expressions, and they were not used in the results reported later in this article. However, these terms are

retained in the software implementation of the model (Tamaro et al., 2024), and can be switched on if desired by the user.

It should also be noted that the present model neglects the effects of the tangential induction, which in fact does not appear in the tangential velocity component expressed by Eq. (5a). This is justified by the fact that the rotor swirl is concentrated close to the hub, and it is small for a large extent of the blade span (Burton et al., 2011), where most of the thrust and power are generated.

- 200 More in general, there are several other effects that are present in a rotor, and that are not modelled here, such as for example radial drag, tip and root losses, blade sweep, prebend and cone, and others. All these effects can be taken into account in detailed BEM models (Hansen, 2015; Burton et al., 2011), but would significantly complicate the present simplified analytical method. However, notwithstanding these limitations, the results of Sect. 4 show a remarkable ability of the proposed approach in predicting the trends of power and thrust as functions of various operating and inflow conditions. Additionally, the ability
- 205 of the model in predicting actual power and thrust values (instead of trends) can be improved by the use of loss functions, as explained in Sect. 3.2.

#### 2.5 Streamtube model

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An expression for the axial induction can be derived using the concept of a streamtube (Hansen, 2015), as shown in Fig. 4 with reference to the present case of a misaligned rotor. Four stations are considered along the stream tube: inlet *i*; outlet *o*; section  $r^-$  located immediately in front of the rotor; and section  $r^+$  located immediately behind the rotor.

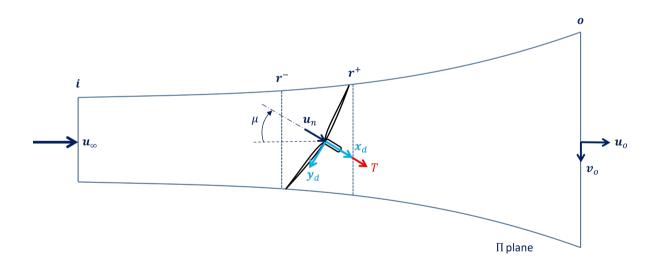


Figure 4. Schematic view of a streamtube around a misaligned wind turbine. Cross-sectional stations: inlet *i*, outlet *o*,  $r^-$  immediately in front of the rotor;  $r^+$  immediately behind the rotor.

The principle of impulse and momentum applied to the streamtube is written as

$$T\cos\mu = \dot{m}(u_{\infty} - u_o),\tag{9}$$

where T is the thrust force,  $u_o$  is the longitudinal flow speed at the streamtube outlet, while  $\dot{m}$  is the mass flux

$$\dot{m} = \rho A u_n. \tag{10}$$

215 By using the thrust coefficient, Eq. (9) yields the non-dimensional longitudinal flow speed at the streamtube outlet:

$$\frac{u_o}{u_{\infty,\text{hub}}} = 1 - \frac{1}{2} \frac{C_T}{1 - a_0}.$$
(11)

Next, Bernoulli's energy conservation theorem is applied between the streamtube inlet and the section immediately upstream of the rotor (stations *i* and  $r^-$  in Fig. 4), and between the section immediately downstream of the rotor and the streamtube outlet (stations  $r^+$  and *o* in the same figure):

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$$p_i + \frac{1}{2}\rho V_i^2 = p_{r^-} + \frac{1}{2}\rho V_{r^-}^2,$$
 (12a)  
 $p_{r^+} + \frac{1}{2}\rho V_{r^+}^2 = p_o + \frac{1}{2}\rho V_o^2,$  (12b)

where p is pressure and  $p_i = p_o = p_\infty$ , while  $p_\infty$  is the ambient value. Additionally,  $V_{r^-} = V_{r^+}$  for continuity, and furthermore  $V_o^2 = u_o^2 + v_o^2$  at the outlet section, where  $v_o$  is the lateral (sidewash) speed component.

Following a customarily text-book assumption used for helicopters rotors in forward flight (Johnson, 1995), only more recently adopted also for wind turbines by Shapiro et al. (2018), the misaligned rotor can be seen as a lifting wing of finite span (albeit of a small aspect ratio  $AR = D^2/A = 4/\pi$ ) operating at an angle of attack  $\mu^1$ . The chord  $C(y_d)$  of the wing in the streamwise direction has an elliptic distribution:  $(C(y_d)/2)^2 + y_d^2 = R^2$ . According to Prandtl's lifting line theory (Tietjens and Prandtl, 1957; Katz and Plotkin, 2001), the wing has consequently an elliptic lift distribution, which induces a spanwiseconstant downwash (in this case, sidewash)  $v_o = \Gamma/4R$ .  $\Gamma = \bar{L}/\rho u_{\infty,hub}$  is the circulation at the wing mid section  $y_d = 0$ , and  $\bar{L} = LC(0)/A$  is the lift per unit span at that same location. Since the wing lift is the rotor side force, i.e.  $L = T \sin \mu$ , it follows that the non-dimensional sidewash at the streamtube outlet can be expressed as (Heck et al., 2023)

$$\frac{v_o}{u_{\infty,\text{hub}}} = \frac{1}{4} C_T \sin \mu. \tag{13}$$

Combining the previous equations, yields an expression for the 0P axial induction  $a_0$  as a function of the misalignment  $\mu$  and thrust coefficient  $C_T$ :

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$$1 - a_0 = \frac{1 + \sqrt{1 - C_T - \frac{1}{16}C_T^2 \sin^2 \mu}}{2\left(1 + \frac{1}{16}C_T \sin^2 \mu\right)}.$$
 (14)

#### 2.6 Thrust force

Equation (14) furnishes an expression for the 0P axial induction as a function of the thrust coefficient. To close the problem, an expression for the thrust coefficient in terms of the operating conditions of the turbine is necessary. To this end, the thrust

<sup>&</sup>lt;sup>1</sup>This interpretation also reveals that the so-called curled shape of laterally deflected wakes (see e.g. Martínez-Tossas et al. (2021) and references therein) is nothing else than the effect of the horseshoe vortex structure generated behind a lifting wing, albeit with the addition of the swirl caused by the rotor rotation.

force T is expressed in terms of the normal sectional force  $F_n$  as

240 
$$T = \frac{B}{2\pi} \int_{0}^{2\pi} \int_{0}^{R} F_n \,\mathrm{d}\psi \,\mathrm{d}r,$$
 (15)

where B indicates the number of blades. Using Eq. (4b) under the assumption of a small inflow angle (i.e.  $\sin \varphi \approx \varphi$  and  $\cos \varphi \approx 1$ ), yields

$$T = \frac{B}{2\pi} \int_{0}^{2\pi} \int_{0}^{R} \frac{1}{2} \rho u^2 c \left( C_D \varphi + C_L \right) d\psi dr.$$
(16)

The lift coefficient ean we written as  $C_L = C_{L,\alpha}\alpha$  is written as  $C_L = C_{L,\alpha}\alpha = C_{L,\alpha}(\alpha_g - \alpha_0)$ , where  $C_{L,\alpha}$  is the list slope, lift slope, and  $\alpha = \varphi - \theta$  is the angle of attack -measured with respect to the zero-lift direction, whereas  $\alpha_g$  is the angle of attack measured with respect to a generic direction. Without any loss of generality, the use of  $\alpha$  is preferred in the following, to avoid carrying along in the derivations the unnecessary extra term  $\alpha_0$ . Furthermore,  $\theta = \theta_p + \beta$  is the local pitch angle (see Fig. 3), where  $\theta_p$  is the blade pitch rotation at the pitch bearing, and  $\beta$  indicates the blade twist -referred to the zero-lift direction.

Neglecting swirl induction, the inflow angle is 
$$\tan \varphi = (1 - a_0)/(\lambda r/R)$$
, where  $\lambda = \Omega R/u_{\infty,\text{hub}}$  is the tip speed ratio. For a  
turbine operating close to optimal induction (i.e.,  $a_0 = 1/3$ ) and a typical tip speed ratio of 8.5, the inflow angle at three-quarter  
span is less than 6°, justifying the small angle assumption. This assumption clearly becomes less accurate for small inductions

and tip speed ratios, or close to the blade root, where however only a modest contribution to the thrust is generated.

Using again the small inflow angle assumption, it follows that  $\varphi \approx u_p/u_t$  and  $u \approx u_t$ , and the thrust T becomes

$$T = \frac{B}{2\pi} \int_{0}^{2\pi} \int_{0}^{R} \frac{1}{2} \rho u_t^2 c \left( C_D \frac{u_n}{u_t} + C_{L,\alpha} \left( \frac{u_n}{u_t} - \theta \right) \right) d\psi dr.$$
(17)

Using Eqs. (3) and (5), solving the double integral and expressing T through the thrust coefficient  $C_T = C_{T_1} + C_{T_2}$ , finally gives

$$C_{T_1} = \frac{\sigma}{2} (C_D + C_{L,\alpha}) \cos \mu \left(\lambda - k \cos \delta \sin \gamma\right) \left(1 - a_0\right), \tag{18a}$$

$$C_{T_2} = -\frac{\sigma}{2} C_{L,\alpha} \theta \left( \sin^2 \mu + \frac{2}{3} \lambda^2 - \frac{k \cos \delta}{12} \left( 8\lambda \sin \gamma - k \cos \delta (\cos^2 \gamma \sin^2 \delta + 3 \sin^2 \gamma) \right) \right), \tag{18b}$$

where  $\sigma = BcR/A$  is the rotor solidity, and  $\lambda = \Omega R/u_{\infty,\text{hub}}$  is the tip speed ratio.

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For null shear, i.e. k = 0, this expression simplifies to

$$C_T = \frac{\sigma}{2} \left( (C_D + C_{L,\alpha}) \cos \mu \left( (1 - a_0) \lambda \right) - C_{L,\alpha} \theta \left( \sin^2 \mu + \frac{2}{3} \lambda^2 \right) \right).$$
(19)

Notice that the terms in Eqs. (18) depending on shear k also depend on the angles  $\gamma$  and  $\delta$ , whereas Eq. (19) only depends on the total misalignment angle  $\mu$ . This is because the wind shear is defined with respect to the ground frame, which is mapped into the nacelle frame by the  $\gamma$  and  $\delta$  angles, whereas  $\mu$  only depends on the relative orientation of the wind vector with the rotor axis, as explained in Sect. 2.1.

Furthermore, we note that – differently from the approach of Heck et al. (2023) – these expressions for thrust are applicable to any desired control policy, as they depend explicitly on the tip speed ratio  $\lambda$  and the pitch setting  $\theta$ .

#### 2.7 Power

The aerodynamic power P generated by a wind turbine is  $P = Q\Omega$ , where the aerodynamic torque Q writes

270 
$$Q = \frac{B}{2\pi} \int_{0}^{2\pi} \int_{0}^{R} F_t r \,\mathrm{d}\psi \,\mathrm{d}r.$$
 (20)

Considering small angles, power can be written as

$$P = \frac{B}{2\pi} \int_{0}^{2\pi} \int_{0}^{R} \frac{1}{2} \rho u_t^2 c \left( -C_D + C_{L,\alpha} \left( \frac{u_n}{u_t} - \theta \right) \frac{u_n}{u_t} \right) \Omega r \, \mathrm{d}\psi \mathrm{d}r.$$

$$\tag{21}$$

Using Eqs. (3) and (5), expressing the angular velocity as  $\Omega = \lambda u_{\infty} / R \Omega = \lambda u_{\infty,\text{bub}} / R$ , and solving the double integral yields the power coefficient  $C_P$ :

275 
$$C_{P} = \frac{\sigma}{2} \lambda \left( C_{L,\alpha} (1 - a_{0}) \cos \mu \left( (1 - a_{0}) \cos \mu - \frac{2}{3} \lambda \theta \right) - \frac{1}{2} C_{D} (\lambda^{2} + \sin^{2} \mu) + k \cos \delta \sin \gamma \left( \frac{2}{3} \cos \mu C_{L,\alpha} \theta (1 - a_{0}) - \frac{1}{2} \lambda C_{D} \right) + \frac{1}{4} k^{2} \cos^{2} \delta \left( \cos^{2} \mu C_{L,\alpha} (1 - a_{0})^{2} - \frac{1}{4} C_{D} (\sin^{2} \mu + 2 \sin^{2} \gamma) \right) \right).$$
(22)

For null shear, i.e. k = 0, the power coefficient simplifies to

$$C_P = \frac{\sigma}{2} \lambda \bigg( C_{L,\alpha} (1 - a_0) \cos \mu \bigg( (1 - a_0) \cos \mu - \frac{2}{3} \lambda \theta \bigg) - \frac{1}{2} C_D (\lambda^2 + \sin^2 \mu) \bigg).$$
(23)

Due to the explicit dependency of  $C_P$  on  $\cos \mu$  and of  $1 - a_0$  on  $\sin^2 \mu$  (see Eq. 14), it follows that – in an unsheared inflow – the aerodynamic power does not depend on the misalignment direction.

#### 2.8 Dependency on misalignment direction

The power model reveals that vertical shear is the culprit for the observed lack of symmetry with respect to yaw misalignment. In fact, because of the second term in Eq. (22) (which is proportional to  $k \sin \gamma$ ), yawing a rotor out of the wind in a sheared inflow will produce a non-symmetric behavior with respect to positive and negative yaw angles  $\gamma$ , i.e.  $P(-\gamma) \neq P(+\gamma)$ . The yaw angle sign associated with the larger power capture depends on the balance of multiple terms, which in turn depend on some rotor design parameters but also on the pitch setting  $\theta$  and tip speed ratio  $\lambda$ . For negative or low  $\theta$ , negative yaw angles can lead to slightly higher power, whereas for increasing pitch – and hence a reduced thrust coefficient –, positive yaw angles produce more. This complex balance of effects is probably the cause for the lack of agreement in the literature on which misalignment direction yields more. As shown by the model, there is no simple answer, and the behavior depends on the rotor

290 design and on how it is operated.

Very similar conclusions apply also to the thrust coefficient. According to Eqs. (18),  $C_{T_1}(-\gamma) > C_{T_1}(+\gamma)$ , because  $C_{T_1}$  depends on  $-k \sin \gamma$ ; whereas  $C_{T_2}(-\gamma) < C_{T_2}(+\gamma)$  (when  $\theta > 0$ ), because  $C_{T_2}$  depends on  $+\theta k \sin \gamma$ . Therefore one can

expect a slightly higher thrust for negative yaw angles at low pitch settings, and viceversa at the higher pitch values (the effect being more pronounced at larger tip speed ratios).

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To illustrate these findings in an exemplary case, Fig. 5 shows the thrust and power coefficients as functions of the misalignment angle  $\gamma$ , for different shear coefficients k and blade pitch angles  $\theta q_{p}$ . All coefficients have been normalized by their respective value in aligned conditions.

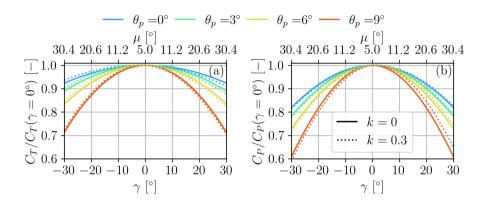


Figure 5. Normalized thrust  $C_T/C_{T_0} - C_T/C_T(\chi = 0^\circ)$  (a) and power  $C_P/C_{P_0} - C_P/C_P(\chi = 0^\circ)$  (b) coefficients, plotted as functions of the misalignment angle  $\gamma$ , for different shear coefficients k and pitch angles  $\theta \theta_P$ .  $C_{T,0}$  and  $C_{P,0}$  respectively indicate the thrust and power coefficients in aligned conditions, i.e. for  $\gamma = 0$ . The plots were generated considering the following values:  $\lambda = 8.5$ ,  $C_D = 4.4 \times 10^{-3} C_D = 5.2 \times 10^{-3}$ ,  $C_{L,\alpha} = 4.82 C_{L,\alpha} = 4.76$ ,  $\beta = -3.16^\circ \beta = 3.35^\circ$ ,  $\sigma = 4.16\%$ ,  $\delta = -5^\circ$ , R = 65 m. An interactive version of the figure that allows one to plot the thrust and power coefficients for user-defined values of the model parameters is available as a Jupyter notebook at the link https://tinyurl.com/btcl-fig-5.

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The lack of symmetry of the rotor with respect to misalignment direction is in general rather small. In a typical field implementation of wake steering, various uncertainties - e.g., due to limits in the knowledge of the ambient conditions, actual yaw orientation of the rotor, asymmetric behavior of the onboard anemometry, etc. - and other model errors probably dominate the problem, making the asymmetric behavior of misalignment a negligible effect, especially for small pitch values and moderate tip speed ratios.

The analysis can be conducted also for a horizontal shear, for example as produced by a wake impingement, leading to similar conclusions.

#### 305 2.9 Implementation

#### 3 Implementation

#### 3.1 Integration with a wake model

The analytical model derived in the previous pages can be readily implemented in engineering flow models (NREL, 2023b; Pedersen et al., 2019). An open-source implementation in FLORIS (NREL, 2023b) is available on Github (Tamaro et al., 2024).
As a summary, we report here for convenience the governing Eqs. (14) and (18), which write

$$1 - a_{0} = \frac{1 + \sqrt{1 - C_{T} - \frac{1}{16}C_{T}^{2}\sin^{2}\mu}}{2\left(1 + \frac{1}{16}C_{T}\sin^{2}\mu\right)},$$

$$C_{T} = \frac{\sigma}{2}\left((C_{D} + C_{L,\alpha})\cos\mu(\lambda - \cos\delta\sin\gamma k)\left(1 - a_{0}\right) - C_{L,\alpha}\theta\left(\sin^{2}\mu + \frac{2}{3}\lambda^{2} - \frac{k\cos\delta}{12}\left(8\lambda\sin\gamma - k\cos\delta(\cos^{2}\gamma\sin^{2}\delta + 3\sin^{2}\gamma)\right)\right)\right).$$
(24a)
(24a)
(24b)

This represents a closed system of equations that can be solved for the axial induction  $a_0$  and thrust coefficient  $C_T$ , given the 315 yaw misalignment  $\gamma$ , pitch setting  $\theta$ , and tip speed ratio  $\lambda$ , the pitch setting  $\theta_p = \theta - \beta$ , and the yaw misalignment  $\gamma$ . Crucially, the presence of  $\lambda$ . The solving system depends also on the parameters  $\sigma$ ,  $C_D$ ,  $C_{L,\alpha}$ ,  $\beta$  and  $\theta_p$  enable using any desired control policy when misaligning the turbine. Having obtained  $a_0$  and  $C_T$ , the rotor power is finally obtained as

$$P(\gamma, \theta, \lambda) = \frac{1}{2} \rho A u_{\infty, \text{hub}}^3 C_P(\gamma, \delta, \theta, \lambda),$$

where  $C_P$  is given power coefficient  $C_P(\gamma, \theta_P, \lambda)$  is obtained by Eq. (22) –

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It should be noted that, through Eq. (24b) and (25), the analytical model depends on  $C_D$ ,  $C_{L,\alpha}$  and  $\beta$ . These are average parameters, which represent in the model the "equivalent" effect caused by corresponding quantities that in reality vary spanwise (but possibly also azimuthally) over the rotor disk. Furthermore, the , finally, the rotor power is computed as

$$P(\lambda,\theta_p,\gamma) = \frac{1}{2}\rho A u_{\infty,\text{hub}}^3 C_P(\gamma,\theta_p,\lambda).$$
(25)

#### **3.2** Improved accuracy by the use of loss functions

325 The analytical derivation of the equations implies that the model lacks many of the features that are present in more sophisticated BEM implementations, such as radially and azimuthally non-uniform induction, swirl induction, tip and root losses, radial flow, spanwise varying geometric characteristics, prebend, etc. Clearly, this lack of accuracy could be resolved by numerically implementing the same model in a BEM code (Hansen, 2015). However, this way the use of the misaligned rotor model in combination with an engineering wake model would become much more complex and numerically expensive. To address this problem, the following implementation is recommended, which was used in the validation and the examples reported in the following section.

First, the A power loss function  $\eta_P$  is computed by using EqsEq. (24) and (25) to yield 22) to yield

$$\eta_P(\underline{\gamma}\underline{\lambda}, \underline{\delta}, \underline{\theta}_p, \underline{\lambda}\underline{\gamma}) = \frac{P(\gamma, \delta, \theta, \lambda)}{P(0, \delta, \theta, \lambda)} \frac{C_P(\lambda, \theta_p, \gamma)}{C_P(\lambda, \theta_p, 0)}.$$
(26)

Next, a refined power estimate P' coefficient  $C_P^{\text{rf}}$  is obtained as

335 
$$\underline{P'}C_{P}^{\text{rf}}(\underline{\gamma}\underline{\lambda},\underline{\delta},\underline{\theta}_{p},\underline{\lambda}\underline{\gamma}) = \eta_{P}(\underline{\gamma}\underline{\lambda},\underline{\delta},\underline{\theta}_{p},\underline{\lambda}\underline{\gamma})\underline{P^{*}}C_{P}^{\text{hf}}(\underline{0}\underline{\lambda},\underline{\delta},\underline{\theta}_{p},\underline{\lambda}\underline{0}),$$
(27)

where  $P^* C_P^{\text{hf}}$  is the power coefficient computed in aligned conditions through a higher-fidelity model, for example based on a sophisticated BEM implementation or even on experimental measurements, when available. In other words, the analytical model is used not to predict the actual power output coefficient, but only the fraction of power it that is lost by misalignment. The actual total capture power coefficient is obtained by applying the loss model to a more accurate power coefficient model in aligned conditions

340 in aligned conditions.

The same approach is adopted for thrust: a thrust change. First, a thrust coefficient loss factor is computed through the model as  $\eta_T(\gamma, \theta, \lambda) = T(\gamma, \theta, \lambda)/T(0, \theta, \lambda)$ , and a refined thrust by using the proposed model as  $\eta_T(\lambda, \theta_p, \gamma) = C_T(\lambda, \theta_p, \gamma)/C_T(\lambda, \theta_p, 0)$ . Next, a refined estimate in misaligned conditions is obtained as  $T'(\gamma, \theta, \lambda) = \eta_T(\gamma, \theta, \lambda)T^*(0, \theta, \lambda)$  by computing  $C_T^{\text{rf}}$  from a higher-fidelity aligned thrust model  $T^*$ . value  $C_T^{\text{hf}}$  i.e.

345 
$$C_T^{\rm rf}(\lambda,\theta_p,\gamma) = \eta_T(\lambda,\theta_p,\gamma)C_T^{\rm hf}(\lambda,\theta_p,0).$$
(28)

In the following, we always adopt this approach for the power and thrust coefficients. However, to simplify the notation, we drop the superscript  $(\cdot)^{\text{rf}}$ . Hence, for example, when we write  $C_P(\lambda, \theta_p, \gamma)$ , we in reality imply that Eq. (27) is used; the same holds for  $C_T$ .

#### 3.3 Implementation of arbitrary control strategies

350 When a wind turbine yaws out of the wind, the inflow seen by the rotor changes with respect to the aligned condition. The controller reacts to the changed inflow, modifying the setpoint, which in turn affects the power captured by the rotor and its loading. Hence, the problem is implicit, in the sense that the misalignment model has to be solved together with the controller. This general implicit approach should be contrasted with the explicit one proposed in Heck et al. (2023), which assumes a rotor performance parameter, C'<sub>T</sub> = 2T/(\rho Au\_n^2), to remain constant even in misaligned conditions. This section explains how arbitrary control laws can be integrated with the present more general model.

The parameters control of a modern variable-speed wind turbine is typically based on the definition of two or three main operational regions.

In region II (also called the below-rated or partial-load regime), the turbine should maximize its power output. This is achieved by operating at the maximum power coefficient  $C_P^*(\lambda^*, \theta_p^*)$ , which corresponds to the optimal tip speed ratio  $\lambda^*$ 

and pitch setting  $\theta_p^*$ . As the tip speed ratio must remain constant at its value  $\lambda^*$  throughout this control region, the rotor speed increases linearly with wind speed, i.e.  $\Omega = \lambda^* u_{\infty}/R$ . The aerodynamic torque  $Q_a$  is readily computed as  $Q_a = K(\rho)\Omega^2$ , with  $K(\rho) = \frac{1}{2}\rho AR^3 C_P^*/\lambda^{*3}$ . Once the aerodynamic torque is known, the torque provided by the generator  $Q_g$  is obtained from the expression  $Q_a = \eta_m \eta_e Q_g$ , where  $\eta_m$  and  $\eta_e$  are the mechanical and electrical efficiencies, respectively.

When the ambient wind speed is above the rated value  $u_{\infty r} = \Omega_r R/\lambda^*$ , there is enough power carried by the wind for the iturbine to produce its maximum (rated) output  $P_r$ . This is called region III (also termed the above-rated or full-load regime), and the turbine operates at the constant (rated) rotor speed  $\Omega_r$ . Hence, the aerodynamic torque is constant, i.e.  $Q_a = P_r/\Omega_r$ , whereas blades are progressively pitched into the wind to reduce  $C_P$  as wind speed increases.

To implement these standard region II and III control strategies in the proposed model, the following power equation is introduced:

370 
$$\frac{1}{2}\rho A u_{\infty}^{3} C_{P}(\lambda(\Omega), \theta_{p}, \gamma) = Q_{a}(\Omega) \Omega.$$
 (29)

The equation has three unknowns: the rotor speed  $\Omega$ , the blade pitch  $\theta_p$ , and the aerodynamic torque  $Q_a$ . Given ambient conditions  $u_{\infty}$  and  $\rho$ , two additional conditions are necessary before the three unknowns can be computed. To this end, one can first assume that the machine operates in region II. Hence, Eq. (29) is solved by appending to it the following two constraints:

$$\begin{array}{c} \theta_p = \theta_p^*, \\ \vdots \\ \vdots \\ \vdots \\ \vdots \\ \vdots \\ \end{array} \tag{30a}$$

$$375 \quad Q_a = K(\rho)\Omega^2. \tag{30b}$$

If the computed rotor speed exceeds the rated value, i.e.  $\Omega > \Omega_r$ , then it means that – for the given ambient conditions and misalignment angle – the turbine operates in region III and not region II. Hence, the solution is discarded, and Eq. (29) is solved again by appending this time the following two constraints:

$$\Omega = \Omega_r, \tag{31a}$$

$$Q_a = P_r / \Omega_r. \tag{31b}$$

This same approach can be used for curtailment and derating strategies (Juangarcia et al., 2018).

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Figure 6 shows the application of this approach to the IEA 3.4 MW reference wind turbine, a typical onshore machine with contemporary design characteristics (Bortolotti et al., 2019). In this example the turbine is exposed to an inflow characterized by a wind speed  $u_{\infty} = 10.5 \text{ ms}^{-1}$ , an air density  $\rho = 1.22 \text{ kgm}^{-3}$ , and a linear vertical shear coefficient k = 0.2. For these ambient conditions, the turbine operates in region III when it is aligned with the wind ( $\gamma = 0^{\circ}$ ). As the turbine starts yawing out of the wind, it initially keeps operating in region III. Accordingly, the tip speed ratio  $\lambda$  (Fig. 6d) and the power coefficient  $C_P$  (Fig. 6b) remain constant, while the thrust coefficient increases (Fig. 6a) and the blades pitch back (Fig. 6c). However, at around  $|\gamma| \approx 15^{\circ}$ , the turbine enters into region II, because the rotor-orthogonal component of the wind speed is not anymore large enough to maintain the rated power output. As the misalignment keeps increasing, the pitch angle remains fixed at its

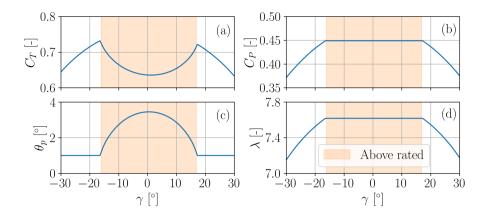


Figure 6. Behavior of the IEA 3.4 MW reference turbine as it yaws out of the wind, transitioning between the above-rated region III and the below-rated region II. Thrust coefficient  $C_T$  (a), power coefficient  $C_P$  (b), blade pitch angle  $\theta_p$  (c), and tip speed ratio  $\lambda$  (d). An interactive version of the figure is available as a Jupyter notebook at the link https://tinyurl.com/btcl-fig-6.

390 optimal value (Fig. 6c), whereas the tip speed ratio drops on account of the slowing rotor speed (Fig. 6d), in accordance with the region II policy.

Often turbines present an additional intermediate operating regime, called region II1/2, which occupies a wind speed interval across the rated value  $u_{\infty}^{lb} \leq u_{\infty r} \leq u_{\infty}^{ub}$ . In such cases, the turbine operates in region II when  $u_{\infty} < u_{\infty}^{lb}$ , in region III when  $u_{\infty} > u_{\infty}^{ub}$ , and in region II1/2 when  $u_{\infty}^{lb} \le u_{\infty} \le u_{\infty}^{ub}$ .

395 Differently from regions II and III, where controllers only require knowledge of the rotor speed, the control policy in region II1/2 typically requires prescribing the desired pitch and torque settings as functions of wind speed. In other words, one has to provide the schedules  $\theta_p = \theta_p(u_\infty)$  and  $Q_a = Q_a(u_\infty)$  in the desired range  $u_\infty^{lb} \le u_\infty \le u_\infty^{ub}$ . Two common examples of region II1/2 control policies are provided by load and noise alleviation techniques.

Load alleviation is often necessary because thrust reaches a sharp maximum at rated wind speed,  $T_r = \frac{1}{2}\rho A u_{\infty}^2 C_T(\theta_n^*, \lambda^*)$ . To reduce the effects of this large load on the sizing of various turbine components, thrust clipping (or peak shaving) is used, 400 where blades are pitched to feather according to a desired schedule  $\theta_p = \theta_p(u_\infty)$ . This has the effect of reducing the angle of attack and hence the thrust (Zalkind et al., 2022), at the price of some reduced power. To minimize power losses for a given pitch schedule, the optimal power coefficient schedule can be computed as  $C_P(u_\infty) = \max_{\Omega} C_P(\Omega R/u_\infty, \theta_P(u_\infty))$ under the constraint  $\Omega \leq \Omega_r$ , which also returns the rotor speed schedule  $\Omega(u_{\infty})$ . Consequently, the torque schedule becomes

405 
$$Q_a(u_\infty) = \frac{1}{2}\rho A u_\infty^3 C_P(u_\infty) / \Omega(u_\infty)$$

One effective way of constraining noise emissions is to limit the rotor speed (Leloudas et al., 2007; Bottasso et al., 2012) to a maximum noise-acceptable value  $\Omega_n$ . When  $\Omega_n < \Omega_r$ , the increase in rotor speed as a function of wind speed that characterizes region II is stopped before the machine reaches rated power. In this case, region II1/2 is entered when  $u_{\infty} > u_{\infty}^{lb} = \Omega_n R / \lambda^*$ and, above this wind speed, the rotor operates at the constant speed  $\Omega_n$ . To minimize power losses, the blade pitch setting can 410 be computed for each wind speed  $u_{\infty}$  as  $\theta_p(u_{\infty}) = \arg \max_{\theta_p} C_P(\lambda(u_{\infty}), \theta_p)$ , where  $\lambda(u_{\infty}) = \Omega_p R/u_{\infty}$ . The corresponding aerodynamic torque schedule is readily obtained as  $Q_a(u_{\infty}) = \frac{1}{2}\rho ARC_P(\lambda(u_{\infty}), \theta_p(u_{\infty}))/\lambda(u_{\infty})$ . The end of region II1/2 is reached when, for sufficiently high  $u_{\infty}$ , the turbine reaches rated power, finally entering into region III.

In order to implement a given control policy for region II1/2 with the proposed model, we assume that the controller will implement the desired schedules  $\theta_p(u_{\infty})$  and  $Q_a(u_{\infty})$  (whether computed as explained above or according to different criteria)

415 by reacting to the rotor-orthogonal wind speed component  $u_{\infty} \cos \gamma$ . In the absence of specific details on the implementation, this is a reasonable assumption, as region II1/2 controllers are typically based on rotor-effective wind speed estimates (Bottasso et al., 2012; that, in a misaligned condition, will sense  $u_{\infty} \cos \gamma$  and not  $u_{\infty}$ . Therefore, when  $u_{\infty}^{lb} \leq u_{\infty} \cos \gamma \leq u_{\infty}^{ub}$ , Eq. (29) is solved by appending to it the following two constraints:

$$\theta_p = \theta_p(u_\infty \cos \gamma), \tag{32a}$$

(32b)

420 
$$Q_a = Q_a(u_\infty \cos \gamma)$$

In summary, using Eq. (29) in combination with the constraint Eqs. (30), (31), or (32), yields the setpoint achieved by the turbine for given ambient conditions and a given misalignment, no matter what region it corresponds to and what control strategy is implemented by the controller.

#### 425 3.4 Model calibration

Through Eq. (24b) and (25), the analytical model depends on  $C_D$ ,  $C_{L,\alpha}$  and  $\beta$ . These are average parameters, which represent in the model the "equivalent" effect caused by corresponding quantities that in reality exhibit a spanwise variability.

When numerical or experimental measurements are available, the parameters  $C_D$ ,  $C_{L,\alpha}$ , and  $\beta$  are can be calibrated to minimize the error produced by the model in the prediction of the power loss factor  $\eta_P$  and of the thrust coefficient  $C_T$ . Notice 430 that  $C_T$  is preferred to  $\eta_T$  for this scope, because it was found that the informational content of  $\eta_T$  is very similar to the one of  $\eta_P$ , reducing the quality of the tuning.

Tuning is Calibration is here performed by numerically solving the following minimization problem

$$\min_{C_D, C_{L,\alpha}, \beta} \quad \sqrt{\frac{1}{N} \sum_{i=1}^{N} \left( \eta_{P,i}^{\text{obs}} - \eta_{P,i}^{\text{mod}} \right)^2} + \sqrt{\frac{1}{N} \sum_{i=1}^{N} \left( C_{T,i}^{\text{obs}} - C_{T,i}^{\text{mod}} \right)^2},\tag{33}$$

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where  $(\cdot)_i^{\text{obs}}$  are N numerical or experimental observations, and  $(\cdot)_i^{\text{mod}}$  the corresponding model predictions. For each data set, tuning was performed solving N/2 times problem the problem expressed by Eq. (33) using a gradient based optimization, each time with a different random 50% subset of the available data, and finally averaging the resulting parameters.

## 3.5 Simplified choice of model parameters in the absence of calibration data

When calibration is not possible, the equivalent model parameters  $C_D$ ,  $C_{L,\alpha}$  and  $\beta$  must be estimated from the corresponding actual spanwise distributions  $C_D(r/R)$ ,  $C_{L,\alpha}(r/R)$  and  $\beta(r/R)$ .

- Examining the expression for thrust given by Eq. (16), neglecting drag, it appears that the lift force has roughly a spanwise triangular distribution. In fact, inspecting Eqs. (5),  $u_t$  is proportional to r, whereas  $u_n$  does not depend on r. Additionally, the leading term of the Taylor series of the optimal twist distribution is 1/r (Burton et al., 2011). Similarly, inspecting the expression for power given by Eq. (21), again neglecting the contribution of drag, it appears that – for the same reasons – also the spanwise power capture has a triangular distribution. This suggests to evaluate the spanwise integrals at the centroid of the
- triangle, which is located at r/R = 2/3. Adopting this approach, the model parameters are then set to the following values:

$$C_D = f_d C_D(2/3), \tag{34a}$$

$$C_{L,\alpha} = f_l C_{L,\alpha}(2/3), \tag{34b}$$

$$\beta = \beta(2/3). \tag{34c}$$

Coefficient  $f_d$  is a correction factor for drag, while  $f_l$  is a knockdown factor for lift, which accounts for the finite span of the blades. Based on comparisons with calibrated values (see Sect. 4.2), we recommend a drag correction factor  $f_d = 1$  for moderate yaw (up to 20°) and pitch values, and a smaller value of 0.45 if the model has to be used also for large yaw and pitch settings. This smaller value is probably due to the approximation of a small inflow angle used in the model, which is partially corrected by a smaller drag coefficient. For lift, we recommend the value  $f_l = 2/3$ .

The performance of this simplified choice of model parameters is demonstrated later in Sect. 4.2.

#### 455 4 Model validation

#### 4.1 Validation with respect to LES-ALM numerical modelsimulations

LES-ALM simulations are used for testing the accuracy of the model in representing misaligned conditions, similarly to what done by other authors (Gebraad et al., 2016; Liew et al., 2020; Nanos et al., 2022). The effects of the rotor on the flow are modelled with the filtered ALM of Troldborg et al. (2007) and Martínez-Tossas and Meneveau (2019), by projecting forces computed along the lifting lines onto the LES grid. The Cartesian mesh consists of approximately 3.5 million cells, and uses

460

four refinement levels. The smallest cells measure 1 m, and are located in correspondence of the rotor.

Operational scenarios for the LES-ALM simulations. Scenario # 1 2 3 4  $\lambda$  -8 9.5 8.38 8.38 k -0 0 0.06 0.19  $\theta_p$  deg

Simulations were conducted for the IEA 3.4 MW reference wind turbine, a typical onshore machine with contemporary design characteristics. The technical specifications of the machine whose complete technical specifications are reported in

Bortolotti et al. (2019). Here we only note that the turbine has a 5° uptilt angle, i.e.  $\delta = -5^{\circ}$ . The four operational scenarios of Table 1 were considered, each corresponding to a different parameters of the proposed model were calibrated as explained in Sect. 3 based on the LES-ALM simulations described in Sect. 4.1.2, but not the ones of Sect. 4.1.1, obtaining the values  $C_D = 0.0052 \pm 0.0001$ ,  $C_{L,\alpha} = 4.759 \pm 0.007$  rad<sup>-1</sup>, and  $\beta = -3.345 \pm 0.007^{\circ}$ , for a 95% confidence level.

# 4.1.1 Simulations in control regions II and III

470 First we demonstrate the integration with a standard controller, including operations in region II, III, and the transition between the two as the turbine is progressively yawed out of the wind. For the LES-ALM simulations, setpoints were computed using a controller in the loop, based on an implementation similar to the one of Bortolotti et al. (2019). For the proposed model, setpoints were obtained from Eq. (29) in combination with the constraint Eqs. (30) and (31).

We consider laminar inflows with four wind speeds: one below-rated speed of 8.5 ms<sup>-1</sup>, and three above-rated speeds of

- 475 10.5, 11, and 13 ms<sup>-1</sup>. Figure 7 reports the results in term of the thrust loss factor  $\eta_T$  (panel a), power loss factor  $\eta_P$  (panel b), blade pitch  $\theta_p$  (panel c), and tip speed ratio  $\lambda$  and shear coefficient k. The flow is steady in all scenarios, but only cases 3 and 4 represent a sheared inflow, as shown in Fig. ??. In each scenario, (panel d), all plotted as functions of the misalignment angle  $\gamma$ . The solid markers are the results of the LES-ALM simulations, while the lines represent predictions of the proposed model.
  - For the lowest wind speed of  $8.5 \text{ ms}^{-1}$ , the turbine always operates in region II. This is the only case where the method of
- 480 Heck et al. (2023) is strictly applicable. In fact, their method does not contain a generic thrust model, but rather it is formulated in terms of the modified thrust coefficient  $C'_T$ , which is constant when a turbine yaws out of the wind in region II. The results of this alternative model are shown with a dashed orange line in the figure. The benchmark LES simulations feature a negative uptilt  $\delta = -5^\circ$ , which is not modelled in the approach of Heck et al. (2023). To avoid cluttering the results with this additional effect, here and in the following examples the total true misalignment  $\mu$  (instead of  $\gamma$ ) is provided as input to the model of
- 485 Heck et al. (2023). The figure shows that both methods are in excellent agreement with the CFD results. For the highest wind speed of 13 ms<sup>-1</sup>, the thrust coefficient was changed by varying the pitch angle according to the values  $\theta_p = \{1.4, 4.9, 6.7, 8.1\}^\circ$ , using the derating controller of Campagnolo et al. (2023) turbine operates in region III for all misalignment angles. On the other hand, for a wind speed of 11 ms<sup>-1</sup> the machine enters region II around  $\gamma = 27^\circ$ , and for 10.5 ms<sup>-1</sup> at about  $\gamma = 16^\circ$ .
- 490 In general, there is a very good agreement of the model with the higher fidelity CFD results, not only in terms of loss factors, but also on the calculation of the setpoints.

The model parameters were calibrated

#### 4.1.2 Simulations with fixed tip speed ratio and pitch setting

Next, we present a second set of results obtained by varying the misalignment while keeping the tip speed ratio and pitch

495 constant, for different inflows. These conditions are meant to provide a more general view of the performance of the proposed method in a variety of conditions, although – as explained in Sect. 3, obtaining the values  $C_D = 0.0040 \pm 0.0001$ ,  $C_{L,\alpha} = 4.796 \pm 0.038$  rac  $\beta = -3.177 \pm 0.005^\circ$ , with a 95% confidence level 3.3 – tip speed ratio and pitch in general do not both remain constant when a turbine yaws out of the wind.

The four operational scenarios of Table 1 are considered. The flow is laminar and steady in all scenarios. Cases 1 and 2 have 500 no shear and different tip speed ratios, whereas cases 3 and 4 are sheared and have the same  $\lambda$ .

Figure ?? reports

Figures 8 and 9 report the power loss factor  $\eta_P$  in the range of yaw misalignment angles  $-30^\circ < \gamma < 30^\circ$ , for the four scenarios for different pitch settings, each corresponding to a different thrust coefficient  $C_{T,0}$  in aligned conditions. Figure 8

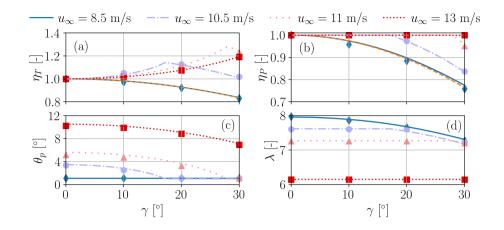


Figure 7. Vertical inflow wind Thrust loss factor  $\eta_T$  (a), power loss factor  $\eta_P$  (b), blade pitch  $\theta_P$  (c), and tip speed profiles used in ratio  $\lambda$  (d), plotted as functions of the LES-ALM simulations misalignment angle  $\gamma$ , with corresponding best-fitted linear shears (dotted for various below, above and around rated wind speeds. Present model: lines; solid markers: LES-ALM; dashed orange lines: Heck et al. (2023) (only for the 8.5 ms<sup>-1</sup> case).

Table 1. Operational scenarios for the LES-ALM simulations.

Scenario #	1	2~	3~~~~	<u>4</u>
<u>λ[-]</u>	<u>8</u>	<u>9.5</u>	8.38	8.38
<u>k</u> [-]	0	$\underbrace{0}_{\sim}$	0.06	0.19

corresponds to scenarios 1 and 2 of Table 1and different pitch settings. Model predictions are indicated with lines, and
 505 LES-ALM results with markers, i.e. no shear, while Fig. 9 reports the solution for the sheared cases 3 and 4 of that same table.

In the figures, LES-ALM results (shown with black circular markers) are compared with the proposed approach (shown with blue solid lines) and the method proposed by Heck et al. (2023) (shown with orange dashed lines). For the latter, the modified thrust coefficient  $C'_T$  was obtained directly from each LES simulation at the corresponding  $\gamma$  value.

- 510 Overall, there is a very good match between model predictions the predictions of the proposed model and numerical simulations. Results for inflow scenarios 1 and 2, which are characterized by a null vertical shear (see For the null shear cases of Fig. ??), are reported in Figs. ??a and ??b. As predicted by the model, in these cases 8, results are reported only for positive yaw angles, as power is symmetric with respect to positive and negative yaw angles. On the other hand, power is not symmetric in Figs. ??e and ??d, which correspond to for the sheared inflow cases of scenarios 3 and 4. In particular, Fig. ??e 9, which
- shows clear evidence of the complex behavior described in Sect. 2.8. At high  $C_T$  (low pitch), the curve is curves are very nearly symmetric with respect to  $\gamma$ . However, as thrust is decreased (and pitch increased), power capture at positive is larger at

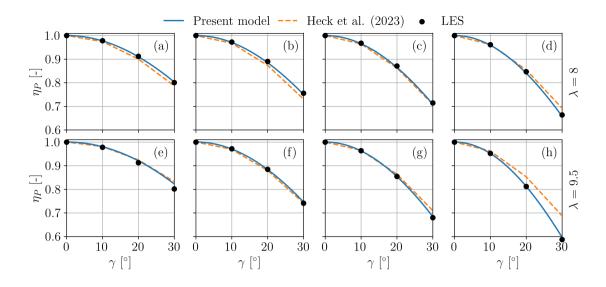


Figure 8. Power loss factor  $\eta_P$  vs. misalignment angle  $\gamma$  . Model predictions: lines; LES-ALM simulations: markers. The different line in the unsheared scenarios 1 ( $\lambda = 8$ ) (**a**-**d**) and marker styles correspond 2 ( $\lambda = 9.5$ ) (**e**-**h**). Each subplot corresponds to the a different value of the thrust coefficients  $C_T$ . Scenario 1 coefficient in wind-aligned conditions:  $C_{T,0} = 0.74$  (**a**); scenario 2  $C_{T,0} = 0.54$  (**b**); scenario 3  $C_{T,0} = 0.44$  (**c**); scenario 4  $C_{T,0} = 0.36$  (**d**),  $C_{T,0} = 0.86$  (**e**);  $C_{T,0} = 0.60$  (**f**);  $C_{T,0} = 0.47$  (**g**);  $C_{T,0} = 0.35$  (**h**).

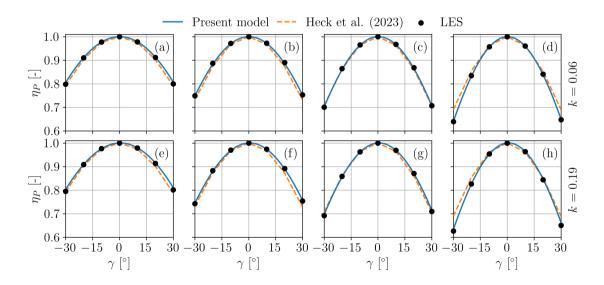


Figure 9. Power loss factor  $\eta_P$  vs. misalignment angle  $\gamma$  in the sheared scenarios 3 (k = 0.06) (**a-d**) and 4 (k = 0.19) (**e-h**). Each subplot corresponds to a different value of the thrust coefficient in wind-aligned conditions:  $C_{T,0} = 0.77$  (**a**);  $C_{T,0} = 0.56$  (**b**);  $C_{T,0} = 0.45$  (**c**);  $C_{T,0} = 0.36$  (**d**),  $C_{T,0} = 0.77$  (**e**);  $C_{T,0} = 0.56$  (**f**);  $C_{T,0} = 0.45$  (**g**);  $C_{T,0} = 0.36$  (**h**).

positive than negative  $\gamma$  values is larger than for negative misalignments. As predicted by the model, this effect is more visible in Fig. ??e than in .

The model of Heck et al. (2023) performs similarly well at high and moderate rotor loading, when  $C'_T$  is roughly constant.

520 However, as the  $C_T$  is reduced, the model tends towards the solution  $\cos^3 \gamma$ , and therefore its accuracy is compromised. Moreover, the model fails to predict the shear-induced asymmetry (see Fig. ??d, because the former has a higher tip speed ratio than the latter 9).

As predicted by the proposed model, the power asymmetry increases with shear (see Table 1 and the explanation given in Sect. 2.8).

- 525 The results of To facilitate the visualization of this effect, Fig. ?? clearly indicate a strong dependency of the power loss factor on the trust coefficient. This can be appreciated even more clearly in Fig. ??, where the results for all scenarios-10 shows the difference  $\Delta \eta_{P,\gamma=\pm 30^\circ}$  between the two values of  $\eta_P$  at  $\gamma = \pm 30^\circ$  were interpolated to plot the average  $\bar{\eta}_P$ as a function of  $\lambda$  for four different values of the thrust coefficient. The plot clearly shows that low shear, for varying thrust coefficients. The asymmetry exhibits also a noticeable dependency on the thrust coefficient, larger asymmetries being
- 530 <u>observed for lower values of  $C_T$  values increase power losses in misaligned conditions, in agreement with Heck et al. (2023)</u> and Campagnolo et al. (2023). Similarly, the tip speed ratio has also a strong influence on power losses, which decrease when  $\lambda$  is reduced.

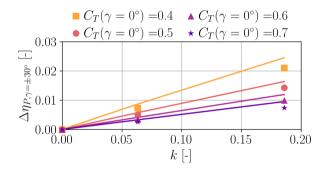


Figure 10. Power Difference  $\Delta \eta_{P,\gamma \equiv \pm 30^\circ}$  between power loss factor  $\bar{\eta}_P$  (averaged among all scenarios) factors  $\eta_P$  evaluated at misalignments  $\gamma = \pm 30^\circ$  vs. tip speed ratio  $\lambda \gamma \equiv 30^\circ$  and at  $\gamma \equiv -30^\circ$ , as a function of vertical linear shear coefficient k, for varying thrust coefficient  $C_T$ . Proposed model: solid lines; LES-ALM simulations; markers.

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Figure ?? reports the thrust change Figures 11 and 12 report the thrust loss factor  $\eta_T$  as a function of yaw misalignment, for the same four scenarios and different thrust settings. Here again, model predictions are indicated with lines, and LES-ALM results with markers. There is a consistently good match, for all scenarios, and for all yaw and pitch values. The lack of symmetry is again consistent with the model, similarly to the case of power discussed above. Figure ??c shows a Figures 12a to 12d show a higher thrust for positive yaw angles at low thrust coefficients (high pitch values), because of the high tip speed

ratio of scenario 3, indicating that term  $C_{T_2}$  prevails over  $C_{T_1}$ . The opposite happens in Fig. ??d12e to 12h, due to the lower  $\lambda$  of scenario 4.

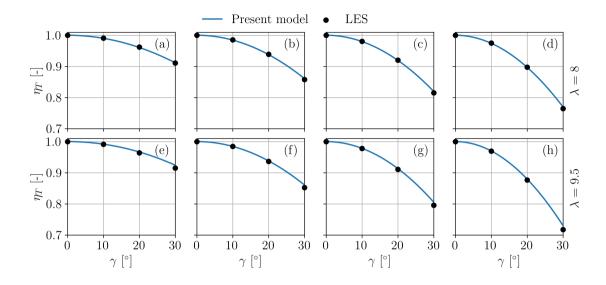


Figure 11. Thrust ehange loss factor  $\eta_T$  vs. misalignment angle  $\gamma$  -Model predictions: lines; LES-ALM simulations: markers. The different line in scenario 1 with  $\lambda = 8$  (a-d) and marker styles correspond scenario 2 with  $\lambda = 9.5$  (e-h). Each subplot corresponds to the a different pitch settings. Scenario 1 value of the thrust coefficient in wind-aligned conditions:  $C_{T,0} = 0.74$  (a); scenario 2  $C_{T,0} = 0.54$  (b); scenario 3  $C_{T,0} = 0.44$  (c); scenario 4  $C_{T,0} = 0.36$  (d),  $C_{T,0} = 0.86$  (e);  $C_{T,0} = 0.60$  (f);  $C_{T,0} = 0.47$  (g);  $C_{T,0} = 0.35$  (h).

Overall, it appears that the performance of the rotor is strongly dependent on thrust coefficient and tip speed ratio, and hence on the way it is controlled when it yaws out of the wind. Hence Therefore, the standard power law  $\cos^{p_p}$  may oversimplify the complex aerodynamics that are typical of this problem. On the other hand, notwithstanding its simplicity, the proposed model is in very good agreement with sophisticated CFD simulations, and it is capable of describing even relatively minor effects of the complex behavior of a misaligned wind turbine rotor in a sheared inflow.

#### 545 4.2 Wind tunnel measurements Validation of the simplified choice of model parameters

The simplified choice of model parameters described in Sect. 3.5 is based on Eqs. (34), which include the correction factors  $f_d$  and  $f_l$ . To verify the existence of typical values for these factors, we considered four different wind turbines: IEA 3.4 MW (Bortolotti et al., 2019); NREL 5 MW (Jonkman et al., 2009); G178, which is a modified version of the DTU 10 MW (Bak et al., 2013; War ; and the small-scale G1 turbine (Bottasso and Campagnolo, 2022a; Campagnolo et al., 2020). For the three full-scale machines,

the model parameters were first calibrated using the LES-ALM simulation results of Sect. 4.1.2, whereas for the G1 model the calibration was performed using wind tunnel measurements (see later Sect. 4.3).

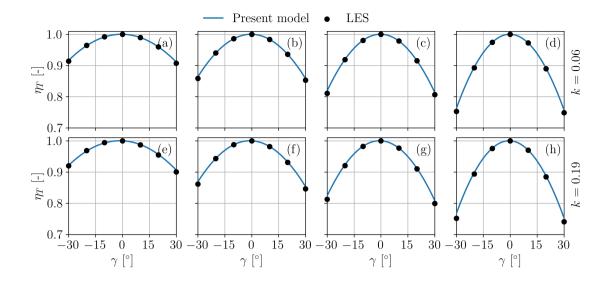


Figure 12. Thrust loss factor  $\eta_T$  vs. misalignment angle  $\gamma$  in scenarios 3 with k = 0.06 (a-d) and scenario 4 with k = 0.19 (e-h). Each subplot corresponds to a different value of the thrust coefficient in wind-aligned conditions:  $C_{T,0} = 0.77$  (a);  $C_{T,0} = 0.56$  (b);  $C_{T,0} = 0.45$  (c);  $C_{T,0} = 0.36$  (d),  $C_{T,0} = 0.77$  (e);  $C_{T,0} = 0.56$  (f);  $C_{T,0} = 0.45$  (g);  $C_{T,0} = 0.36$  (h).

The parameters calibrated this way were then compared to the ones based on the simplified approach of Eqs. (34), leading to the recommended values reported in Sect. 3.5. The  $C_D(r/R)$  and  $C_{L,\alpha}(r/R)$  coefficients were obtained by averaging over the interval of angles of attack 2° below the negative and positive stall limits. In all cases, the calibrated value of the twist corresponded remarkably well with the actual twist at 2/3 span, i.e.  $\beta(2/3)$ .

The simplified choice of the model parameters was then applied to the NREL 5 MW and G178 10 MW wind turbines. Simulations were performed with a steady inflow, at different misalignments and for two different blade pitch settings. The proposed model was calculated with the parameters based on Eqs. (34), using the default correction coefficients  $f_d = 0.45$  and  $f_l = 2/3$  (in other words, without using LES-ALM calibrated values, replicating what one could do in the absence of suitable tuning data).

The results in terms of  $\eta_T$  and  $\eta_P$  for the two turbines are reported in Fig. 13, and compared with LES-ALM simulations. For both turbines there is an excellent match between model predictions and CFD results. This seems to indicate that the even a simplified choice of the model parameters is sufficient for a good performance of the model.

# 4.3 Validation with respect to wind tunnel measurements

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565 Next, the model is compared to data recorded during wind tunnel experimental campaigns performed with a G1 wind turbine (Campagnolo et al., 2016). This scaled machine has a diameter of 1.1 m, a rated rotor speed of 850 RPM, and null tilt. The de-

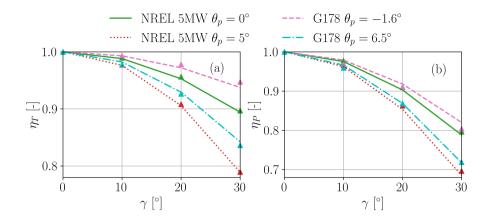


Figure 13. Thrust  $\eta_T$  (a) and power  $\eta_P$  (b) loss factors as functions of the misalignment angle for two pitch settings. Results obtained with the NREL 5 MW (Jonkman et al., 2009) and G178 10 MW (Bak et al., 2013; Wang et al., 2021) wind turbines, using the simplified calculation of the model parameters based on Eqs. (34). Proposed model: lines; LES-ALM simulations; markers.

sign of the G1 is described in Bottasso and Campagnolo (2022b)Bottasso and Campagnolo (2022a), and its rotor aerodynamic and wake characteristics have been reported in Wang et al. (2021) and references therein.

Tests were performed in a boundary layer wind tunnel (Bottasso et al., 2014) with three different inflows: the first one, 570 termed Low-TI, has no shear and a very low turbulence intensity (approx. approximatively equal to 1%); the other two, termed Mod-TI and High-TI, have respectively TIs of approx. about 6% and 13% at hub height, and vertical linear shears in the rotor region equal to k = 0.11 and k = 0.15, respectively. Figure 14a reports the vertical profiles of the longitudinal wind speed component u measured by means of CTA probes (Bottasso et al., 2014), normalized by the wind speed  $u_{pitot}$  measured by a Pitot tube placed at hub height. Figure 14b shows the vertical profiles of the turbulence intensity, as measured with the same 575 instrumentation.

Two different campaigns were conducted for characterizing. The experimental characterization of power losses in misaligned conditions : in the first one the turbine was set at full power, whereas in was performed based on the three campaigns of Table 2, totalling 119 observations of a duration of 2 minutes each. The measured average  $\theta_p$  and  $\lambda$  are reported in Appendix C for campaigns 1 and 2, and in Fig. 19 for campaign 3.

580 The first campaign was conducted in region II (i.e., the below-rated partial-load regime), using the classical variable-rotor-speed maximum-power tracking strategy. Tests were conducted in all three Low-TI, Mod-TI, and High-TI inflow conditions, with hub-height wind speeds of 5.86, 5.69, and 5.40 ms<sup>-1</sup>, respectively.

The second campaign was also conducted in region II, but in this case the second at different levels of derating. For the second campaign the turbine was derated in the range  $P_d \in [50, 100]\%$  while adopting two different strategies: iso- $\lambda$ , where the

tip speed ratio is held constant (Campagnolo et al., 2023), and min- $C_T$ , where the thrust coefficient is minimized (Juangarcia et al., 2018). The first campaign was conducted in all three Low-TI, Tests were conducted only in the Mod-TI , and High-TI

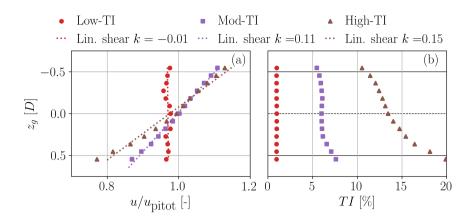


Figure 14. Wind tunnel inflows. Vertical wind speed profiles, with corresponding best-fitted linear shears (dotted lines) (a); vertical profiles of turbulence intensity (b).

inflowconditions, with inflow, with a hub-height wind speeds of 5.86, 5.69, and 5.40 speed of 5.62 ms<sup>-1</sup>, respectively. The second

The third campaign was conducted only in the in region III (i.e., the above-rated full-load regime) in the Low-TI and Mod-590 TI inflow, with a inflows, with hub-height wind speed of 5.62speeds of 6.97 and 6.11 ms<sup>-1</sup>. All test conditions correspond to region II operation, i.e. the below-rated partial-load regime of the turbine, respectively.

Table 2.	Characteristic	s of the wind	tunnel	campaigns.

Campaign #	Inflow wind speed		Control region	Used for tuning	
Campaign #	Low-TI	Mod-TI	High-TI	Control region	Used for tuning
<u>1 - Max P tracking</u>	$\underbrace{5.86\text{ms}^{-1}}_{\bullet$	$5.69 \text{ ms}^{-1}$	$5.40 \text{ ms}^{-1}$	Ĩ	Yes
2 - Derating	~	$5.62 \text{ ms}^{-1}$	-~	$\mathbf{I}_{\sim}$	Yes
3 - Above rated	$\underline{6.97}\mathrm{ms}^{-1}$	$6.11 \text{ ms}^{-1}$	-~	ĨĨĬ	No

Various sources of error affect the experimental observations. These include measurements of the wind speed  $u_{\text{pitot}}$  upstream

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of the model (obtained by a Pitot tube placed at hub height 3D in front of the turbine), of the air density  $\rho$ , of the rotor speed  $\Omega$ , of the shaft torque Q, of the bending moment at tower base (which is used to estimate thrust), of the blade pitch angle, and of the nacelle orientation with respect to the wind tunnel (i.e., of the misalignment angle). The error in  $u_{\text{pitot}}$  is related to the uncertainty associated with the measurements of flow density and dynamic pressure. This latter quantity is measured with a MKS Baratron-Type 226A transducer (MKS Instruments, Inc., 2022) with full span equal to 1 Torr, characterized by an accuracy of  $\pm 0.4$  Pa. Density is instead derived from measurements of air pressure, temperature and humidity, and it is affected by an error equal to  $\pm 0.01$  kgm<sup>3</sup> (Wang et al., 2020). Torque is measured with a load cell installed on the rotor shaft, and it is affected by an uncertainty of  $\pm 0.005$  Nm. The rotor speed measurement, provided by an optical incremental encoder, is instead

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affected by an error equal to  $\pm 1.5$  RPM. The measurement uncertainty on power  $P = Q\Omega$  is derived by adding in quadrature the uncertainties on Q and  $\Omega$ . Finally, thrust Thrust T is obtained by correcting the measurements of the bending moments at tower base by the effects induced by the drag of the tower, nacelle and hub spinner (Wang et al., 2020). The calibration of the load cell at tower base reveled an uncertainty on revealed an uncertainty in the thrust of  $\pm 0.14$  N. Blade pitch and nacelle

605 <u>orientation are measured by optical encoders, affected by uncertainties of  $\pm 0.2^{\circ}$ .</u> In turn, all these effects are used to quantify uncertainties in the tip speed ratio  $\lambda$ , thrust coefficient  $C_T$ , and yaw-induced power and thrust losses  $\eta_P$  and  $\eta_T$ , again by adding errors in quadrature.

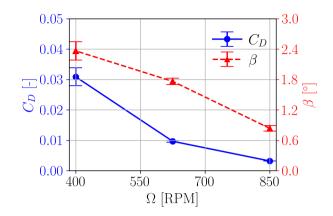
In the two test campaigns, In the following, the resulting uncertainties are reported for a 95% confidence level. Uncertainties in some experimental measurements affect also the predictions of the proposed model. The uncertainties of the

- 610 four model inputs tip speed ratio, blade pitch, rotor speed, and yaw misalignment were propagated forward throughout the model by Latin hypercube sampling with ten thousand sample points, using the UQLab software (Marelli and Sudret, 2014). Tuning of the model parameters  $C_D$ ,  $C_{L,\alpha}$  and  $\beta$  was performed with Eq. (33), using a total of 94 observations were conducted, each one for a duration of 2 minutes. The measured average  $\theta_p$  observations from the experimental campaigns 1 and  $\lambda$  are reported in Appendix C, and were used as inputs to the model equations. 2.
- Given the small size of the G1 wind turbine, the Reynolds number at its blade sections is particularly low. Although special low-Reynolds airfoils are used in the design of the G1 blades (Bottasso and Campagnolo, 2022b), their lift and drag coefficients (Bottasso and Campagnolo, 2022a), their aerodynamic characteristics are particularly sensitive to the operating conditions of the turbine (Wang et al., 2020). In order to account for this effect, the fact, the Reynolds number has a significant effect on the drag and on the zero-lift direction, which in turn affects the parameter  $\beta$ , whereas the effect on the lift slope  $C_{L,\alpha}$  is negligible
- 620 (Wang et al., 2020). Accordingly, the model parameters  $C_D$  and  $C_{L,\alpha}\beta$  are assumed to depend on the rotational speed  $\Omega$ , since the relative speed at the airfoils is close to the tangential speed ( $u \approx u_t$ , see Fig. 3). Tuning was performed with Eq. (33), using as unknown parameters the The values of  $C_D$  and  $C_{L,\alpha}\beta$  at  $\Omega = [850, 625, 400]$  RPM ; were assumed as unknowns, and a piecewise linear interpolation was used at other intermediate values of the rotor speed. Clearly, it is not necessary to consider a Reynolds-dependency for  $\beta$ , and therefore a single value for this parameter was considered.
- Figure 15 reports the tuned  $C_D$  and  $C_{L,\alpha}$  parameters, the shaded areas  $\beta$  parameters, whiskers representing the corresponding 95% confidence intervals. As expected, drag decreases whereas lift slope increases for increasing rotor speed, i.e. for increasing sectional Reynolds number. The tuned twist  $\beta$  also exhibits the same trend, since the zero-lift direction rotates nose-up as the Reynolds number increases (Wang et al., 2020). The tuned parameter  $C_{L,\alpha}$  is equal to  $1.4472 \pm 0.1408^{\circ}4.5033 \pm 0.0459$ .

For full-power operation (first test campaign maximum power tracking operation in region II (test campaign 1), Fig. ?? 16 re-

630 ports a comparison between model-predicted (lines) and measured (markers) power losses and thrust coefficients. and measured power and thrust losses. The present model results are indicated with blue solid lines, while measurements are indicated by black circles. Whiskers indicate the respective 95% confidence intervals. There is a very good match between experiment and experimental measurements and the present model, the latter falling within the uncertainty range of the measurements in most cases. As predicted by the model, the sheared inflow conditions Mod-TI and High-TI exhibit the expected non-symmetric

635 behavior with respect to positive and negative yaw angles.



**Figure 15.** Tuned model parameters  $C_D$  and  $C_{L,\alpha}$  as functions of rotational speed  $\Omega$ .

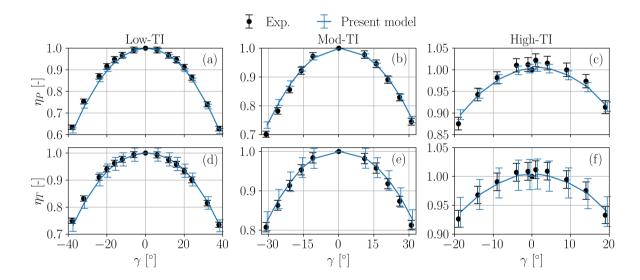


Figure 16. Experimental campaign 1. Power loss factor  $\eta_P$  (top row) (a-c) and thrust change loss factor  $\eta_T$  (bottom row) (d-f) vs. yaw misalignment  $\gamma$ , in full-power region II operation for the three different inflowsinflow conditions. Model predictions: solid lines; experiments: markers. The bars\_Whiskers indicate measurement uncertainties the 95% confidence intervals.

The same quantities In principle, the model of Heck et al. (2023) would be applicable to these tests in region II. However, their method does not directly consider the aerodynamic characteristics of the blades, as it expresses their behavior through the single parameter represented by  $C'_T$ . Therefore, it is blind to the variability of twist and drag with respect to the Reynolds number, which drops significantly as the misalignment angle increases. Since this strong Reynolds-dependency is specific to

640 the small scale of wind tunnel models, the results of the method of Heck et al. (2023) are not shown here, because its poor match with the measurements would be misleading, as these effects would not be present at full scale. The present method is not affected by this issue, because it uses a lifting line approach and specifically includes the blade aerodynamic characteristics in the governing Eqs. (22) and (24b).

The loss factors are reported for derated operation (second test campaign test campaign 2) in Fig. 17. Here again the match

- between experimental data and model predictions predictions of the present model is very good, the latter being mostly within the uncertainty band of the measurements. Slightly larger deviations are observed for the min- $C_T$  case at  $P_d$ =50%. This can be explained by the fact that the machine operates at significantly low  $\lambda$  values, with consequent low rotational speeds. This results in particularly high angles of attack (Juangarcia et al., 2018) and very low chord-based Reynolds. Both have significant impacts on the airfoil performance, which are likely not properly captured by the analytical model. Overall, it appears that the
- 650 model is capable of capturing the reduction in the thrust coefficient as derating  $P_d$  increases, as well as the lack of symmetry with respect to the misalignment angle.

The effects of thrust and shear are visualized in Fig. 18 in terms of the average and the difference of the power loss factors at  $\gamma \pm 20^{\circ}$ , respectively noted  $\bar{\eta}_{P,\gamma=\pm20^{\circ}}$  and  $\Delta \eta_{P,\gamma=\pm20^{\circ}}$ . It appears that power losses tend to decrease with increasing thrust coefficients (panel a), whereas there is no significant dependency on shear (panel b). The power loss asymmetry grows with increasing shear (panel d). On the other hand, the asymmetry is roughly constant with respect to the thrust coefficient (panel c).

In the third test campaign, the wind turbine is operated above rated conditions. Figure 19 reports the 2-minute average tip speed ratio and pitch angles measured during the experiment, and plotted as functions of the misalignment angle γ. For the Low-TI case, the turbine operates in region III for all misalignment angles. Recalling that the tip speed ratio is defined as
\$\lambda = \OR/u\_{\omega\_{\omega\_{\omega\_{\omega\_{\omega}}\omega\_{\omega}}}\$, since both the ambient wind speed \$u\_{\omega}}\omega\_{\omega}}}} and rotor speed \$\Omega\$ are constant, when the turbine yaws away from the wind \$\lambda\$ (indicated by red circles in Fig. 19a) remains constant, while the blades are pitched back (red circles in Fig. 19b) in order to keep the power output equal to the rated value. The same happens for the Mod-TI case. However, when \$\gamma\_{\infty} < -25^\circ,\$}\$, the turbine exits region III and enters region II. Therefore, as blade pitch (purple squares in Fig. 19b) reaches the value for maximum power coefficient, \$\lambda\$ starts decreasing (purple squares in Fig. 19a).</li>

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Figure 20 shows the results for the power and thrust loss factors in this scenario. Once again the proposed model exhibits a very good match with the experiments, falling within the uncertainty bands in most cases.

These results confirm the ability of the method to correctly represent the effects of different control approaches, covering both regions II and III, including derating. This is crucially important because, as shown, control laws have a strong impact on the behavior of power and trust in misaligned conditions.

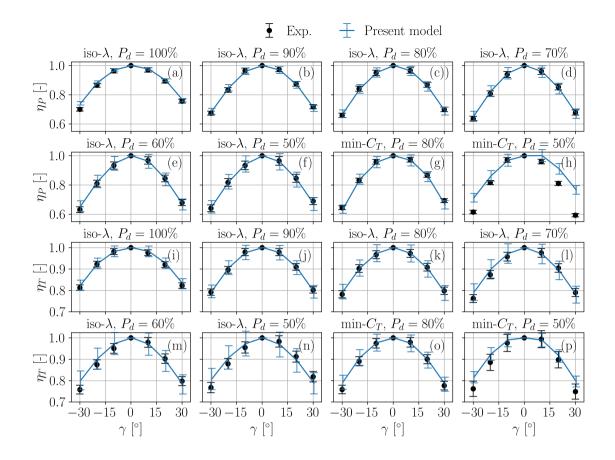


Figure 17. Experimental campaign 2. Power loss factor  $\eta_P$  (two top rows) (a-h) and thrust change loss factor  $\eta_T$  (two bottom rows) (i-p) vs. yaw misalignment  $\gamma$ , in derated operation in region II for the Mid-TI-Mod-TI inflow case. Model predictions: solid lines; experiments: markers. The bars-Whiskers indicate measurement uncertainties the 95% confidence intervals.

## 670 5 Optimal wake steering

The insight provided by the new model suggests two questions:

- What is the power-optimal way to yaw a single turbine out of the wind?
- And does the new model affect the way wake steering should be conducted?

We try to give some initial answers to these questions in the following two sections.

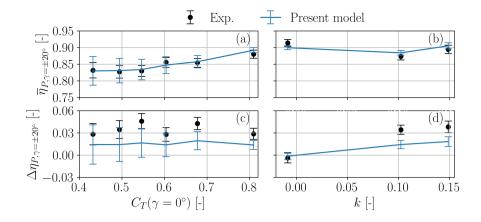


Figure 18. Power loss factor  $\bar{\eta}_{P,\gamma=\pm 20^{\circ}}$  (averaged between  $\gamma = \pm 20^{\circ}$ ) as a function of  $C_T(\gamma = 0^{\circ})$  (a) and of shear k (b). Difference  $\Delta \eta_{P,\gamma=\pm 20^{\circ}}$  between power loss factors  $\eta_P$  evaluated at misalignments  $\gamma = 20^{\circ}$  and at  $\gamma = -20^{\circ}$ , as a function of  $C_T(\gamma = 0^{\circ})$  (c) and of shear k (d). Proposed model: solid lines; experimental measurements: black circles. Whiskers indicate the 95% confidence intervals.

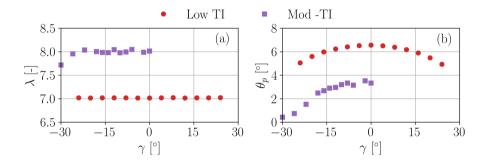


Figure 19. Experimental campaign 3. Blade pitch angle  $\theta_{P}$  (a) and tip speed ratio  $\lambda$  (b) in Low-TI and Mod-TI inflows.

#### 675 5.1 Optimal power capture of a single misaligned turbine

The new model was used to compute the optimal power of a wind turbine when it is misaligned with respect to the wind. The analysis was conducted for the same IEA 3.4 MW wind turbine used for the previous numerical validation of the model.

The optimal control strategy was computed by numerical optimization using an adaptive Nelder-Mead algorithm (Gao and Han, 2012), and results are shown in Fig. 21. The figure reports also the standard region II control approach, which consists in

680 holding the pitch angle fixed while the generator torque is varied proportionally to the square of the rotor speed, i.e.  $Q \sim \Omega^2$ . , as explained in Sect. 3.3. Assuming as a first approximation that  $P = \cos \gamma^{p_p}$ , and considering that  $Q = P/\Omega$ , it follows that  $\Omega \sim \cos \gamma^{p_p/3}$ . The figure reports the solution computed for a coefficient  $p_p = 1.88$ , following Fleming et al. (2015).

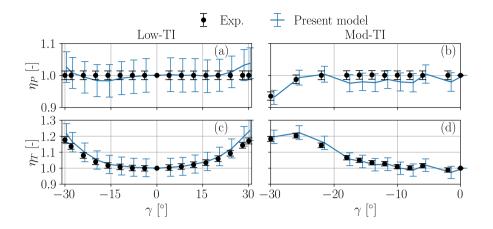


Figure 20. Experimental campaign 3. Power loss factor  $\eta_P$  (a-b) and thrust loss factor  $\eta_T$  (c-d) vs. yaw misalignment  $\gamma$ , in above-rated-speed operation for the Low-TI and Mod-TI inflow cases. Model predictions: solid lines; experimental measurements: black circles. Whiskers indicate the 95% confidence intervals.

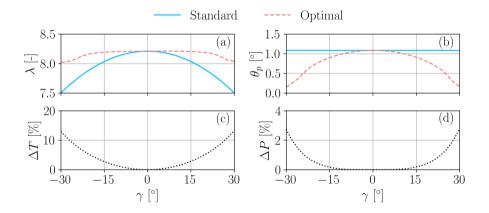


Figure 21. Comparison between standard and optimal control strategies for different yaw angles  $\gamma$ . Tip speed ratio  $\lambda$  (a); pitch angle  $\theta_p$  (b); percent thrust difference between the optimal and standard strategies (c); percent power difference between the optimal and standard strategies (d).

As the yaw misalignment increases, the tip speed ratio drops for the standard control strategy, driven by the reduced rotororthogonal component of the wind. Since the pitch angle remains fixed, the reduced  $\lambda$  leads also to a decreased thrust coeffi-

685 cient. Although this might be beneficial for reducing loading on the yawed turbine, the resulting drop in power is significant. On the other hand, the optimal strategy governs the turbine to keep a much more constant tip speed ratio and thrust coefficient, while the blade pitches back a little. This results into some power boost, which is small for moderate angles but reaches above 3% around  $\pm 30^{\circ}$ . These findings are in line with results presented by Cossu (2021a) and Heck et al. (2023). While the higher  $C_T$  implies that the turbine is loaded more than in the standard case, it has also an effect on the wake that will be felt

690 downstream, as explored in the next section.

# 5.2 Optimal power capture of two turbines

The previous section showed that a single turbine can extract more energy from the wind in misaligned conditions when its  $C_T$  is increased compared to a standard region II control approach. This so-called overinductive yaw control (Cossu, 2021a) increases the velocity deficit in the wake, but it also affects its recovery and enhances its deflection. It is therefore necessary to find the optimal tradeoff among these complex effects when considering wake steering wind farm control (Meyers et al., 2022)

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FLORIS v3 (NREL, 2023b), modified with the present model, was used to optimize the power capture of a cluster of two IEA 3.4 MW wind turbines placed at a distance of 5 diameters. The wake was modelled with the Gauss-Curl-Hybrid model (King et al., 2021). The inflow is characterized by an ambient wind speed  $u_{\infty,hub} = 9.7 \text{ ms}^{-1}$ , a shear of 0.12, and a turbulence intensity of 6%, and a. A 60° range of wind directions  $\Phi$  was considered, in order to realize different degrees of overlap between the wake and the downstream rotor. The optimal wind farm control strategy was computed by numerically maximizing the cluster power with the same adaptive Nelder-Mead algorithm used for the single-turbine case of the previous

section.

- Results are shown in Fig. 22. The plots report in green <u>dotted lines</u> the results obtained with greedy control (i.e., each turbine maximizes its own power capture), in blue <u>solid lines</u> the solution obtained with wake steering control based on the  $\cos^{p_p}$  law using  $p_p = 1.88$ , and in <del>orange red dashed lines</del> with wake steering control based on the present model. For the three control strategies, results were validated with LES-ALM simulations run for five different wind directions, namely  $\Phi = \{270 \pm 5.74, 270 \pm 2.5, 270\}^\circ$ , corresponding to rotor overlaps of 50%, 78.2%, and 100%, respectively. The LES-ALM results are indicated in the figure with markers, where the colors correspond to the control strategy.
- Figures 22a, 22b, and 22c respectively show the front turbine tip speed ratio  $\lambda$ , pitch angle  $\theta_p$ , and absolute misalignment angle  $|\gamma|$ , all plotted as functions of wind direction  $\Phi$ . The solution for the present model is characterized by a fairly constant tip speed ratio that, in conjunction with some pitch back of the blades at the highest misalignments, results also in a roughly constant thrust coefficient (not shown for brevity). This is in contrast with the solution based on the  $\cos^{p_p}$  law, where both tip speed ratio and thrust coefficient drop at the higher misalignments that correspond to the strongest wake overlap conditions. In
- 715 addition, the present model also results into slightly larger misalignment angles, as shown by Fig. 22c.

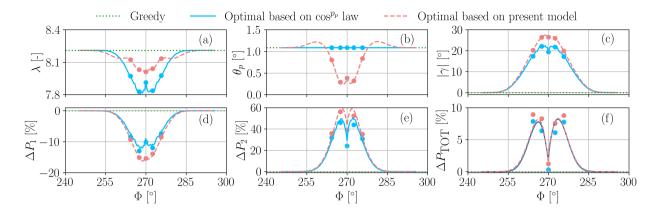


Figure 22. Control of a cluster of two turbines in wake interference conditions. Green: greedy policy; blue: optimal wake steering solution based on  $\cos^{p_p}$ ; orangered: optimal wake steering solution based on <u>new-the proposed</u> model. Lines: FLORIS engineering wake model; markers: LES-ALM CFD. Tip speed ratio  $\lambda$  (a); thrust coefficient  $C_T$  (b); absolute yaw misalignment  $|\gamma|$  (c); percent power changes with respect to the greedy policy for the upstream wake-steering turbine (d); percent power changes with respect to the greedy policy for the downstream turbine (e); overall percent power changes for the cluster of two turbines (f).

The bottom three plots show the effects of the various control strategies on power as function of wind direction  $\Phi$ . Figures 22d and 22e report the power changes with respect to the greedy strategy for the front and back turbines, respectively. It appears that the upstream machine, thanks to a higher tip speed ratio and different pitch control, looses less due to a larger misalignment, looses more power than in the cos<sup>*p*</sup> case. AdditionallyConversely, it also appears that the second machine gains more power with the strategy based on the new model, thanks to a higher the larger misalignment of the upstream turbine but also due to its larger thrust coefficient. Finally, Fig. 22f shows the overall gain at the cluster level. Results indicate a fairly consistent improvement, in excess of roughly 1%, for almost the entire wake-overlap range.

The LES-ALM results confirm the findings based on the FLORIS engineering wake model: less more power losses for the front turbine, and more gains for the downstream one, resulting in a positive net gain for the cluster.

# 725 6 Conclusions

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We have presented a new model to estimate the power performance of a misaligned wind turbine rotor. The model is a modified version of the classical blade element momentum theory, where the rotor is considered as a lifting wing of finite span operating at an angle of attack.

The new model reveals the following characteristics of the behavior of a misaligned rotor:

- Power does not depend on the misalignment angle according to the  $\cos^{p_p}$  law, a formula in widespread use in the literature.

- The true effective misalignment angle that drives wake behavior is a combination of both yaw and tilt. Therefore, a two-dimensional wake model should be described in the plane formed by the rotor axis and wind vectors, not on the horizontal plane as commonly assumed.
- Power depends on the true misalignment angle, but crucially also on the way the rotor is governed as it is pointed out of the wind, a fact that probably explains the widely different performance observed by various authors. This fact also means that power losses due to misalignment can be mitigated by using a suitable control strategy.

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- According to the model, the observed lack of symmetry between positive and negative misalignment angles is caused by the interaction with a sheared inflow. In these conditions, there is a complex interplay of various effects that may lead to various different outcomes in terms of which yaw sign yields more or less power. In general, one can expect a small asymmetry at high thrust coefficients, while a more pronounced asymmetry emerges for low thrust and high tip speed ratios, where a higher power is generated for positive yaw angles. However, in general the behavior of power (but of thrust too, which also exhibits an asymmetric behavior) depends not only on the rotor design characteristics but also on the way it is governed, through the values of the pitch setting and of the tip speed ratio. Additionally, in the field other effects may be present (e.g., due to an asymmetric behavior of the onboard wind vane), which may add to the phenomena described by the model.
  - A constant-over-the-rotor induction is sufficient to accurately describe the power and thrust behavior of a misaligned rotor in a sheared inflow. In fact, under classical small angle assumptions, the tilting of the inflow due to misalignment and shear has only a negligible effect on the quality of the results.
- The model was derived in a semi-analytical form, leading to a closed system of equations that can be directly integrated with engineering wake models, at an irrelevant computational cost. To improve its accuracy, we proposed a specific implementation that overcomes the intrinsic limits brought by the analytical solution of some model integrals. The proposed implementation corrects for the effects of misalignment a higher-fidelity power model obtained in aligned conditions, and calibrates the model parameters based on measurements.
- The model was validated in a broad range of cases, considering LES-ALM numerical simulations of a-various multi-MW machine-machines as well as experimental observations on a scaled wind tunnel model, in different inflows (from unsheared laminar to sheared highly turbulent conditions), in full-power and operating with controllers in the loop in regions II, III, as well as in derated conditions. In all cases, the model achieved a very satisfactory agreement with the numerical and experimental reference values. Power and thrust values. Additionally, we demonstrated how the model can be integrated with given control.
- 760 laws, achieving an excellent match in the calculation of the setpoints. The model was also compared with a similar model recently developed by Heck et al. (2023), limitedly to the control region II where it is applicable, consistently improving on its predictions and exhibiting a wider applicability to arbitrary control strategies.

Using the proposed model, we maximized the power capture of a wind turbine for a range of misalignment angles, obtaining the optimal power strategy in terms of pitch setting and tip speed ratio. Results indicate that the maximum power extraction is

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obtained by keeping an almost constant tip speed ratio and by slightly reducing the blade pitch as the turbine yaws out of the wind. This also implies a roughly constant thrust coefficient, which will increase the loading on the yawed turbine, but will also have an effect on its wake.

Next, we applied the new model to the maximization of the power by wake steering for a cluster of two turbines. The resulting control strategy was compared to the one obtained by the classical  $\cos^{p_p}$  power loss model, and validated by means

of LES-ALM simulations for a few selected cases. Results indicate that the proposed model results in smaller slightly greater power losses for the wake-steering turbine, and which are more than compensated by greater power gains for the wake-affected one, achieving a small but consistent gain in power at the cluster level for the full range of possible wake overlaps.

Future work should investigate the effects of the new model and its resulting control strategy in more complex conditions. Of particular interest is the analysis of the effects on loads, which might increase because of the eliminated drop in thrust 775 coefficient as the turbine is yawed out of the wind.

## **Appendix A: Transformation matrices**

## A1 Transformation from ground to nacelle frame of reference

The nacelle-fixed frame of reference is obtained from the ground-fixed frame by a first rotation  $\delta$  about the horizontal axis  $\boldsymbol{y}_g$ , followed by a rotation  $\gamma$  about the vertical axis  $\boldsymbol{z}_g$ . The components of a generic vector  $\boldsymbol{v}$  are noted  $\underline{v}_g$  when measured in the ground frame  $\mathcal{F}_g$ , and  $\underline{v}_n$  when measured in the nacelle frame  $\mathcal{F}_n$ . Combining the two successive rotations, one obtains the transformation of components from one frame to the other as

$$\underline{v}_n = \begin{bmatrix} \cos\delta & 0 & -\sin\delta \\ 0 & 1 & 0 \\ \sin\delta & 0 & \cos\delta \end{bmatrix} \begin{bmatrix} \cos\gamma & -\sin\gamma & 0 \\ \sin\gamma & \cos\gamma & 0 \\ 0 & 0 & 1 \end{bmatrix} \underline{v}_g = \begin{bmatrix} \cos\delta\cos\gamma & -\cos\delta\sin\gamma & -\sin\delta \\ \sin\gamma & \cos\gamma & 0 \\ \sin\delta\cos\gamma & -\sin\delta\sin\gamma & \cos\delta \end{bmatrix} \underline{v}_g.$$
(A1)

The inverse transformation is simply given by the matrix transpose.

Using Eq. (A1), the components of the ambient velocity vector  $\boldsymbol{u}_{\infty}$  in the nacelle-attached frame are readily found to 785 be  $\underline{u}_{\infty n} = u_{\infty} \{\cos \delta \cos \gamma, \sin \delta, \sin \delta \cos \gamma\}^T$ , where the scalar wind speed is  $u_{\infty} = |\boldsymbol{u}_{\infty}|$ , while  $\underline{x}_{n_n} = \{1, 0, 0\}^T$ . Hence, it follows that

$$\frac{u_{\infty}}{u_{\infty}} \cdot \boldsymbol{x}_n = \cos \mu = \cos \delta \cos \gamma. \tag{A2}$$

#### A2 Transformation from wake-deflection to ground frame of reference

The nacelle and wake-deflection frames share the same unit vector  $\boldsymbol{x}_n = \boldsymbol{x}_d$ , which corresponds to the rotor axis of rotation. 790 The  $\boldsymbol{z}_d$  unit vector is orthogonal to the plane composed by  $\boldsymbol{u}_{\infty}$  and  $\boldsymbol{x}_d$ , and therefore it can be written as  $\boldsymbol{z}_d = z\boldsymbol{x}_d \times \boldsymbol{u}_{\infty}$ , where z is a normalization scalar such that  $\boldsymbol{z} \cdot \boldsymbol{z} = 1$ . Performing the cross product and the normalization, one finds  $\underline{z}_{d_n} = z\{0, -\sin\delta\cos\gamma, \sin\gamma\}^T$  and  $z = 1/\sqrt{\cos^2\gamma\sin^2\delta + \sin^2\gamma} = 1/\sin\mu$ . A right-handed triad is completed by setting  $\boldsymbol{y}_d =$   $-x_d \times z_d$ , which yields  $\underline{y}_{d_n} = z\{0, \sin\gamma, \sin\delta\cos\gamma\}^T$ . The transformation matrix between the wake-deflection and nacelle-fixed components is therefore readily obtained as

$$795 \quad \underline{v}_n = \begin{bmatrix} 1 & 0 & 0 \\ 0 & z\sin\gamma & -z\sin\delta\cos\gamma \\ 0 & z\sin\delta\cos\gamma & z\sin\gamma \end{bmatrix} \underline{v}_d.$$
(A3)

Finally, the transformation between wake-deflection and ground-fixed components follows by using Eq. (A3) and Eq. (A1), which yields

$$\underline{v}_{g} = \begin{bmatrix}
\cos\delta\cos\gamma & \sin\gamma & \sin\delta\cos\gamma \\
-\cos\delta\sin\gamma & \cos\gamma & -\sin\delta\sin\gamma \\
-\sin\delta & 0 & \cos\delta
\end{bmatrix}
\begin{bmatrix}
1 & 0 & 0 \\
0 & z\sin\gamma & -z\cos\gamma\sin\delta \\
0 & z\cos\gamma\sin\delta & z\sin\gamma
\end{bmatrix}
\underline{v}_{d},$$
(A4a)
$$= \begin{bmatrix}
\cos\delta\cos\gamma & 1/z & 0 \\
-\cos\delta\sin\gamma & z\cos^{2}\delta\sin\gamma\cos\gamma & -z\sin\delta \\
-\sin\delta & z\sin\delta\cos\delta\cos\gamma & z\cos\delta\sin\gamma
\end{bmatrix}
\underline{v}_{d}.$$
(A4b)

800 The inverse transformation is simply given by the matrix transpose.

Using Eq. (A4b), the longitudinal (given by Eq. 11) and lateral (sidewash, given by Eq. 13) flow velocity components at the streamtube outlet can be transformed into the corresponding longitudinal and lateral components in the ground frame:

$$\frac{v_{o,g}}{u_{\infty}} = \frac{C_T}{4} \cos \delta \sin \gamma, \tag{A5a}$$

$$\frac{w_{o,g}}{u_{\infty}} = \frac{C_T}{4} \sin \delta. \tag{A5b}$$

# 805 Appendix B: Effect of a non-uniform axial induction on the rotor disk

As mentioned in Sect. 2.4, misalignment and shear cause a non-uniform distribution of the induction over the rotor disk. Following the classical approach used for helicopter rotors in forward flight (Johnson, 1995), the simplest model of non-uniform axial induction is the one expressed by Eq. (6), based on the 1P harmonics  $\kappa_{1s}$  and  $\kappa_{1c}$ . For the present application, however, it appears that the inclusion of these terms is not necessary. To show this, we consider the  $\kappa_{1s}$  term, which is triggered by

810 the misalignment  $\mu$  and results in the largest induction in the most downstream portion of the rotor disk. Figure B1 presents the loss factors  $\eta_T$  and  $\eta_P$  predicted by the model with (red dotted line) and without (solid blue line) the 1P sine term  $\kappa_{1s}$ . Differences appear to be negligible, especially for  $\eta_T$ . Neglecting  $\kappa_{1s}$  leads to a slight decrease in  $\eta_P$  as the misalignment increases, reaching a maximum difference of 0.21% at  $\gamma = -30^\circ$ .

# Appendix C: Experimental data set

815 Figures C1 and C2 report the blade pitch  $\theta_p$  and tip speed ratio  $\lambda$  measured in the wind tunnel experimental campaigns 1 and 2, respectively. The same quantities for experimental campaign 3 are reported in Fig. 19.

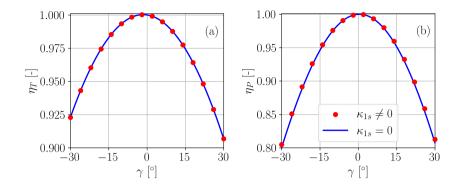


Figure B1. Thrust loss factor  $\eta_T$  (a), and power loss factor  $\eta_P$  (b) computed with the proposed model, with and without the 1P sine harmonic  $\kappa_{1,s}$  in Eq. (6). Results are computed for the IEA 3.4 MW reference wind turbine, subjected to a vertical shear k = 0.2, operating at a tip speed ratio  $\lambda = 8.5$ , and with a blade pitch angle  $\theta = 1^{\circ}$ .

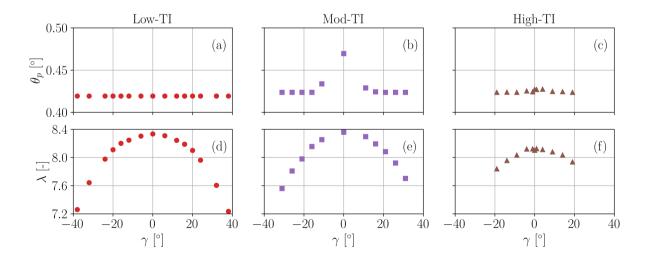


Figure C1. Average blade Experimental campaign 1. Blade pitch angle  $\theta_p$  (top row) (a) and tip speed ratio  $\lambda$  (bottom row), for the full-power case (b) in the three inflow cases Low-TI, Mid-TIMod-TI, and High-TI inflows.

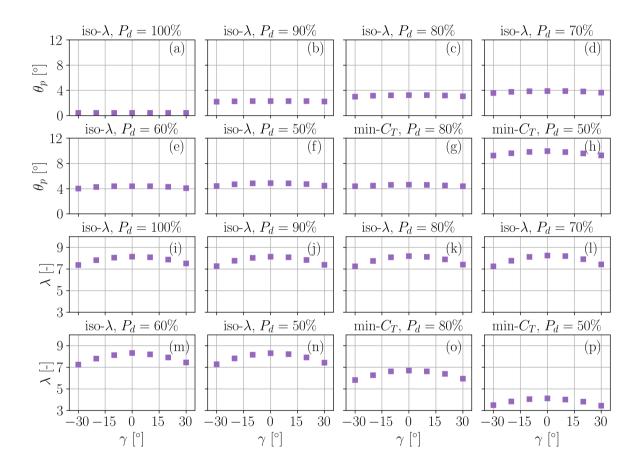


Figure C2. Average blade Experimental campaign 2. Blade pitch angle  $\theta_p$  (two top rows) (a) and tip speed ratio  $\lambda$  (two bottom rows), for the derating cases (b) in the Mid-TI Mod-TI inflow.

# **Appendix D: Nomenclature**

	A	AreaRotor disk area
	a	Axial induction
820	В	Number of blades
	$C_D$	Drag coefficient
	$C_L$	Lift coefficient
	$C_{L,lpha}$	Lift slope
	$C_P$	Power coefficient
825	$C_T$	Thrust coefficient
	$\widetilde{D}^{L}$	<sup>~</sup> Modified thrust coefficient of Heck et al. (2023) Rotor diameter
	$F_n$	Normal force
	$F_t$	Tangential force
830	$\frac{K}{k}$	<sup>C</sup> Coefficient relating aerodynamic torque and squared rotor speed in control region II Linear vertical wind shear coefficient
	$\dot{m}$	Mass flux
	p	Pressure
	Р	Power
835	$P_d$	Power demand (derating)
	Q	Rotor torque
	R	Rotor radius
	r	Spanwise coordinate
	T	Thrust force
840	$u_{\infty}$	Free-stream wind speed
	$u_{\infty,\mathrm{hub}}$	Free-stream wind speed at hub height
	u	Longitudinal velocity component
	$u_n$	Rotor-orthogonal velocity component
	$u_t$	Rotor-tangential velocity component
845	v	Lateral velocity component
	x	Cartesian coordinate
	y	Cartesian coordinate
	z	Cartesian coordinate
850	α	Angle of attack

	eta	Blade twist angle
	$\delta$	Rotor tilt angle
	$\eta_P$	Power loss factor
	$\eta_T$	Thrust change loss factor
855	$\gamma$	Rotor yaw angle
	$\lambda$	Tip speed ratio
	$\mu$	Rotor total (true) misalignment angle
	Ω	Rotor angular speed
	arphi	Inflow angle
860	$\Phi$	Wind direction
	$\psi$	Rotor azimuth angle
	ρ	Air density
	heta	Local pitch angle
	$ heta_p$	Blade pitch anglerotation at the pitch bearing
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	$\underline{v}_f$	Components of vector $\boldsymbol{v}$ in frame $f$
	$(\cdot)_d$ $(\cdot)_g$	<sup>~</sup> Quantity evaluated in the wake-deflection intrinsic frame of reference Quantity evaluated in the ground frame of reference
070	$(\cdot)_g$	
870	$(\cdot)_r(\cdot)_n$	Quantity evaluated in the nacelle frame of reference
	$(\cdot)_{d}(\cdot)_{1c}$	Quantity evaluated in the wake-deflection intrinsic frame of referenceOnce-per-revolution cosine
	$(\cdot)_{1s}$	harmonic Once-per-revolution sine harmonic
	$(\cdot)_{1c}$ OP	Once-per-revolution cosine harmonic Zeroth (constant) harmonic
875	1P	One per revolution harmonic
	ALM	Actuator Line Method
	BEM	Blade Element Momentum
	CFD	Computational Fluid Dynamics
	СТА	Constant Temperature Anemometry
880	FLORIS	FLOw Redirection and Induction in Steady State
	LES	Large Eddy Simulation
	RPM	Revolutions Per Minute
	SOWFA	Simulator fOr Wind Farm Applications

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Code and data availability. An implementation of the model described in this article in the FLORIS framework is available on Github at https://github.com/sTamaroTum/Beyond\_the\_cosine\_law/ (Tamaro et al., 2024). The repository also contains all the data and the Jupyter notebooks used to generate the figures. The code and the scripts to reproduce the figures can be run on Binder at the link https://tinyurl.com/btcl-figs. The notebook of Fig. 5 can be used to interactively plot the thrust and power coefficients for other user-defined values of the model param-

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eters, while the one of Fig. 6 can be similarly used to visualize different control trajectories. The notebooks of Figs. 21 and 22 contain also the code used for computing the optimal control policies. The complete data sets from the LES simulations and wind tunnel experiments are available upon request.

Author contributions. CLB developed the formulation of the misalignment model, and supervised the overall research. ST implemented the model, performed the LES simulations and the corresponding model validation, and conducted the wake-steering analyses with FLORIS. FC 895 performed the validation with respect to the experimental measurements. All authors contributed to the interpretation of the results. CLB and ST wrote the manuscript, with contributions by FC in the experimental section. All authors provided important input to this research work through discussions and feedback and by improving the manuscript.

Competing interests. The authors declare that they have no conflict of interest, except for CLB who is the Editor in Chief of the Wind Energy Science journal.

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