Wind turbine rotors in surge motion: New insights into unsteady aerodynamics of FOWT from experiments and simulations

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Abstract. An accurate prediction of the unsteady loads acting on floating offshore wind turbines (FOWT) under consideration of wave excitation is crucial for a resource-efficient turbine design. Despite a considerable number of simulation studies in this area, it is still not fully understood, which unsteady aerodynamic phenomena have a notable influence on the loads acting on a wind turbine rotor in motion. In the present study, investigations are carried out to evaluate of the most relevant unsteady aerodynamic phenomena for a wind turbine rotor in surge motion. As a result, inflow conditions are determined for which a significant influence of these phenomena on the rotor loads can be expected.

The experimental and numerical investigations are conducted on a two-bladed wind turbine rotor subjected to a tower top surge motion. A specialised wind tunnel test rig has been developed to measure the aerodynamic torque response of the rotor subjected to surge motions with moderate frequencies. The torque measurements are compared to two free vortex wake methods, namely a panel and a lifting-line method. Unsteady contributions that cannot be captured using quasi-steady modelling have not been detected in either the measurements or the simulations in the covered region of motion frequencies. The surge motion frequencies were limited to a moderate range due to vibrations occurring in the experiments. Therefore, a numerical study with an extended the range of motion frequencies using the panel and the lifting-line method was performed. The results reveal significant unsteady contributions of the surge motions to the torque and thrust response that cannot be modelled in a quasi-steady way. Furthermore, the results show the presence of the returning wake effect, which is known from helicopter aerodynamics. Additional simulations of the UNAFLOW scale model and the IEA 15 MW rotor demonstrate that the occurrence of the returning wake effect is independent from the turbine, but determined by the ratio of rotor speed and surge motion. In case of the IEA 15 MW, a notable impact of the returning wake effect was found at surge motion frequencies in the range of typical wave periods. Finally, a comparison with OpenFAST simulations reveals notable differences in the modelling of the unsteady aerodynamic behaviour in comparison to a FVW method.

1 Introduction

The rapid development of floating offshore wind turbines has been enabled by elaborate simulation techniques that are improving continuously with the needs of new wind turbine designs. In case of new, large scale FOWT, the reliability of these simulation methods becomes more and more crucial for the evaluation of their feasibility and safety. In order to assess whether

25 a numerical method is sufficiently reliable for such FOWT concepts, the underlying physical phenomena must be understood and characterised. Due to increasing rotor diameters, the relevance of various aerodynamic phenomena to wind turbine rotor loads must be continuously re-evaluated. Therefore, many current research activities in the field of floating wind turbine aerodynamics focus on the occurrence and effects of unsteady aerodynamic phenomena caused by the motions of the tower top of large wind turbines. Although a considerable number of studies have been carried out on this subject, no consensus
30 has been reached in the research community on which unsteady aerodynamic phenomena are relevant and how much they can influence the loads acting on the wind turbine. Accordingly, the present work is intended to contribute to the understanding and characterisation of such unsteady phenomena and their influence on the loads of FOWT rotors using a novel experimental approach and comprehensive numerical investigations.

In the first step, an experimental study on a model rotor subjected to a surge motion is presented and load measurements
35 are compared to simulations using two different free vortex wake (FVW) methods. A special test rig was developed for the experimental investigation to minimise vibrations and an elaborate data acquisition procedure was applied to gain reliable and precise measurement results at high surge motion frequencies.

In the second step, the results of further numerical simulations exceeding the physical limits of the test rig are presented. A specialised set of load cases is used to clearly differentiate between quasi-steady and unsteady aerodynamic phenomena
40 impacting the simulation results. Apart from the well-known unsteady airfoil effect and the dynamic inflow effect, the simulations show a significant impact of the returning wake effect on the loads of the surging model rotor. This effect is known from helicopter aerodynamics, but has received very little attention in the context of FOWT to date. In order to investigate the relevance of the returning wake effect for different turbines, the analysis was extended to the rotor used in the UNAFLOW (Fontanella et al., 2021) measurement campaign and the large scale IEA 15 MW wind turbine (Gaertner et al., 2020).

45 Finally, a comparison of the free vortex wake simulation results of the IEA 15 MW turbine and those obtained by Aero-Dyn/OpenFAST (Jonkman, 2022) provides insight into the capabilities and limitations of a state-of-the-art blade element momentum theory (BEMT) method in the context of a surging wind turbine rotor and the returning wake effect.

The present work is structured as follows. Sections 2 and 3 briefly summarise the theory and current state of research regarding moving and especially surging wind turbine rotors. The panel method *pan*MARE and its lifting line module as well
50 as the simulation setups are described in section 4. A summary of the experimental setup, the test rig and the data acquisition followed by a comparison of experimental and numerical results is given in section 5, while the simulation studies on the three rotors are presented in sections 5.5 and 6. In the subsequent section, the characterisation and scaling of the observed unsteady phenomena and their impact on the rotor loads using dimensionless numbers are discussed. Finally, the main results of the experimental and numerical investigations are summarised in section 8.

55 ## 2 Theoretical background

2.1 Quasi-steady and transient parts of a rotor's load response to a surge motion

The load response of a wind turbine rotor to a tower top surge motion is generally unsteady as the surge motion velocity varies the inflow conditions over time. However, in case of a very slow motion, this unsteady response can be described in a *quasi-steady* way. Considering a very fast motion, it is likely that the behaviour of the rotor loads will be affected by additional aerodynamic phenomena. Such behaviour is referred to as *transient* in this work, because the frequently used term *unsteady* may lead to confusion as it also includes quasi-steady behaviour.

As a consequence, the aerodynamic power of a wind turbine in motion can be divided into a quasi-steady and a transient part. For a harmonic surge motion with a velocity amplitude V_m at constant rotational speed and constant wind speed, these parts can be described as follows:

$$P(t) = \underbrace{\frac{\rho A C_p\big(\lambda(t)\big)}{2}(v_0 + V_m sin(2\pi f t))^3}_{\text{quasi-steady}} + \underbrace{P_{trans}(t)}_{\text{transient}}, \tag{1}$$

where P, C_p, λ and f describe the aerodynamic power, power coefficient, tip speed ratio and motion frequency of the wind turbine. ρ and v_0 denote air density and wind speed, respectively. It has to be noted, that λ can be determined by the time dependent motion velocity in this case. All potential transient influences on the unsteady power are summarised in P_{trans}.

Introducing the dimensionless number b_{vel} as the ratio of *motion to inflow velocity*, the power amplitude can be expressed as:

$$b_{vel} = \frac{V_m}{v_0} \tag{2}$$

$$P_{amp} = \underbrace{\frac{\rho A v_0^3}{2}\frac{1}{2}\Big(C_p\big(\lambda_{min}\big)(1 + b_{vel})^3 - C_p\big(\lambda_{max}\big)(1 - b_{vel})^3\Big)}_{\text{quasi-steady}} + \underbrace{\Delta P_{trans}}_{\text{transient}} \tag{3}$$

A detailed derivation of equation (3) and the following equations is given in appendix A. The same derivation is valid for the rotor torque at constant rotational speed. Similarly, the thrust T_{amp} amplitude can be expressed as:

$$T_{amp} = \underbrace{\frac{\rho A v_0^2}{2}\frac{1}{2}\Big(C_t\big(\lambda_{min}\big)(1 + b_{vel})^2 - C_t\big(\lambda_{max}\big)(1 - b_{vel})^2\Big)}_{\text{quasi-steady}} + \underbrace{\Delta T_{trans}}_{\text{transient}}, \tag{4}$$

where C_t describes the thrust coefficient. It has to be noted that λ_{min} and λ_{max} are also determined by b_{vel}. This is due to the assumption of constant rotational speed and wind speed. Therefore, it can be concluded that the quasi-steady part of the power amplitude as a response of a harmonic surge motion is exclusively dependent on the dimensionless number b_{vel}. This knowledge will be used in section 5.5 to derive a set of load cases, which is suitable to distinguish the quasi-steady part of the load response from the transient part.

2.2 Transient aerodynamic phenomena

Leishman (2002) describes three transient aerodynamic phenomena that were found to be most significant in the context of wind turbine rotors. Namely, the *unsteady airfoil* effect (attached flow), the *dynamic inflow* effect and *dynamic stall*. This is in line with Schepers' selection of relevant transient aerodynamic phenomena (Schepers, 2012). In addition to that, Eliassen
85 (2015) showed that the *returning wake* effect may also influence the aerodynamic loads acting on a wind turbine in case of tower top vibrations. For the sake of simplicity, the experimental and numerical setups in the present work are designed so that stall has no or minor influence on the rotor loads. As a consequence, only the other three effects are described briefly in the following, while details can be found in the above mentioned publications.

Unsteady airfoil effect

90 The unsteady airfoil effect for attached flow describes two physical phenomena that are referred to as *circulatory* and *non-circulatory*. The circulatory effect is based on the fact that vortices are shed from the trailing edge of an airfoil when its circulation (i.e. lift force) changes. In turn, these vortices influence the angle of attack of the airfoil, which results in a mutual interaction between the angle of attack and the change of the lift force. This is a consequence of Kelvin's theorem, which claims that the cumulated circulation in a closed fluid volume cannot change. The shed vortices are then transported downstream,
95 which causes their influence on the angle of attack to vanish over time. The non-circulatory effect refers to added mass forces that act on the airfoil surface, which are caused by an acceleration or deceleration of the flow field around the airfoil. Such forces may occur e.g. when the airfoil is pitching or plunging. In case of a sinusoidal variation of the angle of attack in a frequency range that can be expected in the context of wind turbines, the circulatory effect is most prominent. It causes a time lag and a decrease of the lift amplitude with increasing frequency.

100 A reduced frequency f_a was described by Sebastian and Lackner (2013) to characterise the influence of the unsteady airfoil effect on a wind turbine blade section:

$$f_a = \frac{\pi f c(r)}{\sqrt{v_0^2 + (r\Omega)^2}}. \tag{5}$$

Here, v_0 and Ω describe the wind velocity and the angular speed of the rotor while r and c are the radius and chord length of the radial section under investigation, respectively. f denotes the frequency with which the inflow conditions vary (i.e. surge
105 motion frequency). f_a is referred to as the *airfoil reduced frequency* in this work.

Leishman proposed that the flow around an airfoil may be considered as quasi-steady up to $f_a = 0.05$ (Leishman, 2000). As the airfoil reduced frequency varies strongly over the blade span, it is difficult to use the value of f_a as a clear indicator for the strength of unsteady airfoil effect acting on the whole rotor.

Dynamic inflow effect

110 The helical vortex structure in the wake of a wind turbine induces an axial velocity component at the rotor plane, which is referred to as the induction. This induction has significant influence of the angle of attack at the blade sections and consequently

on the rotor loads. A change of the inflow situation of the rotor (i.e. due to a blade pitch event) leads to an adaptation of the wake structure and vortex strength, which in turn causes a change of the induction of the rotor. As the wake is transported downstream, the induction is influenced by the newly shed vortices as well as the 'old' wake structure. The influence of the
115 'old' wake decays over time until a new steady state of the wake and induction is reached. Consequently, the induction of the rotor can only change gradually after a change of the inflow, which is the main impact of the dynamic inflow effect.

In the context of a surge motion, the dynamic wake effect can be characterised by a normalised 'advection wave length' introduced by Sebastian and Lackner (2012). A more convenient way is to utilise its inverse, which known as wake or *rotor reduced frequency* f_r:

$$f_r = \frac{fD}{v_0} \tag{6}$$

where D denotes the rotor diameter. Typical values for the rotor reduced frequency corresponding to wave induced motions may reach up to 8[1].

Returning wake effect

As already mentioned in the context of the unsteady airfoil effect, vortices are shed from the trailing edge of an airfoil when
125 the circulation around it changes rapidly. In case of a rotor, these vortices may not only have an influence on the blade that was shedding them, but can also affect the flow situation around the following blades passing the same azimuth angle. In case of an oscillatory excitation of the inflow, the returning wake effect leads to a significant decrease of the load amplitude when the frequency of the vortex shedding matches the blade frequency.

This effect can be illustrated when considering the influence of the shed vortices of an airfoil on the flow field of a second
130 airfoil behind it. In Figure 1, the history of the shed vortices during a cycle of a periodic surge motion [2] of two airfoils in a row is illustrated schematically. In the presented case, the period of the motion is assumed to be equal to the time a vortex shed from the first airfoil needs to be transported to the second airfoil. The four rows shown in the figure correspond to four different positions during the surge motion. The first row represents the zero crossing of the surge motion during the downwards motion. In the second row, the maximum position is reached, while the third and fourth correspond to the zero crossing in upward
135 direction and the minimum position respectively. During the downward motion, the presence of the motion velocity leads to an increase of the the angle of attack and finally to a maximum of the lift force. The opposite is the case during the upward motion. The strength of the shed vortices is determined by the rate of change of the angle of attack, which is correlated to the actual rate of change of the surge motion velocity. In case of the downward motion in the first row, the motion velocity is maximum, which results in a zero rate of change of the angle of attack and no vortex shedding. When the airfoil reaches the
140 maximum position in the second row, the rate of change of the angle of attack is negative. Therefore, clockwise vortices are shed from the trailing edge of the airfoil. In the third row, where the zero position is reached during the upward motion, the

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[1] A rotor reduced frequency of 8 approximately corresponds to a 20 MW turbine at $5\,\mathrm{m\,s^{-1}}$ wind speed and a wave induced motion period of $7\,\mathrm{s}$.

[2] In the reference frame of the airfoil the surge motion of the rotor would be a plunge motion. However, for the sake of simplicity, the term surge motion is used here.
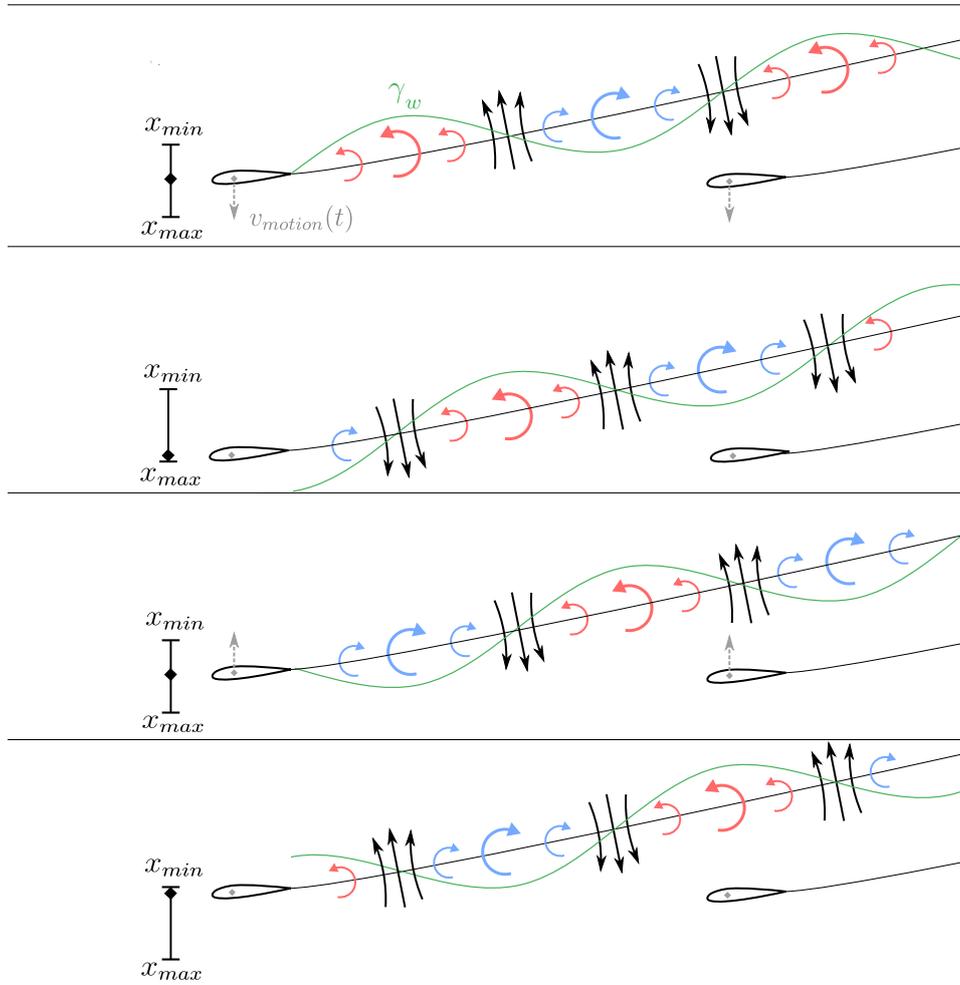
Figure 1. Schematic sketch of the vortex shedding at four steps during a surge motion cycle in the two-dimensional projection. The actual surge motion position varies between x_{min} (turbine moved in wind direction) and x_{max} (turbine moved againt wind direction). Dashed, grey arrows illustrate the moiton velocity. Full, black arrows indicate the local flow direction induced by the vortices. The green line represents the strength of the shed vorticity, which varies in a sinusoidal manner.

rate of change of the angle of attack and the vortex shedding is zero again. Finally, the shedding of counterclockwise vortices in the fourth row is caused by the positive rate of change of the angle of attack at the minimum position. This procedure is the same in every surge motion period, which results in an alternating shedding of clockwise and counterclockwise vortices with 145 the motion frequency.

The lift amplitude of both airfoils oscillate due to the surge motion. In the example, the vortices shed in one time instant reach the second airfoil exactly after one surge motion period. The superposition of the flow fields of the vortices yield an alternating
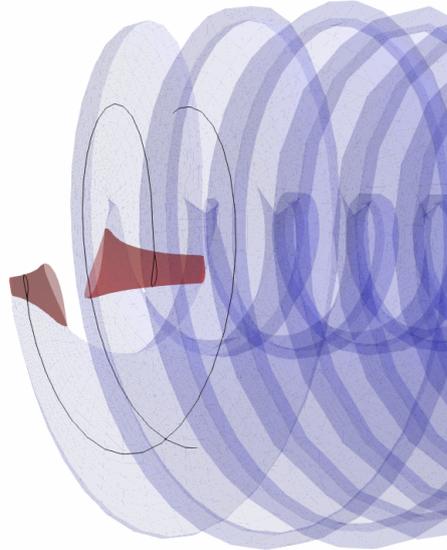
Figure 2. Illustration of blades and wake of the TUHH rotor as modelled in the panel method. A sectional view of blades and wake at a relative radius of 62 % is shown in black.

upwards and downwards flow. In case these flow regimes are near the second airfoil, its angle of attack is deviated by them. In the first row, the vortex induced velocity leads to a decrease of the angle of attack, when the airfoil has its maximum downward

150 motion velocity. Concurrently, the lift force is maximum due the downward motion motion of the airfoils. Therefore, the vortex induced velocity causes a decrease of the maximum lift force of the second airfoil. The opposite is the case in the third row, where the vortex induced velocity increases the angle of attack during the upward motion. As the minimum lift force occurs during the upward motion, this minimum increases due to the presence of the shed vortices. Consequently, the lift amplitude of the second airfoil caused by the surge motion decreases due to the presence of the shed vortices in case the vortices reach the

155 second airfoil after one motion period. In a similar way, it can be argued that a maximum of the lift force occurs when vortices reach the second airfoil after half a motion period.

The illustrated situation in Figure 1 can be adopted to a wind turbine rotor in surge motion. In this case, the two airfoils represent the blade sections of the wind turbine blades at the same radial position (see Figure 2). The distance between the airfoils is assumed to be equal to the distance one blade section travels until the next blade passes in the three-dimensional case.

160 Therefore, the time until the second blade passes the position where the vortices were emitted from the first blade is equal to $1/2$ of the rotation period of the rotor (for a two-bladed rotor). As a consequence, the above described drop of the lift amplitude occurs, when the motion period equals $1/2$ of the rotation period.

This effect is known as returning wake effect from helicopter aerodynamics (Leishman, 2000) and was described mathematically by Loewy (1957). It can be described by the ratio of both frequencies:

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$$q_b = \frac{2\pi f}{n_b \Omega}, \tag{7}$$

where n_b and Ω denote the number of blades and the angular velocity of the rotor respectively. In case of a harmonic surge motion of a wind turbine, this means that a drop of the lift amplitude of the blades occurs when the motion frequency is equal to or a multiple of the blade passing frequency. q_b would be an integer in this case.

In addition, the influence of the shed vortices on the lift force of the following blade is dependent on the distance from the blade to the vortices emitted by the preceding blade. A more detailed explanation of the physical principle is given in Eliassen (2015).

2.3 Suitability of FVW methods for wind turbine aerodynamics

Lifting line FVW methods have proofed their ability to predict the loads acting wind turbine rotors under typical, stationary operation conditions in several comparisons with experimental studies, see Schepers (2012) for example. The same is true for FVW panel methods (van Garrel, 2016; Roura et al., 2010), except for the occurrence of stalled blades. FVW methods are also generally capable of modelling the circulatory unsteady airfoil effect. This is due to the fact that the vortices emitted into the wake due to a change of circulation around the airfoil are directly modelled in these methods. However, the non-circulatory part caused by the accelerated fluid around the airfoil can only be modelled directly by unsteady FVW panel methods representing the whole blade surface instead of a single lifting line. This effect can be introduced in lifting line methods by means of empirical corrections. The dynamic inflow effect and the returning wake effect are caused by a varying strength and motion trajectory of the vortices in the wake due to unsteady changes of the circulation around the blades and/or the inflow conditions. This is directly modelled in FVW methods, which means that both effects can be captured by these methods. In practise, the ability of a lifting line method to explicitly model the dynamic inflow effect has also been shown in the past, e.g. by Schepers (2012) in conjunction with a pitch step of the NREL Phase VI experiment. As there is no fundamental difference in the wake treatment between lifting line and panel methods, this can also be transferred to panel methods.

3 Previous numerical and experimental works

3.1 Numerical investigations

A large number of simulation studies on the aerodynamic behaviour of wind turbine rotors undergoing typical FOWT motions have been presented in recent literature. A selection of these studies is listed in table 1. While a wide range of motion conditions is covered, the variety of wind turbine models is fairly limited among the listed studies. Most frequently, the NREL 5 MW turbine (Jonkman et al., 2009) with a rotor diameter of $126\,\mathrm{m}$ is considered, which is a comparably small size from today's perspective. Recently, the rotor utilised in the UNAFLOW test campaign is considered more frequently.

The characterisation of the aerodynamic load response to a rotor motion and the influence of transient phenomena is addressed in most of the listed studies. Often, a harmonic tower top surge motion is chosen in order to simplify the analysis of the simulation results. Among these, some studies utilising FVW or RANS methods report a linear increase of the normalised

Table 1. Selection of numerical investigations on rotor loads of surging and pitching wind turbines based on mid and high fidelity simulation approaches.

Authors	Methods	Turbine	Motions	Rotor reduced frequency* [-]	Motion to inflow velocity ratio* [%]	Norm. surge amplitude* [%]
Sebastian et al. (2012)	FVW, BEMT	NREL 5 MW	surge + pitch			
De Vaal et al. (2014)	AD, BEMT	NREL 5 MW	surge	0.23 - 0.90	4 - 71	3 - 13
Micallef et al. (2015)	AD, BEMT, GDW	NREL 5 MW	surge	1.23	6	0.8
Tran et al. (2015)	CFD, BEMT, GDW	NREL 5 MW	surge	0.23 - 1.35	4.5 - 108	3 - 13
Tran et al. (2015)	CFD, BEMT, GDW	NREL 5 MW	pitch	0.38 - 2.29		
Farrugia et al. (2015)	FVW, BEMT	NREL 5 MW	surge	0.83 - 2.21	5 - 15	1.1
Sant et al. (2015)	FVW, BEMT	scale model	surge	0.36 - 0.83		
Sant et al. (2016)	FVW, BEMT, GDW	NREL 5 MW	surge + pitch			
Wen et al. (2017)	FVW	NREL 5 MW	surge	0.55 - 5.53	- 41	
Wu et al. (2017)	FVW, CFD, GDW	NREL 5 MW	pitch	1.97 - 15.75		
Cormier et al. (2018)	FVW, CFD, BEMT	UNAFLOW	surge	0.60 - 1.19	2 - 6	- 1.5
Lienard et al. (2020)	CFD	NREL 5 MW	surge, pitch	1.11	44 - 88	6 - 13
Fang et al. (2020)	CFD	scale model	pitch	0.52 - 4.18		
Mancini et al (2020)	CFD, FVW, BEMT	UNAFLOW	surge	0.07 - 1.19	2 - 5	- 5
Chen et al. (2021)	CFD	NREL 5 MW	pitch + yaw			
Ferreira et al. (2022)	CFD, Semi-FVW, BEMT	NREL 5 MW	surge	0.16 - 3.19	10 - 200	10
Mancini et al. (2022)	FVW, BEMT	UNAFLOW	pitch	0.15		
Ribeiro et al. (2023)	FVW (panel)	UNAFLOW	surge	1.19 - 4.76	- 19	- 2
Bergua et al. (2023)	BEMT, FVW, GDW, CFD	UNAFLOW	surge	0.07 - 1.19	2 - 5	- 5
Schulz et al. (2023)	FVW (panel), BEMT	Vestas V80	surge	0.04 - 3.57	9	
Schulz et al. (present)	FVW (panel + LL), BEMT	scale model	surge	see table 3		
		UNAFLOW	surge	see table 3		
		IEA 15 MW	surge	see table 3		

* at operation conditions near optimum TSR

thrust force amplitude with the motion frequency or motion to inflow velocity ratio b_{vel} (see equation (2)). Wen et al. (2017) and Sant et al. (2015) found that the thrust force amplitude is linearly dependent on b_{vel}, as long as the b_{vel} stays in a moderate range below approximately 15 %. This can be explained by assuming a constant C_t and neglecting the higher order terms in equation (4) (and (A6)). However, these assumptions are only valid for moderate b_{vel} and the absolute limits are dependent on the special case. In the FVW and RANS simulations of Mancini et al. (2020) and Ribeiro et al. (2023), a linear rise of the thrust

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amplitude ΔT, which is normalised by the motion amplitude A, with the motion frequency is present. This can be expressed as:

$$\frac{\Delta T}{A} \propto f. \tag{8}$$

205 When considering that the motion to inflow velocity ratio is proportional to the product of motion frequency and amplitude,

$$b_{vel} \propto fA \tag{9}$$

it can be easily shown that this is equivalent to the above mentioned linear relation between the thrust amplitude and b_{vel}.

$$\Delta T \propto b_{vel} \tag{10}$$

In consequence, a constant motion to inflow velocity ratio results in an approximately constant power and thrust amplitude
210 which is confirmed by Wen et al. (2017). In the above mentioned cases, the thrust response amplitude can therefore be described in a quasi-steady way so that a transient contribution is most likely not present in a moderate range of b_{vel}. This linear trend could also be confirmed during the OC6[3] Phase III (Bergua et al., 2023). In this project, a considerable number of different low, mid and high fidelity methods were compared against each other and experiments utilising the UNAFLOW rotor. In contrast to this, lifting line FVW simulations of the NREL 5 MW wind turbine in surge motion conducted by Farrugia et al. (2015) show
215 a strong decrease of torque and thrust amplitudes from $0.15\,\mathrm{Hz}$ to $0.2\,\mathrm{Hz}$ at equal motion amplitudes. However, very similar simulation cases were also covered by Wen et al. (2017), who observed a linearly increasing trend again.

For higher motion to inflow velocity ratios (i.e. $b_{vel} > 15\%$) of the surging NREL 5 MW rotor around the design operation condition, Wen et al. found that the rise of the power amplitude becomes slightly nonlinear. A similar observation was found by Ribeiro et al. (2023) for the thrust force, where a FVW panel method was used to simulate the UNAFLOW rotor. In addition,
220 a rising phase shift between the surge velocity and rotor thrust was reported independent from b_{vel}, which was also observed by Mancini et al. (2020) earlier. Ribeiro et al. argue that this phase shift may lead to the lower-than-expected amplitude at high motion frequencies: As the strength of the unsteady airfoil effect, which most likely causes the phase shift, changes over the blade span, the phase shift also varies. Therefore, the lift contributions from the blade sections do not reach their maximum at the same time, which results in a lower cumulated amplitude. Although this reasoning seems plausible, a proof for this
225 hypothesis is not given. However, in this case a high motion velocity was investigated concurrently with a high b_{vel}. Therefore it cannot be distinguished whether the deviation from the linear trend indicates that the quasi-steady part of equation (4) cannot be approximated as linear due to the high b_{vel} or a transient contribution caused the deviation.

In a recent work of the authors (Schulz et al., 2023), a small contribution of transient aerodynamic phenomena to the torque and thrust response of the Vestas V80 rotor to surge motions was identified. While the unsteady airfoil correction of a BEMT
230 method predicted a decrease of the load amplitudes due to the unsteady airfoil effect at high motion frequencies, panel method simulations showed a slight increase. This contradicting behaviour of the results of the methods was attributed to unsteady, three-dimensional wake effects that cannot be covered by the BEMT.

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[3]Offshore Code Comparison, Collaboration, Continued, with Correlation and uncertainty, IEA Task 30

Table 2. Experimental investigations on rotor loads of surging wind turbines.

Authors	Load measurements	Rotor diameter [m]	Wind speed [m/s]	Rotor reduced frequency* -	Motion to inflow velocity ratio* [%]	Norm. surge amplitude* [%]
Farrugia et al. (2014)	torque	0.46	8.0	0.02 - 0.05		
Sivalingam et al. (2018)	thrust, torque	1.00	0.8 - 1.2	0.18 - 0.89	3 - 53	2 - 12
Bayati et al. (2016)	thrust, torque	2.38	2.33 - 3.67	0.16 - 2.15	3 - 7	- 4
Bayati et al. (2018) *and*						
Mancini et al. (2020) *and*						
Fontanella et al. (2021)	thrust, torque	2.38	2.5 - 6.0	0.07 - 1.19	2 - 5	- 5
Meng et al. (2022)	thrust, power	0.18	4.9 - 7.1	- 0.01	- 4	10 - 20
Wei et al. (2022)	torque	1.17	8.0	0.36 - 0.83		
Fontanella et al. (2022), (2023)	thrust, torque	2.40	2.95 - 5.0	0.15 - 2.54	4 - 5	- 6
Schulz et al. (present)	torque	0.93	8.5	0.55 - 1.09	2 - 9	- 2

* at operation conditions near optimum TSR; measurement series 'RATED' for UNAFLOW campaign

When increasing the motion to inflow velocity ratio to extreme values, minimum and maximum power become more and more asymmetric to the mean value. Such behaviour is present in Tran et al. (2015) and Chen et al. (2021), where b_{vel} of 55 % or even higher have been investigated. This may be explained by the terms containing the third power of b_{vel} in the quasi-steady part of equation (A5) and does therefore not necessarily indicate a transient contribution.

In the above mentioned studies, the amplitude of the rotor thrust (and torque) response to a surge motion was mostly determined by a quasi-steady behaviour. However, some hints were found that the amplitudes may be deviated by transient effects at high motion frequencies. Unfortunately, the influence of the potential transient effects could not be investigated accurately in most cases, because the simulation cases did not allow for a clear distinction between these and the quasi-steady effects.

3.2 Experimental investigations

A number of investigations considering FOWT rotors in motion have been conducted in the last decade. Most studies either address the influence of the motion on the wake development or the rotor loads. The most relevant studies that focus on the unsteady rotor loads are listed in table 2. As the model turbines are significantly downscaled, a scaling parameter for the surge motion is mandatory. Many studies utilise the rotor reduced frequency for this purpose. However, it has to be noted that from the three above mentioned transient aerodynamic phenomena, only the dynamic inflow effect is covered by this choice.

Most studies utilise rotor diameters of $1\,\mathrm{m}$ and higher, because higher rotor diameters lead to higher rotor reduced frequencies at the same motion frequency. This is also necessary to maintain a high local Reynolds number at the blade, which is a prerequisite for reaching realistic power coefficients and good comparability between experiment and full scale behaviour. As an exception, Farrugia et al. (2014) presented torque measurements of a combined wind tunnel and wave tank experiment of a model FOWT with a rotor diameter of $0.46\,\mathrm{m}$. As the wind speed has been chosen to be comparatively high for wave tank experiments $(8\,\mathrm{m\,s^{-1}})$, a low maximum rotor reduced frequency of 0.047 resulted. Mean values and standard deviation of the measured torque were analysed for a large number of tip speed ratios (TSR) and different wave frequencies. Finally, no evidence for motion-induced torque fluctuations was found in the measurements, while FVW simulations clearly showed such fluctuations. Farrugia et al. noted that this mismatch is likely caused by the limited capabilities of the utilised torque meter.

One simple way to reach high wake reduced frequencies is to make use of an excessively low inflow wind speed. However, this usually leads to a significantly reduced flow quality and high uncertainties in the average wind speed as most wind tunnels are unable to reach and control a homogeneous flow at low wind speeds. In addition, the local Reynolds number at the blades becomes very small so that viscous contributions cause significant scale effects. Sivalingam et al. (2018) solved these issues by using water as flow medium, which increases the Reynolds number massively due to its low kinematic viscosity. The model rotor with a diameter of $1\,\mathrm{m}$ was mounted on a rigid carriage and towed through a towing tank at travel speeds around $1\,\mathrm{m\,s^{-1}}$. An actuator mounted on the carriage realised the surge motion with different frequencies and amplitudes. In this way, a maximum rotor reduced frequency of 0.89 could be achieved at a relatively low surge frequency of $0.89\,\mathrm{Hz}$ during normal operation of the rotor at a TSR of 6. Torque and thrust have been measured directly at the shaft, but it was not mentioned whether a correction of the inertial forces due to the surge movement was used. In addition to the experimental results, three kinds of simulations utilising a RANS, a classical BEMT and a BEMT method with unsteady corrections are presented. While the experiment at optimum TSR consistently shows a significantly higher thrust amplitude (between approx. $10\,\%$ and $25\,\%$) than the RANS and the classical BEMT simulations, the unsteady BEMT results do not follow a clear trend in comparison the experiments. The measured mean thrust force scatters in a range of around $5\,\%$, but no clear trend with the motion frequency or motion to inflow velocity ratio could be identified. For torque oscillations at optimum TSR the simulation results are in better agreement with each other, while the experiment again shows significantly higher amplitudes. It is interesting to note that negative values for the measured rotor torque already occur at a motion to inflow velocity ratio of $13.5\,\%$. RANS simulations by Tran et al. (2015) and Lienard et al. (2020) show zero power output at motion to inflow velocity ratios above $70\,\%$ and $80\,\%$ respectively. Unfortunately, a quantitative analysis of transient contributions to torque and thrust amplitudes is not given.

A series of experimental campaigns has been performed in the large wind tunnel of the Politecnico di Milano (PoliMi) using a rotor with a diameter of $2.38\,\mathrm{m}$. Maximum rotor reduced frequencies of up to 2.15 could be achieved for surge and pitch motions in the first study (Bayati et al., 2016). Later, it was found that tower vibrations had a significant influence on the measurements and a second campaign with a stiffer tower was performed within the UNAFLOW project (Fontanella et al., 2021; Mancini et al., 2020), which focused on surge motions. The rotor thrust was measured by two triaxial force-moment sensors at the tower top and tower base. At high motion frequencies, the measured thrust force was dominated by inertial loads resulting from the tower top acceleration due to the surge motion. To remove the inertial loads from the thrust signal,

two compensation strategies were utilised: First, the inertial loads were determined in an experiment without wind and later
subtracted from the signal with wind (difference-based compensation). Second, the tower top acceleration was measured with
an accelerometer near the force-moment sensor and the inertial contribution was calculated from mass and actual acceleration
(acceleration-based compensation). However, in both cases the resulting thrust signal is extremely noisy due to tower vibrations
and other sources at higher surge frequencies (Mancini et al., 2020; Bayati et al., 2018), so that only amplitude and phase of
the first harmonic of the thrust signal were used as measurement result. Such data processing requires that the signal can be
considered as a mono-harmonic sine function in order to be accurate. Both compensations seem to be limited to a maximum
motion frequency as a result of the tower vibrations. While the thrust measurements were found to be reliable up to a reduced
frequency of approximately 0.9 for the difference-based compensation, a maximum reduced frequency of approximately 1.2
could be reached by the acceleration-based compensation (Mancini et al., 2020). In Fontanella et al. (2021), two measurement
series at rated turbine conditions (i.e. optimum TSR) are presented. The main difference between these series is the inflow
wind speed, which translates to a variation of the local Reynolds numbers at the blades. This difference seems to cause strong
deviations between the two series in the amplitude of the thrust force. The series with the higher wind speed is considered as
more reliable as unwanted viscous phenomena should have a reduced influence and a higher thrust force can be measured in
this case. An essentially linear relation between and the thrust amplitude per motion amplitude and the rotor reduced frequency
was found up to a reduced frequency around 0.6. For small motion amplitudes, this corresponds the a linear relation between
the thrust amplitude and the motion to inflow velocity ratio, which was also observed in some of the simulation studies. For
a higher reduced frequency, the scattering between the results for the different motion amplitudes is approximately 20 % of
their mean value. As the results for the highest and the lowest motion amplitude are very similar while the two intermediate
motion amplitudes caused the highest and the lowest thrust amplitude, no physically meaningful interpretation of the scattered
results could be found. Therefore, these differences may be considered as measurement uncertainty. In Mancini et al. (2020),
where the acceleration-based compensation is applied, a very similar trend was observed. However, the scattering seems to be
slightly lower in this case. In both cases, no significant phase shift between the thrust force and the surge velocity could be
observed, while the interpretation of the phase angles was biased by a scattering of 5° to 10°. Mancini et al. noted that the
presented rotor torque measurements were strongly disturbed by a drive train resonance, which was at least present for the two
highest reduced frequencies. However, no clear, independent criterion was presented, which could identify the border between
the valid and the invalid measurement points. Therefore, also the results below a reduced frequency of 0.6 could be slightly
affected by this distortion. An empirical model was tuned in order to compare the expected quasi-steady load amplitudes
against the measurements. The quasi-steady model tends to underpredict the measured load amplitudes in both cases, however
the scattering of the measurements at higher motion frequencies makes an evaluation difficult.

In the context of the COREWIND project, another model rotor was designed and subjected to surge motions in the wind
tunnel of PoliMi. The COREWIND rotor is very similar to the UNAFLOW rotor as the same airfoil, nearly the same rotor
diameter and a similar operation parameters are utilised. Another test rig is used to impose the surge motion, which is also able
to perform 6-DOF motions in a limited frame. In Fontanella et al. (2022), motions in surge, sway, roll, pitch and yaw were
performed. For the surge motions, the difference-based inertia compensation is applied to the thrust measurements. Due to a

lower wind speed, a higher rotor reduced frequency compared to the UNAFLOW experiments could be reached in the tests. In summary, three rotor reduced frequencies were tested: 0.14, 0.28 and 2.54. At the highest frequency, the measured thrust and torque amplitudes were found to be around 30 % higher than the quasi-steady prediction of the empirical model. In case the prediction is able to accurately represent the quasi-steady behaviour of the model rotor, this is a clear proof for a significant contribution of transient aerodynamic phenomena. Recently, another work of Fontanella et al. (2023) presents an experimental investigation with a the same rotor containing a control system. Fontanella et al. mention a significant influence of small rotor speed oscillations on the measured rotor torque amplitude. As a consequence of these oscillations, the torque measurements are altered by inertial loads. The torque measurements are therefore corrected in a post processing step.

The investigations by Meng et al. (2022) and Wei et al. (2022) also include load measurements of a surging rotor. However, these are not directly comparable to the present work, because of major differences in the measurement and/or data processing.

The above mentioned studies show that the measurement of rotor loads in surge motion is a challenging task. Vibrations lead to a high random uncertainty or even biased measurement results at comparatively low rotor reduced frequencies. Nevertheless, the linear trend of the rotor thrust amplitude with the motion to inflow velocity ratio could be proofed for low rotor reduced frequencies in the PoliMi experiments. Latest results from the COREWIND project indicate that this is also the case for moderate rotor reduced frequencies up to 1.7. A strong indication for transient effects could be found in another presentation of the COREWIND campaign, but no confirmation by simulations or other sources is available so far.

4   Numerical methods and setups

The panel method *pan*MARE and its submodule for lifting line calculations are utilised for the simulation of the three wind turbine rotors under consideration. In case of the IEA 15 MW rotor, additional BEMT simulations using OpenFAST are performed in order to evaluate the modelling of transient phenomena in state-of-the-art BEMT methods during a surge motion. For the sake of simplicity, the numerical methods are exclusively used to calculate aerodynamic loads, while inertia loads, rotor speed variations or controller actions are excluded from the analsys. The utilised methods and setups are described briefly in the following sections.

4.1   *pan*MARE: Panel method

The first-order panel method *pan*MARE is an in-house development of the Hamburg University of Technology (TUHH) and has been utilised for floating structures and ship propellers (Bauer and Abdel-Maksoud, 2012) before it was extended to the simulation of FOWTs by Netzband et al. (2018). Validations were carried out with different wind turbine rotors and a verification with the results of the Offshore Code Comparison Continuation (OC4) campaign showed that combined aerodynamic, hydrodynamic and mooring loads as well as platform motions can be accurately captured (Netzband et al., 2018). *pan*MARE has also been utilised to develop and simulate the yaw mechanism of the passively yawing FOWT concept CRUSE Offshore SelfAligner (Netzband et al., 2020). A validation and verification with full-scale measurement data and simulations of the Floatgen prototype has been performed during the VAMOS project (Netzband et al., 2023).

In *pan*MARE, the blade surface is discretised by panels. While source and doublet distributions are assigned to these panels, the three-dimensional wake is represented by doublet panels transported with the local wind speed (free wake). Panel methods are based on the potential theory, which allows to describe the flow field as a boundary value problem. Therefore, viscous drag forces need to be considered by a viscous correction. In the present case, this correction is based on empirical airfoil drag coefficients of the blade sections. At every blade section, the inflow speed is calculated from the stagnation pressure. The effective angle of attack is then found by a comparison of the sectional lift force with the lift force of inviscid, two-dimensional simulations of the airfoil. Finally, the drag force of a section is calculated on the basis of local inflow velocity, effective angle of attack and the empirical coefficients. A more detailed description of the panel method in conjunction with wind turbine rotors can be found in Netzband et al. (2018)

The TUHH model rotor is discretised by 2,940 blade surface panels and 9,600 wake panels, where the wake deformation is frozen after one rotor rotation. This represents a wake length of approximately 4 rotor diameters. The discretisation of the wake is coarsened in 4 steps. Starting with a resolution of $1.26°$ azimuth angle per wake panel in tangential direction in the direct vicinity of the trailing edge, which corresponds to a time step size of $0.0002\,\mathrm{s}$. The tangential resolution of the near wake is approximately $10°$ per wake panel, while the discretisation of the far wake is $20°$ per wake panel in tangential direction. It has to be noted that the time step size is driven by the necessity to capture the surge motion in sufficient detail here. The cylindrical and transition blade sections near the root cannot be modelled in potential theory. Therefore, the blade starts at a relative radius of $r/R = 0.2$, where the blade is aerodynamically shaped. As the panel method is limited to inviscid flow, low Reynolds number regimes as seen in experiments often cause a slight overprediction of the lift force of the utilised airfoil. In case of the SD7062 airfoil, a simple lift correction could be applied in the panel method, which is described in appendix B.

For the model of the IEA 15 MW rotor, 3,150 blade surface panels and 17,280 wake panels are used. A wake length of approximately 4 rotor diameters is covered by the wake panels. Three wake coarsening steps are applied so that a resolution of $3°$ per wake panel is reached near the blade. This corresponds to a time step size of $0.1\,\mathrm{s}$. The resolution is $6°$ per wake panel in the near wake and $12°$ per panel in the far wake. The blade is modelled from a relative radius of $r/R = 0.18$.

4.2 *pan*MARE: Lifting line method

A submodule for lifting line simulations was implemented in *pan*MARE and used to efficiently generate initial wake geometries for propeller simulations in Wang and Abdel-Maksoud (2020). This submodule was slightly modified and adopted to wind turbines by the authors. In the submodule, the discretised blade surface is replaced by a simple lifting line at the 1/4 chord position and the wake is shed from the end of the chord line. In every time step, the local inflow velocity and angle of attack is determined from the flow field, which includes the influence of the wake and all inflow parameters. Then, the lift and drag forces are determined from empirical airfoil coefficients. Finally, the circulation of the lifting line is computed from the lift force. No unsteady corrections are used in the current version of the lifting line submodule. As described in section 2, these might be helpful for the incorporation of the non-circulatory unsteady airfoil effect as well as for the modelling of dynamic

385   stall. However, both phenomena have minor influence in the considered cases.

The TUHH model rotor is discretised with the same resolution as in the panel method. Instead of the blade representation, a lifting line with 20 sections in spanwise direction is utilised. Lift and drag coefficients of the SD7062 were computed using Xfoil (Drela, 1989) for a local Reynolds number of 125 k. The two-dimensional simulations were validated against experimen-

390   tal data presented in Lyon et al. (1997) at slightly different Reynolds numbers and proofed excellent agreement.

The UNAFLOW rotor is divided into 24 spanwise sections and a total number of 14,400 wake panels is utilised. This corresponds to a wake length of approximately 3 rotor diameters. A time step size of $0.002\,\mathrm{s}$ is applied, which corresponds to a angular resolution of $2.88°$ per time step. The wake resoultion is approximately $6°$ in the near wake and $23°$ in the far wake. Lift and drag coefficients are chosen according to the description in Bergua et al. (2023).

395   ### 4.3   OpenFAST

In BEMT methods, a momentum equilibrium on every annular ring of a wind turbine rotor is assumed. With this assumption, the rotor loads can be computed based on empirical lift and drag coefficients of the utilised airfoils. In contrast to FVW methods, the direct modelling of the flow field in the wake is not part of the method. The BEMT method AeroDyn v15, which is part of OpenFAST (v3.1.0), contains two unsteady corrections to correct for the unsteady airfoil and dynamic inflow effect. The

400   unsteady airfoil correction is based on the Leishman-Beddoes model (Leishman and Beddoes, 1989) with slight modifications. The dynamic inflow correction based on Øye (see Snel et al. (1995)) prevents the axial induction from undergoing rapid changes. Further information on AeroDyn and its unsteady corrections is presented in Moriarty and Hansen (2005); Damiani and Hayman (2019) and Branlard et al. (2022).

In the model of the IEA 15 MW rotor, Prandtl tip and hub loss corrections as well as the unsteady unsteady airfoil and

405   dynamic wake corrections are applied. The blade geometry and parameters for the unsteady airfoil correction published in the official github repository of the IEA 15 MW turbine[4] were utilised in the simulations. It has to be noted that the coefficients A_1, A_2, b_1 and b_2 for the unsteady airfoil correction have originally been derived for a NACA0012 airfoil (Leishman and Beddoes, 1989). However, individualised coefficients for the various airfoil shapes at the blade sections have not been discovered in literature. For the dynamic inflow correction, the time constant τ_1 is chosen according to Øye (see e.g. Branlard et al. (2022)).

410   The time step size is set to $0.1\,\mathrm{s}$. A study on smaller time steps showed no noteable influence of this particular choice on the results.

5   Measurement campaign and simulations

Figure 3. Photograph of the wind tunnel setup.
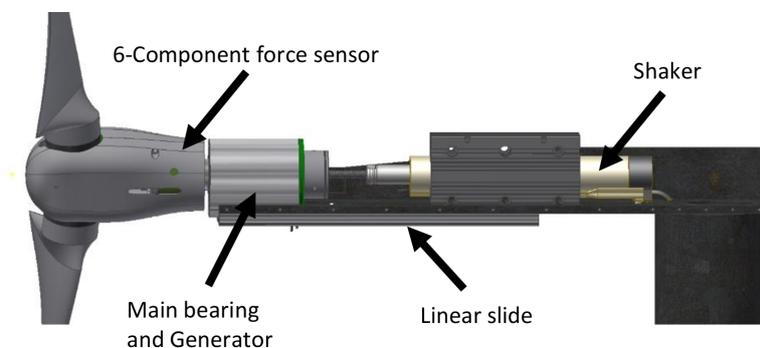
Figure 4. Sketch of the tower top of the test rig. The rotor nacelle assembly is mounted on a carriage, which can be moved in surge direction by the linear actuator.

5.1 Experimental setup

The experimental investigations were performed in the wind tunnel of the TUHH. Its test section is 2 m high, 3 m wide and 5 m long and the flow is characterised by a turbulence intensity below 0.5 %. A description of the wind tunnel environment and the TUHH model rotor can be found in a previous study of the yaw moment of the rotor (Schulz et al., 2022). Therefore, only a brief description on these topics is given in the following.

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[4]https://github.com/IEAWindTask37/IEA-15-240-RWT, accessed at May 3rd, 2023
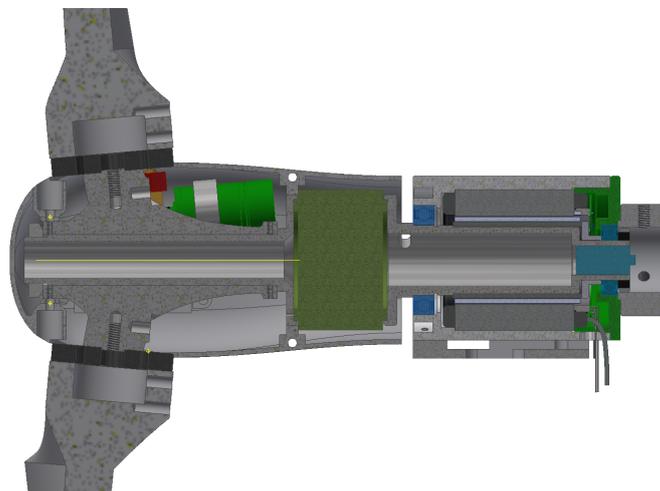
Figure 5. Sectional view of the nacelle. The load sensor (dark green) is integrated in the rotating shaft. The generator is illustrated in dark grey and the main bearings are coloured in blue. A slip ring is placed inside the shaft at its right end.

The two-bladed model wind turbine (see Figure 3) has a rotor diameter of $0.93\,\mathrm{m}$. It is designed to be rigid so that no notable influence of blade bending and torsion on the load measurements is expected. The accuracy of the blade manufacturing was
420   checked using a high-accuracy 3D surface scan. As scaled wind turbines generally suffer from a very low Reynolds number, the chord length is often significantly increased in comparison to the geometric scaling of full scale turbines. This leads to a redesign of the complete blade geometry. Therefore, scale models can usually not be considered as certain turbine type anymore but serve as representatives for a typical modern wind turbine. In this case, a Reynolds number of approximately $125\,\mathrm{k}$ could be maintained over major parts of the blades due to the increased chord length and a comparatively high rotational speed
425   of $1050\,\mathrm{RPM}$ during the measurements. In addition, the SD7062 airfoil was used for the whole blade, as laminar-turbulent transition occurs earlier compared to conventional wind turbine airfoils, which has a positive effect on aerodynamic properties at low Reynolds numbers.

Due to the specialised blade design, the occurrence of stall is not expected except very near to the root. All data to set up a numerical model of the rotor is given in Schulz et al. (2022).
430   A novel test rig (see Figure 3) was developed in order to impose surge motions with high frequencies while minimising tower top vibrations. The key feature of the test rig is that the nacelle is mounted on a linear slide at the tower top (see Figure 4), which is connected to a magnetic linear motor. This allows to make use of a heavy and rigid tower design, while keeping the weight of the moving parts low. The steel tower with a height of $1\,\mathrm{m}$ consists of a base plate, a main column with a diameter of $0.1\,\mathrm{m}$ and a thickness of $6\,\mathrm{mm}$ and bracings that support the main column. The lowest eigenfrequency of the tower structure
435   was found to be approximately $800\,\mathrm{Hz}$ in finite element simulations. A sketch of the nacelle is given in Figure 5. In order to reduce the inertia loads disturbing the thrust measurement, a 6-component force-moment sensor (dark green in Figure 5) was integrated into the rotating hub. Therefore, a wireless transmission of the measurement data was necessary. In addition, hall
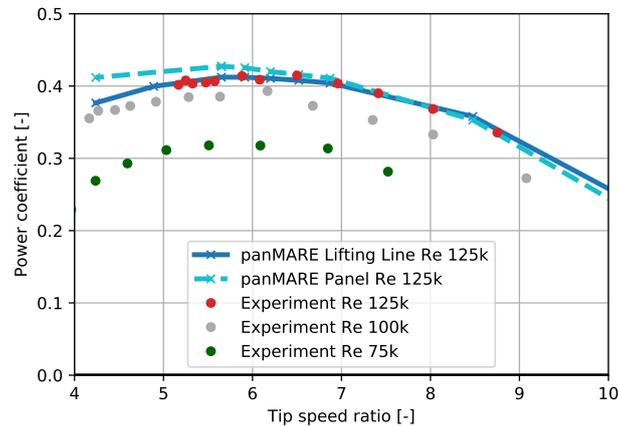
Figure 6. Power coefficient of the model rotor in dependence of the TSR. Lift and drag coefficients used in the simulations correspond to a Reynolds number of 125k.

sensors, a surge position sensor and an accelerometer were placed in the nacelle. Details on the sensors and data acquisition can be found in appendix C.

440 ## 5.2 Steady operation conditions

In order to validate the capability of the simulation models to capture the quasi-steady behaviour of the model rotor, multiple test series with constant rotational speed and varying inflow wind velocity were performed and the experimental results were compared with those of the simulations. The three test series with rotational speeds of 600, 800 and 1050 RPM approximately correspond to local Reynolds numbers of 75 k, 100 k and 125 k at the blade sections. The measured values of the rotor torque

445 were filtered for outliers, averaged over a period of 15 s and then utilised to compute the power coefficients, which are shown in Figure 6. For the highest Reynolds number, a maximum power coefficient of approximately 0.41 is reached, which confirms a comparatively low influence of undesirable viscous effects induced by the scaling. Therefore, a low sensitivity of local variations of the Reynolds number on the measured aerodynamic loads can be concluded. In addition, the smoothness of the curves gives a hint on a comparatively low repetition error, which was also observed in a previous investigation of this rotor

450 (Schulz et al., 2022).

The lifting line simulations show a nearly exact agreement with the corresponding measurement series, while the panel method overpredicts the power coefficient slightly at low TSR. The operating point at which the surge tests were performed was selected to be at the highest rotational speed and a wind speed of $8.51\,\mathrm{m\,s^{-1}}$. This corresponds to a TSR of 6 and optimal power generation.

5.3 Thrust measurements in surge motion

Sinusoidal surge motions were applied to the model rotor at constant rotational speed and wind speed. An overview about the experimental load cases is given in table 3. The motion amplitudes and frequencies were chosen so that realistic motion to inflow velocity ratios for a FOWT in typical environmental conditions were covered. However, vibrations of the tower top limited the rotor reduced frequency to 1.09.

Although thrust measurements were recorded, a sufficiently low uncertainty could not be achieved. Accelerometer-based and difference-based inertia removal procedures were applied to separate the aerodynamic thrust force from the inertial forces of the rigid body. In the accelerometer based procedure, the nacelle acceleration was multiplied by the accelerated mass of the hub and subtracted from the axial rotor force. In the difference based procedure, the axial rotor force in surge motion was measured with and without wind. To gain the aerodynamic rotor thrust, the signal without wind was subtracted from the signal with wind. The inertia corrected thrust force signals were phase averaged over at least 10 periods and the thrust force amplitudes were extracted.

To check the integrity of the inertia force compensation procedures, a number of selected load cases were run multiple times without wind. Both procedures were applied to evaluate the measured values of test runs, which should result in a vanishing rotor thrust as no aerodynamic load was applied. In the covered motion range, the residual thrust amplitudes deviated up to 10 % of the nominal thrust from zero. Since the magnitude of the uncertainty is in the order of magnitude of the total thrust force amplitude at low b_{vel}, the determination of the transient parts would be pursued as the identification of these parts of the load response require higher accuracy than in the measurements performed.

Forced tower top oscillations are considered as the main reason for the insufficient accuracy of the inertia removal procedures. These oscillations could not be filtered, because they occur exact at the same frequency as the inertia and aerodynamic forces. Finally, it was found that the tower and support structure could not maintain a sufficiently low level of vibrations, although these were designed in a very rigid manner in comparison to other similar experimental set-ups. In contrast to this, torque measurements are less sensitive to tower top vibrations.

5.4 Torque measurements in surge motion

Large effort was undertaken to keep an ideally constant rotor torque during the surge motion, however, a sinusoidal variation of the rotor speed in a range of 0.1 to 0.3 % of the nominal rotor speed could not be avoided. Due to the acceleration and deceleration of the rotor, the measured rotor torque contained a significant contribution of the corresponding rigid body inertia loads. In the present case, this contribution counteracted the aerodynamic rotor torque and therefore compensated the measured torque fluctuations to surprisingly small values. The same phenomenon was observed in the COREWIND experiments by Fontanella et al. (2023). As a consequence, an inertia compensation procedure needs to be applied to the torque measurements

Table 3. Surge motion load cases for the three different rotors. For the simulation studies, only minimum and maximum values for one series are given. Maximum values of periods and minimum values of amplitudes correspond to the same load cases and vice verca.

		absolute			*dimensionless*		
	Wind speed	Rotational speed	Motion period	Motion amplitude	Rotor reduced frequency	Norm. amplitude	Motion to inflow velocity ratio
TUHH rotor experiment	$8.51\,\mathrm{m\,s^{-1}}$	1050 RPM	0.2 s	5.10 mm	0.546	0.59 %	1.88 %
			0.2 s	10.1 mm	0.546	1.09 %	3.74 %
			0.2 s	15.1 mm	0.546	1.62 %	5.57 %
			0.125 s	5.30 mm	0.874	0.57 %	3.13 %
			0.125 s	10.5 mm	0.874	1.13 %	6.20 %
			0.125 s	15.7 mm	0.874	1.69 %	9.27 %
			0.1 s	5.50 mm	1.093	0.59 %	4.06 %
			0.1 s	10.9 mm	1.093	1.17 %	8.03 %
TUHH rotor extended	$8.51\,\mathrm{m\,s^{-1}}$	1050 RPM	$0.0217 - 2.17\,\mathrm{s}$	$2.18 - 218\,\mathrm{mm}$	5.04 - 0.0504	0.2344 - 23.44 %	7.42 %
			$0.0217 - 2.17\,\mathrm{s}$	$3.26 - 326\,\mathrm{mm}$	5.04 - 0.0504	0.3505 - 35.05 %	11.09 %
UNAFLOW rotor	$4.19\,\mathrm{m\,s^{-1}}$	240 RPM	$0.0625 - 8.00\,\mathrm{s}$	$2.20 - 282\,\mathrm{mm}$	9.09 - 0.071	0.0924 - 11.83 %	5.28 %
IEA 15 MW rotor	$7\,\mathrm{m\,s^{-1}}$	5 RPM	$2.5 - 300\,\mathrm{s}$	$0.30 - 36.0\,\mathrm{m}$	13.7 - 0.114	0.125 - 15.0 %	10.77 %
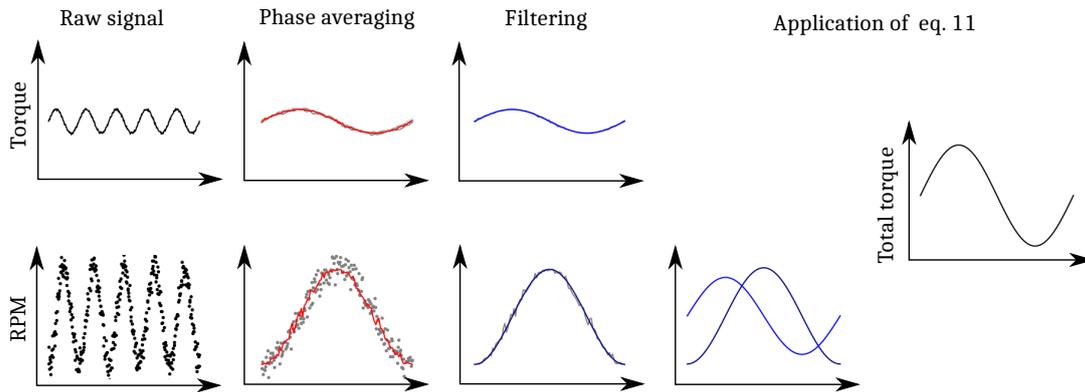
Figure 7. Illustration of the data acquisition and inertia removal procedure for the torque measurements. Signal parts from load sensor measurements (top) and the rotational speed measurements (bottom) are added up in the last step.

even for very small variations of the rotor torque. The aerodynamic rotor torque Q can be written as:

$$Q(t) = Q_{meas}(t) + J\frac{d\Omega}{dt}. \tag{11}$$

Where $Q_{meas}(t)$ is the measured torque and J describes the mass moment of inertia of the rotating part of hub and rotor behind
490   the sensor. As a prerequisite for the application of such compensations, the rotor speed oscillations need to be measured with

high accuracy. The calculation procedure of the aerodynamic torque is illustrated in Figure 7. In the procedure, the rotational speed signal is phase averaged over at least 15 periods and than low pass filtered with an edge frequency of 2.5 times the motion frequency. Then, a central differencing scheme is applied to calculate the numerical derivative and the result is fed into equation (11). The measured rotor torque is also phase averaged and filtered in the same way. Finally, the aerodynamic ro-
495   tor torque is calculated from equation (11). An estimation of the systematic uncertainty of this procedure is given in appendix D.

In Figure 8, the measured, phase averaged and filtered rotational speed variations as well as the compensated rotor torque are shown for all load cases. Due to the low amplitude of the rotational speed oscillation, the signal is largely scattered. However, the phase averaged results follow a sinusoidal trend in every case. Small disturbances of the gradient are disregarded using the
500   low pass filtering. The aerodynamic torque signal follows a mostly mono-harmonic trend, which is not a result of the filtering but the measured characteristic of the signal.

The extracted torque amplitudes shown in Figure 9 are normalised by the torque measured (or simulated) in steady conditions at the operation point. Only marginal differences between the experimental and numerical results are present, no matter if panel method or lifting line simulations are considered. Every measurement was repeated at least twice[5]. The results for the same
505   load case show very similar values, which indicates a comparatively small repetition error. For selected load cases, also a third run was performed, which resulted in similar values but are not shown here.

The same amplitudes are plotted over the corresponding motion to inflow velocity ratio b_{vel} and coloured by the rotor reduced frequency in Figure 10. A linear ratio between the rotor torque amplitude and b_{vel} can be observed. This linear trend is captured nearly exactly by both simulation methods. Interestingly, the variation of the rotor reduced frequency does not seem
510   to have any impact on this relation. Therefore, it can be clearly seen that b_{vel} determines the rotor torque amplitude, when rotor speed and inflow speed are kept constant. This can be considered as proof that no transient contributions to the torque amplitude are present in these cases. It has to be noted that the absence of transient behaviour can be concluded from the independence of the linear trend from the motion frequency, however not necessarily from the presence of a liner trend itself.

In Figure 11, the average rotor power derived from the rotor torque measurements normalised by the steady state power is
515   plotted over b_{vel}. A slight increase of the power can be observed with rising b_{vel}. In addition, comparatively strong scattering is visible, because the deviation from the steady state power is in a range of only 1 %. Again, no systematic influence of the motion frequency is detectable, which proofs the absence of a transient influence on the average power. Both simulations agree closely with the measurements, which is not surprising as it has been shown in Figure 6 that the quasi-steady power of the model rotor can be accurately captured by the simulations.
520

In summary, a precise agreement of measurements and the lifting line and panel method simulations in the steady and unsteady cases can be observed. No hint on transient contributions to the rotor torque or any other transient behaviour could be found in the experiments and simulations. Therefore, the rotor torque oscillations can be considered as quasi-steady and can

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[5]Two cases are missing in the graph, because the rotational speed contained a significantly higher scattering in comparison to the other test runs.
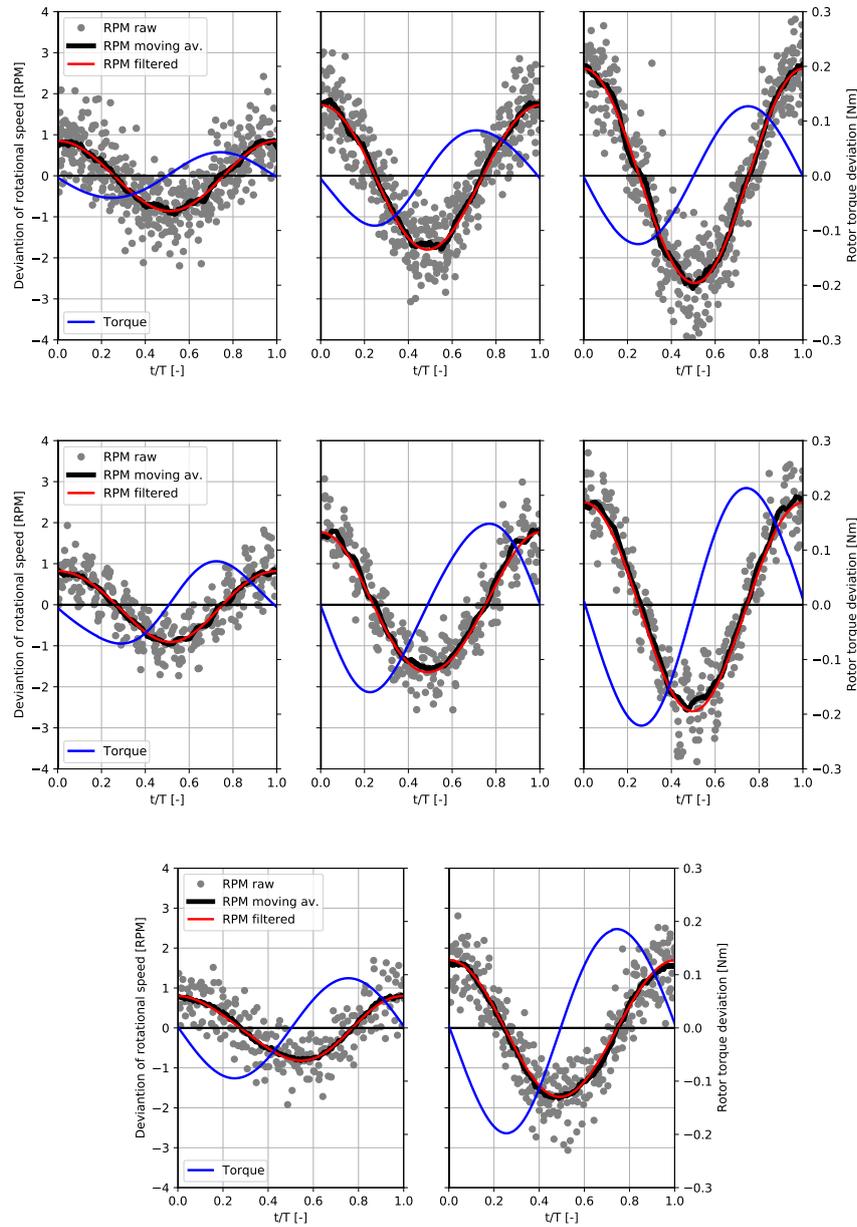
Figure 8. Raw, phase averaged and filtered variation of the rotational speed measurements (left y-axis) and phase averaged, inertia-compensated rotor torque fluctuations (right y-axis).

be determined by b_{vel} exclusively in the applied range of motions and at constant rotor speed and wind inflow conditions. It has to be noted that only aerodynamic loads and no controller actions are considered here and in all following results.
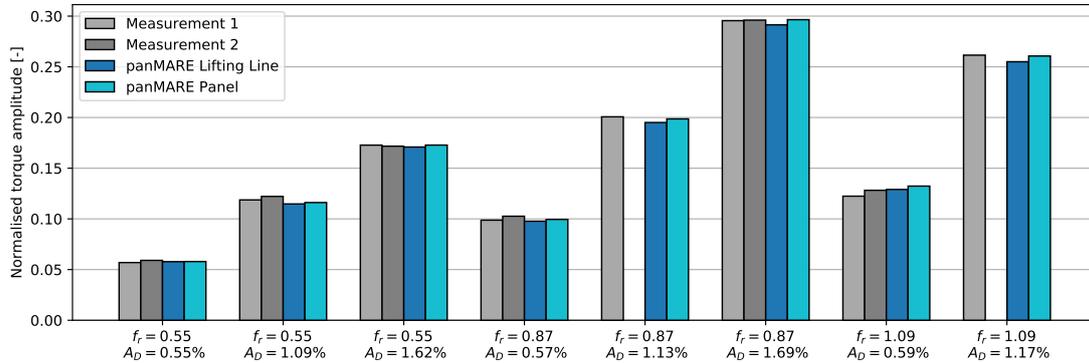
Figure 9. Measured and simulated rotor torque amplitudes as response to 8 different surge motions. Values are normalised by the corresponding average rotor torque without motion. Two measurement runs under identical conditions are shown for most surge motions. A_D describes the motion amplitude normalised to the rotor diameter.
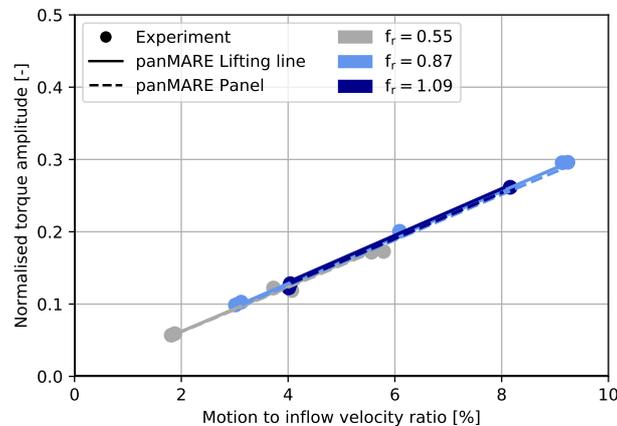
Figure 10. Measured and simulated rotor torque amplitudes as response to the surge motions plotted over the motion to inflow velocity ratio.

5.5 Simulation of extended motion parameters

The test rig was limited to moderate motion frequencies. In reality, significantly higher rotor reduced frequencies up to 8 or even higher may occur when considering 20 MW wind turbines moving with typical wave frequencies. Therefore, the surge motion frequency range was extended to investigate whether transient aerodynamic effects may occur at very high motion
530 frequencies. Figure 12 shows the normalised rotor torque amplitude over b_{vel} comparable to Figure 10, but with a higher range of the rotor reduced frequencies and b_{vel}. As in the previous case, a linear trend between the amplitude and the motion to inflow velocity ratio is present. The grey line representing a rotor reduced frequency of 0.5 is approximately equal to the previous experimental and numerical results and can therefore be considered as quasi-steady. The slope of the linear relation
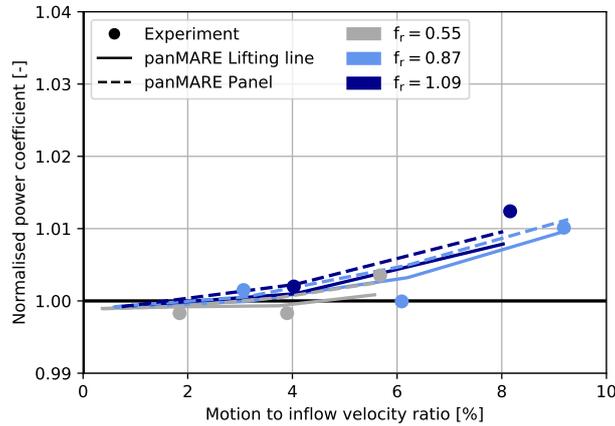
Figure 11. Measured and simulated average power coefficient during the surge motions plotted over the motion to inflow velocity ratio.
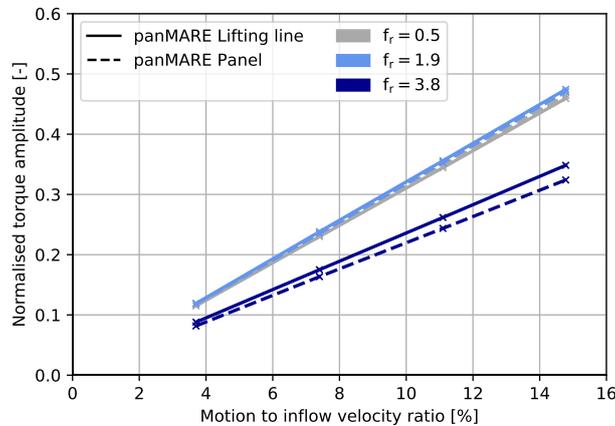
Figure 12. Simulated rotor torque amplitudes of the model rotor as response to the extended surge motions plotted over the motion to inflow velocity ratio.

slightly rises for an increased reduced frequency of 1.9. Interestingly, a further increase to a reduced frequency of 3.8 yields a
significant decrease of the slope. Finally, the results show a significant dependency of the rotor torque amplitude on the rotor
reduced frequency at the same motion to inflow velocity ratio b_{vel}. This can be considered as proof of a transient contribution
to the load response of the simulations.

In order to characterise this transient behaviour, a different kind of load case series is applied, which allows to investigate
the influence of the motion frequency independent from b_{vel}. In these series, b_{vel} is kept constant, while the motion frequency
varies (see Table 3). This results in a constant quasi-steady behaviour for every load case in the series while the influence of
transient phenomena may vary with the motion frequency. The load case with the lowest motion frequency is considered as

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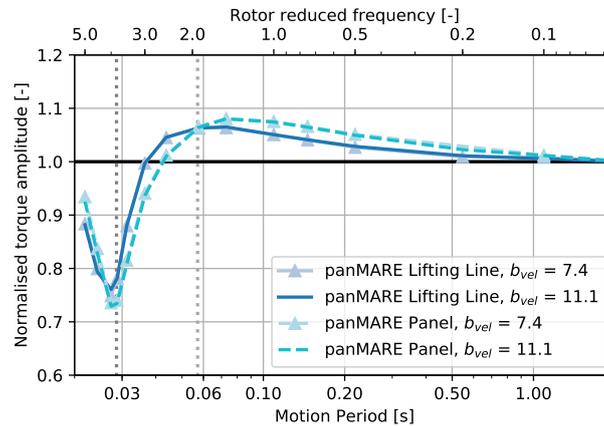
Figure 13. Rotor torque amplitudes of the model rotor from simulations as response to the extended surge motions plotted over motion period and rotor reduced frequency. Torque amplitudes are normalised to the case with the highest motion period. The full (dark grey) and half (light grey) blade passing periods are indicated by veritical, dotted lines.

an approximately quasi-steady case, because one surge motion period lasts as long as 35 rotor revolutions. The extremely low motion frequencies and their corresponding amplitudes therefore represent somewhat exaggerated cases, as such behaviour is not expected in reality.

545   In Figure 13, the torque amplitude is displayed over the motion period and rotor reduced frequency. Two series with exemplary motion to inflow velocities were simulated with the panel method and the lifting line method. For visual purposes, the x-axis is chosen to be logarithmic. All amplitudes are normalised to the quasi-steady case, which corresponds to the amplitude at a rotor reduced frequency of 0.02 (or a period of 2 s). This results in the fact that all curves reach 1 at this period. Starting on the right at the lowest reduced frequency, both methods show a slowly increasing torque amplitude up to a reduced frequency

550 of 2. A rapid drop of approximately 30 % occurs towards a reduced frequency of 4, which is followed by another increase. This behaviour clearly indicates the presence of transient aerodynamic phenomena. Otherwise, all curves would follow a constant trend over the rotor reduced frequency. From the displayed results it is difficult to conclude, which of the above described transient aerodynamic phenomena may cause such behaviour. However, the minimum amplitude is reached at the surge motion frequency that corresponds to the blade passing frequency (35 Hz i.e. a period of 0.0286 s). This provides a strong indication

555 that the returning wake effect is present.

    The returning wake effect can be modelled in a two-dimensional manner using the cascade approach, which was demonstrated for wind turbines by Eliassen (2015). Therefore, an equivalent two-dimensional setup approximately corresponding to the flow situation on a cylindrical surface with a relative radius of 62 % was created in *pan*MARE (see Figure 14). For

560 the sake of simplicity, only two airfoils representing the two blade sections were considered here. However, for a complete representation of the returning wake effect in such setup, multiple airfoils and their wakes would be necessary. The two airfoils
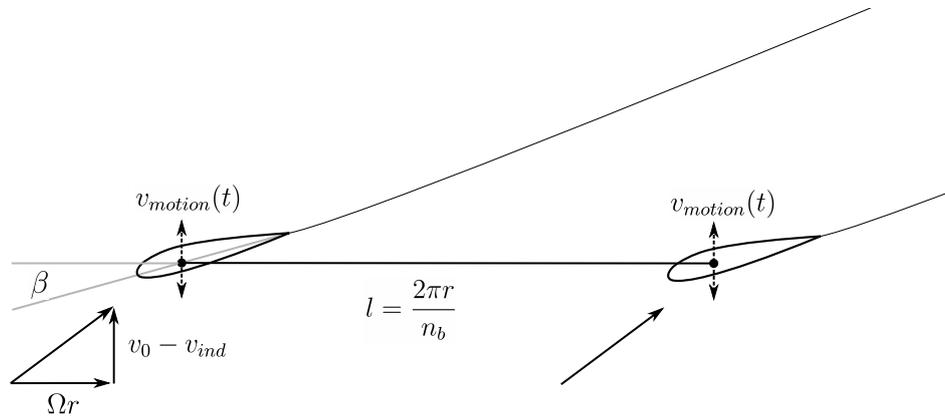
Figure 14. Schematic setup of the two-dimensional simulations of a blade section. The setup can be considered as unrolled version of the sectional view illustrated in black in Figure 2. l and β describe the distance between the two airfoils and the local twist angle at the corresponding rotor radius.

in Figure 14 represent cuts through the two blades at a constant radius as indicated by the black contours in Figure 2[6]. Their distance in lateral direction to each other equals the arc length from one blade to the other in the three-dimensional case. In consequence, the interaction of the shed vortices from the first blade with the lift force of the second blade at a given blade

565   section can be represented in this way. However, due to the limitation to two airfoils, only the vortices shed during one rotor rotation are considered in the two-dimensional simulations. The inflow speed and angle of attack of the airfoils approximately correspond to those in the three-dimensional simulations. The surge motion is represented as a harmonic vertical motion of the airfoil pair.

     The lift force amplitude of the second airfoil for different motion frequencies is shown Figure 15. In addition to that, the

570   results for the same setup with only one airfoil, where no returning wake effect is present, can be seen in the graph. Again, the amplitudes are normalised to the largest motion period (corresponding to the lowest rotor reduced frequency), which can be considered as a quasi-steady case. This assumption can be confirmed by the nearly horizontal slope of the amplitudes at the highest motion period. On the top border of the graph, a second scale indicating the airfoil reduced frequency is inserted. The lift amplitude of the single airfoil follows a decreasing trend from an airfoil reduced frequency of 0.02. This decrease is in

575   accordance with simulations presented by Leishman (2000) and can be explained by the circulatory unsteady airfoil effect. The slight differences between the panel method and the lifting line simulations may be caused by the insufficient representation of the blade utilising a point vortex in the lifting line method. When two airfoils are considered, the characteristic of the lift amplitude is similar to the torque amplitude of the rotor. The maximum is reached when the motion frequency equals half the blade frequency and a drop of around 30 % is present when the motion frequency equals the blade frequency. As the setup

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[6]The present analysis is limited to the influence of the shed wake of the first blade to the second blade during the first rotor rotation. However, due to the modelling constraints of the cascade approach, a third airfoil has to be placed in the row in order to evaluate the lift force of the second airfoil in a physically correct way (see Eliassen (2015) for details). This third airfoil is present in the simulations but not shown in the figure for the sake of simplicity.
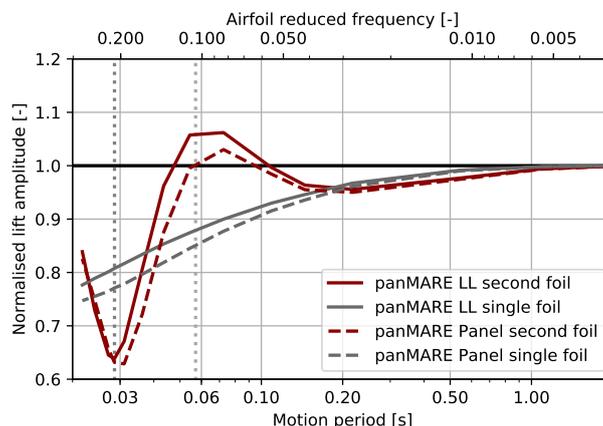
Figure 15. Lift amplitudes of an airfoil undergoing a plunging motion. Amplitude and frequency of the motions are chosen corresponding to the flow situation at a rotor radius of 62 % of the model rotor undergoing the surge motions presented in Table 3. Results are shown for the a setup with one and two airfoils. In case of two airfoils, the lift amplitude of the second airfoil is plotted. Lift amplitudes are normalised to the case with the highest motion period.

580   shown in Figure 15 corresponds to the schematic illustration in Figure 1, the occurrence of the maximum and minimum can be explained in the same way as done in section 2.

Finally, the characteristic of the rotor torque amplitude can be approximately reconstructed at a representative two-dimensional blade section. The occurrence of the minimum at the blade frequency can be clearly attributed to the returning wake effect. This observation provides a strong indication that the driving aerodynamic phenomena are the unsteady airfoil and the return-

585   ing wake effect. The less pronounced maximum may be explained by fact the torque is a the summation of the sectional load responses, which may not have the exact same characteristic at different blade spans. Another reason might be the limitation of the two-dimensional simulations to two airfoils.

It is unclear at this point whether or in which frequency range the dynamic wake effect comes into play. Regarding the two different motion to inflow velocity ratios tested, it seems that the chosen b_{vel} has negligible influence on the occurrence of the

590   transient aerodynamic behaviour, because nearly no difference between the corresponding curves is present. This is in line with the conclusions drawn from similar simulations of the FLOATGEN wind turbine using a panel method and a BEMT method (Schulz et al., 2023).

In Figure 16, the phase shift between the torque oscillation and the surge position is shown. In case of the rotor torque, an

595   initial phase shift of 90° is present at the lowest rotor reduced frequency, which means that the torque oscillation is in phase with the motion velocity. The phase shift stays constant up to a rotor reduced frequency of 1 and then increases until a maximum is reached slightly below 4. With further rising reduced frequency, the phase shift drops approximately 20°. Lifting line and panel method simulations agree well up to a reduced frequency of 3. At higher reduced frequencies, a difference of around
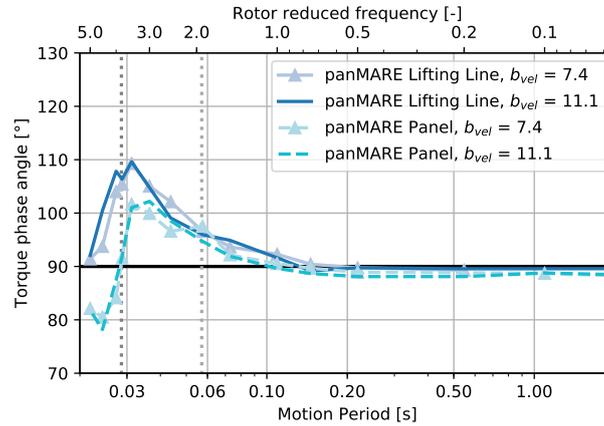
Figure 16. Phase shift between rotor torque and surge position correspiong to amplitudes of the model rotor shown in Figure 12.
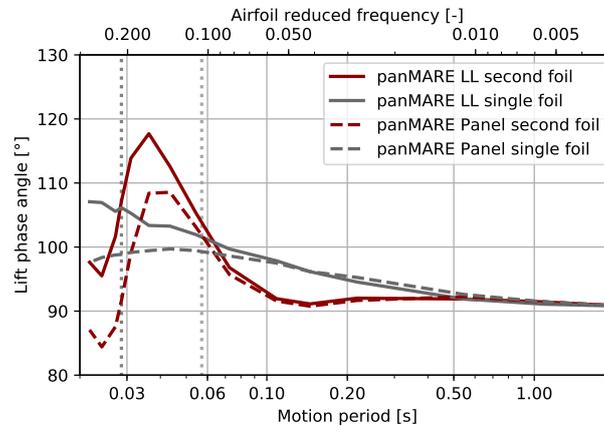
Figure 17. Phase shift between lift and surge position correspiong to amplitudes of two-dimensional simulations shown in Figure 15.

600 10° evolves. The characteristic behaviour is in line with the phase shift of the lift force in the two-dimensional case (see Figure 17), when the second airfoil is considered. Therefore, the characteristic of the phase can also be attributed to a superposition of the unsteady airfoil and the returning wake effect. Interestingly, the difference between the lifting line and the panel method simulations is also present in the simulations, where only one airfoil is considered. In this case, the unsteady airfoil effect determines the transient behaviour. The difference at higher airfoil reduced frequencies can most likely be attributed to the non-circulatory unsteady airfoil effect, which cannot be modelled by the lifting line method.

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When considering the thrust force oscillation caused by the imposed surge motion, a nearly identical behaviour can be observed in the graphs in appendix E. This is straight forward, because both, torque and thrust, are driven by the lift force fluctuations.

6   Simulation studies on the UNAFLOW and the IEA 15 MW rotors

610   In order to investigate whether the occurrence of the returning wake effect is caused by the individual characteristic of the TUHH model rotor or generally needs to be taken into account, the behaviour of two other exemplary rotors is analysed.

6.1   UNAFLOW rotor

The *pan*MARE lifting line model, which was set up in the OC6 project, is utilised to run a load case set with extended motion frequency range in comparison to previous studies. As most previous investigations on wind turbine rotors in surge motion

615   focus on the thrust force, the same is done in the following investigation. However, the behaviour of the torque amplitudes was found to be very similar and is shown in appendix F.

In Figure 18, the normalised thrust amplitude of the UNAFLOW rotor as a response to a series of surge motions with a constant motion to inflow ratio and varying rotor reduced frequency is shown. The motion to inflow velocity was chosen to be $b_{vel} = 5.28\%$, which corresponds to a load case from the measurement campaign. The load cases are listed in Table 3. As

620   in the previous figures, the amplitudes are normalised to the case with the lowest rotor reduced frequency i.e. highest motion period. It is shown, that the amplitude stays approximately constant in the range of low rotor reduced frequencies. From a rotor reduced frequency of 0.5, the curve rises again slightly and reaches a flat maximum at half the blade passing frequency followed by a rapid decrease with a minimum at the blade passing frequency. Apart from the slight decrease at low motion frequencies, a very similar behaviour as seen for the TUHH rotor can be identified. The same is true in case of the phase shift of

625   the thrust oscillation shown in Figure 19. However, the maximum phase shift is about $10°$ higher in comparison to the TUHH rotor. As the lifting line method showed an overprediction of the maximum phase angle in case of the TUHH rotor, it is likely that the same is true for the UNAFLOW rotor. Due to the characteristic behaviour of the thrust amplitude, it can be concluded that the returning wake effect also has a significant influence on the UNAFLOW rotor in surge motion.

A hint on such behaviour can also be found in Ribeiro et al. (2023). Panel method simulations showed a lower-than-expected

630   thrust amplitude at a rotor reduced frequency slightly below 5. This is in line with the decrease of the amplitude in the range of the rotor reduced frequencies between 3 to 7 shown in the present simulations. Ribeiro et al. assumed that this drop is caused by a varying phase shift of the lift force variation over the blade span (see section 3). However, from the two-dimensional investigations shown in the previous section and the characteristic occurrence of minimum and maximum it seems more likely that the returning wake effect induces this behaviour.

635   In the measurements performed during the UNAFLOW project, no such behaviour of the thrust amplitude was observed. This is in line with the present simulation results as the measurements in Mancini et al. (2020) were limited to reduced a maximum reduced frequency of 1.19. In this region, only limited influence of transient effects can be seen in Figure 18. Also
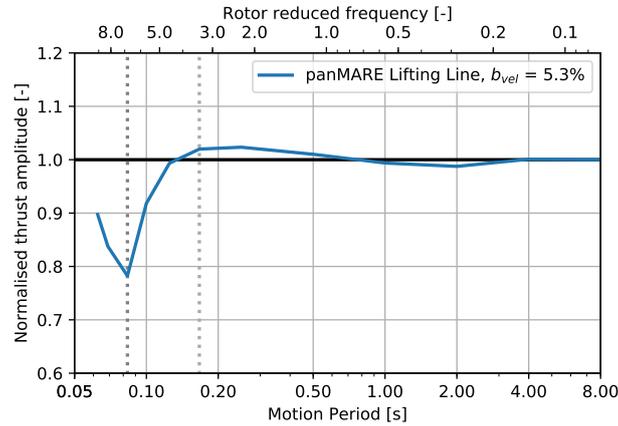
Figure 18. Rotor thrust amplitudes of the UNAFLOW rotor from simulations as response to surge motions plotted over motion period and rotor reduced frequency. Thrust amplitudes are normalised to the case with the highest motion period.
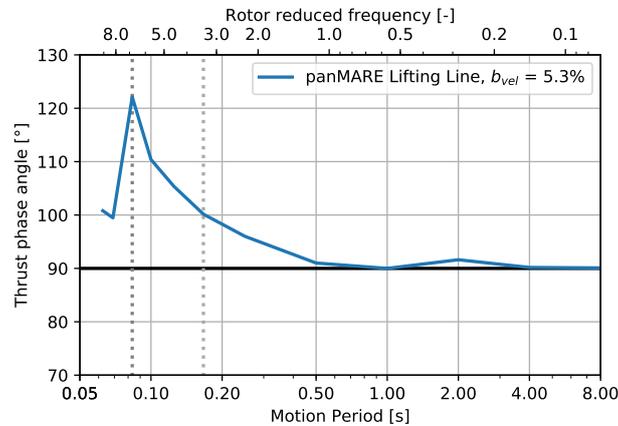
Figure 19. Phase shift between rotor thrust and surge position correspiong to amplitudes of the UNAFLOW rotor shown in Figure 18.

other previous simulation studies including OC 6 Phase III (Bergua et al., 2023) were limited to a similar motion frequency range, so that - apart from a slight phase shift - no transient behaviour could be noticed.

640   In contrast to this, unexpected thrust and torque amplitudes were observed during the COREWIND measurement campaign (Fontanella et al., 2022). At a rotor reduced frequency of approximately 2.5, these were approximately 30 % higher than expected from a quasi-steady model. As the rotor, operational conditions, wind speed and load case set are quite similar to the UNAFLOW campaign, it can be assumed that at least a similar characteristic behaviour of both rotors should be present. However, the present simulations seem to contradict the measurements. The reason for this is not quite clear and further

645   simulations on this specific rotor as well as continued experimental efforts may shed light on this interesting observation.
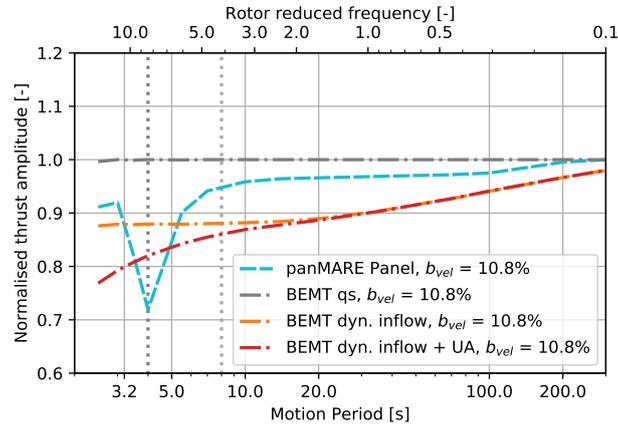
Figure 20. Calculated rotor thrust amplitudes of the IEA 15 MW as a response to surge motions plotted over motion period and rotor reduced frequency. Thrust amplitudes are normalised to the scenario with the highest motion period in case of the panel method and to the quasi-steady simulation in case of the BEMT.
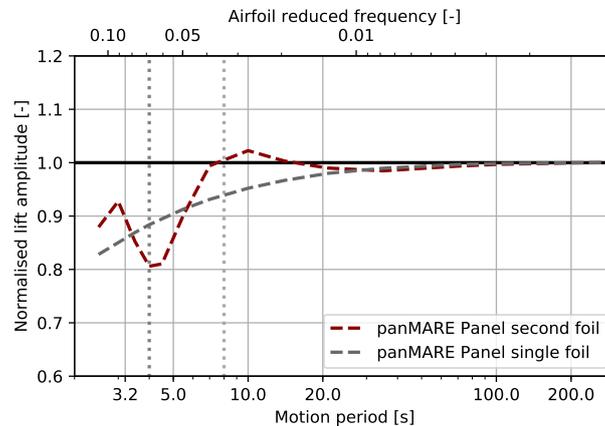
Figure 21. Lift amplitudes of an airfoil undergoing a plunging mnotion. Amplitude and frequency of the motions are chosen corresponding to the flow situation at a rotor radius of 64 % of the IEA 15 MW rotor undergoing the surge motions presented in table 3. Results are shown for the a setup with one and two airfoils. In case of two airfoils, the lift amplitude of the second airfoil is plotted. Lift amplitudes are normalised to the case with the highest motion period.

6.2   IEA 15 MW

The IEA 15 MW wind turbine is investigated as it is a representative for modern, large size wind turbines. A set of load cases with a constant motion to inflow velocity ratio and varying surge motion frequency is applied to the rotor (see Table 3). The thrust force amplitudes calculated by the panel method and the BEMT method over a wide range of motion periods are shown
650   in Figure 20. The characteristic change of the thrust force amplitude is similar to those of the model rotors. However, no distinct

maximum can be identified at the surge motion frequency, which equals half the blade passing frequency. In order to investigate this difference, two-dimensional flow simulations around the blade sections of the IEA 15 MW rotor at an exemplary relative radius of 0.64 were performed. The setup was chosen in the same way as in section 5.5. In Figure 21, the lift amplitude of the second airfoil as a response to the surge motion is shown. A characteristic similar to the one in the previous two-dimensional simulations can be identified. However, the minimum and maximum are less pronounced in this case. Similarly, the decrease of the lift amplitude in the case of a single airfoil is less strong. This can be explained by the lower range of the airfoil reduced frequency (see upper x-axis) in comparison to Figure 15. For example, at the blade passing frequency, the airfoil reduced frequency is 0.07 for the IEA 15 MW airfoil while it is 0.2 for the TUHH rotor airfoil[7]. The unsteady airfoil effect is therefore much stronger in case of the TUHH rotor at the same ratio of motion and blade passing frequency. As the returning wake effect is driven by the same vortices shed from the trailing edge of the upward blade, it is straight forward that it is also less pronounced in this case. The influence of the returning wake effect may be considered as a perturbation of the unsteady airfoil effect on the lift amplitude of an oscillating airfoil.

It has to be noted that the two-dimensional simulations only represent the flow at one specific rotor radius. Therefore, the lower height of the maximum peak in these simulations may be understood as a hint that the returning wake effect is less pronounced in case of the IEA 15 MW rotor, which most likely leads to the absence of a maximum in the rotor thrust amplitude. In addition, the two-dimensional simulations indicate that unsteady airfoil and returning wake effect already act on the sectional blade loads between 10 s and 20 s although the trend of the rotor thrust amplitude is nearly constant in this region. Therefore, a simulation method that only covers the unsteady airfoil effect, might predict erroneous amplitudes in this region.

The results of the BEMT simulations are illustrated by three different lines in Figure 20. A normalisation of the thrust force amplitudes to the quasi-steady results is applied. The BEMT without unsteady corrections (BEMT qs) shows a constant trend, which due to its quasi-steady nature. With the application of the dynamic inflow correction (BEMT dyn. inflow), the normalised amplitude shows a continuously decreasing trend with decreasing motion period. This decrease changes to a constant trend for motion periods lower than 20 s. The unsteady airfoil correction causes an additional decrease of the thrust amplitude at motion periods below 15 s. While the effect of the unsteady airfoil correction is similar to the trend of the single airfoil in Figure 21, the returning wake effect is naturally not captured by the unsteady corrections. Furthermore, it is remarkable that the dynamic inflow correction causes such strong decrease of the thrust amplitude, because this is not in line with the panel method. In addition, the panel method seems to converge against a quasi-steady value for periods above 200 s, while the dynamic inflow correction in the BEMT simulations still contributes to the amplitude at a period of 300 s. As the panel method inherently captures the dynamic inflow effect, it is likely that the dynamic inflow correction of the BEMT method may not be suitable in this case. The unsteady airfoil correction of the BEMT method seems to capture the general trend of the phase shift well, while the returning wake effect is naturally not captured.

In Figures 22 and 23, the phase shift of the rotor thrust oscillation and the lift force in 2D in comparison to the surge position are shown. Again, a similar characteristic can be identified.

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[7]Note that the airfoil reduced frequency is dependent on the chord length and consequently on the rotor radius. Therefore, this comparison is only valid because both simulations represent the situation at approximately the same relative radius.
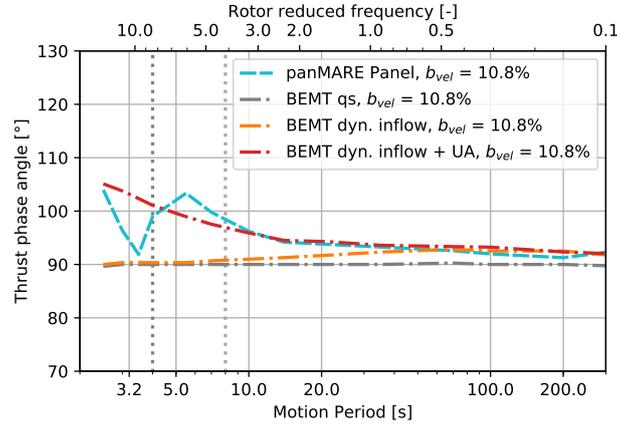
Figure 22. Phase shift between rotor thrust and surge position correspiong to amplitudes of the IEA 15 MW rotor shown in Figure 20.
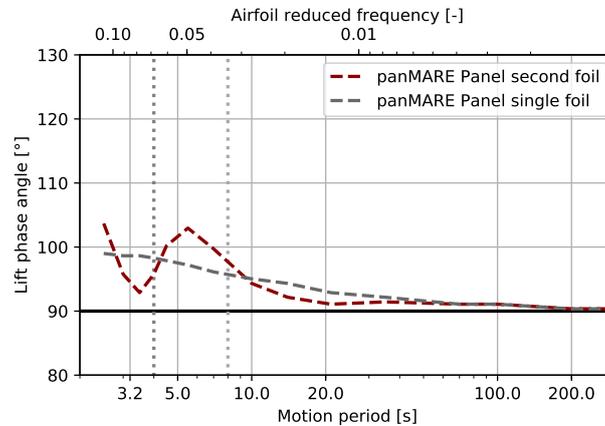
Figure 23. Phase shift between lift and surge position correspiong to amplitudes of two-dimensional simulations shown in Figure 21.

7 Characterisation and occurrence of transient aerodynamic phenomena

7.1 Characterisation

The unsteady airfoil effect and the returning wake effect could be identified as major drivers for transient contributions to the load response of surging wind turbine rotors. In particular, the circulatory part of the unsteady airfoil effect was dominant over the non-circulatory part. A significant contribution of the dynamic inflow effect could not be found in the free vortex wake simulations. However, no special investigations aiming at this were performed in this study.

690   The occurrence and strength of these transient aerodynamic phenomena can be estimated by the dimensionless numbers described in section 2. In case of the airfoil reduced frequency, it is difficult to find a representative value for the rotor as it varies over the blade span due to its dependency on the chord length. Transient contributions to the load response were identified at an airfoil reduced frequency of 0.02 and higher in the investigated cases. This is differs slightly from the limiting value (0.05) given by Leishman (2000). The strength of the returning wake effect is strongly influenced by the airfoil reduced frequency

695   and the ratio of surge motion frequency and blade passing frequency. In case the airfoil reduced frequency is sufficiently high, the returning wake effect comes into play when the motion frequency rises above half the blade passing frequency or even at slightly lower values. However, it seems that in practise, an airfoil reduced frequency of above 0.02 is a prerequisite for the occurrence of the returning wake effect. The strength of the returning wake effect is also affected by the distance between the blade sections and the wake sheet from the upward airfoil i.e. from the pitch of the helical wake.

700   Finally, an experimental wind turbine rotor would need to maintain similarity with all three dimensionless numbers from section 2 in order to model the occurrence of these transient aerodynamic phenomena comparable to a full scale turbine. The consideration of the rotor reduced frequency only is therefore not sufficient. This can be easily recognised by comparing the load response amplitudes of the three rotors (Figures 13, 18 and 20). The minima in these cases clearly not occur at the same rotor reduced frequencies.

705   **7.2   Occurrence in full scale**

In practise, the investigated pure surge motion is superposed with a rotor speed oscillation in below rated conditions. The rotor speed oscillation has shown to trigger a strong dynamic inflow effect even at low motion frequencies in a previous work (Schulz et al., 2023). However, the observed phenomena will most likely also occur, when a concurrent rotor speed variation is present. The returning wake effect may occur at large turbines in low wind speed conditions, where half the blade passing frequency

710   may be in a range of the wave spectrum. As seen in the case of the IEA 15 MW at $7\,\mathrm{m\,s^{-1}}$ wind speed, the returning wake effect impacts the sectional lift force at a motion period of $10\,\mathrm{s}$. This is within the range of typical wave periods. However, a prediction of the quantitative impact of the returning wake effect from the dimensionless numbers is challenging and therefore exemplary FVW simulations at low wind speeds might give a more reliable insight.

8   Summary and conclusions

715   An experimental and numerical investigation on the occurrence of transient aerodynamic phenomena caused by a surge motion of a wind turbine rotor is presented. The investigation is limited to aerodynamic loads and basic surge motion sceanrios excluding rotor speed variations or controller actions. To impose a surge motion to the two-bladed TUHH model rotor, a novel test rig was designed. The surge motion is applied to the nacelle only, which allows for a very rigid tower design. In addition, the load sensor was placed in the shaft of the rotor to reduce undesirable inertia forces in the measurement signal. The

720   results of FVW lifting line and panel method simulations utilising the software *pan*MARE were compared against those of the measurements. In a second step, the simulations were extended to a frequency range exceeding the capabilities of the test rig

and certain load cases were investigated to distinguish between the quasi-steady and the transient part of the load response to a surge motion. In order to proof the generality of the findings, the load case set was additionally applied to the UNAFLOW and the IEA 15 MW rotors. Additional BEMT simulations of the IEA 15 MW rotor with and without unsteady corrections were
725 conducted. Finally, a characterisation of the observed transient aerodynamic phenomena is given and the applicability of three dimensionless numbers to identify their occurrence is discussed.

The most relevant findings from the numerical and experimental investigations can be summarised as follows:

– Despite the comparatively rigid tower design and two different approaches to remove inertial loads from the axial force measurements, a sufficient accuracy of the compensated rotor thrust force could not be achieved due to forced vibrations
730 of the tower top.

– Slight oscillations of the rotor speed were found to have major influence on the measured rotor torque. Therefore, an inertia removal procedure was developed and applied. The compensated measurements proofed to have a low repetition error and very good agreement with the FVW simulations. This is the case for steady and unsteady scenarios.

– The measured torque oscillation amplitudes during surge motions were found to be determined by the motion to in-
735 flow velocity ratio b_{vel} and can therefore be considered as quasi-steady in the motion frequency range covered by the experiments.

– The extension of the motion frequency range in the FVW simulations revealed a significant influence of the returning wake effect on both amplitude and phase of the torque and thrust force. Therefore, transient aerodynamic phenomena are clearly present in the simulations.

740 – The occurrence of the returning wake effect could also be proofed in case of the UNAFLOW and the IEA 15 MW rotor. In case of the IEA 15 MW, transient contributions to the torque and thrust response were found within the frequency range of common seaways. The returning wake effect had a clear influence on the airfoil loads at motion frequencies in the range of half the blade passing frequency and higher. A prerequisite for its occurrence seemed to be an airfoil reduced frequency of more than 0.02.

745 – The transient aerodynamic behaviour was determined by the circulatory unsteady airfoil effect and the returning wake effect in the investigated cases. A contribution of the dynamic inflow effect was not investigated in detail. Therefore, a proper scaling of these effects would need to maintain similarity between the corresponding dimensionless numbers described in section 2. Utilising the rotor reduced frequency only for this purpose was found to be not sufficient.

– BEMT simulations with and without common unsteady corrections were naturally unable to capture the returning wake
750 effect. Instead, an impact of the dynamic inflow effect was present at comparatively low motion frequencies i.e. rotor reduced frequencies. This is not in line with the FVW simulations and may indicate a limited suitability of the dynamic inflow correction in the investigated cases.

It has to be noted that the above mentioned findings regarding the transient aerodynamic phenomena are based on two FVW methods that utilise the same software routines in major parts. In addition, attached flow conditions were assumed in all cases. In case of the IEA 15 MW rotor, this might have deviated the results slightly. Therefore, all findings need to be evaluated using other FVW methods, higher fidelity simulations and experiments. Special attention should be paid to a comparison of high fidelity simulations and/or experiments with the COREWIND measurements, where a contradicting trend in comparison to the present simulations was found.

In practise, the returning wake effect is most likely present in case of large rotors and low inflow wind speeds. Most state-of-the-art BEMT methods are not capable of modelling the returning wake effect. Therefore, FVW simulations are strongly recommended to check whether its influence is significant and the application of such methods is valid in these cases.

Appendix A: Derivation of equation (3)

In section 2, the amplitude of the rotor power oscillation caused by a harmonic surge motion at constant wind speed and constant rotational speed is stated (see equation (3)). As a basis, the well-known empirical formulation for the steady power of a wind turbine is adopted to a sinusoidal variation of the inflow speed in equation (1), where potential transient effects are summarised in the extra term $P_{trans}(t)$. When introducing the dimensionless number b_{vel}, equation (1) can be rewritten as:

$$P(t) = \frac{\rho A v_0}{2} C_p\big(\lambda(t)\big)\big(1 + b_{vel} sin(2\pi f t)\big)^3 + P_{trans}(t). \tag{A1}$$

Furthermore, the quasi-steady amplitude of the rotor power can be computed by the difference of the minimum and the maximum power during a surge motion cycle, which occur at minimum and maximum motion velocity.

$$P_{amp} = \underbrace{\frac{\rho A v_0^3}{2}\frac{1}{2}C_{p,v_{max}}(1 + b_{vel})^3 - C_{p,v_{min}}(1 - b_{vel})^3\big)}_{\text{quasi-steady}} + \underbrace{\Delta P_{trans}}_{\text{transient}}. \tag{A2}$$

As the steady power coefficient is determined by the actual inflow velocity and the rotational speed, its values at the higest and lowest motion velocities can be determined simply.

$$C_{p,v_{max}} = C_p\big(v_0(1 + b_{vel}), \Omega\big) = C_p(\lambda_{min}) \tag{A3}$$

$$C_{p,v_{min}} = C_p\big(v_0(1 - b_{vel}), \Omega\big) = C_p(\lambda_{max}) \tag{A4}$$

λ_{min} and λ_{max} correspond to the minimum and maximum TSR that is reached during a surge motion cycle. When finally introducing equations (A3) and (A4) into (A2), equation (3) results.

For small b_{vel}, C_p may be assumed to be constant. In this case, equation (3) can be simplified to:

$$P_{amp} = \frac{\rho A v_0^3}{2} \frac{1}{2} C_p(\lambda)(3b_{vel} + b_{vel}^3) + \Delta P_{trans} \tag{A5}$$

780   As the third power of b_{vel} may be disregarded in this case, a linear trend of the power amplitude with b_{vel} results in the quasi-steady case. Due to the constant rotor speed, the torque amplitude is linearly dependent on the power amplitude. Analogously, the thrust amplitude can be expressed as:

$$T_{amp} = \frac{\rho A v_0^2}{2} \frac{1}{2} C_t(\lambda)(2b_{vel} + b_{vel}^2) + \Delta T_{trans} \tag{A6}$$

However, the limitations of these simplifications need to be evaluated in the considered application case.

785   **Appendix B: Lift correction in *pan*MARE**

Panel methods often suffer from an overprediction of the lift force, when airfoils at low Reynolds numbers are considered. In case of the SD7062 airfoil used for the TUHH model rotor at a Reynolds number of $125\,\mathrm{k}$, this overprediction can be approximated as a constant offset up to an angle of attack of $5°$ (see Figure B1). The angles of attack along the blade span during the surge motion tests range from approximately $1°$ to $4.5°$ at $90\,\%$ of the blade. In this region, an offset of the angle of

790   attack of about $0.8°$ is sufficient to correct the overprediction of the lift coefficient (see 'panMARE panel, corrected' in Figure B1), which is applied in the presented simulations of the TUHH model rotor.
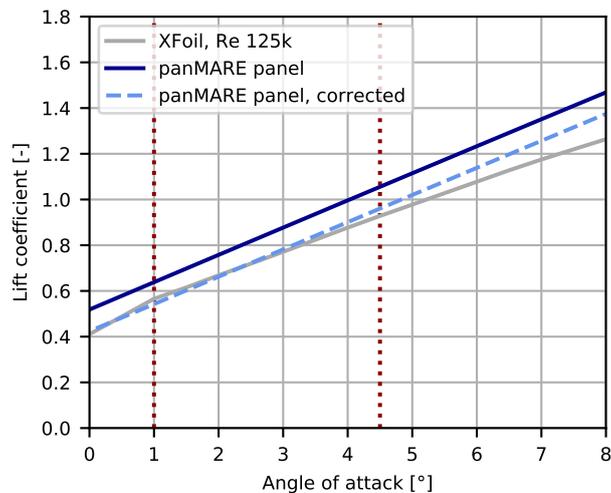
Figure B1. Lift coefficient of the SD7062 airfoil computed with XFoil and *pan*MARE in comparison with the applied correction. Dotted red lines indicate the range of realistic angles of attack of the TUHH model rotor in the present operating conditions.

Appendix C: Data acqusition

C1   Actuation and sensing system

The six-component force-moment sensor K6D40 supplied by ME-Meßsysteme GmbH was mounted inside the rotating shaft
795   and in front of the the generator bearings. Apart from a thin cable connection for the power supply of the measurement amplifier
and sensor, no other loads than aerodynamic and inertia loads were measured by the sensor. The miniature measurement
amplifier GSV6-BT (ME-Meßsysteme GmbH) has been placed inside the rotating hub and was connected to the force-moment
sensor, while sending the acquired measurement data to the main measurement computer via a wireless Bluetooth connection.
The power supply cable for the rotating hub was placed inside the hollow shaft of the rotor and connected to a slip ring on the
800   backward end of the nacelle.

The Kollmorgen brushless DC-Motor TBMS-6051 was utilised as a generator. The signals of the three inbuilt hall sensors
were fed into the Kollmorgen AKD-P01206 Motor controller and in parallel into the HBM measurement amplifier.

Nacelle accelerations were measured with a TE Connecticity A3S 4030 tri-axial acceleration sensor, which was cable-
connected to the HBM measurement amplifier.

805   The LinMot MS01-1/D magnetic position sensor was mounted on the backward end of the nacelle while the magnetic band
was placed on the steel frame of the tower top. It was used for a closed loop control of the linear motor as well as the position
measurement via the HBM measurement amplifier. The magnetic linear motor LinMot PS01-37x120 was used as actuator to
apply the surge motion of the nacelle. A feedback control with the position sensor was realised with the LinMot C1100-GP
motor driver.

810

Due to the limited data rate, the sampling frequency of the wireless measurement amplifier was limited to $600\,\mathrm{Hz}$. A low
pass filter (IIR) with an edge frequency of $200\,\mathrm{Hz}$ was applied to the measurement signal before transmitting it. Torque mea-
surements were additionally filtered as described in section 5.

For hall sensors, position sensor and acceleration sensor, a sampling frequency of approximately $10\,\mathrm{kHz}$ was chosen.

815   C2   Reconstruction of rotational speed

The rotational speed signal was reconstructed from the signals of three inbuilt, equidistant hall sensors inside the motor. The
sensors deliver a rising edge in the signal, when an electrical pole of the motor crosses it. Due to the presence of 6 pole pairs, 6
rising edges can be detected during one rotation in one sensor signal. The rotational speed can therefore be calculated from the
fact that a rising every edge occurs $60°$ of rotation and the time between two rising edges. As the signal takes approximately
820   two times steps to rise at the sampling rate of $10\,\mathrm{kHz}$, the point in time, when the signal starts to rise is detected via linear
interpolation between the samples. As a consequence, the resolution of the rising edge detection can be considered as about 10
times higher than the sampling resolution.

C3 Determination of the mass moment of inertia of the rotor

The mass moment of inertia of the rotor and hub geometry was determined experimentally. When rotor torque and rotational

825   acceleration is known while no aerodynamic load is applied, the mass moment of inertia can be calculated by the following equation.

$$J = -\frac{Q_{meas}(t)}{\frac{d\Omega}{dt}}. \tag{C1}$$

The rotor was accelerated with a constant torque from 0 to 300 RPM. A linear fit of the approximately linear rotor speed increase between 200 and 300 RPM was computed. The steepness gave the average angular acceleration during this process.

830   Concurrently, the measured rotor torque, which was approximately constant, was averaged over the same period of time. Finally, the mass moment of inertia was computed by the average of 4 repetitions of this procedure and the above equation. In order to validate the result, the same procedure was repeated with a higher and a lower level of torque yielding a faster and a slower acceleration of the rotor. A maximum deviation of 1.4 % between the results was reported.

Appendix D: Sources of random and systematic uncertainty

835   The rotor torque amplitude consists of the direct torque measurement and the inertia compensation procedure. Major sources of random and systematic uncertainty of the relevant quantities for the measurement of the rotor torque amplitude are listed below.

Uncertainty of the direct torque measurement

On the torque axis, the force-moment sensor has an uncertainty of $0.008\,\mathrm{N\,m}$ (with a confidence level of 95 %), which cor-

840   responds to below 1 % of the rated rotor torque. As the sensor is mounted with a fit into the rotor shaft, inaccuracies caused by the mounting position and angle can be considered as negligible. A few very thin wires for power transmission connect the measuring side of the sensor to the generator side. A change of the parasitic moment induced by the wires during one measurement cycle may result in a minimal contribution to the random uncertainty.

Uncertainty of rotor speed measurement

845   Sources of random uncertainty in the rotor speed measurement are any geometrical inaccuracies of the placement of the motor poles and hall sensors as well as the rise characteristics of the hall sensors. These uncertainties do not result in systematic errors, because the influence of these parameters on the rotor speed measurement will be the same in any rotation cycle. A systematic influence on the phase averaged rotor speed amplitude would therefore require the rotor rotation and surge motion to be phase locked, which is not the case.

850   The measurement of the time between two rising edges due to the sampling is the most crucial contributor to the random uncertainty (see dots in Figure 8). As this time is in the magnitude of $\frac{1}{105Hz}$, and can be detected with a resolution of approximately $\frac{1}{10*10kHz}$, an error of approximately 0.1 % for every measurement point can be expected. Due to the fact that the measured peak-to-peak rotor speed oscillations range from 0.2 to 0.6 % of the rated rotor speed, this is quite significant.

However, it can be assumed that this error is of purely random nature and is therefore drastically reduced due to the phase averaging.

A systematic influence on the rotor speed amplitude measurement is that the actual rotor speed is averaged over 1/6 of a rotation. This results in the fact that magnitudes of maxima and minima are slightly underpredicted, because the rotor speed is averaged over a phase range of 60°. For the highest motion frequency, this effect yields a systematic underprediction of the rotor speed amplitude of approximately 2 %.

Uncertainty from synchronisation of rotor speed and torque measurement

Rotor speed and direct torque measurements were acquired with two different measurement amplifiers. A synchronisation was achieved by a comparison of the rotor load in wind direction (including inertia) and the nacelle acceleration. Both signals were time-shifted so that the onset of the motion occurs at the same time in both signals. The accuracy of this procedure is difficult to evaluate, but is assumed to be maximum 2 % of the motion period. This results in a maximum phase shift of 7° between rotor speed and direct torque measurements. As the contribution of the direct torque measurements to the rotor torque amplitude is below 20 %, the influence of this phase shift is assumed to be comparatively small.

Uncertainty of the determination of the mass moment of inertia

The two sources of uncertainty are the rotor speed and the rotor torque measurement (both discussed above). In both, no significant systematic uncertainty is expected. As the measurements are repeated only 4 times, the above described sources of random uncertainty did not completely average out. The procedure resulted in a maximum deviation of 1.4 % between the results for three independent runs with different rotor accelerations.

Uncertainty of position and velocity amplitude measurement

For the reproduction of the measurements with numerical model, the measurement of the actual motion amplitude is as crucial as the torque measurement itself. The positioning sensor has an overall uncertainty of $0.01\,\mathrm{mm}$. This is about 0.2 % of the lowest motion amplitude. However, the most crucial uncertainty comes from the deviation of the actual motion trajectory between different motion cycles. Deviations of about 3 % have been observed when considering approximately 15 motion cycles. These deviations have been accepted and no detailed analysis on the consequences of these was performed.

Finally, a theoretical investigation of the combined systematic and random uncertainty of the rotor torque, surge motion and data processing with a sufficient level of detail was found to be very complex and prone to errors. However, an experimental investigation of these uncertainties would have required the construction of an elaborate calibration rig. This was out of scope of the current work. Therefore, the authors had to rely on the above theoretical analysis, where systematic errors are shown to be generally in a low single-digit percentage range. In addition, a hint that the random uncertainty is also within a low single-digit percentage can be concluded from the comparison of the shown repetition results.

885   **Appendix E: Thrust force amplitudes of the simulation study of the TUHH model rotor**
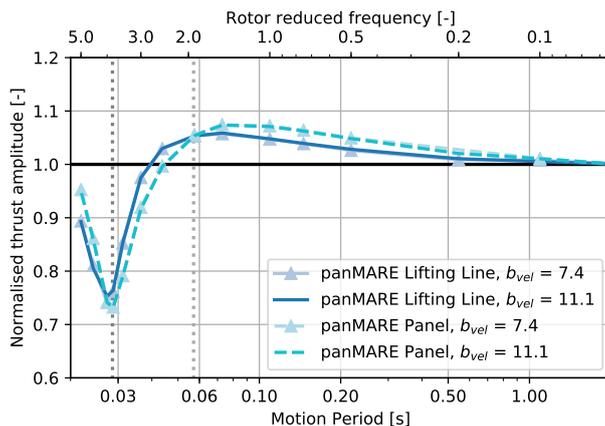
Figure E1. Aerodynamic rotor thrust amplitudes of the model rotor from simulations as response to the extended surge motions plotted over motion period and rotor reduced frequency. Thrust amplitudes are normalised to the case with the highest motion period.

Appendix F: Torque amplitudes of the simulation study of the UNAFLOW and the IEA 15 MW rotor
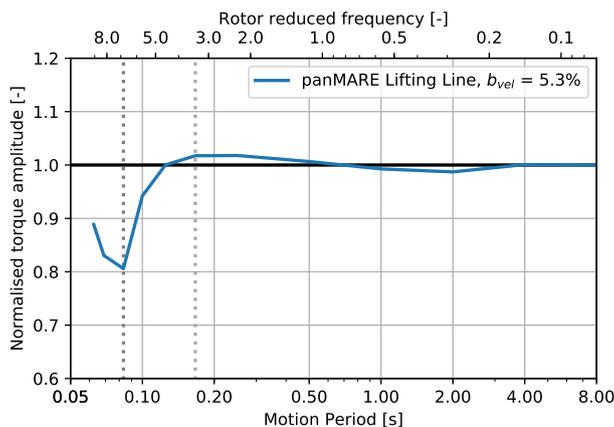
Figure F1. Aerodynamic rotor torque amplitudes of the UNAFLOW rotor from simulations as response to surge motions plotted over motion period and rotor reduced frequency. Torque amplitudes are normalised to the case with the highest motion period.
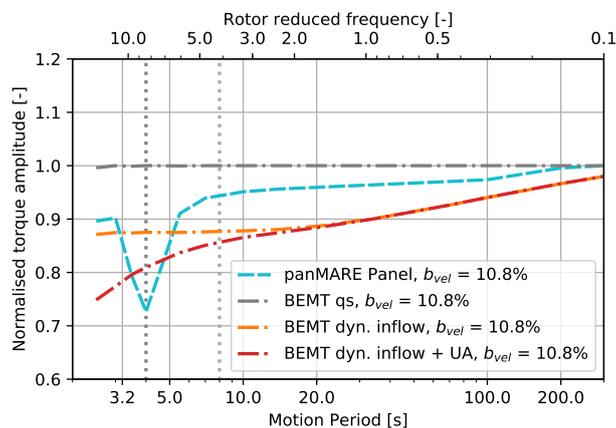
Figure F2. Aerodynamic rotor torque amplitudes of the IEA 15 MW as a response to surge motions, plotted over motion period and rotor reduced frequency. Torque amplitudes are normalised to the scenario with the highest motion period in case of the panel method and to the quasi-steady simulation in case of the BEMT.

Author contributions. CWS planned, designed and conducted the experiment, which includes the turbine design and data processing. In addition, CWS developed the load case sets, performed the FVW simulations, led the analysis of numerical and experimental results and wrote the mauscript. UÖ performed OpenFAST simulations, supported the analysis of the simulations and reviewed the manuscript. SN
890   implemented the extension of *pan*MARE for wind turbines, supported the FVW simulations, contributed to the analysis of their results and reviewed the manuscript. PWC contributed to the theoretical discussions, supervised the works of UÖ and aqcuired funding. MM contributed to the theoretical discussions, supervised all works of CWS and SN, reviewed the manuscript and acuried funding.

Competing interests. The authors declare no competing interests.

Acknowledgements. The authors kindly thank the German Federal Ministry for Economic Affairs and Climate Action (BMWK) for funding
895   the HyStOH project (03SX409A-F) and the VAMOS project (03EE2004A-C). Special thanks go to our partners CRUSE Offshore GmbH, aerodyn engineering GmbH, JÖRSS-BLUNCK-ORDEMANN GmbH, DNV-GL and the Institute for Ship Structural Design and Analysis at the Hamburg University of Technology for the excellent cooperation in the HyStOH project as well as the partners from the VAMOS consortium, BW IDEOL, Ecole Centrale de Nantes, sowento GmbH, UL International GmbH, and DNV GL. The authors greatly appreciate the work of Klaus Wieczorek to support the experimental investigations.
900   Publishing fees supported by Funding Programme Open Access Publishing of Hamburg University of Technology (TUHH).

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