

Dear reviewers,

The authors would like to thank you for the time and effort that you have dedicated to providing valuable feedback on our manuscript. We have been able to incorporate changes to reflect your suggestions. Please find a point-by-point response to your comments and a version of our manuscript highlighting all made changes below.

Reviewer 1

1. Refer to the Table 2: The scaling factor for the rotor diameter D is 133.33, while for the root radius it stands at 50. Drawing from my experience, I speculate that the difference in the scaling factor could stem from the necessity to accommodate electronics within the rotor root. This adjustment may have disrupted the maintenance of a consistent scaling factor. However, such intricacies might not be readily apparent to individuals not directly involved in the experimental process. Hence, I advise to present the rationale behind the difference in the scaling factor. Furthermore, it is important to address the potential ramifications of this difference on the ultimate conclusions drawn from the study. To bolster these assertions, referencing relevant studies would provide additional support to your arguments.

Your speculations are correct. This region requires mechanical parts strong enough to carry centrifugal loads and any imbalances in the rotor. Therefore, the mechanical components cannot be scaled by the same factor as the rotor itself. A study of existing literature reveals that the root radius is almost never reported and/or the chord distribution is plotted all the way to $r/R = 0$, which is unrealistic. One example reporting the root radius is the work by Fontanella et al. (2022). Here, a ratio of root to tip radius of 0.075 is reported, while ours is 0.067. We have referenced their work, see Page 5, Line 244. Regarding the impact on the studies result, see our answer to your next comment, which we feel is closely related.

2. Refer to the line 100: In the study, the authors noted a manual reduction of the chord to 4cm below $r/R = 0.25$. However, the term "manually" lacks clarity regarding its intended meaning. It appears to indicate a deviation from utilizing the scaling law specified in equation 3. If this interpretation holds true, what alternative method did the authors employ? Furthermore, what factors contribute to their confidence that the results would not significantly diverge from those obtained by precisely following equation 3? The authors are encouraged to expound upon their approach, providing support from relevant literature.

We agree that the term manually is not precise enough here. We used a cubic spline to reduce the chord to the cylindrical root region, as explained now on Page 6, Line 273. This reduction of the chord is common procedure for wind tunnel experiments. While not always explicitly reported, it can be deducted from images/graphs that this is done for most of the papers referenced in our literature review. It is expected to have little impact on rotor aerodynamics since the aerodynamic forces in this region are low when compared to the outboard region. This line of argumentation has been added to the text as well.

3. Upon re-deriving equation 6, I discovered that it was not that straightforward. The derivation of this equation entails several algebraic steps. Given the significance of this equation in determining the blade's twist angle, it would be beneficial to include its derivation in either the main text or an appendix. Additionally, I recommend using the term "twist angle" to define β , as "pitch angle" typically denotes the rotation of the entire blade at the root along the blade axis with respect to the rotor plane.

Thank you for this comment. We can see that the derivation of the equation should be made more clear to the reader. Therefore, we have added additional algebraic steps between Page 6, Line 278 and Page 7, Line 292. Furthermore, we have changed the definition of β in the nomenclature to twist angle.

4. Refer to the line 112: The authors are requested to precisely indicate the simulations from which the parameter values were extracted. Furthermore, it is advisable to provide insight into the reasons behind selecting those specific papers and elaborate on their relevance to the current study.

The simulations were conducted using an inhouse BEM algorithm developed by the authors. The algorithm has been

45 used in published work to simulate the IEA 15 MW RWT with straight and swept blades. In that work, the straight blade simulations were validated against the established TNO-inhouse lifting line algorithm AWSM. Thus, we have confidence in the accuracy of the numerical results used in the scaling approach presented in this study. We have specified the used algorithm in the text, see Page 7, Line 296.

50 5. Refer to lines 130 and 131: Please specify the value of the time delay and the way the authors calculated it. The time delay value was 41 ms. It was determined (rather than calculated) by comparing images captured during turbine operation to standstill images with the blade fixed in the desired position. By conducting this comparison close to the tip, where the sensitivity to the time delay is largest due to the high rotational velocity, we ensured that all measured cross-sections were in the desired rotor position. We have added this information to the text, see Page 8, Line 320.

55 6. Suggestion for Equation 14: It is noted that you have presented the equation in vector form. For mathematical accuracy, it would be more appropriate to use $n \cdot (u - u_B)$ instead of $n(u - u_B)^T$. This adjustment should also be applied to the third term. While Euclidean inner products, which are relevant in this context, can be calculated as the matrix product of a row and column vector, they are distinct concepts in precise mathematical terms. Thank you for pointing this out. We have applied the suggested changes to Equation 14. We have additionally adjusted Equation 15 to be aligned with the original paper by Noca.

60 7. Refer to the Figure 12:

(a) Blade 3: If the airfoil maintains a consistent shape across all sections, one would expect identical lift coefficients (c_l) for a given Angle of Attack (AOA). However, a difference is observed in the initial two points, suggesting other factors at play. It would be beneficial to explore potential causes for these differences. On a positive note, it is commendable that the remainder aligns well with the design lift curve.

65 (b) Blade 2: The observation that the lift coefficient (c_l) remains constant despite increasing AOA, especially at lower angles, is unexpected. This outcome warrants further investigation to determine the underlying cause, as it does not closely adhere to the anticipated design lift curve, leading to some reservations about its accuracy.

70 (c) Blade 1: This blade shares the same issue as Blade 2, with an added discrepancy at a 6° AOA, where the c_l value significantly deviates from the design lift curve. For Blades 1 and 2, it is plausible that the airfoil's shape diverges from the standard SD7032 profile, possibly in the curvature at the leading edge. Such variations might also stem from inadequate resolution in the velocity field analysis.

Thank you for this comment. We agree with your observations that some experimentally determined c_l values deviate from the design polar or from their expected relative position to other measurement points. We believe there are two major factors influencing these results.

75 Firstly, the Reynolds number varies along the span such that a comparison between the experimental data and a design airfoil polar at a single Reynold number can only be drawn based on trends. This was also pointed out by your fellow reviewer, so please also refer to Reviewer 2, Comment 6 and our corresponding answer. The additional analysis proposed by Reviewer 2 led us to the new insight that there is a slight deficit in lift production between the blades used in this experiment and the design airfoil's polars in the root and tip region.

80 Secondly and as you mentioned already yourself, some of these discrepancies are likely due to small deviations in geometry that might have been introduced during the manufacturing process. Furthermore, the airfoil model on which the SD7032 polars were measured had a chord of 130 mm while the majority of our measurements are taken at sections with chord lengths smaller than 50 mm. With such dimensions, small errors in geometry have a relatively larger impact on the results. Lastly, potential differences in surface finish could also play a role. Based on these arguments, we believe that our analysis, while acknowledging existing deviations, still demonstrates a high level of accuracy in a challenging experimental setup.

85 8. Refer to lines 314 and 315: This scaled-down version of the original wind turbine (WT) replicates the non-dimensional thrust by design, as it was developed based on thrust similarity principles. However, the scaled-down WT is tailored to

90 a specific thrust configuration, determined by factors such as blade pitch angle (where pitch angle refers to the angle
by which the entire blade rotates around its axis relative to the rotor plane). Altering the pitch angle of blades in the
original WT would necessitate a redesign of the scaled WT. For instance, consider the minimum function defining
the twist angle β (as shown in equation 6). Changing the inflow angle to the blade through pitch adjustment would
consequently alter the radial twist distribution in the scaled-down WT. It's important to exercise caution, as blade shapes
95 and types differ between the original and scaled-down WTs, resulting in varying Reynolds numbers. Consequently, flow
physics, such as flow transition and separation at a given radial position (r/R), differ at the airfoil level. Therefore, while
global aerodynamic properties like non-dimensional thrust may exhibit similarity, it's essential to recognize that local
aerodynamics at each radial position between the original and scaled-down WTs may not necessarily align.

We fully agree with your argumentation. We felt a statement along these lines was best placed at Page 8, Line 301 rather
than in the conclusions section and hope you agree with this assessment.

100 Reviewer 2

1. In the introduction the authors mention other scaled experiments with e.g. the DTU 10 MW model turbine. There are
also a few experiments with a model turbine scaled down from a 5MW reference turbine (<https://doi.org/10.5194/wes-6-1341-2021>). In these experiments they also use the method by Herreaez to determine the axial and tangential induction
factor along the rotor blade.

105 Thank you for pointing out this gap in our literature review. The research done at ForWind is relevant to this study and
should be mentioned in the introduction. We have added references to relevant publications, see Page 4, Line 209. Since
you mention their use of the Herreaez method, let me explain two reasons why we did not choose this method. Firstly,
we were interested in deriving blade loads from the flow field around the blade. Thus, we would have had to take twice
as many measurements (or introduce a second measurement technique like LDA) if we also wanted to capture the flow
110 in the bisectrix. Secondly, we ran an analysis very similar to the one presented here on a turbine with swept blades
(currently under review: <https://doi.org/10.5194/wes-2024-11>). For swept blades, the symmetry condition of the Herreaez
method is not given anymore, and the method would fail. To ensure comparability between the two campaigns, we went
with a method that was applicable to both.

2. Figure 3 can easily be removed since it doesn't provide any extra information for the rest of the paper.

115 You are right. Upon reviewing this section, we agree that the figure indeed adds little content. We have removed the
figure and the reference to it, see Page 8, Line 306.

3. On page 9 line 150 the authors state that in the post-processing they stitch together the results from the pressure and the
suction side of the profile to get the total flow around the profile. Are these images averaged over several events or are
they temporally highly resolved? In the later case, can the authors say anything about variations in the flow field on the
120 upper and lower side for measurements between single rotations?

The former is the case. We first individually average the pressure and suction side flow fields over the 120 phase-locked
images taken at these locations. Then, the two average flow fields are stitched. We have adjusted the text to be more
precise about this procedure, see Page 9, Line 342.

4. Line 200, page 11: The model turbine was running at constant rotational speed. Was that actively controlled or is that
given due to the fact that the inflow was constant ? I can image, that there might have been some fluctuations in the
125 rotational speed also due to the fact that the blades were all different. Passing the tower will cause different aerodynamics
for each of the blades which could be seen in fluctuations in the rotational speed. Did the authors observe something like
that ?

130 Indeed, we actively control the rotor speed. Instead of a pitching mechanism to regulate torque and rpm, the turbine is
driven by a motor that closely follows the set rotational speed. In the future, we hope to expand the turbine's capability
to be driven by its aerodynamic torque rather than a motor. We have added this information, see Page 8, Line 307. Given
this setup, we did not observe azimuth-dependent variations in the rotor speed.

- 135 5. In section 3.1 it is not quite clear if the authors corrected the mean offset on the pitch (twist) for each blade or if they just measured the offset. If they did not correct the mean offset, why not? The authors say that the turbine is equipped with a manual pitch system which should allow the correction of the offset, right?
Thanks for pointing out that this is not fully clear. The differences in pitch angle and twist deformation were unfortunately only found in postprocessing after the campaign had ended. Thus, the pitch angle could not be corrected anymore. We have added a statement to clarify this, see Page 13, Line 439.
- 140 6. I do not fully understand why the authors show figure 12. Here they compare the calculated lift coefficient for each blade and compare it to the one from a reference for a specific Reynolds number. Why not going the other way and calculating the normal and tangential forces for each radial position on the blades using the angle of attack, acting velocities and the corresponding lift and drag coefficients from figure 1 for the correct Reynolds number and add the results to figure 11? That would minimise the effect of the Reynolds number in the end should show how well all methods agree — or maybe I misunderstood the intension of figure 12.
145 The intention of this plot is to demonstrate that the experimentally derived lift polar follows the trend of the design airfoil's lift polar. We do, however, agree with you pointing out the dependency on the local Reynolds number. We would like to keep Section 3.3 unchanged as we would rather not mix up quantities directly derived from measurements with those indirectly derived (via the input polars). Additionally, Figure 10 (a) (formerly Figure 11 (a)) already contains six datasets and we do not want to overload this figure. Thus, we alternatively present a comparison between the c_l derived from the measurements and that derived via the polars in Section 3.4, see Page 17, Line 491 and Figure 11 (b).
150 This analysis reveals that there are small deficits in lift production in the root and tip region between the blades used in the experiment and what the polars would suggest. This is a new insight, and we would like to thank you for the suggestion that led to it. We have also added a statement in the conclusions section to reflect this insight, see Page 19, Line 513. To reflect the Reynolds number variation along the span, we have also included the SD7032 lift polars for two more Re numbers in Figure 11 (a). We hope these changes are in agreement with your suggestion.
155
7. Line 316, page 18, the authors mention that in future research they would like to reduce the impact of blade deflection on the results. In the paper they never mentioned the problem or determined the deflection and the impact on the results. While you mentioned it here, what is the impact of blade deflection on the results presented in the paper?
160 Firstly, we need to clarify that we meant deformations, but wrote deflections. To be more consistent with the rest of this paper, we have adjusted the text, see Page 19, Line 521. Secondly, we acknowledge that you raise a good point by asking about deflections. Maximum flapwise tip deflections were determined to be 14 mm. In relation to the 900 mm blade tip radius, the influence of these deflections on the aerodynamic analysis presented in this paper is considered negligible.

165 We would like to thank the reviewers again for their detailed and constructive feedback. Please find a version of our manuscript highlighting all the changes made on the following pages. We look forward to hearing from you in due time regarding our submission and to responding to any further questions and comments you may have.

Sincerely,
Erik Fritz, André Ribeiro, Koen Boorsma, Carlos Ferreira

Aerodynamic characterisation of a thrust-scaled IEA 15 MW wind turbine model: Experimental insights using PIV data

Erik Fritz^{1,2}, André Ribeiro², Koen Boorsma¹, and Carlos Ferreira²

¹Wind Energy, TNO Energy Transition, Petten, Netherlands

²Faculty of Aerospace Engineering, Technical University of Delft, Delft, Netherlands

Correspondence: Erik Fritz (e.fritz@tno.nl)

Abstract. This study presents results from a wind tunnel experiment on a three-bladed horizontal axis wind turbine. The model turbine is a scaled-down version of the IEA 15 MW reference wind turbine, preserving the non-dimensional thrust distribution along the blade.

Flow fields were captured around the blade at multiple radial locations using Particle Image Velocimetry. In addition to these flow fields, this comprehensive dataset contains spanwise distributions of bound circulation, inflow conditions and blade forces derived from the velocity field. As such, the three blades' aerodynamics are fully characterised. It is demonstrated that the lift coefficient measured along the span agrees well with the lift polar of the airfoil used in the blade design, thereby validating the experimental approach.

This research provides a valuable public experimental dataset for validating low to high-fidelity numerical models simulating state-of-the-art wind turbines. Furthermore, this article establishes the aerodynamic properties of the newly developed model wind turbine, creating a baseline for future wind tunnel experiments using this model.

180 Nomenclature

Latin letters		continues on next page...
a, a'	Axial and tangential induction factor	
C_c	Chord scaling constant	
C_T	Thrust coefficient	
c	Chord	
c_l, c_d	Lift and drag coefficient	
c_l^0	Lift coefficient at zero angle of attack	
D	Rotor diameter, drag force	
D_{root}	Diameter of the blade root section	
\mathbf{F}	Force vector	
F_N, F_T	Normal and tangential force	

Latin letters

...continued

I	Identity matrix
Kl	Lift slope
L	Lift force
\mathcal{N}	Dimensional constant
n	Normal vector
R	Blade tip radius
Re_c	Chord Reynolds number
r	Radial coordinate
r_{root}	Blade root radius
S, S_B	Outer and inner boundary curve of a control volume
t	Time
U_∞	Free stream velocity
\mathbf{u}	Velocity vector
u, v	Velocity components
V_{rel}	Relative inflow velocity
V_{rot}	Rotational velocity
\mathbf{x}	Position vector

Greek letters and other symbols

α	Angle of attack
β	Blade twistpitch^{EF} angle
Γ	Circulation
γ	Flux term
λ	Tip speed ratio
λ_L	Geometric scaling factor
ρ	Density of air
$\boldsymbol{\tau}$	Reynolds stress tensor
ϕ	Inflow angle
ω	Angular velocity
$\boldsymbol{\omega}$	Vorticity vector
∇	Nabla operator

Subscripts

<i>CV</i>	Control volume
<i>ind</i>	Induced
<i>KJ</i>	Kutta-Joukowski
<i>M</i>	Model
<i>O</i>	Original

pol^{EF} Based on design polars^{EF}

1 Introduction

Wind tunnel experiments are vital in progressing horizontal axis wind turbine (HAWT) technology. They help in improving the understanding of, e.g. the turbine's aerodynamic, aeroelastic or acoustic characteristics. Equally important, the gathered data can be used to validate and improve numerical models that aim to simulate reality as closely as possible.

185 In light of these two goals, arguably, the two most relevant experiments on HAWTs are the Unsteady Aerodynamics Experiment (UAE) and the Model Rotor Experiment in Controlled Conditions (MEXICO). NREL executed the UAE in multiple phases. While Phases I - IV, conducted between 1989 and 1997, were field experiments (Butterfield et al., 1992; Simms et al., 1999), Phase VI was a wind tunnel experiment conducted in 2000. A two-bladed rotor of 10 m diameter was heavily instrumented and placed in the NASA Ames wind tunnel (Hand et al., 2001). The MEXICO experiment was conducted in 2006 in
190 the German-Dutch Wind Tunnel (DNW). Detailed aerodynamic measurements, including pressure, loads and 3D flow field characteristics using Particle Image Velocimetry (PIV) were taken on a three-bladed rotor with 4.5 m diameter (Schepers and Snel, 2007; Boorsma and Schepers, 2009). Its successor project "New Mexico" was conducted in 2014 to obtain additional data (Boorsma and Schepers, 2015). The results of these two experimental campaigns have been analysed in great detail and have been used for the validation/calibration of simulation tools of varying fidelity. For an extensive review of the literature
195 related to these two experiments, the reader is referred to the work of Schepers and Schreck (2018).

Given the success of these two experiments, the existing databases were extended by conducting further experiments on scaled versions of the two rotors. The wake of a 1:8 scaled version of the UAE Phase VI rotor was measured using PIV by Xiao et al. (2011). At the Korean Aerospace Research Institute (KARI), Cho and Kim (2014) tested the Reynolds number effect on torque and power on a 1:5 scaled model of the UAE Phase VI turbine. Comparable experiments were done by the
200 same researchers for a 2:4.5 scaled version of the MEXICO rotor (Cho and Kim, 2012). The Spanish National Institute for Aerospace Technology (INTA) tested a 1:4 scaled MEXICO rotor. Results of the scaled models are compared against the original MEXICO data in IEA Task 29 by Schepers et al. (2012).

Complementary to experimental investigations, HAWTs are studied extensively using numerical simulations. To enable numerical benchmarks between different simulation tools and to facilitate collaboration between academic and industrial research,
205 multiple reference wind turbine (RWT) models have been developed in recent years, e.g. the NREL 5 MW RWT (Jonkman

et al., 2009), the DTU 10 MW RWT (Bak, 2013) and the IEA 15 MW RWT (Gaertner et al., 2020). While not representing existing wind turbines, these open-source reference models reflect current trends and developments of HAWT technology. Wind tunnel campaigns with scaled versions of these reference wind turbines have been conducted to provide experimental datasets that can be used to validate numerical simulations. Berger et al. (2018) developed a model turbine based on the NREL 5 MW RWT, which has since been used to study dynamic inflow phenomena due to pitch steps (Berger et al., 2021) and fluid-structure interaction by means of photogrammetry (Langidis et al., 2022; Nietiedt et al., 2022).^{EF} Fontanella et al. (2021a) ran experiments on a scaled DTU 10 MW wind turbine mimicking the motions of a floating offshore wind turbine (FOWT). In addition to load cell measurements on the turbine, the wake was characterised using PIV measurements. Similar experiments were conducted by Taruffi et al. (2024), extending the mimicked floater motions to six degrees of freedom and larger amplitudes and frequencies. Fontanella et al. (2022) performed another set of experiments on a 1:100 scaled model of the IEA 15 MW RWT developed by Allen et al. (2020). Here, rotor loads were measured using load cells, and the wake was characterised using hot wire velocity measurements. A 1:70 scaled model of the IEA 15 MW was tested by Kimball et al. (2022) with a focus on verifying thrust and torque curves and validating the utilised pitch controller. While these studies on scaled-down versions of the RWTs provide valuable data regarding rotor-level aerodynamics, they lack more detailed data on the blade-level.

Such blade-level data can be obtained using non-intrusive measurement techniques such as Laser Doppler Velocimetry (LDV) or Stereoscopic Particle Image Velocimetry (SPIV). Phengpom et al. (2015a, b, 2016) studied the flow field in the direct vicinity of the blade using LDV. Akay et al. (2013) researched the vortex structure around the blade root of a two-bladed wind turbine with a 2 m diameter based on SPIV measurements. Similarly, Lignarolo et al. (2014) investigated the wake development of a smaller model (two blades, 0.6 m diameter) focusing on the tip vortices. Continuing this line of research, the generation of the tip vortex was investigated in more detail on a different two-bladed wind turbine model of 2 m diameter by Micallef et al. (2014, 2015). Furthermore, and most relevant to the present work, SPIV was employed to derive the spanwise blade load distribution of a HAWT in axial and yawed inflow by del Campo et al. (2014, 2015).

The present work studies the spanwise aerodynamic characteristics of a 1:133 scaled model of the IEA 15 MW RWT, thus, of the most recent available reference wind turbine. SPIV is used to measure the flow field around various radial sections of the blade and, consequently, to derive the spanwise aerodynamic properties of this model wind turbine. By characterising the blades in terms of induction values, inflow angle and angle of attack, circulation, and blade loads, this study provides a more complete dataset of blade-level aerodynamics than previous wind tunnel experiments. As such, this research aims to enable further multi-fidelity numerical benchmarking as well as to establish a reference dataset that can be used as a starting point for future experimental studies on this wind tunnel model turbine.

This paper is built up as follows: Section 2 describes the scaling approach used to develop the wind tunnel model's blades. Furthermore, details of the experimental setup are given and the equations used to derive blade aerodynamic quantities from the measured flow fields are provided. Section 3 initially details the challenge of estimating deviations from the design twist distribution. It then presents the results of the experimental campaign in terms of flow fields, distributed blade aerodynamics and airfoil polars. Finally, conclusions are drawn in Section 4 and an outlook at further research is given.

2.1 Scaled wind turbine model

The model HAWT tested in this experiment is a scaled version of the IEA 15 MW RWT (Gaertner et al., 2020), preserving non-dimensional thrust. The main model characteristics are given in Table 2 alongside their full-scale equivalents.

Parameter		IEA 15 MW RWT	Wind tunnel model
Rotor diameter	D	240 m	1.8 m
Blade root radius	r_{root}	3 m	0.06 m
Design tip speed ratio	λ	9 -	9 -

Table 2. Specifications of the IEA 15 MW RWT and the scaled wind tunnel model

The geometric scaling factor of 1:133 applied to the rotor diameter cannot be maintained at the blade root. Here, mechanical and electronic components necessitate a larger blade root radius leading to a scaling factor of 1:50. The ratio of root to tip radius is in close agreement with comparable wind tunnel models, see Fontanella et al. (2022).^{EF}

Multiple challenges occur when creating a scaled-down wind tunnel model of a wind turbine. Arguably, the largest challenge lies in the fact that the chord Reynolds number Re_c present on a full-scale wind turbine can generally not be achieved in a wind tunnel. A difference in Re_c of multiple orders of magnitude necessitates the use of airfoils designed explicitly for low Reynolds numbers. One such airfoil is the *SD7032* airfoil, which has a maximum relative thickness of 10% and was characterised experimentally by Fontanella et al. (2021b). The lift and drag coefficient of the *SD7032* airfoil for different Reynolds numbers is given in Figure 1. A characteristic of this airfoil, making it useful for small-scale wind turbines, is the relative insensitivity of the lift polar to the Reynolds number over a large range of angles of attack. The well-documented wind tunnel polars, as well as the airfoil's application in comparable wind tunnel campaigns (Kimball et al., 2022; Fontanella et al., 2022) motivated the choice for the *SD7032* airfoil.

Instead of using various airfoils along the span, the developed model blades are defined by this single airfoil, which transitions into a cylindrical section at the blade root. The polars of the wind tunnel model differ from those of the airfoils used on the full-scale turbine. Thus, even at identical angles of attack, the non-dimensionalised lift distribution will differ between model and original. Bayati et al. (2017) detail a scaling approach designed to ensure comparable non-dimensionalised blade loads. In this approach, the model chord distribution c_M is calculated as

$$c_M = \frac{c_O}{\lambda_L} \frac{Kl_O}{Kl_M} \quad (1)$$

where c_O is the original chord distribution, λ_L is the geometric scaling factor, and Kl_O and Kl_M are the lift slopes in the linear region of the original and model airfoil polars, respectively. The model twist distribution β_M is calculated as

$$\beta_M = \beta_O - \frac{c_{l,O}^0}{Kl_O} + \frac{c_{l,M}^0}{Kl_M} \quad (2)$$

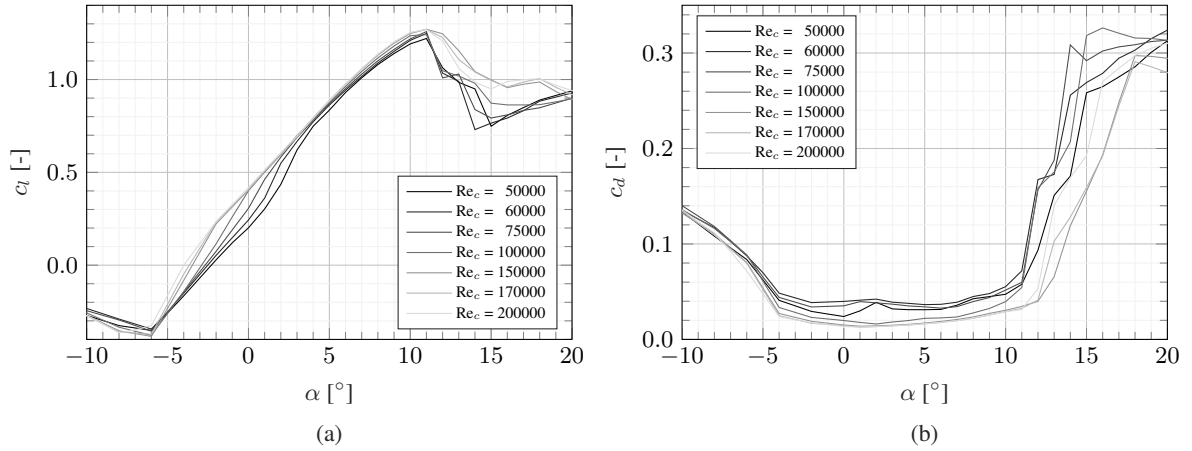


Figure 1. Lift coefficient c_l (a) and drag coefficient c_d (b) of the *SD7032* airfoil for varying Reynolds numbers (Fontanella et al., 2021b)

265 where β_O is the original twist distribution and $c_{l,O}^0$ and $c_{l,M}^0$ are the lift coefficient values at zero angle of attack of the original and model airfoil polars, respectively.

Since the IEA 15 MW RWT's blade is very slender, applying this scaling approach leads to blades with very small chord values. Using the *SD7032* airfoil to create the geometry, such low chord values entail very thin blades. To avoid unnecessary challenges during the manufacturing process of the wind tunnel model blades, a constant factor is applied to the chord scaling
270 so that

$$c_M = \frac{c_O}{\lambda_L} \frac{Kl_O}{Kl_M} C_c \quad (3)$$

with $C_c = 1.5$. Furthermore, this factor ensures angles of attack well away from the stall margin of the *SD7032* airfoil. Inboard of $r/R = 0.25$, a cubic spline is used to reduce^{EF} the chord ~~is manually reduced~~^{EF} to obtain^{EF} a cylindrical root section with $D_{root} = 4$ cm. This is a common practice for scaled wind tunnel models (Bayati et al., 2017; Muggiasca et al., 2021) motivated
275 by manufacturing and assembly constraints, and it is expected to have little impact on rotor aerodynamics due to the generally lower aerodynamic forces acting in the root region.^{EF}

Rather than matching the lift force, a comparable thrust distribution along the blade is targeted. Therefore, equal thrust coefficient distributions $C_T = \frac{F_N dr}{\frac{1}{2} \rho U_\infty^2 2\pi r dr}$ are enforced

$$\frac{F_{N,M}}{\rho U_{\infty,M}^2 \pi r_M} = \frac{F_{N,O}}{\rho U_{\infty,O}^2 \pi r_O} \quad (4a)$$

280

$$F_{N,M} = \frac{U_{\infty,M}^2}{U_{\infty,O}^2} \frac{r_M}{r_O} F_{N,O} \quad (4b)$$

^{EF}

where F_N is the axial force per unit span, ρ is the density of air, U_∞ is the freestream velocity and r is the radial coordinate. The local axial force coefficient c_N^{EF} can also be expressed as

$$285 \quad c_N = \frac{F_N}{\frac{1}{2}\rho V_{rel}^2 c} = c_l \cos(\phi) + c_d \sin(\phi)^{\text{EF}} \quad (5)$$

with V_{rel} being the local relative inflow velocity, ϕ being the local inflow angle, L and D being the lift and drag force, c_l and c_d the lift and drag coefficients, respectively. Substituting Equation 4b in Equation 5 yields a minimum function with β_M as variable

$$\min_{\beta_M} = c_{N,M}(\alpha_M) - c_{N,O}(\alpha_O) \quad (6a)$$

$$290 \quad \min_{\beta_M} = c_{l,M}(\phi_O - \beta_M) \cos(\phi_O) + c_{d,M}(\phi_O - \beta_M) \sin(\phi_O) - \frac{U_{\infty,M}^2 r_M}{U_{\infty,O}^2 r_O} \frac{F_{N,O}}{\frac{1}{2}\rho V_{rel,M}^2 c_M} \quad (6b)$$

where $V_{rel,M} = \sqrt{(U_{\infty,M}(1 - a_O))^2 + (\omega_M r_M(1 + a'_O))^2}$, with a_O and a'_O being the axial and tangential induction factors, respectively, and ω the angular rotation frequency. Based on Equation 6b, the model twist distribution can be determined. The original flow properties ϕ_O , $F_{N,O}$, a_O and a'_O are taken from numerical simulations of the full-scale IEA 15 MW RWT based on blade element momentum theory (BEM). The underlying algorithm has previously been used for simulations of the IEA 15 MW RWT (Fritz et al., 2022) and was validated in the same work against the established lifting line algorithm AWSM (Grasso et al., 2011).^{EF} For these simulations, an inflow velocity of $U_{\infty,O} = 10$ m/s is chosen, corresponding to operation just below rated. $U_{\infty,M}$ is set to match the targeted wind tunnel inflow velocity. The resulting chord and twist distribution of the wind tunnel model blade are given in Figure 2.

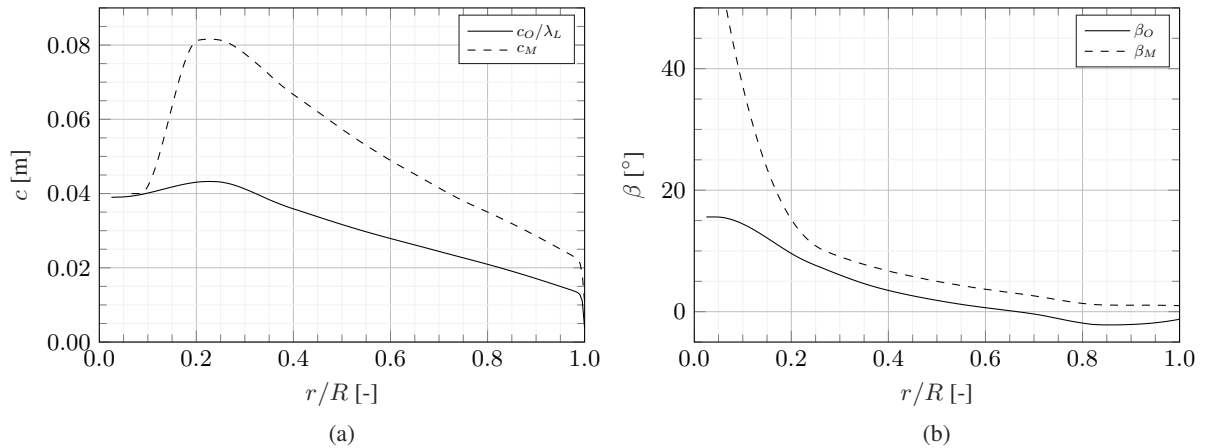


Figure 2. Chord (a) and twist (b) distribution of the geometrically scaled IEA 15 MW RWT and the wind tunnel model

The presented scaling approach ensures close resemblance of the model’s non-dimensionalised thrust distribution to that of its reference. It should, however, be noted that other flow physics, such as flow transition or separation, can be fundamentally different due to the changes in airfoil and chord Reynolds number.^{EF}

2.2 Experimental setup and measurement system

305 The experiments were conducted in the Open Jet Facility at the TU Delft Faculty of Aerospace Engineering, which is a closed-circuit open jet wind tunnel. The jet exit is an octagon of $2.85\text{ m} \times 2.85\text{ m}$. ~~A schematic of the OJF is given in Figure ??~~^{EF}. The turbine was operated at an approximate tip speed ratio of $\lambda = 9$ and an inflow velocity of $U_\infty = 3.75\text{ m/s}$. ~~To achieve the desired tip speed ratio, the turbine is driven by a motor that closely maintains the set rotational speed.~~^{EF} Inflow conditions of the wind tunnel were logged for each measurement point and showed no significant variation. The wind tunnel was kept at a
310 constant temperature of 20°C .

In this campaign, SPIV was used to non-intrusively measure the flow around the blades. A Quantel Evergreen double-pulsed Neodymium-doped Yttrium Aluminium Garnet (Nd:YAG) laser provides the light source. Using laser optics, a thin vertical laser sheet was generated that illuminates the area around the targeted blade cross-section. To reduce reflections of the laser, the blades and most other turbine components were spray-painted matt black. A Safex smoke generator produced smoke particles
315 with a median diameter of $1\text{ }\mu\text{m}$, which were used as tracers. The smoke generator was placed downstream of the tunnel test section, ensuring homogeneous mixing during the flow recirculation.

Two LaVision Imager sCMOS cameras with lenses of 105 mm focal length and an aperture of $f/8$, captured the illuminated particles during the two laser pulses. The laser and cameras were simultaneously triggered by an optical sensor that was activated by a notch in the rotor shaft once per revolution. A time delay between the optical sensor’s signal and the laser/camera
320 trigger ensured the blade was in the horizontal position during its upward movement when taking the images. ~~The delay’s value of 41 ms was determined by comparing images captured during operation with pictures captured during standstill with the blade fixed in the desired horizontal position. This comparison was conducted close to the blade tip where the rotational velocity is highest and, thus, the position of the cross-section in the field of view most sensitive to the time delay value.~~^{EF} The image pairs were taken with a time separation of $150\text{ }\mu\text{s}$, which allowed the tracing of the particles’ movement. This time separation
325 is equivalent to a particle movement of approximately 5 px and a turbine rotation of 0.3° . At each measurement location, 120 phase-locked images were taken, which are used in postprocessing to obtain an average flow field and its standard deviation. The images are acquired and processed using the LaVision Davis 8 software. The field of view (FOV) resulting from this measurement setup is approximately $FOV \approx 297\text{ mm} \times 257\text{ mm}$ and the final image resolution is 8.81 px/mm .

Both cameras and laser were mounted rigidly on a traversing system. This way, velocity measurements could be conducted
330 at multiple radial stations without the need for refocusing the cameras and calibrating the software. Figure 3 shows a schematic of the measurement setup.

A total of 22 measurement planes were placed along the blade span as follows:

- $\Delta r/R = 0.100$ for $0.10 \leq r/R \leq 0.40$

– $\Delta r/R = 0.050$ for $0.40 \leq r/R \leq 0.80$

335 – $\Delta r/R = 0.025$ for $0.80 \leq r/R \leq 1.05$

This selection aims at accurately representing the stronger gradients in blade aerodynamics typically present close to the tip. At four radial locations, namely at $r/R = [0.4, 0.6, 0.8, 0.9]$, measurements were taken for all three blades to evaluate how representative the main measurement blade is for the remaining two blades.

When illuminating a cross-section, the blade casts a shadow where no particles could be traced. Thus, the flow field was captured in two steps. In a first step, the blade's pressure side was evaluated by placing the laser upstream of the turbine and angling the laser sheet downstream. Following that, the laser was relocated downstream of the rotor plane and its laser sheet was tilted upstream to capture the suction side (as shown in Figure 3). In a postprocessing step, [the two flow fields averaged individually over the phase-locked upstream and downstream images](#)^{EF} were stitched together, resulting in the entire flow field around a blade cross-section.

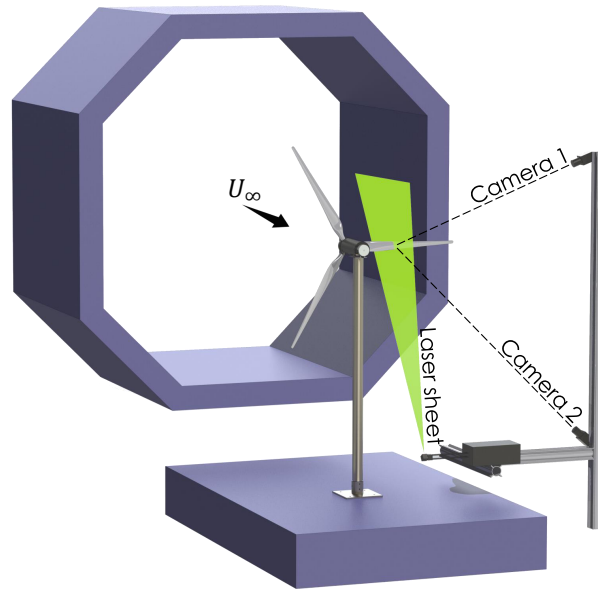


Figure 3. Experimental setup and measurement system

345 2.3 Deriving blade level aerodynamics from PIV measurements

This section presents the equations used to derive the distributed blade aerodynamics regarding bound circulation, induction, inflow angle and angle of attack, and blade loads. Based on these quantities, it is possible to calculate the experimental lift polar, too. In this study, the equations presented below are applied under the assumption of local two-dimensional flow, i.e. only the velocity components in the measurement plane are considered.

350 2.3.1 Determination of bound circulation

The bound circulation Γ at each measurement location can be calculated as the line integral of the measured velocity field \mathbf{u} along a curve S enclosing the blade cross-section (e.g. Anderson, 2017, p. 176):

$$\Gamma = - \oint_S \mathbf{u} \cdot d\mathbf{s} \quad (7)$$

A study of the sensitivity to the bounding curve's size is presented in Appendix A. It revealed that the circulation, and also
 355 the forces calculated using Noca's method (see Section 2.3.3), do not exhibit perfect convergence with varying control volume size. As a consequence, the methods presented in this section are applied for multiple control volumes with different sizes, from which a mean value and standard deviation are calculated.

2.3.2 Determination of induced velocities, inflow angle and angle of attack

Several methods for determining the local inflow conditions exist. The inverse BEM approach (Bruining et al., 1993; Snel et al.,
 360 1994; Bak et al., 2006) uses measured/simulated forces as input to the blade element momentum equations and iteratively solves for the inflow conditions. Other methods characterise the inflow based on the annulus average flow field (Hansen and Johansen, 2004; Johansen and Sørensen, 2004) or based on the wake induction at the plane exactly between two blades (Herráez et al., 2018). Other approaches use the bound circulation strength to estimate local induced velocity and consequently the inflow conditions (Shen et al., 2007, 2009; Jost et al., 2018). Several benchmarks of these methods have been conducted based on
 365 CFD and/or experimental data (Guntur and Sørensen, 2014; Herráez et al., 2018; Rahimi et al., 2018).

The approach denoted as the Ferreira-Micallef method in Rahimi et al. (2018) is used here. It relies on potential flow theory to estimate the induced velocities at each spanwise location. This theory states that the velocity at any point can be expressed by the sum of the relative velocity and the velocities induced by free and bound vorticity, such that the measured velocity at a point p is given as

$$370 \quad \mathbf{u}_p = \sum \mathbf{u}_{ind} + \mathbf{V}_{rel} \quad (8)$$

Biot-Savart law is employed to determine the sum of the induced velocities at a set of control points located along S so that

$$\sum \mathbf{u}_{ind} = \sum \frac{\Gamma}{2\pi} \frac{\mathbf{x}_p - \mathbf{x}}{|\mathbf{x}_p - \mathbf{x}|^2} \quad (9)$$

where \mathbf{x}_p and \mathbf{x} are the position vectors of the control point and inducing vortex element, respectively. By minimising the error
 between \mathbf{u}_p and \mathbf{u}_{ind} using a least squares approach, the relative inflow vector \mathbf{V}_{rel} is determined, yielding the local axial and
 375 tangential induction factors

$$a = 1 - \frac{u_{rel}}{U_\infty} \quad (10)$$

$$a' = \frac{v_{rel}}{\omega r} \quad (11)$$

Knowing the induced velocities, the local inflow angle and angle of attack can then be calculated as

$$\phi = \tan^{-1} \left(\frac{U_\infty(1-a)}{\omega r(1+a')} \right) \quad (12)$$

$$380 \quad \alpha = \phi - \beta \quad (13)$$

2.3.3 Determination of blade loads

Noca's method:

The forces exerted by an immersed body on the surrounding fluid can be evaluated by integrating the change of momentum over a finite control volume. Noca et al. (1999) presented an alternative formulation of the momentum conservation equation, solely relying on surface integrals of flow quantities placed on the boundary of the control volume. The forces can, thus, be derived from the measured velocity field and its spatial and time derivatives. This approach has been successfully applied to PIV data collected on a vertical axis wind turbine by LeBlanc and Ferreira (2022). The force per density is given by:

$$\frac{\mathbf{F}}{\rho} = \oint_S \mathbf{n} \cdot \gamma \, ds - \oint_{S_B} \mathbf{n} \cdot (\mathbf{u} - \mathbf{u}_B) \mathbf{u} \, ds - \frac{d}{dt} \oint_{S_B} \mathbf{n} \cdot (\mathbf{u} \mathbf{x}) \, ds \quad (14)$$

EF

390 where \mathbf{n} is the normal vector of the bounding curves, γ is the flux term, S is the outer boundary curve of the control volume surrounding the immersed body, S_B is the control volume's inner boundary curve prescribed by the immersed body's surface, and \mathbf{u}_B is the velocity vector of the immersed body's surface.

The term $\oint_{S_B} \mathbf{n} \cdot (\mathbf{u} - \mathbf{u}_B) \mathbf{u} \, ds$ is related to the flow through the inner boundary curve S_B . Given the solid airfoil surface, this term is zero. The third term $\frac{d}{dt} \oint_{S_B} \mathbf{n} \cdot (\mathbf{u} \mathbf{x}) \, ds$ describes the force due to acceleration of the inner boundary surface. As 395 the model wind turbine was running at a constant speed during the experiment, the velocity of the airfoil representing the inner boundary surface can be approximated as constant within the measurement plane. Therefore, this term is zero, too. The flux term γ can be determined as ~~The aerodynamic force vector \mathbf{F} non-dimensionalised by the fluid density ρ can then be determined as~~^{EF}

$$\begin{aligned} \gamma = & \frac{1}{2} u^2 \mathbf{I} - \mathbf{u} \mathbf{u} - \frac{1}{\mathcal{N}-1} \mathbf{u} (\mathbf{x} \times \boldsymbol{\omega}) + \frac{1}{\mathcal{N}-1} \boldsymbol{\omega} (\mathbf{x} \times \mathbf{u}) \\ & - \frac{1}{\mathcal{N}-1} \left(\mathbf{x} \cdot \frac{\partial \mathbf{u}}{\partial t} \right) \mathbf{I} + \frac{1}{\mathcal{N}-1} \mathbf{x} \frac{\partial \mathbf{u}}{\partial t} - \frac{\partial \mathbf{u}}{\partial t} \mathbf{x} \\ & + \frac{1}{\mathcal{N}-1} [\mathbf{x} \cdot (\nabla \cdot \boldsymbol{\tau})] \mathbf{I} - \frac{1}{\mathcal{N}-1} \mathbf{x} (\nabla \cdot \boldsymbol{\tau}) + \boldsymbol{\tau} \end{aligned} \quad (15)$$

400 EF

where \mathbf{I} is the identity matrix, \mathcal{N} is the dimensional constant, $\boldsymbol{\omega}$ is the vorticity vector and $\boldsymbol{\tau}$ is the Reynolds stress tensor.

There exist two possible frames of reference in which to apply the equations given above. On the one hand, a stationary reference frame can be chosen, where the measured blade cross-section moves vertically through the control volume, see Figure 4 (a). On the other hand, a reference frame rotating with the investigated cross-section can be used, see Figure 4 (b).

405 While the original PIV data is captured in a stationary reference frame, it can easily be converted to a rotating frame by adding the apparent rotational velocity $V_{rot} = -\omega r$ to the measured vertical velocity component v .

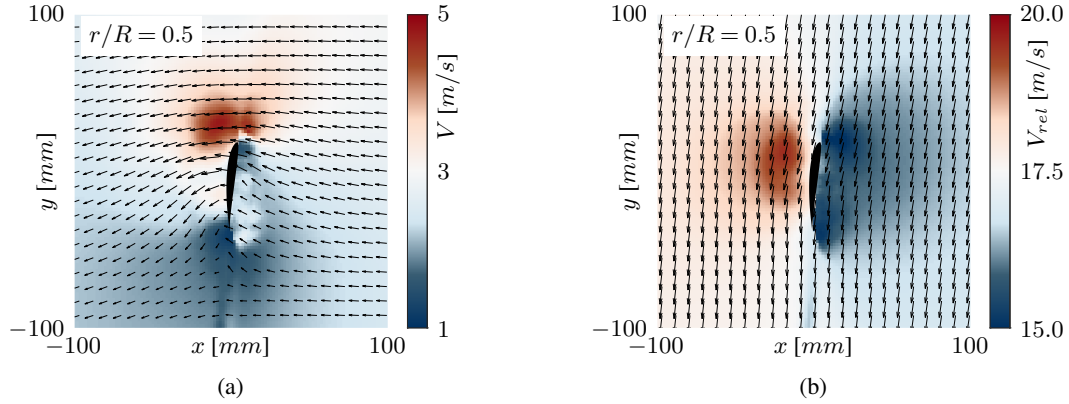


Figure 4. Velocity field in a stationary (a) and rotating (b) reference frame

For the analysis performed in the present work, a rotating frame of reference is chosen. In this reference frame, the time derivatives of Equation 15 are zero.

Kutta-Joukowski theorem (KJ):

410 Alternative to Noca’s method, the forces can be derived from the bound circulation using the Kutta-Joukowski theorem (e.g. Anderson, 2017, p. 282), which states that the sectional lift force is given by $L = \rho V_{rel} \Gamma$. This formulation can be decomposed to yield the forces normal and tangential to the rotor plane:

$$F_N = \rho \omega r (1 + a') \Gamma \quad (16)$$

$$F_T = \rho U_\infty (1 - a) \Gamma \quad (17)$$

415 It should be noted that the Kutta-Joukowski theorem is based on potential flow theory. Thus, e.g. the viscous drag contribution to the tangential force is neglected.

3 Results

3.1 Determination of the combined pitch and twist offset

The blades used in this experiment are made of vacuum-infused carbon fibre-reinforced material. This partially manual manufacturing approach led to minor differences between the three blades. Based on visual inspection, one blade was chosen on which the measurement campaign was mainly conducted, hereafter called blade 1. However, measurements were taken for blades 2 and 3 at $r/R = [0.4, 0.6, 0.8, 0.9]$ to estimate the main measurement blade’s representation of the other two blades.

Early investigations into the gathered data indicated non-negligible differences in blade aerodynamics between the three blades. To explain this behaviour, the blade cross-sections visible in the raw images were visually inspected and compared

425 against the original design of the blade. This approach is visualised in Figure 5 (a), where the blade cross-section is illuminated
 in white. The original design is overlaid as red airfoil shape. Then, the correct local twist is found by rotating this airfoil
 around the trailing edge until its pressure side follows approximately the same curve as the pressure side of the illuminated
 cross-section. This correction was determined with a precision of 0.1° . The corrected airfoil is shown in green. Based on this
 430 comparison, it became apparent that the blade cross-sections were positioned at different angles than designed, resulting in the
 offset in twist/pitch shown in Figure 5 (b).

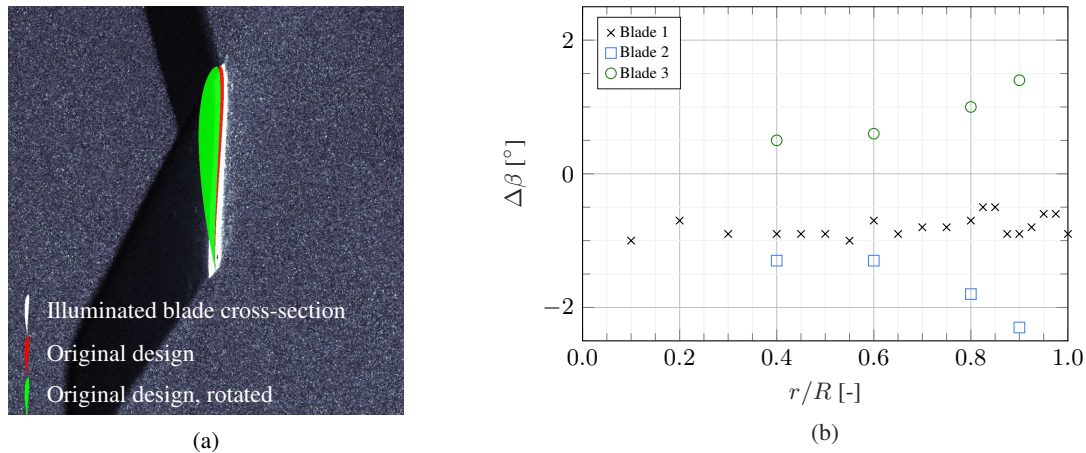


Figure 5. Approach of determining actual local airfoil orientation (a), twist/pitch offset determined by comparing experimentally captured blade cross-sections to the original design (b)

For blade 1, where many data points are available along the span, a quadratic fit is used to describe the trend and balance out the fluctuations likely due to human error in the interpretation of the raw images. Blade 1 appears to have a pitch offset of approximately negative one degree and additionally shows slight twist deformation towards the tip. More extreme twist deformations can be observed for blades 2 and 3, with opposite directions. This shows how challenging the use of vacuum-
 435 infused carbon fibre composite blades is. Despite having the same fibre layup, the manufacturing process is a highly manual task where minor differences can impact the structural properties of the blade. The pitch offset can be explained by the model turbine's connection between blade root and hub: The turbine is equipped with a manual pitch mechanism which is fixed in the desired position using set screws. Despite being used with care, this manual mechanism is likely the origin of the pitch deviations between the three blades. *As these deviations from the intended design were only found in postprocessing after the*
 440 *campaign had ended, no correction to the pitch angle could be made anymore.*^{EF}

3.2 Flow field

The flow fields represent the primary data collected during this experiment using stereoscopic PIV. Figure 6 depicts the measured velocity magnitude fields at the four radial stations where data for all three blades is available. Overall, the general flow

patterns are in good agreement. However, the twist/pitch offset described in the previous section leads to differences in the
445 angle of attack, explaining minor discrepancies in velocity magnitudes. For example, blade 2, exhibiting twist deformations
towards higher angles of attack, induces higher velocities, while the opposite holds for blade 3.

Notably, many measurement points have low-velocity regions close to the suction side surface. Here, laser reflections from
the blade surface reduce the accuracy of the PIV processing. This is less the case on the pressure side, where the concave blade
surface causes lower reflections.

450 3.3 Blade aerodynamics

All plots presented in this section contain error bars. These represent the 95% confidence interval and are based on variations
in the measured velocity field during the capturing of the PIV images as well as in the processing with various control volume
sizes, see Appendix A. This uncertainty is a measure of both the quality of the phase-lock as well as the unsteadiness of the
flow. While almost all data points have very low uncertainty, the measurement point closest to the root suffers from the laser
455 reflecting off the nacelle and hub, increasing measurement uncertainty. This effect is visible to a varying degree in all derived
aerodynamic quantities.

Figure 7 shows the circulation distribution of the three blades. The effect of varying pitch angles and twist deflection ex-
presses itself in the different circulation levels of the three individual blades.

The axial and tangential induction factor distribution is shown in Figure 8. Compared to the circulation distribution, differ-
460 ences in induction are minor between the three blades. This is a significant finding in support of fundamental BEM theory,
which uses a rotor-averaged induction factor.

Figure 9 depicts the local inflow angle and angle of attack distribution. Given that the blades have a cylindrical cross-section
at $r/R = 0.1$, the value of the angle of attack at this location is meaningless and reported for completeness only. The angle of
attack distribution is evidently influenced by the pitch/twist variations between the three blades. Despite these variations, all
465 derived angles of attack are well within the linear region of the design airfoil's lift polar.

The axial and tangential force distributions are presented in Figure 10. Two methods are employed to derive the normal
force distribution, namely Noca's method and the Kutta-Joukowski theorem (KJ). Both methods are in close agreement; a
linear fit between the results of all three blades yields $F_{N,KJ} = 1.016 F_{N,Noca} - 0.3918$ with $R^2 = 0.9965$. By integrating the
normal force distribution, the rotor thrust can be calculated and non-dimensionalised to obtain the thrust coefficient. To this
470 end, piecewise cubic curves are fit to the experimental results. Where no data is available at blade root and tip, zero loading
is assumed. The resulting thrust coefficients are $C_{T,Noca} = 0.8170$ and $C_{T,KJ} = 0.7821$. For a tip speed ratio of $\lambda = 9$, the
IEA 15 MW RWT has a thrust coefficient of $C_T = 0.8$ (Gaertner et al., 2020). Thus, the relative deviation of the thrust-scaled
blades to their reference corresponds to $\Delta C_{T,Noca} = 2.1\%$ and $\Delta C_{T,KJ} = -2.2\%$, respectively.

As demonstrated in Appendix A, Noca's method is, however, unreliable when estimating the tangential force from this
475 experimental dataset. Therefore, only the tangential force derived using the Kutta-Joukowski theorem is presented here. It is
noteworthy, that this method neglects viscous effects and, consequently, misses the contribution of the viscous drag. Overall,
the normal and tangential force trends are consistent between the three blades. However, the magnitude is fairly different, with

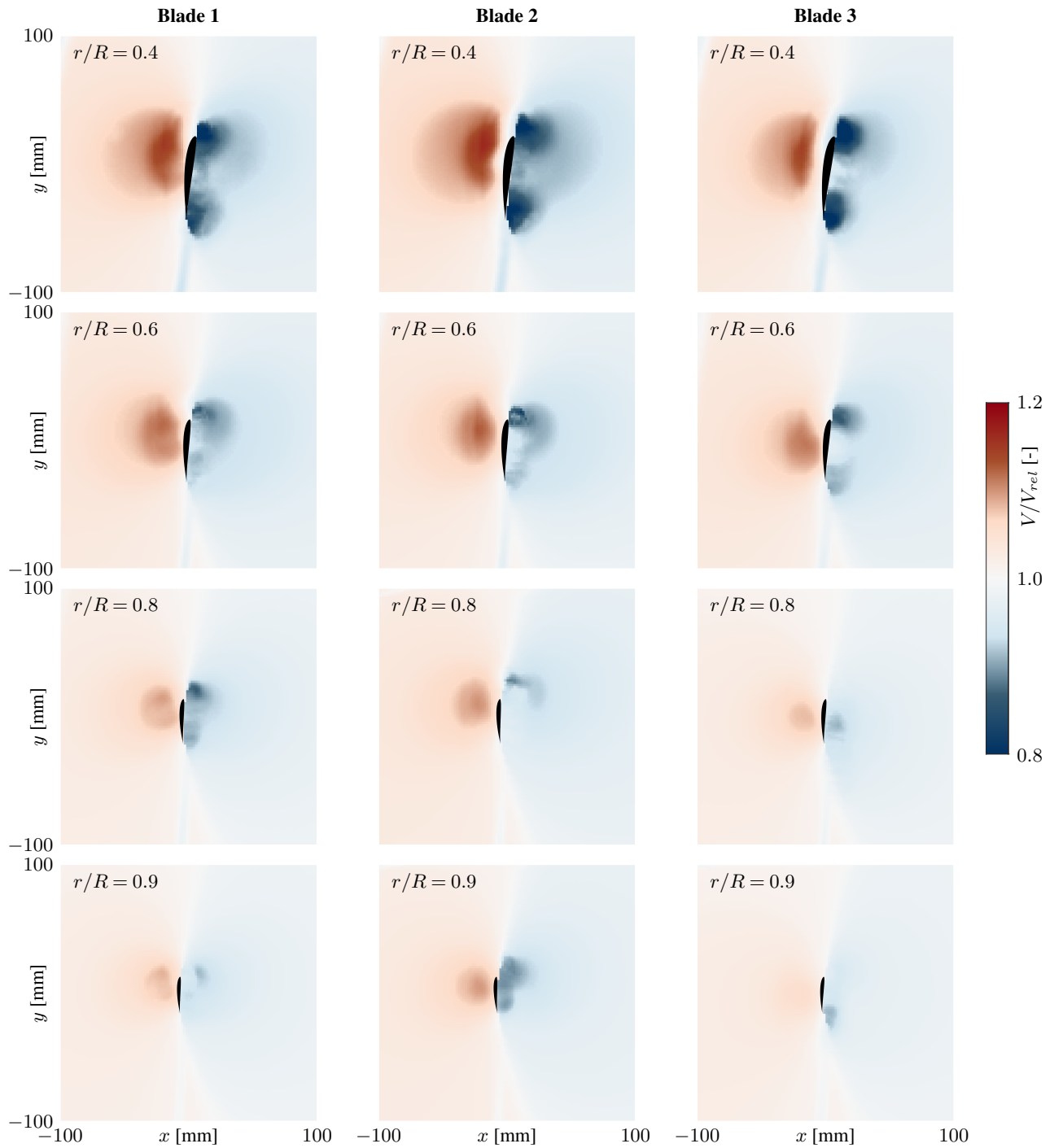


Figure 6. Non-dimensionalised velocity magnitudes at the radial stations measured for all three blades

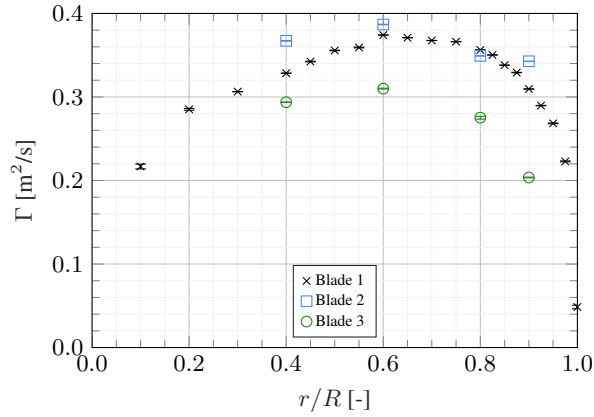


Figure 7. Spanwise distribution of bound circulation, error bars representing the 95% confidence interval

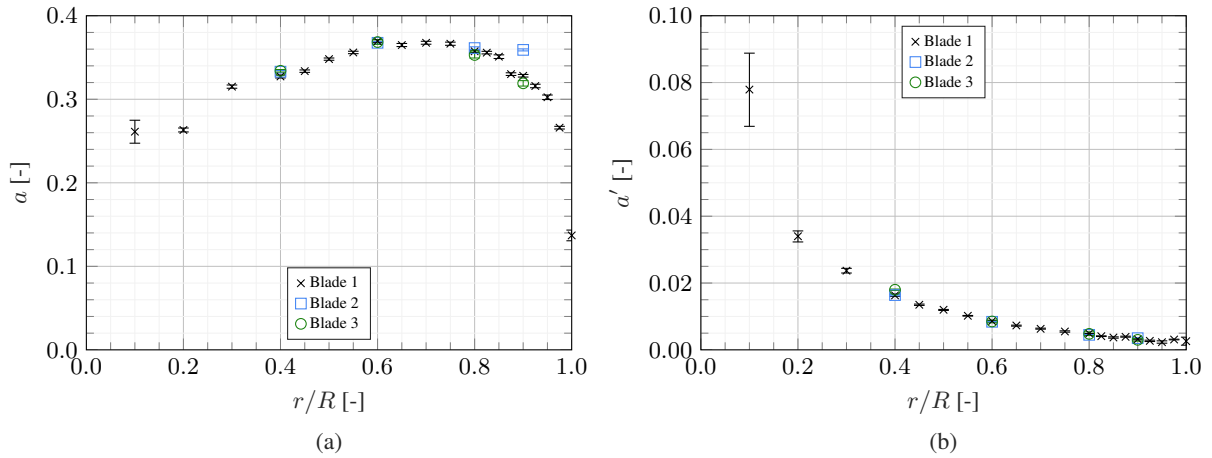


Figure 8. Spanwise distribution of axial (a) and tangential (b) induction factors, error bars representing the 95% confidence interval

blade 2 having, on average, slightly higher values than blade 1, while blade 3 exhibits lower values than the other two blades. These differences are in line with the pitch/twist offset discussed in Section 3.1.

480 3.4 Lift polar

Based on the aerodynamic quantities presented in the previous section, the lift coefficient is derived. The lift force is calculated using the force distributions based on the Kutta-Joukowski theorem

$$c_l = \frac{F_{N,KJ} \cos(\phi) + F_{T,KJ} \sin(\phi)}{\frac{1}{2} \rho V_{rel}^2 c} \quad (18)$$

Figure 11 (a)^{EF} shows the experimental lift polar compared to the *SD7032* airfoil at $Re_c = 50000$ (Fontanella et al., 2021b),
 485 resembling the Reynolds numbers present in this experiment, which vary between approximately 40000 and 65000 depending

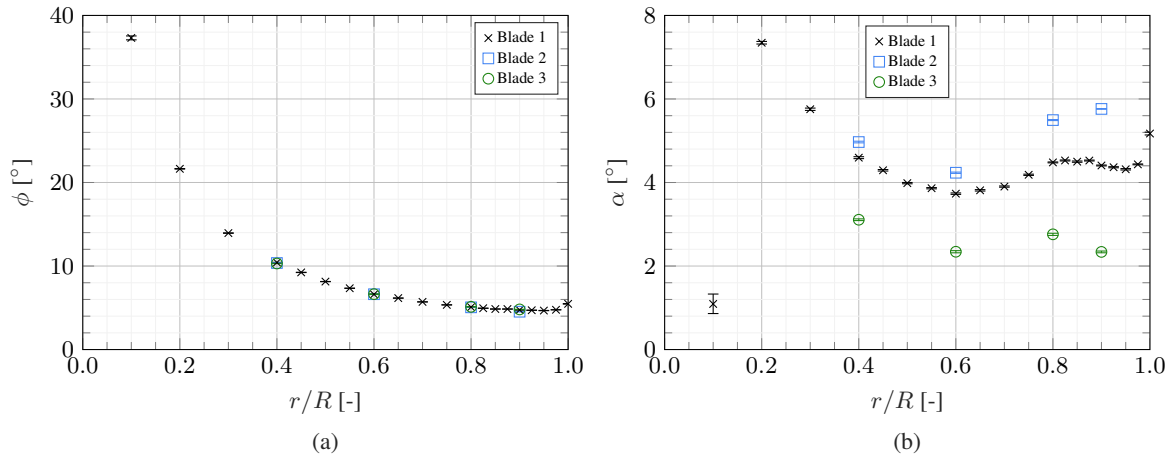


Figure 9. Spanwise distribution of inflow angle (a) and angle of attack (b), error bars representing the 95% confidence interval

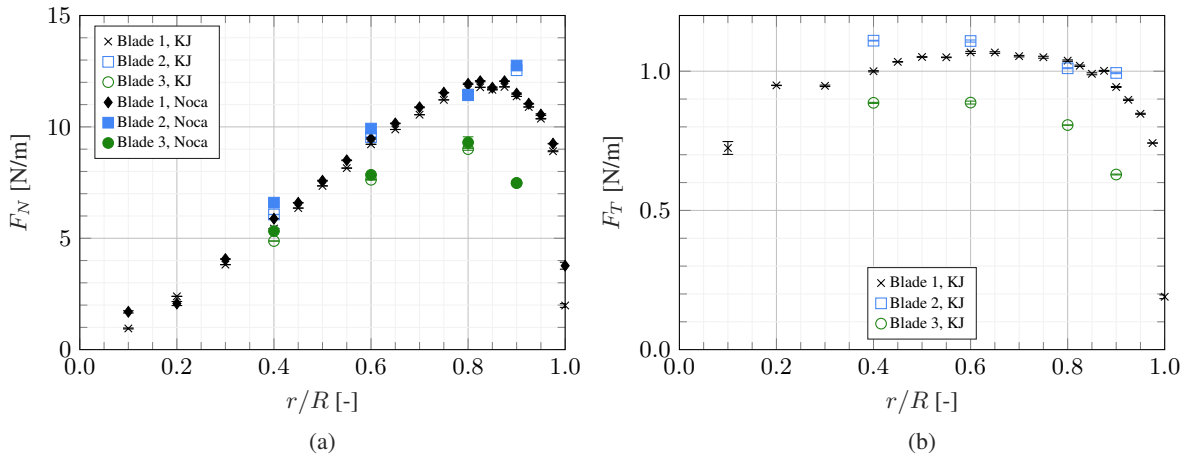


Figure 10. Spanwise distribution of normal (a) and tangential (b) force, error bars representing the 95% confidence interval

on the radial position. For clarity, only the mean values are reported. The two measurements closest to the root are omitted as these cross-sections are defined by a cylinder and a blend between a cylinder and the *SD7032* airfoil. Additionally, the two measurements closest to the tip are omitted because the tip vortex causes highly three-dimensional flow features, which should not be compared to two-dimensional airfoil polars. The remaining measurement points are in good agreement with the lift coefficient curve of the design airfoil.

While giving an indication of the experimentally derived lift polar, Figure 11 (a) does not represent the variable Reynolds number along the blade. Alternatively, the design airfoil polars can be interpolated for the experimentally derived Reynolds number and angle of attack to obtain a polar-based, expected lift coefficient $c_{l,pol}$. These values are plotted alongside the lift coefficient based on the measured forces and the spanwise distribution of the chord Reynolds number in Figure 11 (b). It

495 demonstrates that, in the root and tip region, the blades used in this experiment produce less lift than would be expected. It can be hypothesised that this is a consequence of differences in surface finish between the used blades and the airfoil measured by Fontanella et al. (2021a), as well as minor inaccuracies in the manually produced geometry.^{EF}

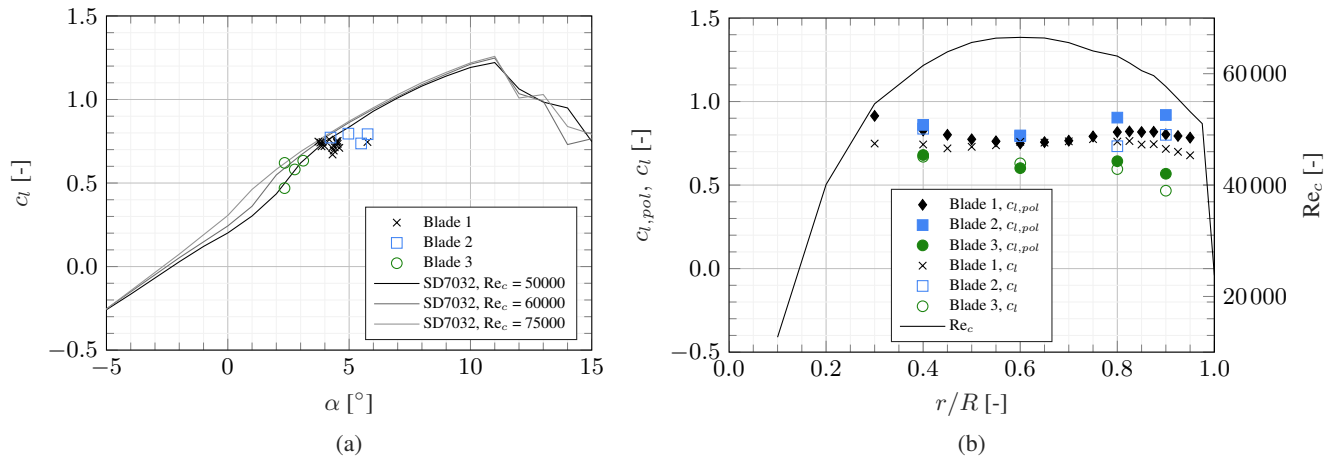


Figure 11. Experimental lift polarefficient^{EF} compared to the SD7032 airfoil lift polar at $Re = 50000$ ^{EF} (a) and comparison between the lift coefficient derived from measured forces to that expected based on the design lift polars, alongside the chord Reynolds number distribution (b)^{EF}

4 Conclusions and outlook

This study presents the results from an experimental campaign on a thrust-scaled version of the IEA 15 MW RWT. Particle
 500 image velocimetry is used to measure the flow field at multiple radial stations around the blade. Various aerodynamic blade properties are derived directly from the measured flow field along a closed curve around the blade cross-sections: The circulation is determined from the velocity integral, the inflow conditions by removing the blade induction from the measured flow field using elemental potential flow solutions and the forces based on Noca's method and the Kutta-Joukowski theorem.

Early analyses revealed that the blades were mounted with minor deviations from the desired pitch angle and, on top of that,
 505 exhibited twist deformations. This leads to considerable differences in the angle of attack and, consequently, blade loads among the three blades, which is consistently reflected in their experimentally derived spanwise distributions. In contrast, the derived induction values remain nearly constant between the three blades, indicating that induction can be considered a rotor-averaged phenomenon. This is an experimental confirmation of one of the fundamental assumptions in blade element momentum theory.

The dataset created in this wind tunnel experiment fully characterises the three blades in terms of the surrounding flow field,
 510 bound circulation, local inflow conditions and blade loads. The normal force distributions derived using Noca's method and the Kutta-Joukowski theorem were found to be in good agreement. Knowing these aerodynamic parameters, it can be demonstrated that the lift coefficient measured along the span follows the trend of agrees well with^{EF} the lift polar of the airfoil^{EF} used in the

blade design. There are, however, slight deficits in lift production in the root and tip regions compared to the expected values based on the design airfoil's lift polar.^{EF}

515 The experimental data presented here can be used in future numeric model validation studies. It provides data relevant for validating low-fidelity models, such as algorithms based on blade element momentum theory or lifting line theory, and for mid to high-fidelity models, such as panel codes and computational fluid dynamics. Since the model blade is based on the IEA 15 MW RWT, the non-dimensionalised loads resemble the current state-of-the-art of real offshore wind turbines and numerical reference models. Furthermore, the newly created model wind turbine can be used in future experiments investigating the
520 aerodynamics of this reference wind turbine.

To reduce the impact of blade deformations^{EF} in future research, it is recommended to either produce a new set of blades with less variation in their stiffness properties or to apply more advanced deformation tracking techniques such as photogrammetry. To improve the accuracy in pitch setting, the manual pitch mechanism could be exchanged for a variable pitch mechanism controlled by a motor. This would then require an initial calibration before a new experimental campaign.

525 *Data availability.* The data presented in this study, as well as information regarding the blade planform and logged wind tunnel operating conditions, are openly available on the 4TU.ResearchData repository at DOI:10.4121/164890ab-39d7-4af8-8b3c-9e21f789b80a.

Appendix A: Sensitivity to chosen control volume

In this study, the blade's aerodynamic quantities are determined by interrogating flow information along a closed curve enclosing the investigated blade cross-section. A circular curve is chosen with the blade cross-section positioned in its centre. To
530 verify the methods presented in Section 2.3, a panel code developed by Ribeiro et al. (2022) based on the work of Katz and Plotkin (2001) is used to replicate the wind tunnel experiment numerically. The panel code simulates the three-dimensional surface of the blade and can be used to derive flow fields at locations equivalent to the measurement planes of the experiment. Such results then offer the opportunity to derive circulation and loads based on the velocity field around the blade ("indirect") but also from the aerodynamic solution on the blade ("direct"). By comparing these two approaches, the methods for deriving
535 aerodynamic quantities from the flow field can be verified before applying them to the experimental data.

Figure A1 shows the sensitivity of the calculated circulation and of the forces based on Noca's method to the control volume's size, given as the ratio of its radius r_{CV} to the local chord, at three radial locations. When calculating the forces based on the Kutta-Joukowski theorem, they are directly proportional to the circulation distribution and are, thus, not presented here.

The sensitivity is investigated for both the experimental data as well as the panel code results. Generally, there is a conflict
540 of interest between the data points per control volume size which favours a large control volume and the approximation of two-dimensional flow in a flat measurement surface which favours a small control volume.

For the panel code results (PC), it can be observed that the indirectly determined circulation converges against the directly determined value with increasing control volume size. For the normal force, this is only true for the two outboard sections

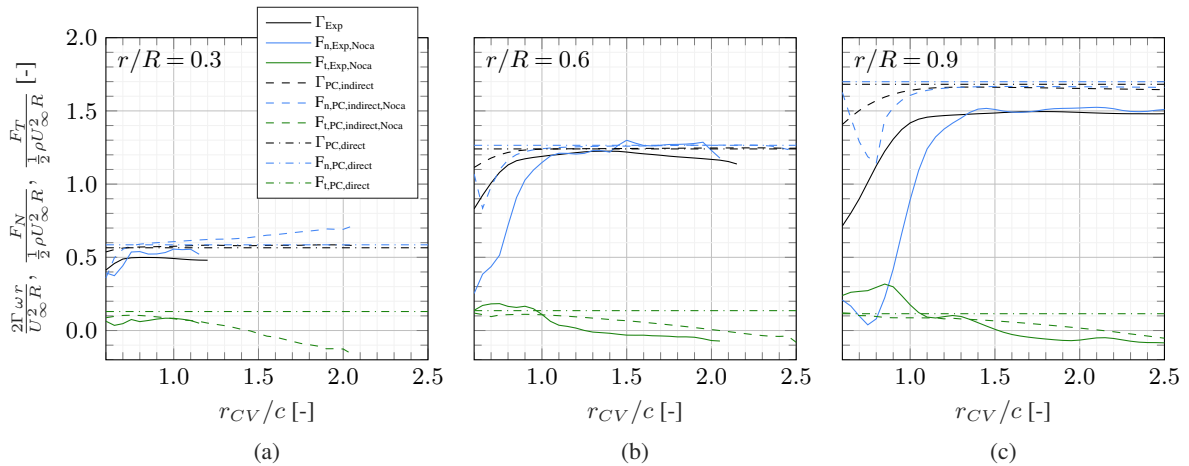


Figure A1. Sensitivity of determined blade loads and circulation to the chosen boundary curve size at various radial stations

shown in Figure A1 (b) and (c). The discrepancy between the direct and indirect approach at the inboard section can be attributed to the increasing flow curvature in this region, which stands in contrast to the two-dimensional control volume.

In contrast to circulation and normal force, the tangential force does not converge anywhere along the span but rather decreases with increasing control volume size. The high tip speed ratio of the model turbine entails very low torque values and the tangential force is very small. As such, the momentum change corresponding to the tangential force is difficult to capture with the Noca method. Based on this finding, only the tangential force calculated via the Kutta-Joukowski theorem is presented in this article.

The circulation and forces determined based on the experimental data largely follow the same trends observed for the panel code results. However, given the less clean flow field, the convergence is not as steady and shows slight deviations even after the initial, clearly unconverged, ramp. This is particularly true for the forces calculated using Noca's method, which relies on sensitive derivatives of the velocity field. To limit the influence of the control volume, the convergence is evaluated individually for each measurement plane and the endpoint of the initial convergence ramp is identified. The aerodynamic quantities are determined for multiple control volumes with sizes beyond the initial convergence ramp and then averaged over these. This approach yields the results presented in Section 3.3. It should further be noted, that for the experimental results, the largest possible control volume is dictated by the available field of view. Thus, the convergence of methods such as Noca's should be taken into consideration when defining the PIV setup and, consequently, the field of view.

Author contributions. EF designed the wind turbine model, built the model blades, planned and executed the experiment, and post-processed and analysed the measurement data. AR contributed to the experiment execution and data analysis and provided numerical simulations to help the development of post-processing methods. KB acquired funding and contributed to the experiment planning and the data analysis.

CF acquired funding and contributed to the experiment planning and execution, the development of post-processing methods, and the data analysis.

565 *Competing interests.* The authors declare that they have no competing interests.

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