Response to Reviewer #1 (Alexander Krimmer)

General Remarks

"The manuscript presents a combined approach to determine the non-proportionality factor for fatigue loads in bond lines of wind turbine rotor blades. It thereby enables to account for both, deviatoric and hydro-static stress contributions. The method is compared to available approaches that are the basis for this combined method and shows very good results. Afterwards the method is applied to three different rotor blade structures available in the literature and the non-proportionality factors are compared and assessed.

To my understanding, there are two fundamental outcomes, that can be highlighted more prominently."

Statement #1:

"Ignoring non-proportionality seems to be making errors on the conservative side."

Response: At first glance, one might actually think that neglecting non-proportionalities results in conservative interpretations. However, this is not the case.

The standard method to perform a fatigue analysis for non-proportional stress time series is the critical plane approach. At many potentially critical planes, a fatigue damage calculation is performed based on local, scalar equivalent stresses, constant life diagrams and linear damage accumulation. Only the stresses acting on the analysed plane are then included in the local equivalent stress criteria. The fatigue damage on the analysed planes is then compared with each other and the plane with the highest fatigue damage is identified as the critical plane. A macroscopic crack will occur on this plane after the fatigue life.

In contrast, the approach of global equivalent stresses, in which the complete three-dimensional stress tensor is included, has become established for proportional stress time histories. Here, the time series of the stress tensor are converted into time series of global, scalar equivalent stresses and then into fatigue damage with the aid of a constant life diagram and linear damage accumulation. This means that damage is accumulated at all effective planes, which is not harmful as long as the effective planes do not change over time. However, this is only the case for proportional stress time series.

It may be obvious to believe that the use of global equivalent stresses for non-proportional time series is conservative, as damage is accumulated on all effective planes. The result should therefore be significantly greater damage than with the critical plane approach, where the damage is only accumulated on the actual critical plane.

However, it has been experimentally proven for an epoxy resin-based rotor blade adhesive at coupon level that neutral material behaviour occurs. This means that the fatigue life does not depend on the degree of non-proportionality. This behaviour can be reproduced with the critical-plane approach, but not with the global approach. On the contrary,
it could be shown that the fatigue damage in the case of the global equivalent stress approach is not higher for increasing non-proportionality, but significantly lower. In this respect, the global approach not only incorrectly predicted the basic fatigue behaviour (i.e., it did not capture the neutral behaviour), but also significantly overestimated the fatigue life. The prediction was therefore substantially non-conservative.

The results mentioned above were published in the following paper, and the reader is asked to refer to it for more details:


This paper does not deal with the fatigue damage calculation itself. It addresses the qualitative classification of stress time series in a trailing edge bondline with respect to non-proportionality and the quantification of these non-proportionalities. The influence on the calculation of fatigue life is currently the subject of research and will be published in due course. We therefore ask for your patience in this regard. However, we included an additional paragraph in the introduction to address this point, which seems to important for the reviewer. The paragraph reads:

"It may be obvious to believe that the use of global equivalent stresses for non-proportional time series is conservative, as damage is accumulated on all effective planes. The result should therefore be significantly greater damage than with the critical plane approach, where the damage is only accumulated on the actual critical plane. However, it has been experimentally proven for an epoxy resin-based rotor blade adhesive at coupon level that neutral material behaviour occurs (Kuhn et al., 2023). This means that the fatigue life does not depend on the degree of non-proportionality (Sonsino, 2020). This behaviour can be reproduced with the critical-plane approach, but not with the global approach. On the contrary, it could be shown that the fatigue damage in the case of the global equivalent stress approach is not higher for increasing non-proportionality, but significantly lower (Kuhn et al., 2023). In this respect, the global approach not only incorrectly predicted the basic fatigue behaviour (i.e., it did not capture the neutral behaviour), but also significantly overestimated the fatigue life. The prediction was therefore substantially non-conservative."

Statement #2:

"Accounting for the S/N curve exponents of the applied adhesives, the relation between the dominant (z-axis) stresses and the other stresses makes the z-axis stresses by far the dominant design relevant contributions. Therefore it is expected that accounting for the non-proportionality within these three designs may have little to no influence on the final fatigue damage."

Response: This is correct. We agree that the longitudinal normal stress is the dominant stress component. This matches observations in the field and in full-scale blade tests, where tunneling cracks, i.e., cracks in the cross-sectional direction, frequently occur. These cracks cannot be prevented by increasing the width of the adhesive joint, which usually is the primary design variable for adhesive joints. The trailing edge girder needs to be stiffened instead in order to reduce the longitudinal strain and with it the longitudinal stress. To the best knowledge of the authors, this is not yet common practise throughout the industry, and is important to keep in mind. As this is an important finding which
is common for all three investigated blades, we have added the following phrases to the conclusions:

"A common finding for all three blades was that the longitudinal normal stress was the predominant stress component. The major goal in blade design with respect to a trailing edge adhesive joint should thus be to avoid excessive longitudinal strain and stress. This cannot be realized by increasing the bondline width, which is the usual design parameter for adhesive joints, but by increasing the stiffness of the trailing edge girder to reduce the longitudinal strain. However, if the longitudinal stress is reduced in this way, the other stress components not affected by the stiffening of the trailing edge girder, will become more significant and the degree of non-proportionality will increase."

The degree of non-proportionality is indeed limited for the blade designs investigated in the paper, but it is finite. There are some spots in the adhesive joint where the non-proportionality is significant, especially after weighting with the wind speed. The evaluation of the stress time series is ongoing and subject of research. We would like to emphasize that it is too early to draw conclusions on the effect of the non-proportionalities on the fatigue life estimate and the choice of the fatigue analysis methodology. This is already included in the last sentence of the conclusions. As argued above, early conclusions can be wrong. Without calculation, it is impossible to say if the non-proportionality is negligible or not, although it may seem obvious.

Further Remarks

"Anyhow, this is a significant learning for future rotor blade designs!

Detailed comments can be found in the attached *.pdf document."

Comment #1:

"I know, it is simple, but in my view it makes sense to state Beltrami here since it is very simple and accounts for compressibility (much more capable then von Mises in my view)."

Response: We included the Beltrami criteria via the following reference:


Comment #2:

"I wonder what is the mechanism that causes these. This may affect the influence on the equivalent stress."

Response: You are right that the damage mechanism can influence the equivalent stress. Unfortunately, at the present time, the presence of tension-compression asymmetry (TCA) in an epoxy-based rotor blade adhesive is a matter of macroscopic observation and cannot be clearly explained yet. In the opinion of the authors, speculation is of little use. However, to substantialize our statement that TCA is present in epoxy-based adhesives, we
included a reference to measurements that were carried out recently in our group, which is the following:


Comment #3:

"But in fact the extent of shear stresses in trailing edge bond lines is surprisingly low."

Response: This is true and is also reported in this paper. It was stated explicitly in the sentence before that "the normal stress in spanwise direction is dominant in trailing edge adhesive joints", which is substantiated by the stress time series presented in Figure 6 and the respective explanations. Moreover, the ratio between longitudinal normal stress and shear stress depends on the blade design, i.e., if the trailing edge girder is stiff enough, the longitudinal normal stress is decreased significantly, while the shear stress remains the same. We thus did not include an additional comment on the small magnitude of shear stresses, which would have been repetitive and example-specific, and kindly ask for your agreement.

Comment #4:

"To my understanding, this indicates that a structural change of the material is involved."

Response: If you mean some sort of microcrack formation by "structural change", and microcracks being responsible for stiffness and strength degradation in fatigue, then we agree. A microcrack evolving for a particular stress state is relevant for fatigue degradation due to a principle stress perpendicular to the microcrack. It is not necessarily relevant for a completely different stress state at another instance of time in a non-proportional stress history. E.g., if a microcrack forms transverse to the longitudinal direction due to longitudinal stress, it will probably not grow if only normal stresses in the cross-sectional direction occur in the following stress time series, and it will potentially not contribute substantially to further fatigue-related degradation.

Although these may be reasonable thoughts, it is again speculation, as it was not yet proven by experiments and microscopic observations. As the scope of the manuscript is the analysis of the degree of non-proportionality and not the detailed explanation of damage mechanisms due to that non-proportionality, we prefer not to go into detail in this context. This may be subject of future work. We hope this finds the agreement of the reviewer.

Comment #5:

"Why do the shear stresses get a factor of sqrt(2)? This should just be 2, correct?"
Response: In the Voigt notation of the stress tensor, no coefficient is present in the shear stress components. A factor of 2 would occur in the Voigt notation of the strain tensor, if $\gamma$ was used instead of $\epsilon$. Bishop actually used the Mandel notation, where a coefficient of $\sqrt{2}$ is present. We corrected this error throughout the manuscript.

Comment #6:

"I feel that this is a very bad term for describing what actually happens. Is there a chance to chose a term that is not already used in a very different meaning in structural mechanics?"

Response: It is a matter of taste, but it may be true that the term results in misunderstandings or confusion of the reader. We have thus introduced the terms stress body and rectangular moment of inertia, which are both described in a clarifying paragraph, which reads:

"The state of stress at a specific point in time can be mapped as a point in the six-dimensional stress space. The collection of these points for a complete stress time series results in a six-dimensional body, which is called stress body in the following. The rectangular moment of inertia (RMOI) of the stress body [...]"

The term rectangular body of inertia, or its abbreviation RMOI, are now used throughout the manuscript.

Comment #7:

"This is another hind [sic], that damaging is driving the behavior since non-linearity in composites usually is caused by damaging."

Response: Yes, we agree. However, this paper is about the analysis of the degree of non-proportionality in stress time series, not on fatigue damage analysis, which would certainly be a logical next step. However, discussions on fatigue damage analysis, damage processes or the analysis of material non-linearities are beyond the scope of the manuscript. A discussion on that may be included in a future publication, where we will try to link the degree of non-proportionality with a proper fatigue damage analysis.

Comment #8:

"I assume, that these factors $\sqrt{2}$ and $\sqrt{3}$ are supposed to account for the the [sic] two assumptions of full compressibility and on the other hand incompressibility. But this does not necessarily go in line with equation (8) (see my comment)."

Response: The factor $\sqrt{2}$ comes from the Mandel notation of the stress tensor. The factor $\sqrt{3}$ originates from independency of hydrostatic pressure. There was indeed an inconsistency in notation, both in this paragraph and equation (8), see also the answer to comment #9 below. The errors have been corrected.
Comment #9:

"Is this just due to an inconsistency in writing? Because $\sqrt{3}$\(\sigma_{13}\) is not equal $\sqrt{3}$\(\tau_{13}\)!

Response: Yes, it was, see answer to your comment #8. The inconsistency has been corrected, both in the text and equation (8).

Comment #10:

"This implies, that hydro-static stresses do not contribute to damaging, correct?"

Response: At this stage, yes, it does. The dependency of hydrostatic stress states is reintroduced later.

Comment #11:

"Maybe rather evolution?"

Response: Yes, evolution has been adapted.

Comment #12:

"of"

Response: The typo has been corrected.

Comment #13:

"As stated above, this underlines that shear stresses (contradicting the design principle of bond lines) do not play a significant role in trailing edge bond line fatigue. Hence, the non-proportionality, that most probably is in favor of the bond line fatigue life (as I deduce from your descriptions), is not driving the design."

Response: We agree that the shear stress is not driving the bondline design in the blades investigated in the manuscript, as the longitudinal normal stress is dominating the time series. However, the stress time series strongly depend on the particular blade design, and for other blades, this may be different.

In any case, it can not necessarily be concluded that non-proportionality is always favorable. It is rather one possible indicator for the choice of the fatigue analysis methodology. As argued above, a global equivalent stress approach works well for proportional loading, but fails for non-proportional loading, as was elaborated in the following paper, which has also been referenced in the manuscript (see also the answer to your general comment #1):

Global equivalent stress approaches give principally wrong fatigue life estimates in case non-proportional stress time series are involved. A comparison of the fatigue life calculated with both methods is not valid in this case. Calculation of the fatigue life with global equivalent stress approaches in presence of non-proportional stress time series may be on the conservative side, or it may not, as was the case in the aforementioned paper. This is not necessarily known a priori. The only thing that is known at the time writing this sentence is that it is wrong.

Again: Fatigue analysis is not the subject of this paper, so we did not include additional comments in this regard here.

Comment #14:

"Question is, what criterion has this bond line been designed to? Because this significantly influences the different means and amplitudes."

Response: Exactly. The influence of each stress component on the non-proportionality factor depends on the blade design, which in turn depends on the design criteria. A trailing edge bondline that is properly designed for longitudinal strain will very likely experience a higher degree of non-proportionality than a blade that is designed only for shear stress, because the longitudinal stress will be lower, increasing the relative influence of shear. And vice versa of course.

To evaluate that was not the scope of this manuscript, but to sensitise that there always is non-proportionality in a rotor blade bondline, that there are methods to evaluate the degree of non-proportionality, and that such methods may serve to select a suitable fatigue analysis framework, i.e., the critical plane or the global equivalent stress approach.

In both approaches, different equivalent stress criteria can be further selected, which is complicating things. But as said: This is beyond the scope of the paper.

In fact, the criteria used in the blade design procedures are not known to the authors, as we did not carry out the design by our own. Hence, we can unfortunately not give information in the paper in this regard.

Comment #15:

"Agree."

Response: Thank you, this is appreciated.

Comment #16:

"... and would still be in favor of the integrety of the bond line if I get all this right."
Response: Not necessarily. See explanations related to your general comment #1 and the additional comment #13. Further research is required in order to draw a general conclusion in this regard – if this is possible at all.

Comment #17:

"Again, isn’t the error on the conservative side?.."

Response: Again: Not necessarily. A general conclusion is not yet possible. There are rather implications in the paper of Kuhn et al. 2023 that it may be strongly non-conservative in presence of substantial non-proportionality. This requires further investigations.